

## ABSTRACT

Title of Thesis:                      EXPERIMENTAL CHARACTERIZATION OF  
ACOUSTIC WAVE PROPAGATION  
THROUGH A SUPERSONIC DUCTED FLOW

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In scramjet combustors, if pressure waves could propagate upstream through subsonic boundary layer flow, it would set up an acoustic feedback mechanism that could lead to self-sustained combustion instability. To investigate the possibility of upstream wave propagation, non-reacting supersonic flow experiments were conducted in a specially-designed supersonic flow duct, which simulated the internal flow path of a dual-mode scramjet combustor. Furthermore, to experimentally simulate combustion instability, large-amplitude pressure oscillations were created by passively exciting the exhaust jet flow using screech mechanism, which resulted in large-amplitude pressure oscillations with dominant frequencies ranging between 2.7kHz and 4.2kHz. Then, the acoustic signal was tracked along the supersonic flow duct using four high-frequency-response Kistler pressure transducers that were flush-mounted at the combustor and isolator walls. Schlieren visualization was conducted to characterize the internal supersonic flow field, and an analytical approach was used to estimate the turbulent boundary layer growth and displacement thickness. Ten sets of experiments were conducted at various stagnation pressure values ranging from 35psi to 125psi, and four sets of experiments

where strong resonances were observed were repeated over ten separate runs for reproducibility. Fast Fourier Transform was used to quantify the changes in pressure oscillation amplitude in each case. The results conclusively show that the downstream disturbances were propagating upstream, and they were being attenuated at different rates depending on flow conditions and duct geometry. Possible reasons for this new phenomenon were examined and discussed.

EXPERIMENTAL CHARACTERIZATION OF ACOUSTIC WAVE  
PROPAGATION THROUGH A SUPERSONIC DUCTED FLOW

By

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Thesis submitted to the Faculty of the Graduate School of the  
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## Dedication

This thesis is dedicated to my father, Clive Oliver Stamp, who has given me the desire and passion for learning as well as the vision to see and believe in the possibilities in life. He has given me the courage to challenge conventional thinking and create pathways that were considered to be impossible. These timeless intangibles are, in essence, the critical reasons for my motivated and assiduous effort in pursuing this scholarly work.

## Acknowledgements

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## Nomenclature

$a$  = speed of sound  
 $F(x)$  = amplifying (positive) or dampening (negative) effects  
 $f$  = frequency  
 $H_0$  = constant area section duct height  
 $K$  = bulk modulus of the medium  
 $M$  = Mach number  
 $n$  = number of measurements.  
 $P_r$  = reference pressure  
 $P_x$  = individual pressure  
 $p$  = acoustic pressure  
 $p'$  = local pressure fluctuations  
 $q'$  = oscillating energy release  
 $R$  = gas constant  
 $Re_x$  = Reynolds number  
 $T$  = temperature  
 $t$  = time  
 $U_\infty$  = freestream velocity  
 $u$  = time average velocity inside boundary layer w.r.t to  $y$   
 $V$  = uniform velocity  
 $\bar{x}$  = average value of the measured data, or best estimate  
 $z$  = displacement along  $x$

### Greek Letters

$\gamma$  = specific heat ratio  
 $\delta$  = boundary layer thickness  
 $\delta^*$  = displacement thickness  
 $\theta$  = momentum thickness  
 $\lambda$  = wavelength  
 $\mu$  = Mach angle or viscosity  
 $v$  = specific volume  
 $\rho$  = density  
 $\sigma$  = standard deviation  
 $\tau$  = compressibility parameter

## Acronyms

AHSTF	Arc-Heated Scramjet Test Facility
CAD	Computer Aided Design
CHSTF	Combustion-Heated Scramjet Test Facility
CHHEBD	Combustion-Heated High Enthalpy Blown-Down
GASL	General Applied Science Laboratory
HHT	High Temperature Tunnel
ICCD	Intensified Charge Coupled Device
NPT	National Pipe Thread
NASP	National Aero-Space Plane
PT	Pressure Transducer

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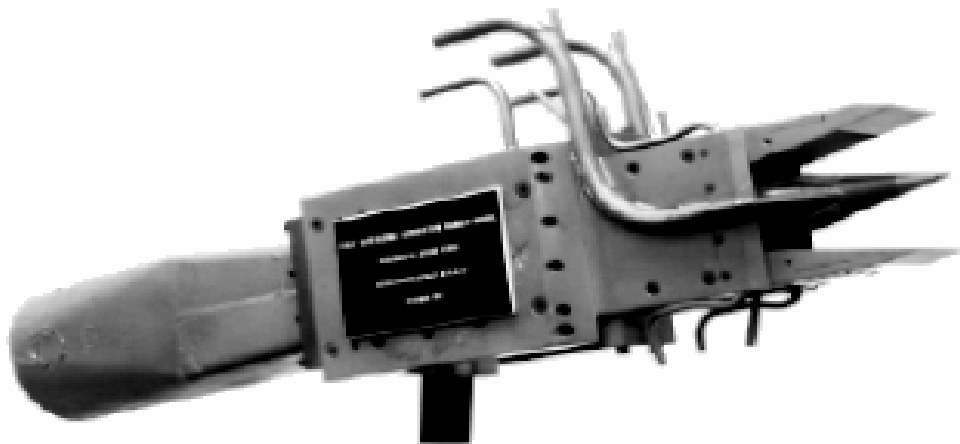
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# Chapter 1: Introduction

## 1.1. Background and Motivation

As the world of aviation moves closer and closer towards the reality of hypersonic flight, the engines that are projected to power these vehicles for this lofty endeavor are predicted to be supersonic combustion ramjets, otherwise known as *scramjets*. Before the start of developmental research on scramjet engines in the 1960's, ramjet engine technology has been the focus of study for high-speed air breathing propulsion since the 1940's in the United States.<sup>1</sup> During this time, other nations were also conducting research on hypersonic vehicle propulsion systems. Mestre and Viaud (1964) of France performed several scramjet engine experiments by burning kerosene/air in a constant area supersonic duct.<sup>2</sup> Scramjet research and development in Russia, or the former Soviet Union, was established primarily by Prof. E.S. Shchetnikov at the Central Aerohydrodynamic Institute.<sup>3</sup> Building upon the successes in the advancement of ramjet engines, Shchetnikov and other scientist at the Central Institute of Aviation Motors continued to examine some of the fundamental issues of scramjet engines. Such problems as fuel-air mixing and burning in a supersonic flowfield over some practical length became central to their research.<sup>3</sup> The earliest research and development of scramjet engines in Japan was carried out during the 1970's. Experiments were primarily conducted in the universities, which focused on the ignition mechanism for a diffusion flame in a supersonic flowfield.<sup>4</sup>



**Figure 1. Illustration of the first US freejet tested scramjet designed by A.Ferri (1963) (Fry 2004)**

The propulsion cycle of scramjet engines allow the vehicle to perform over a broader flight Mach number envelope in comparison to vehicles flying with conventional airbreathing engines. Efficient operation for typical liquid fueled scramjets occurs at flight Mach number range<sup>6</sup> of  $M_\infty = 3$  to 10. In comparison to liquid fueled, gaseous fuel scramjets can be efficiently operated up to orbital speeds. One of the critical constraints for liquid fuel type scramjets is the factor of energy consumption during ionization and dissociation. Academic institutions and other research entities have found a litany of fundamental problems that need to be addressed and if possible completely solved before the realization of efficiently operating scramjet and ramjet engines become successfully integrated in future aircraft. Some of the critical problems facing the successful development of these engines include understanding complex flowpath, efficient fuel injection, mixing and combustion, as well as suppressing combustion instability. With the captured air inside the inlet region, it becomes critical to ascertain the complex flowfield irrespective of the flow channel's geometric configuration<sup>6</sup>. Specifically, according to Waltrup<sup>6</sup>, a thorough comprehension about the compressible flowfield (externally and internally) coupled with shockwave dynamics is needed.

Another critical area of interest is fuel injection. Problems associated with this area can be broken down into two types: wall (normal) injection and axial (transverse) injection.<sup>6</sup> In terms of wall fuel injection into the flowfield, a detailed knowledge of how the fuel breaks down to the smallest possible sizes (atomization), disintegrates, and imbeds into the flowfield becomes necessary.<sup>6</sup> According to Waltrup<sup>6</sup>, the axial fuel injection issues are focused around the dynamics of free shear layer growth. In addition, fuel ignition characteristics with respect to “thermochemical properties” are particular interests that require major research effort. Within the combustor, the focus is on optimizing the amount of heat release and pressure losses, which are the two driving factors of its development.<sup>6</sup> While attempting to maximize the thrust output of the engine, it becomes critical to keep in mind the optimum combustor length in order to accomplish this objective. It is equally vital to grapple with the issue of how the fuel-air entrainment (mixing process) over the specified design combustor length can be carried out efficiently.<sup>6</sup> Therefore, mixing and burning with timescales on the order of milliseconds are problems that have elicited the attentive efforts of many.

According to Ma et al.<sup>7</sup>, there seems to be a general consensus that acoustic wave upstream propagation inside a supersonic flowfield is not possible. The thought is that pressure oscillations inside the combustor will not interact with the flame front to create a closed-loop feedback, which is essential for the prolonged combustion instability.<sup>7</sup> This idea would be correct if the subsonic boundary layer region of the flowfield inside the duct was neglected.

Ma et al.<sup>7</sup> proclaimed that a different point of view was needed and that these combustion oscillations in the form of acoustic waves can propagate upstream via any subsonic portion of the flowfield i.e. boundary layer or recirculation regions. This is, in essence, the primary motivating reason behind this research effort.

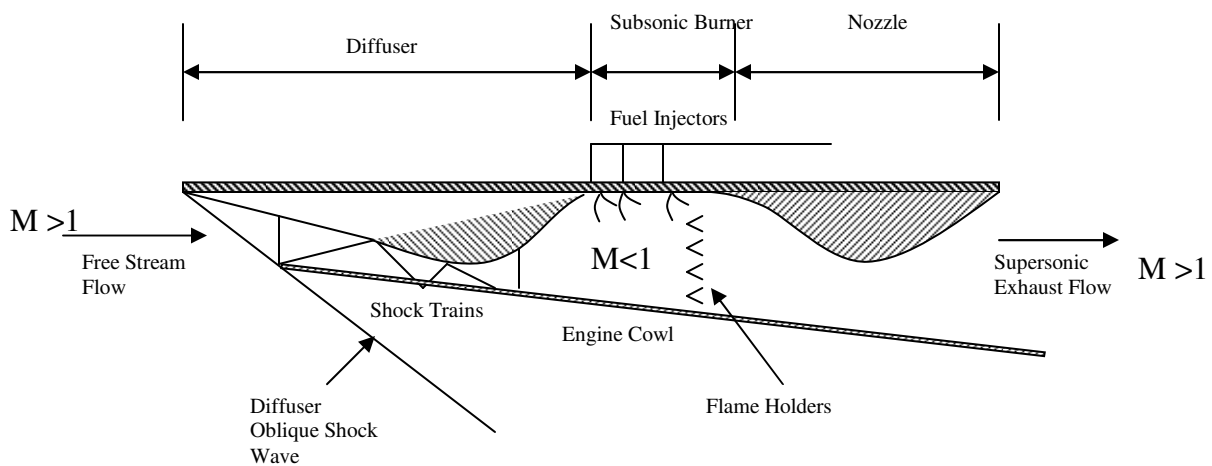
#### 1.1.1. Scramjet and Ramjet Historical Evolution

Ramjet engines, which are subsonic combustion engines, were the main focus of air-breathing propulsion R&D between the 1940's and 1960's in the United States. Research institutions involved in ground testing and flight testing were NASA Langley (Wallops Station), Navy (China Lake), and U.S. Air Force (Marquardt).<sup>8</sup> In the early 1960's, the aerospace research community realized that supersonic combustion ramjets (scramjets) were needed in order to accomplish the goal of achieving hypersonic flight.<sup>8</sup> By the middle of the 1960's, free-jet engine testing programs began to take hold. Testing was conducted mostly in propulsion test facilities that accommodated the high-energy exhaust from the nozzle. At the General Applied Science Laboratory (GASL), the Combustion-Heated High Enthalpy Blown-Down (CHHEBD) Tunnel conducted the first free-jet tests of the first known scramjet engine in 1963, see Figure<sup>8</sup> 1. This scramjet engine was a fixed geometry model, which was designed by Dr. A. Ferri.<sup>8</sup> After a decade of "non-airframe integrated" engine testing, the push towards integrating the engine on an airframe became the primary objective. Testing facilities dedicated to sub-scale engine propulsion were established at NASA Langley. From the mid 1970's to mid 1980's, the Arc-Heated Scramjet Test Facility (AHSTF) and the Combustion-Heated Scramjet Test Facility (CHSTF) were mainly used for testing and development of these sub-scale engines. For large-scale engine testing, such as the National AeroSpace Plane (NASP), the Langley High-Temperature Tunnel (HHT) was used in the early 1990's. The

Advanced Research for Transportation Technology program directed by NASA Marshall continues to push the envelope in sub-scale engine research today.

The cooperative efforts by NASA Dryden and Langley oversee the execution of the Hyper-X Project. The steady progress in scramjet engine research can be attributed to the great strides made in the advanced development of Computational Fluid Dynamics as well as dedicated and in-depth research for scramjet inlet and combustor development.<sup>8</sup>

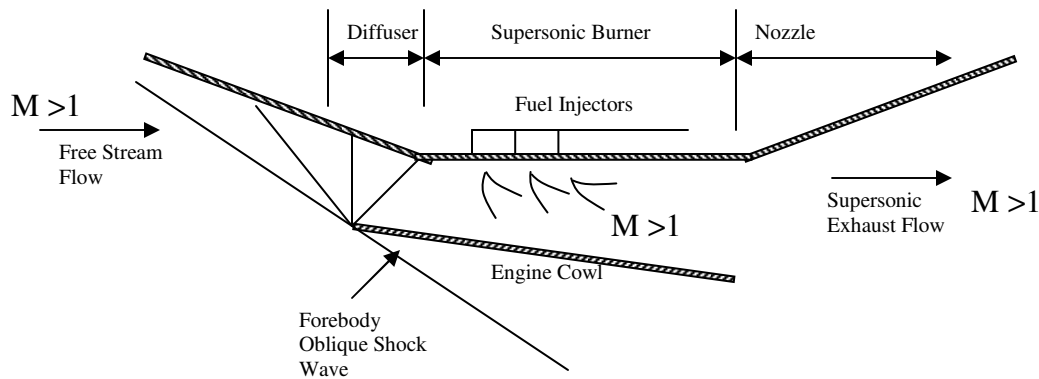
#### 1.1.1.1. Scramjet and Ramjet Engines Flow Dynamics



**Figure 2. Schematic illustration of the 2-D flow dynamics inside a generic ramjet engine**

In describing the overall flow dynamics through a ramjet engine, a schematic diagram, see Figure 2, will be used to illustrate the critical portions of the engine. Starting from left to right with the freestream conditions, the airflow enters the diffuser, which is mainly used to decelerate the flow. At the tip of the vehicle's forebody an oblique shock is used to compress the air, which is the beginning of the flow deceleration process. Deceleration continues through a series of normal shocks, also known as a shock train, and then travels through the divergent portion of the duct just before the entrance of the subsonic burner. Inside the burner, the subsonic air stream mixes and burns with the injected fuel.<sup>9</sup> The combustion products at high pressures are then accelerated to supersonic speed through the convergent-divergent portion of the exhaust nozzle.

With greater momentum at the exit plane of the engine than at the inlet plane, the ramjet engine is able to exert thrust on the vehicle in order to propel it in the forward acting direction.<sup>9</sup>



**Figure 3. Schematic illustration of the 2-D flow dynamics inside a generic scramjet engine**

At the limiting case for flight region of Mach 6, the ramjet engine becomes impractical. The critical reason for this is due in part to what happens to the energy during the deceleration process. The diminishing kinetic energy is transferred into internal energy of the flowfield<sup>9</sup>. Consequently, one can observe the significant increase in temperature, pressure, and density at the burner entrance. These conditions could escalate even more with increasing Mach number. With such disadvantageous conditions, numerous effects such as large heat transfer rates at the wall, diminishing combustor integrity, as well as engine performance losses may ensue. To resolve some of these detrimental effects, the flowfield entering the combustor should be supersonic instead of subsonic as seen in the ramjet engine. The key point in doing this is to reduce the amount of loss in kinetic energy being transferred into internal energy and hence less rate of pressure, temperature, and density with increasing Mach number.

The generic operation of a scramjet engine and its major components can be described using Figure<sup>9</sup>3. Oblique shock waves from the forebody and diffuser compress the supersonic freestream flow to the design burner entrance supersonic Mach number.

Inside the supersonic combustor, fuel is then injected at the optimum location to allow for complete mixing and burning in a short period of time (short combustor length less structural mass).<sup>9</sup> This is arguably one of the most critical aspects of a successful development of scramjet engines. This area of research has been and is currently being pursued at the Advance Propulsion Research Laboratory (University of Maryland, College Park) as well as many other institutions and laboratories across the nation and the world. Finally, the divergent nozzle provides the needed acceleration for the high-speed combustion products.

### 1.1.2. Combustion Instability

As mentioned before, a greater understanding of the fundamental physical phenomena that exists inside the combustor of these high-speed vehicles (i.e. hypersonic vehicles) is essential to successful development and implementation of engines such as scramjets and ramjets. Application requirements of modern propulsion system are the primary driving force behind which of these parameters, for example performance,  $\text{NO}_x$  output, fuel efficiency, and maintenance cost, will take a higher priority level for optimization.<sup>10</sup> Given the increasing demand on contemporary power plants, it has become a top priority in recent years to address some of these phenomena. A critical phenomenon that is of interest is the combustion instability that is recognized in propulsion systems.<sup>11</sup> In general, the combustion instabilities that are observed in propulsion systems are large amplitude pressure oscillations that are intensifying sporadically out of the noise that occur during the combustion process.<sup>11</sup> According to Roy<sup>10</sup>, one may describe these combustion instabilities as a very dynamic and complex feedback process that occur when these oscillations of single or multiple acoustic modes are driven by the perturbation during combustion. Historically, the first studies conducted on combustion instability phenomenon were focused on liquid rocket engines in countries such as Germany, Russia, and the United States during WWII<sup>11</sup>.

Currently, combustion instabilities are not only being addressed in rockets but also in liquid fueled air-breathing propulsion systems (ramjets and, scramjets, and gas turbines) as well in burners.<sup>12</sup> In essence, these combustion instabilities are produced by the amplification of the disturbances in the flow by the coupled effects of the heat release during combustion and the generated acoustic energy.<sup>13</sup> The self-sustainability of the instabilities is driven by the continuous reinforcement of the acoustic energy.

The performance and structural integrity of the propulsion system will be compromised with the detrimentally increasing effects of the expected thermal and mechanical loads from these combustion instabilities.<sup>13</sup> Lord Rayleigh, who addressed this issue in 1878, stated that the acoustic disturbances receive a continuous influx of energy if the energy release during the combustion process is “in phase” with the transient pressure conditions. This concept known as Rayleigh’s Criterion can be expressed mathematically with the following formula, according to Roy<sup>10</sup>.

$$F(x) = \frac{1}{T} \int_T q'(x, t) p'(x, t) dt \quad (1)$$

where T= period of investigation

F(x)= amplifying (positive) or dampening (negative) effects

q'(x,t)= oscillating energy release

p'(x,t)= local pressure fluctuations

The current consensus is that, for theoretical and empirical pursuits, the understanding and implementation of the formulation (1) will bolster the capability of attenuating the disturbances in the combustor, but doesn’t necessarily improve the total performance of the engine.<sup>10</sup> A thorough look at the mixing enhancement methods as well as various control mechanism for pressure fluctuations may be warranted. For a more in-depth understanding of the intricate details that are involved with these instabilities, a list of relatively current numerical and experimental research work involving the flowfield, vortex structures, small scale turbulent structures, and thermoacoustic oscillations is documented by Roy.<sup>10</sup> The promising research work in combustion control, which include active as well as passive control has been at the forefront of combustion instability research.<sup>14-21</sup> One may describe active combustion control as a manipulative technique that utilizes sensor control mechanisms to suppress the amplified oscillations in the combustor. While on the other hand, the means of passive control which is more limited in its implementation, where expenses can accrue rapidly due to constant modification to distinct systems.<sup>14</sup> The mode of oscillations that are typically observed in combustors

falls under the categories of low and high frequency oscillations. In general, low frequencies are associated with longitudinal modes while high frequencies are associated with transverse modes, which produce the very noticeable “screeching” sound.<sup>13</sup>

### 1.1.3 Screech Tones

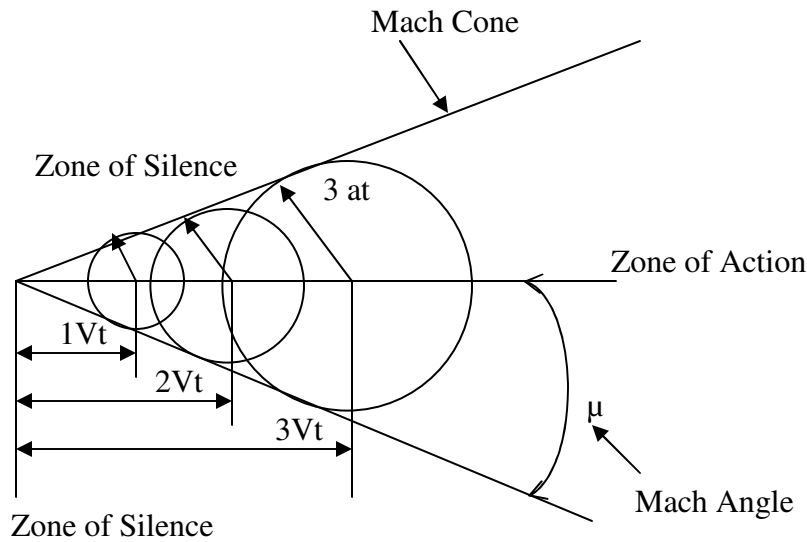
In order to simulate the large pressure oscillations that are associated with these combustion instabilities, an acoustic phenomena known as screech tones have been implemented as a passive excitation mechanism during this experiment. Screech tones were first studied by A. Powell in the early 1950’s at the University of Southampton in England.<sup>22</sup> According to Raman, Powell used the schlieren flow visualization technique on a small-scale jet when this phenomenon was observed. He studied both two dimensional and circular jets and proposed a “phased array model” that brought forth the formulae for the screech frequency and directivity.<sup>22</sup> Powell described these screech tones as “embryonic disturbances” that are produced at the nozzle lip. As they propagated downstream, these vortex structures grow and in the process combine with the “shock cells” in order to generate sound. The generated sound then propagates back upstream to nozzle tip where the resonant loop is established.<sup>22</sup> In this experiment, it was observed that the overexpansion of the supersonic jet at the exit plane of the duct oscillates periodically which produced these very audible screech tones. The supersonic flow was discharged into the tube from the nozzle lip of both the top and bottom components of the supersonic duct. Shear layer growth and shock interactions occur such that a passive acoustic excitation is created. This oscillation can propagate upstream through the fluid medium via the boundary layer. How far upstream and how much dissipation has occurred are the fundamental focuses of this research.

A previous study that dealt with sound waves passing through a turbulent boundary layer examined the problem on two fronts experimentally and theoretically.<sup>23</sup> Experimentally, the test was modeled as boundary layer flow over a flat plate where the attenuation of sound was examined. With Mach numbers reaching up to as high as 0.8, the authors used a flat plate that was attached to one side of this high speed jet.<sup>23</sup> The authors found significant losses of sound transmission for high frequency signal in the forward direction. Theoretically, the authors developed a model of the sound propagation that



took into account refraction due to velocity and various density levels of the turbulent boundary layer.<sup>23</sup> They discovered that the frequency of the sound wave, the measurement angle, and the Mach number of the jet had a strong effect on eddy viscosity and how it relates to acoustic transmission losses. The investigation carried out during this experiment rest on the fundamental basis that upstream propagation of acoustic waves will occur via the subsonic boundary layer and not the supersonic core flow.

This concept that disturbances of any types will not propagate upstream through the purely supersonic flow is a fundamental physical property of supersonic flow.<sup>24</sup> It is well know that a body in a supersonic stream will create a Mach cone with concentric circles that separate regions called zone of action and zone of silence,<sup>25</sup> see Figure 4.



**Figure 4. Schematic diagram of the effects of some point source traveling at supersonic speed**

The schematic diagram in Figure 4 illustrates a critical concept known as the “rule of forbidden signals”.<sup>26</sup> This fundamental principle in supersonic flow simple states that a body moving a supersonic speed will produce pressure perturbations that will fall behind the body itself. The creation of a wave front in the form of Mach cone with the moving body at the vertex point.<sup>26</sup> It is assumed that the body travels with uniform velocity,  $V$ ; the distance traveled is therefore “ $Vt$ ”. The angle, from the central point, of the cone is called the Mach angle,  $\mu$ , which is defined as  $\{\mu = \sin^{-1}(1/M)\}$ , where  $M = V/a$ . This Mach number ( $M$ ) is defined as the local velocity

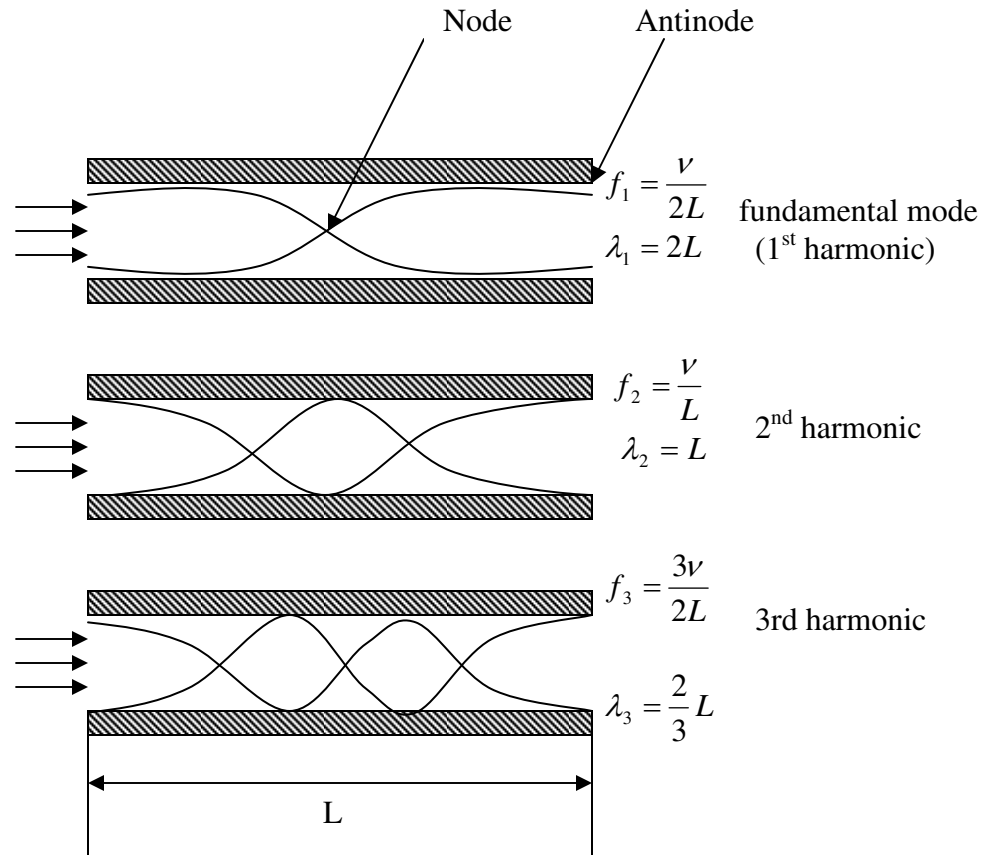
of the moving body divided by the local speed of sound " $a$ ". The zone of action is the region that is located within the Mach cone, where disturbances propagate downstream of the moving point source. Beyond the boundaries of the Mach cone, the zone of silence stretches upstream of the moving body such that no detection of the signal is permitted in this zone.<sup>25</sup> This concept essentially forms the assertion of this research, where it is assumed that the signal propagation will not be felt inside the supersonic stream but only in the subsonic regions of the flow, where the disturbances propagate in all directions.

#### 1.1.4. Research Objectives

In order to give this research effort a well-defined direction, a few realistic objectives are desired to accomplish this mission. First, one would like to examine whether or not the downstream disturbances will propagate upstream through the supersonic duct. To answer the question of how far the perturbations could propagate upstream, one would examine the dissipation effects of the acoustic wave amplitude (intensity) as it propagates upstream through the boundary layer of a supersonic flow duct. Finally, one then would characterize the dampening mechanism and investigate whether or not a control mechanism can be implemented in this experiment. A characterization of the amplitude with respect to the horizontal distance ( $x$ ) was carried out.

## Chapter 2: Theoretical Analysis

### 2.1 Acoustic Waves



**Figure 5. Schematic illustration of the first three modes of longitudinal waves inside a duct**

Waves, in general, come in various types such as water waves that move across the ocean, distortional waves that oscillate within vibrating bodies. They also include sound waves, which transmit or carry tones and noises through the air and electromagnetic waves that transmit light, x-ray, infrared, etc... Wave propagation can be viewed as motion of a disturbance through some medium.<sup>27</sup> A typical example of this is the motion of some impulse of disturbance along a string or stretched out horizontally or vertically. Most waves can be classified as either longitudinal or transverse waves.

Longitudinal waves are, in essence, waves in which the vibrating motion of the particles moving forward and backward parallel to the direction of the wave propagation. On the other hand, transverse waves are generally described as waves in which the particles vibrate perpendicular to the direction the wave motion.<sup>27</sup> One may think of acoustic waves as mechanical waves with physical attributes. As a result, the medium of wave propagation must have inertia and elasticity for successful transmission.<sup>27</sup> Waves also transmit energy along the direction of propagation. The energy maybe transformed in various waves. In the case of a wave traveling from one medium to the next, the waves tend to distribute their energy proportionally. This distribution process may occur as a reflection, transmission, and absorption. This idea is analogous to light waves striking a glass, where part of it is reflected, most transmitted through the glass, while still a small potion is absorb by the glass. When energy of wave is completely absorb by its medium this absorbed energy is usually converted to heat.<sup>27</sup>

#### 2.1.1. Acoustic Wave Theory

It is well known that sound wave in air propagates longitudinally, where the oscillations takes place in the same direction of the wave motion.<sup>28</sup> This type of wave motion can be describe mathematically as the following: Assume one dimensional traveling wave,

$$\frac{\partial^2 z}{\partial x^2} = \frac{1}{c^2} \frac{\partial^2 z}{\partial t^2} \quad (2)$$

where,

$z$  = displacement along  $x$

$c$  = velocity of sound

$t$  = time

Solutions to equation (2) can describe the longitudinal motion of waves in pipes and ducts with specified boundary conditions. The acoustic pressure is related to “ $z$ ” in terms of the following, where  $K$ = bulk modulus of the medium,<sup>28</sup>

$$p = -K \frac{\partial z}{\partial x} \quad (3)$$

Emphasis during this research wasn't placed upon the rigorous determination of the exact solution to equation (2) due to the necessity of more in-depth understanding of the complex flow field. This could be pursued as future research interests, maybe examining the two and three-dimensional solutions.

For this research work, a first cut approximation of the acoustic mode of the supersonic duct was obtained by using the model in Figure 5. This is a double open channel depicting the general shape of the first (fundamental), second, and third harmonics. The oscillating air column produces nodal and anti-nodal points, see Figure 5. A displacement anti-node is observed at each end where the maximum displacement occurs due to the freely moving air. While, the minimal displacement occurs at the nodal points.<sup>29</sup> The frequency and wavelength can be estimated by using the formulas given in Figure 5. Assuming a constant area channel with a length of 12 inches (305mm), the fundamental frequency (first harmonic) was calculated as being approximately 567 Hz. The higher harmonics were estimated as 1134Hz and 1700Hz, 2<sup>nd</sup> and 3<sup>rd</sup> harmonics respectively. These values were based on constant velocity of sound, with constant temperature (298K). This analysis was pertinent in order to gauge what role the supersonic duct's inherent acoustic modes would play in this experiment. Based on the dominant frequency of the passive acoustic mechanism, there is little probability that resonance would occur between the screech tones and natural frequency of the supersonic duct.

## 2.2. Boundary Layer Theory

The concept of a boundary layer development along a surface was first presented by Ludwig Prandtl in Heidelberg, Germany (1904).<sup>30</sup> The significance of this concept is the fact that it allowed engineers and scientist to execute viscous flow analysis. The practical application for such a concept was used to generate drag and flow separation theoretical data for aerodynamic principles.<sup>30</sup> The boundary layer equations, which are obtained from the full Navier–Stokes Equations were derived based on the assumption that boundary layer is adjacent to some aerodynamic body. Solving these boundary layer equations produced solutions for heat

transfer and distributions of shear stress across the surface.<sup>30</sup> The boundary layer is essentially the region of flow that is adjacent to the surface is considerably thin in comparison to the core flowfield.<sup>29</sup> In ideal fluid analysis, the viscous terms in the momentum equation as well as the heat conduction for energy equation in the governing equations of fluid dynamics are usually ignored in order to allow for rapid calculations this assumption leads to the Euler equations.<sup>31</sup> One may think this ideal or inviscid case as a flow in which the infinitely approaching Reynolds number becomes a dominant constraint.

In this experiment the boundary layer theory is focused on the assumption of flow over a flat plate. A few critical assumptions are that the boundary layer is much thinner than the parameterize length of the flat plate and the non-slip condition at the surface. Also, the Reynolds number is assumed very large and its inverse is on the same order of magnitude as the boundary layer thickness squared.<sup>32</sup> Another critical assumption is that at a given point  $x$  along the flat plate the pressure gradient perpendicular to the surface is negligible as follows:

$$\frac{\partial p}{\partial y} = 0 \quad (4)$$

In essence, pressure can be assumed constant along the vertical axis from the surface to the outer edge of the boundary layer at the lower limit of the main core flow.<sup>32</sup> The growth of the boundary layer reduces the cross-sectional area for the core or central flowfield in a channel flow. Hence, for supersonic wind tunnel, the flow will increase for a subsonic flow field and decreases for supersonic according to the Area-Mach number relation.<sup>31</sup>

### 2.2.1. Laminar and Turbulent Boundary Layer

Fundamental solutions to the boundary layer equations are based upon whether the flow is laminar or turbulent. Typically, laminar flow can be considered as smooth fluid motion where the mass transport between the adjacent fluid layers is negligible.<sup>33</sup> In essence the fluid elements seem to steadily glide over each other instead of exhibiting some random motion. In contrast to laminar flow, turbulent flow, as the name suggest, is more unpredictable and erratic motion of the fluid particles. The oscillation of the velocity occurs along the normal and along the direction of the average core flow.<sup>33</sup> According to Anderson,<sup>30</sup> the increased energy of the

turbulent structures translate into larger velocity profile for turbulent boundary layers than in the case of laminar boundary layers. The less likely hood of flow separation for turbulent boundary layers than laminar boundary layers can also attribute to the fact of increasing energy of the turbulent flow. The approximated analysis for the boundary layer thickness ( $\delta$ ) and displacement thickness ( $\delta^*$ ) was based on the turbulent boundary layer analysis. Kreider<sup>34</sup> provided an approximated solution with use of polynomial data fit of the velocity profile, which is known as the power law to the one-seventh scale, given as follows:

$$\frac{u}{U_{\infty}} = \left( \frac{y}{\delta} \right)^{1/7} \quad (5)$$

where  $u$ = time average velocity inside boundary layer w.r.t to  $y$

$U_{\infty}$ = freestream velocity

$\delta$ = boundary layer thickness

The boundary layer thickness in this analysis is based on the assumption that the time average velocity ( $u$ ) is 99% of the velocity at its outer edge.<sup>30</sup> The boundary layer thickness, displacement thickness, and Reynolds number were approximated with the following equations,

$$\delta_{turb} = \frac{0.368x}{Re_x^{1/5}} \quad (6)$$

$$\delta_{turb}^* = \frac{0.046x}{Re_x^{1/5}} \quad (7)$$

$$Re_x = \frac{\rho U x}{\mu} \quad (8)$$

### 2.3. Compressible Flow

Compressible flow is usually defined as “variable density flow”.<sup>24</sup> In comparison to incompressible flow, where the density remains relatively constant, compressible flow is mainly considered as any flow regime where the density changes with time. Let’s examine the compressibility parameters of a fluid, where the compressibility is defined as:

$$\tau = -\frac{1}{v} \frac{dv}{dp} \quad (9)$$

where  $\tau$  is the compressibility parameter,  $v$  is the specific volume, while  $p$  is the pressure. The physical meaning behind the compressibility can be observed as the fractional change in volume of the fluid element per unit change in pressure.<sup>24</sup> To obtain an even more accurate definition of compressibility, one would take into account the ratio of partial change in specific volume and pressure with respect to temperature or isentropic changes. In terms of change in density, where density ( $\rho$ ) is defined as the inverse of the specific

volume,  $\rho = \frac{1}{v}$ . Given the definition of density and equation 9, the change in density with respect to the change in pressure and the compressibility parameter is as follows:<sup>24</sup>

$$d\rho = \rho \tau dp \quad (10)$$

It can be shown that speed of sound, “ $a$ ”, defined as the square root of the partial change in pressure and density through an isentropic process.<sup>24</sup>

$$a = \sqrt{\left( \frac{\partial p}{\partial \rho} \right)_s} \quad (11)$$

Equation 11 is considered the general relation for the speed of in a gas. For a calorically perfect gas and in terms of specific heat ratio  $\gamma$ , gas constant  $R$ , and the temperature  $T$ , an alternate definition can be obtained for the speed of sound, where

$$a = \sqrt{\gamma RT} \quad (12)$$



## 2.4 Error Analysis

All experimentalists understand that whenever an experiment is conducted it is subjugated to errors and uncertainties. Therefore it becomes critical to quantify the errors associated with experimental data collection. In essence one looks at the accuracy, precision, and sensitivity to describe the quality of the experimental data.<sup>34</sup> Accuracy is generally defined as a measure of the relative proximity of the measured data to the true value. Basically how close the collected data comes to the true value. Hence, the closer the measured value to the true value the greater the accuracy. In terms of precision, it is usually describe as a measure of repeatability. How repeatable are one's measurements of the same test multiple times sequentially is answered by the level of preciseness of the data. In addressing the sensitivity, it is often considered as being the relative measured affect produced by a change in the value of the quantity that was measured. Basically, how much some parameter changes as a result of the change in the measured quantity. A quantitative measure of preciseness can be carried out by statistical analysis. The general idea is that as you increase the number of times a measurement is taken the more precise the measurement. In order to address this issue, statisticians develop what is known as the "Student's t Table", which relates the standard deviation of a certain set of measurements of the same measured parameter.<sup>24</sup> Mathematically the standard deviation is given by

$$\sigma \equiv \sqrt{\frac{\sum (x - \bar{x})^2}{n - 1}} \quad (13)$$

where x = measured data

$\bar{x}$  = average value of the measured data, or best estimate

n= number of measurements.

The standard deviation tells how much the data is distorted around the true value. In this experiment the error analysis focuses mainly on the standard deviation of the measured quantity. The standard deviation is used in calculating the product or quotient of a desired value. For example, an investigation of the ratio of the dynamic

pressure to static pressure was carried in this experiment. Due to the fact that each individual value was subjected to errors it was necessary to account for such errors the quotient analysis as well. In doing so the proper formulation for the product or quotient is as follows:

$$\sigma_{\frac{x}{y}} = \left(\frac{x}{y}\right) \sqrt{\left(\frac{\sigma_x}{x}\right)^2 + \left(\frac{\sigma_y}{y}\right)^2} \quad (14)$$

where

$\sigma_{\frac{x}{y}}$  = Quotient error

x = measured parameter

y= measured parameter

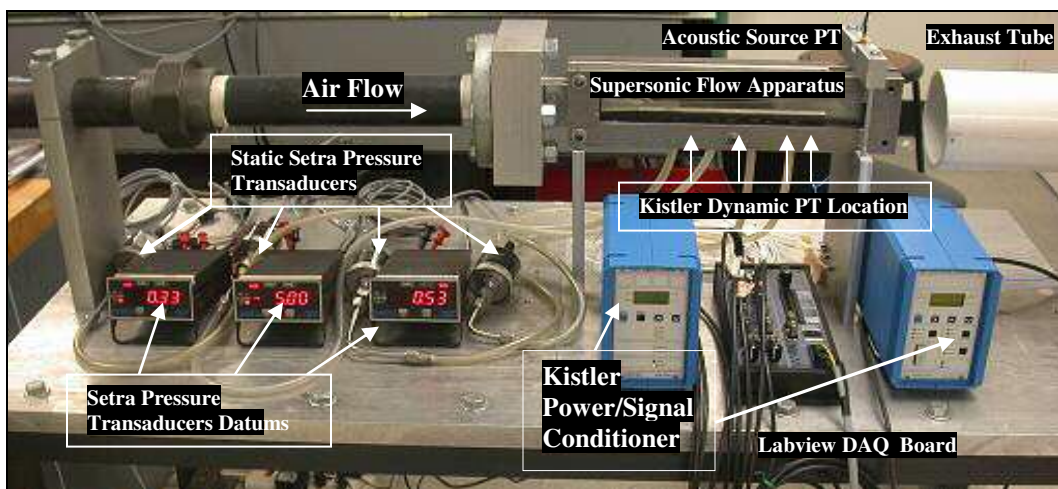
$\sigma_x$  = standard deviation of parameter x

$\sigma_y$  = standard deviation of parameter y

## Chapter 3: Experimental Setup and Approach

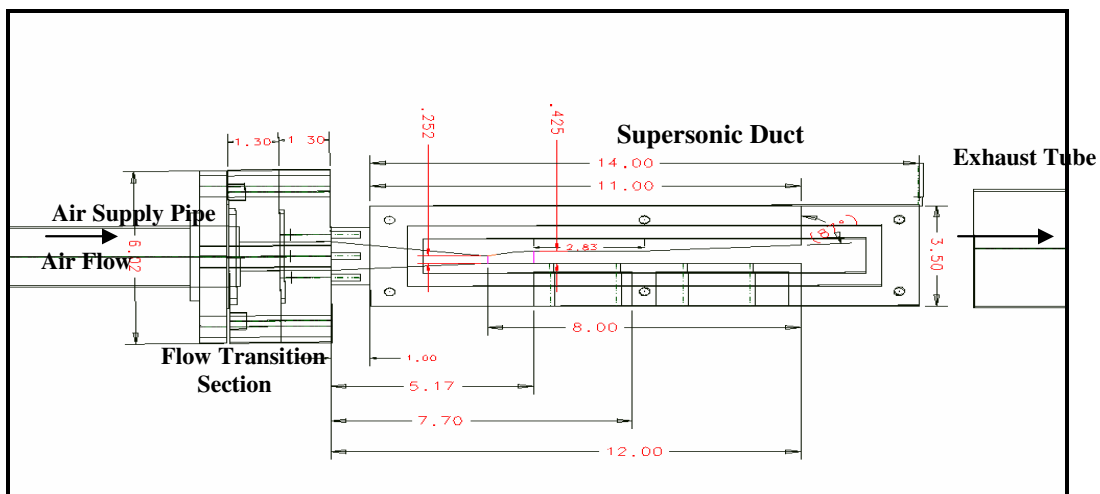
### 3.1. Experimental Facility

In order to investigate this problem of acoustic wave propagation through a supersonic flow duct, a two-dimensional channel that simulates scramjet a scramjet engine was designed and manufactured to examine this critical issue. As mentioned before, this experiment was a non-reacting (cold flow) investigation that mainly focused on the characterization of acoustic wave transmission inside the supersonic duct. This experiment was conducted using a Mach 2 convergent-divergent nozzle. Through a two inch NPT circular steel pipe, high pressure air was delivered from a stationary, single-stage, oil-injected screw type compressor into the supersonic duct inlet with a  $0.5\text{-in}^2$  cross-sectional area. The Atlas Copco Compressor is capable delivering a maximum volumetric flow rate of air at  $358\text{ ft}^3/\text{min}$  for pressures up to 150psi. A settling tank, which is connected to the compressor, provides a means of removing particle deposits and oil residue. Branching directly from the settling tank is a compressor line to a dryer, which basically eliminates the condensate by reducing the temperature to freezing levels. Before air is delivered to the inlet of the test rig, it was ultimately passed through a gas/air filter. A quarter ton valve upstream of duct inlet provides the mechanical means of turning on and off the flow of air, which subsequently goes through a heavy duty Wilkerson screw type regulator, which is rated for a pressure range of 0 to 180psi.



**Figure 6. Pictorial view of experimental setup of supersonic flow duct in laboratory**

The airflow is then transitioned from a circular pipe to a rectangular cross-section area with an O-ring seal providing airtight sealing on the transition block, see Figure 6 for a pictorial view of experimental set up. From the inlet region of the supersonic duct in Figure 5, airflow is then accelerated to Mach 1 at the throat of the convergent-divergent (c-d) nozzle. At approximately 78.74-inches (2m) upstream from the throat area of the nozzle, a 120 psig rated Setra Model 206 pressure transducer was used to measure the stagnation pressure at the choked flow regulator. The airflow is expanded from the sharp-cornered throat to a Mach 2 flow at the nozzle exit area with use of a nozzle profile, which was designed from the method of characteristics calculation. From the nozzle exit in the duct, the airflow travels through a constant area section of 0.213-in<sup>2</sup> (137.419 mm<sup>2</sup>), which can be viewed as an isolator region. Flow is then expanded through a diverging section with an angle of 3° from the horizontal. The reason for this was mainly to avoid the Fanno effect, where unwanted choking due to friction becomes prominent. Once the upstream flow conditions are established, the static pressure and dynamic pressure measurement can be obtained separately via the Setra Model 206 pressure transducers and the Kistler high-frequency response PT. The static pressure measurements were displayed using Setra's dual channel Datum 2000, while the dynamic pressure signals were transmitted to the Kistler Power and Conditioning Units. A detailed pictorial illustration of the experimental setup with the data acquisition is depicted in Figure 6. A National Instrument DAQ board served as the interface between the raw signal and Labview, which is essentially a real time based virtual laboratory software.



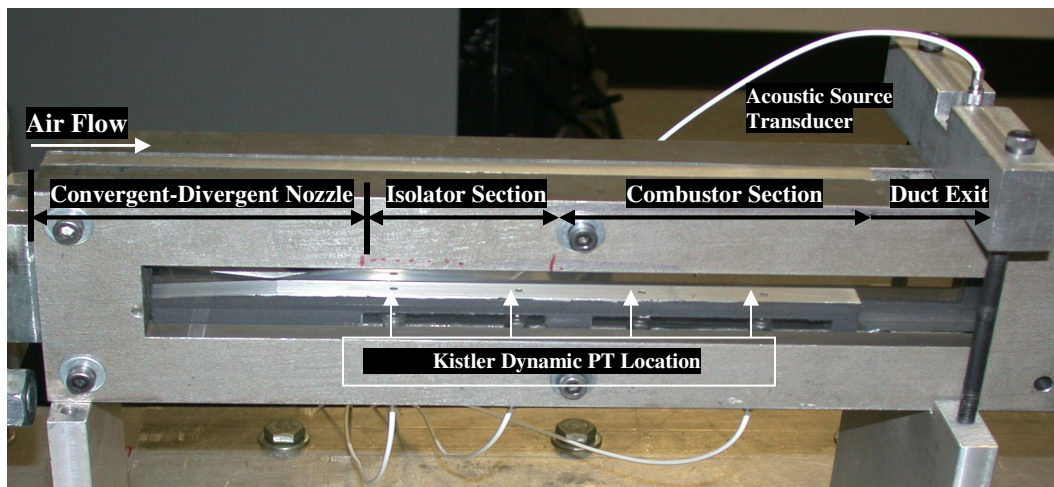
**Figure 7. Schematic illustration of the supersonic flow duct dimensional drawings layout with all dimensions in inches.**

A 2-D view of the assembled supersonic duct with dimensions at critical areas such as the throat area,  $0.126\text{-in}^2$  ( $81.29\text{ mm}^2$ ) of the c-d nozzle, the constant area section, and expansion region are shown in Figure 7.

### 3.2. Experimental Apparatus

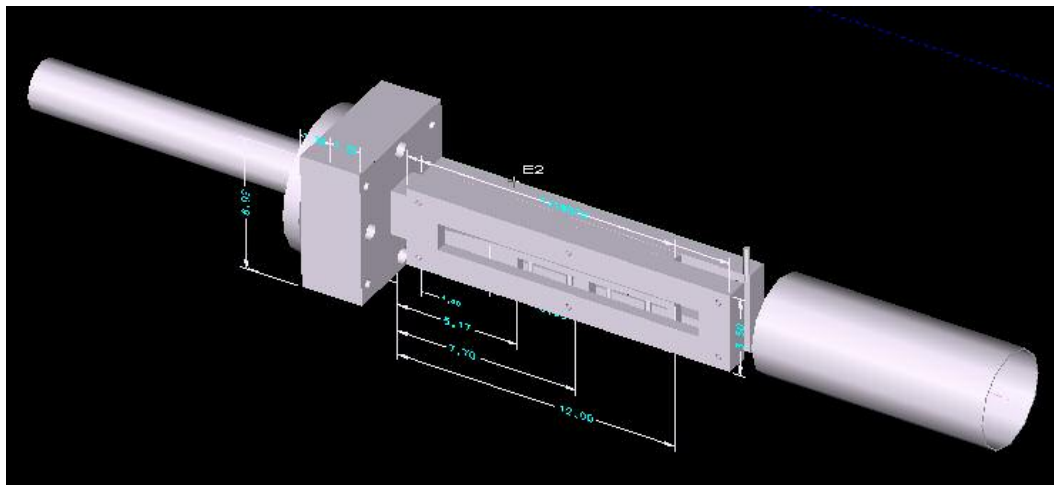
#### 3.2.1. Convergent-Divergent Duct

The top and bottom components of the duct were inserted and fixed between two one-inch (25.4mm) thick steel metal plates. The plates were designed with flow visualization area dimensions of 1.2 inches (30.48mm) high and 11.8-inches (299.72mm) long that is sealed on both sides with two (12-in x 2-in x 0.5-in) polished quartz glass. Below the test section (isolator and expansion regions), as seen in Figure 8, a bank of the Kistler high frequency response pressure transducer was strategically placed. The location of the pressure transducers were established based on the height of the constant area section at the convergent divergent nozzle exit. Transducers were placed at  $1H_o$ ,  $5H_o$ ,  $9H_o$ ,  $13H_o$ , where  $H_o=0.425\text{-in}$  (10.795mm), away from the Mach 2 c-d nozzle exit in the duct.

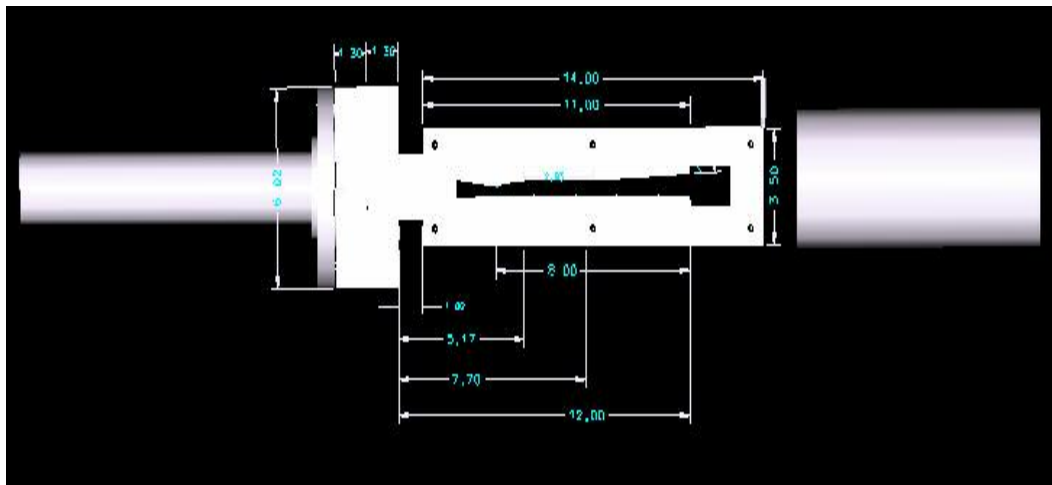


**Figure 8. Close-up view pressure transducer location inside the supersonic duct**

Following the c-d nozzle is a constant-area rectangular duct with internal dimension of 0.50-inches (12.7mm) wide, 0.425-inches (10.795mm) high, and approximately 3-inches (76.2mm) long, which serves as an isolator, see Figure 8. After the isolator section, there is a simulated combustor section, which expands on the top wall at a  $3^\circ$  angle. The supersonic duct simulates a scramjet combustor internal flowfield. A Kistler 211B5 piezoelectric pressure transducer was mounted vertically at 1.06-inches (26.92mm) downstream of the jet exit plane. This pressure transducer was used to measure the dominant frequency of the acoustic source pressure oscillations that were generated by the passive acoustic mechanism. An accurate measure of the amplitude was not obtained due to inability to account for uncertainties involved with the mounting of the transducer normal to the exhaust flow. It was difficult to ascertain at what angle the acoustic pressure waves were impinging on the pressure transducer. Tracking the acoustic source's dominant frequency and how its amplitude changes inside the duct as it propagates upstream irrespective of the initial amplitude from the acoustic source were the primary focus.

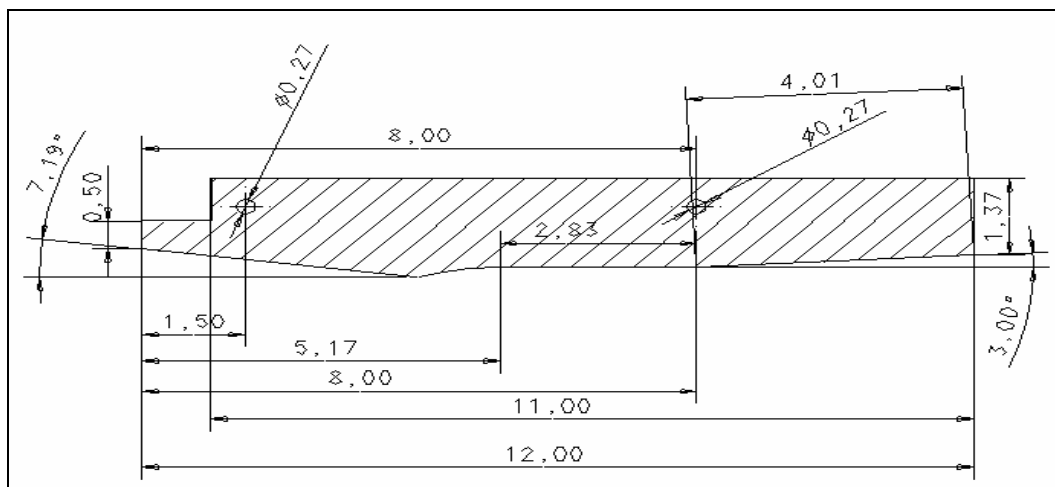


**Figure 9. 3-D isometric view of supersonic duct test setup**

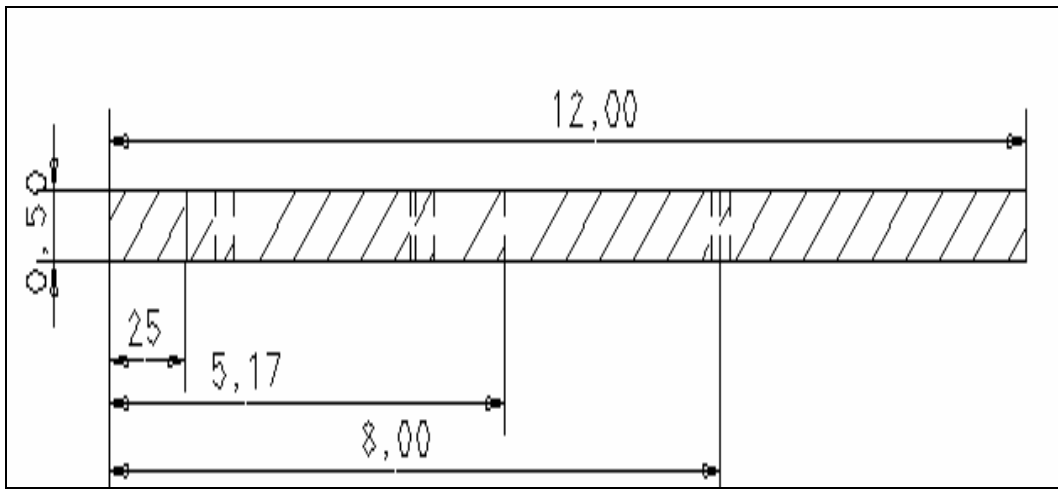


**Figure 10. 3-D side view of supersonic duct test setup**

The other four 211B5 pressure transducers were mounted flush with the internal surface plane of the supersonic duct's lower section. They are used to detect pressure oscillations propagating upstream. The location of the pressure transducers were established based on the height of the constant area region. The design of the supersonic duct was created using IDEAS-9, which is a 3D modeling computer aided design (CAD) software program. Typical three dimensional model views such as the isometric and side view are shown in Figures 9 and 10, respectively. In focusing upon the critical components of this apparatus, a detailed view of the key dimensions for the upper and lower components are shown in Figures 11 and 12, respectively. In Figure 11a, the side view of upper component of the supersonic duct is shown with the very noticeable convergent-divergent section of the duct.

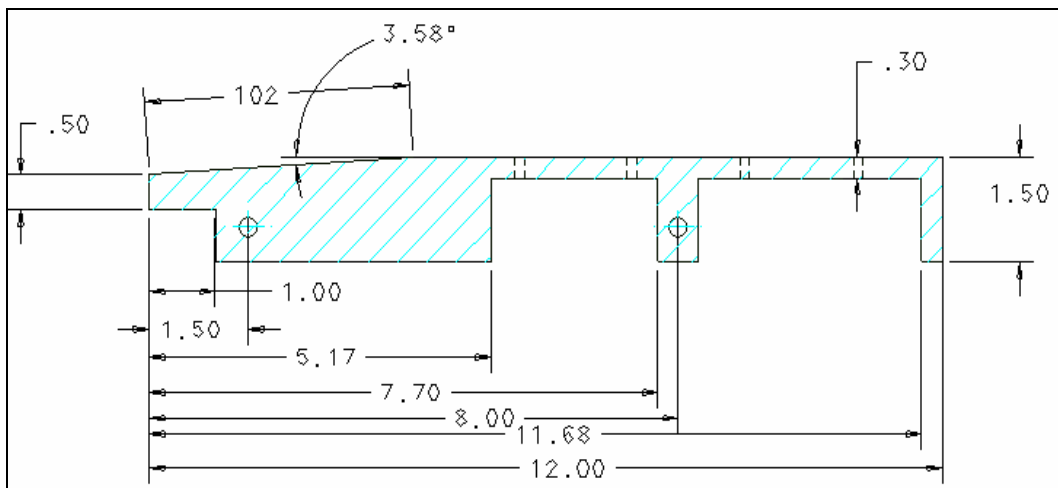


**a)**



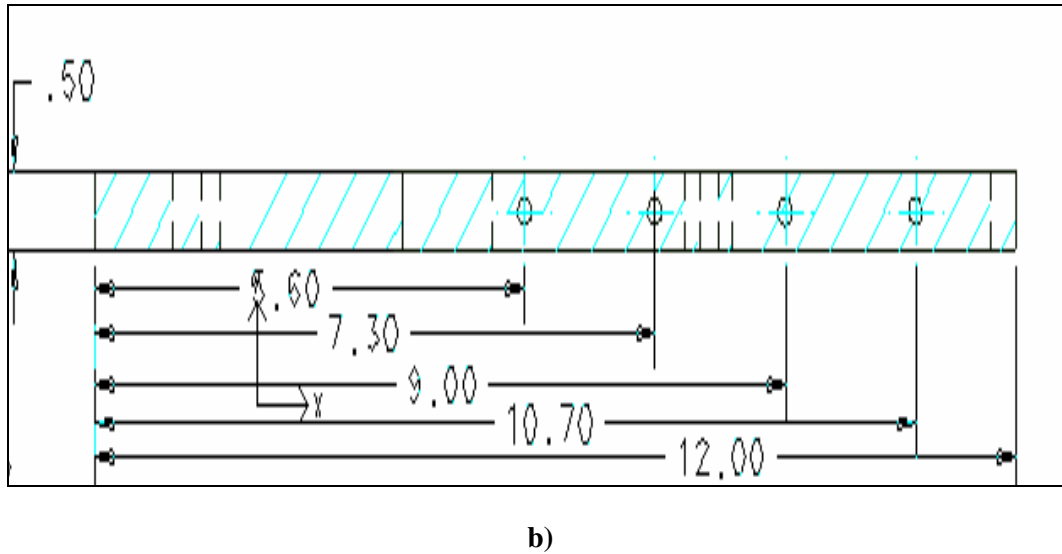
b)

**Figure 11. Schematic illustration of upper component of supersonic flow duct a) side view and b) top view with all dimensions in inches**



a)

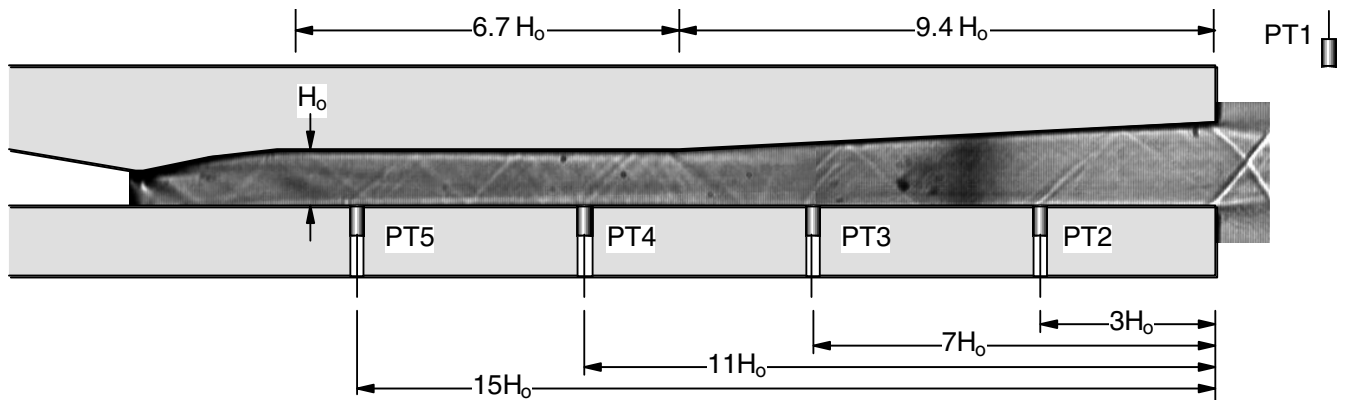




**Figure 12. Schematic illustration of lower component of supersonic flow duct a) side view and b) top view**

The supersonic flow exiting the combustor section is discharged into a 4-in (101.6mm) diameter exhaust pipe after passing through a series of mounting brackets. This arrangement for supersonic jet exhaust flow resulted in strong acoustic resonance of the discharged jet under certain conditions, creating high-amplitude screech tone noise. The mechanism generating screech noise appeared to be related to hole-tone resonance in supersonic jets. While the study of screech mechanism is not in the scope of current investigation, this phenomenon allowed us to investigate the transmission and attenuation characteristics of resulting pressure oscillations through the supersonic duct upstream.

### 3.2.2. Acoustic Wave Pressure Measurements



**Figure 13. Schematic drawing of 2-D view of convergent-divergent supersonic duct depicting pressure transducer location relative to duct height**

The notation PT1 in Figure 13 denotes the location of the acoustic source designated location point. A typical schlieren image was imbedded inside the schematic drawing of c-d supersonic duct to illustrate the dynamics of the flowfield taking place at each pressure transducer. The first 6.7 duct height ( $6.7H_o$ ) downstream of the convergent-divergent nozzle corresponds to isolator section, and the remaining 9.4 duct height ( $9.4H_o$ ) is analogous to the expanding combustor section. The “combustor” flow then discharges into ambient air and to the exposed exhaust duct. Again, the test section is equipped with quartz windows where the internal flow can be visualized using schlieren or shadowgraph flow visualization techniques. As shown in Figure 13, the transducers were placed at a distance of  $3H_o$ (PT2),  $7H_o$ (PT3),  $11H_o$ (PT4), and  $15H_o$ (PT5), where  $H_o=0.425\text{in}$ , away from duct exit. Power and signal conditioning for the pressure transducers were provided by the four channel Kistler 5134A Power Supply and Signal Conditioner, which essentially serves as an interface between the piezoelectric pressure transducers and the analyzing instrumentation.

### 3.3 Flow Visualization Methods

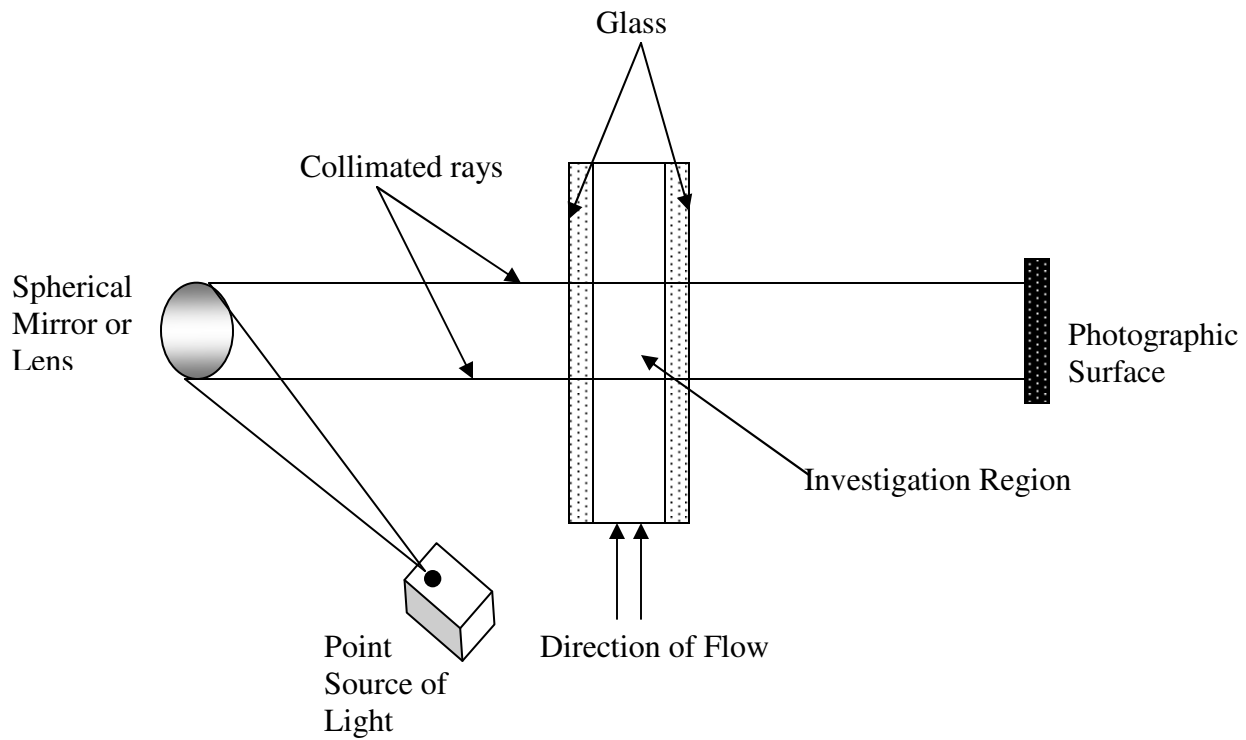
Qualitative investigation of the internal flowfield as well as the acoustic phenomenon at the duct exit was carried out by using a well known flow visualization technique. Optical techniques are very common experimental tools for investigating the intricate details of fluid flow. In particular, highly compressible fluid flows are in a sense ideal for some of these methods due to the fundamental physical principles involve between the gas flow and the optics. Typically, the three most common methods in flow visualization for compressible gas flow are the Mach-Zehnder(MZ) interferometer method, the schlieren method, and the shadowgraph.<sup>25</sup> In principle, each one of those methods, afore mentioned, is tied to the physical phenomenon of the speed of light and its dependency on the index of refraction of the fluid medium as it propagates through. In essence, the density gradient affects how light bends in the fluid medium because of the index of refraction. The larger the density gradient of the fluid the larger index of refraction of the medium, hence creating favorable conditions for flow visualization by optical means. The interferometer method differs from both the shadowgraph and schlieren techniques with its ability to render reliable quantitative measurements of the density of flowfield.<sup>25</sup>

Other critical difference for the interferometer are its inherent difficulty to operate and being very costly.

The repeating dark and light bands also known as fringes are the distinguishing factors for the interferometer method in comparison to the shadowgraph and schlieren flow visualization methods. Basically, one is able to extract the density data based upon these color striations where the darker bands signify higher density and less density for lighter bands.<sup>25</sup>

### 3.3.1. Shadowgraph Method

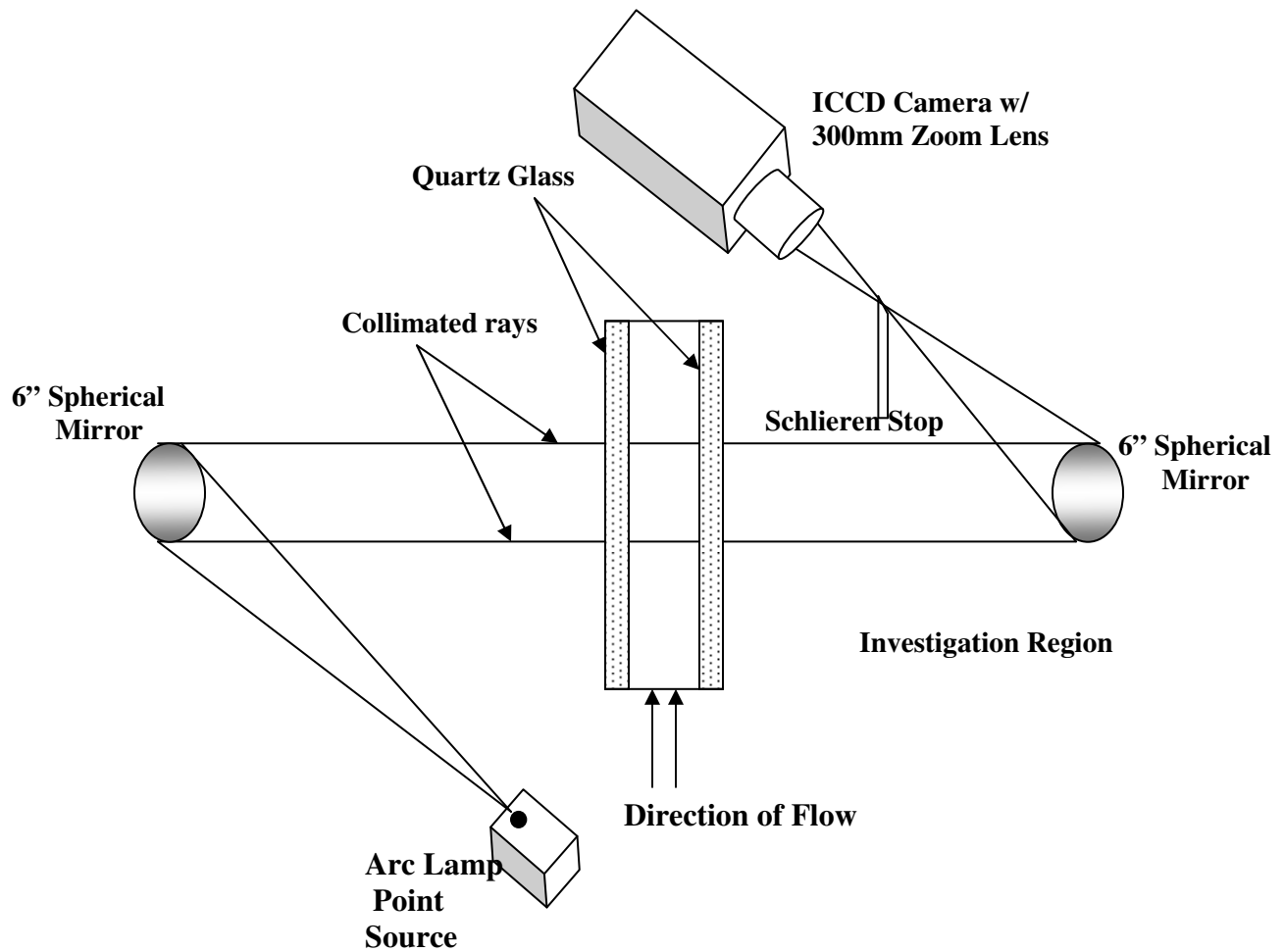
Of the three flow visualization that focuses on the density change in the flow, the shadowgraph is the simplest to setup but yields the least amount of concrete information.<sup>25</sup> This optical technique focuses on the how light is refracted as it travels through the compressible flow field. This method is only effective if there is a gradient of the density gradient in the flow. This can be interpreted as the changes in the brightness pattern on the photographic surface is direct function of the 2<sup>nd</sup> derivative of the density gradient.<sup>25</sup> This basically means that with a constant deflection of light in a constant density gradient flow field one would observe no change in brightness of the flowfield. Now, if there exist a gradient of the density gradient inside the flowfield very distinct flow structures will become apparent on the viewing surface. As a result of this, the shadowgraph has been a very effective flow visualization technique for observing shock wave patterns in a highly compressible flowfield.<sup>25</sup> It is well known in fluid dynamics that across a shock wave density changes dramatically hence the change in density will be illuminated instantly with this optical method.<sup>24</sup> The general observations are bright lines that follow dark lines wherever shocks are located, or any rapidly changing region of density gradient.<sup>25</sup> In Figure 14, a typical shadow graph setup is shown where light is projected from a point source then on to a spherical mirror or through a lens. The light will be collimated, or made parallel, as it passes through the lens or reflected from the mirror. The collimated rays then travel through the investigation region, where transparent material such as polished quartz glass serves as boundary surface for the confinement fluid medium and transmission passage for the collimated light rays on to some type of photographic surface.<sup>25</sup>



**Figure 14. Schematic drawing of shadowgraph flow visualization method**

### 3.3.2. Schlieren Method

For this experiment, the schlieren method was extensively employed to characterize the internal flowfield of the supersonic duct. In comparison to the MZ-interferometer method, one is able to get, in theory, quantitative density gradient data but the accuracy is known to be far less reliable than the values obtained using MZ-interferometer technique.<sup>25</sup> The images of the flowfield especially the turbulent structure, vortex structures and shock waves are more profound using the schlieren method than the MZ-interferometer principle. Unlike the shadowgraph technique, the shock waves that observed in the schlieren method are not accompanied by dark and bright lines.<sup>25</sup> The basic setup of the shadow graph is very similar to the schlieren setup, but typically two additional critical components are needed for the schlieren setup.



**Figure 15. Schematic drawing of schlieren flow visualization method**

In comparing the shadowgraph and schlieren methods setup in Figures 14 and 15, respectively, one is able to observe an additional spherical mirror, or lens and a schlieren stop, which is typically done with a knife-edge. The purpose of the knife edge is to allow a portion of the light from the image at the focal point to be uniformly transmitted on to the photographic device. The fundamental principle behind this method is that light rays perpendicular to the density gradient will have an angular deflection rendering the flowfield in shades of gray.<sup>25</sup> In this experiment the high speed schlieren method employed a Spectral Physics arc lamp as the light source at one of ends of the conventional “Z” arrangement schlieren technique. The light is then projected on to a 6-inches (152.4mm) diameter spherical mirror with a focal length of 60-inches (1.524m).

The collimated light rays travel through the test section bounded by two highly polished quartz glass and then on the second spherical mirror, which has identical physical parameters as the first spherical mirror. From the second spherical mirror the collimated rays are brought to a focal point where a schlieren stop was intentionally placed.

A radial schlieren stop was used in order to cut the image at all angles, which rendered a more uniform distribution of light but at the same time capturing the density gradient in the flow. Two types were used to capture the images of the flowfield in this experiment. The reason for this arrangement was in part due to the significant gain in details of the flowfield, which will be evident in the results. The initial camera used was the Photron Ultima 1024 CMOS camera. This camera operates with an acquisition rate of 60-16000 frames per seconds (fps) and shutter gate speeds ranging from 0.016 to  $7.8 \times 10^{-5}$  seconds. In this experiment the Photron camera was operated typically 250 fps and with shutter speeds of 1/32000 seconds. Images rendered by this camera can be seen in Figures 38 and 39. The second camera that was used during this experiment for schlieren flow visualization technique was a DiCam Pro camera. The images passing through the radial schlieren stop was projected on to this intensified charged couple device (ICCD) camera. This specialized CCD camera was able to significantly capture the flowfield structures in an apparent freeze frame mode due to the high shutter speed capabilities. Live images of the flowfield were seen on a desktop monitor and then recorded frame-by-frame using the DiCam software.

### 3.4 Experimental Methodology

#### 3.4.1. Experimental Approach

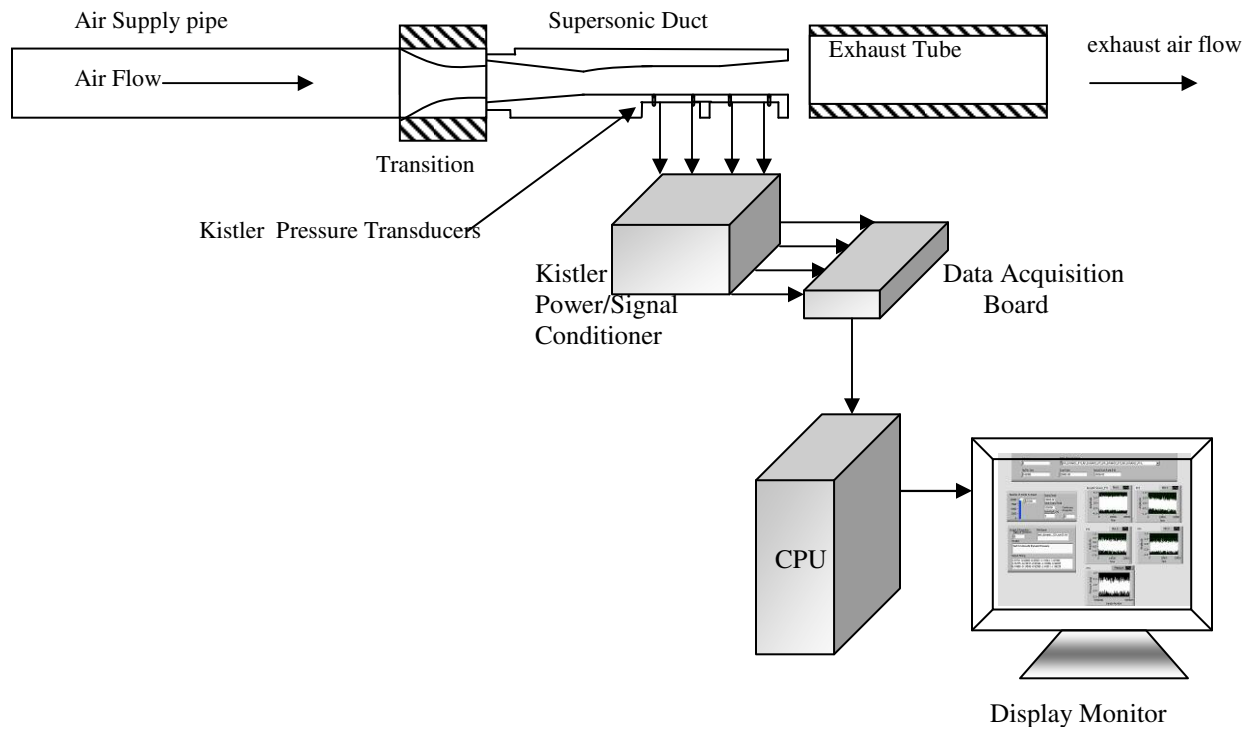
Initial characterization of the distinctive acoustic signal emanating from the exit plane of the supersonic duct was produced using a Tecktronix TDS 3014 digital phosphor oscilloscope. It was conjectured that the fundamental frequency would be below the 20kHz threshold of the hearing capability of the average person. With that in mind, it was decided that sampling should be done at a sampling rate that was twice the signal

frequency in order to not only capture the signal but also reduces the chance of aliasing. Aliasing is well known as a means of distorting the real signal during the sampling process. Sampling was taken at 250kHz and the Niquist criterion was effectively executed via the oscilloscope were a complete frequency bandwidth of 125kHz was captured on the monitor of the oscilloscope. The fluctuation of the signal was averaged using the averaging mode on the oscilloscope. Essentially this method was designed to reduce the Gaussian noise in the signal such that the distinct signal of interests can be captured. It is executed in sequential manner where the current signal is compared to the previous such that abnormalities are discarded as noise and the consistent dominant signal was rendered.

Once the basic idea of what type of frequency range was expected to be created by the designated test conditions, a more accurate approach to the data acquisition was carried using Labview. The data acquisition procedure was carried out with the use of National Instrument (NI) Labview version 6 software. The dynamic and static pressures were taken simultaneously for 10 consecutive test runs. The flow of data from the Kistler Piezotron<sup>TM</sup> pressure transducers (Model 211B5) inside the duct is illustrated in Figure 15. The pressure transducers were secured in a flush mounted adapter with a sensor port of approximately 0.1in (2.54mm) diameter. The pressure transducers are miniature in size, very sensitive, low impedance rating, and high frequency response. The 211B5 is pressure rated for up to 100psi with frequency range up to 50kHz. The signal from the pressure transducer is transmitted to the Kistler Piezotron<sup>TM</sup> Coupler (Model 5134A). This four channel input/output piezoelectric sensor power and signal conditioning unit is a robust unit that provides the necessary excitation and signal processing to each dynamic pressure transducer. The max gain for the Piezotron<sup>TM</sup> Coupler is 100 with a bandwidth of 0.036Hz to 10kHz. In this experiment, the gain was set to one with a higher frequency bandwidth of 30kHz. Cutoff frequency was set to 30kHz for the system imbedded low pass filter.

For static pressure measurement the raw signal was transmitted from the Setra pressure transducer (Model 206) directly in to the NI-DAQ board. The Setra (Model 206) series pressure transducers are pressure rated up to 250psig with a response time of 5miliseconds. Pressure values were also displayed on the Datum 2000<sup>TM</sup> dual channel meter. The DAQ board serves as the interface between the Labview software and raw acoustic signal. Each channel was appropriately scaled for their respective pressure transducers. The sensitivity scaling

values for each Kistler dynamic PT as well as each Setra static PT were provided by the manufacture's calibration sheet.



**Figure 16. Schematic diagram of experimental setup of supersonic flow duct in laboratory**

A close up view of the typical Labview front panel for dynamic pressure data acquisition is shown in Figure 16. One can observed the oscillations for given number of data points approximate 200,000. In this experiment a sampling rate of 20kHz for 10seconds each test run. A Fast Fourier Transform (FFT) was applied to obtain the power spectrum of the signal as it propagates upstream. The raw data that was acquired using Labview was loaded into a MATLAB code, which produced the plots showing the dominant frequency with its respective amplitude for all pressure transducers.



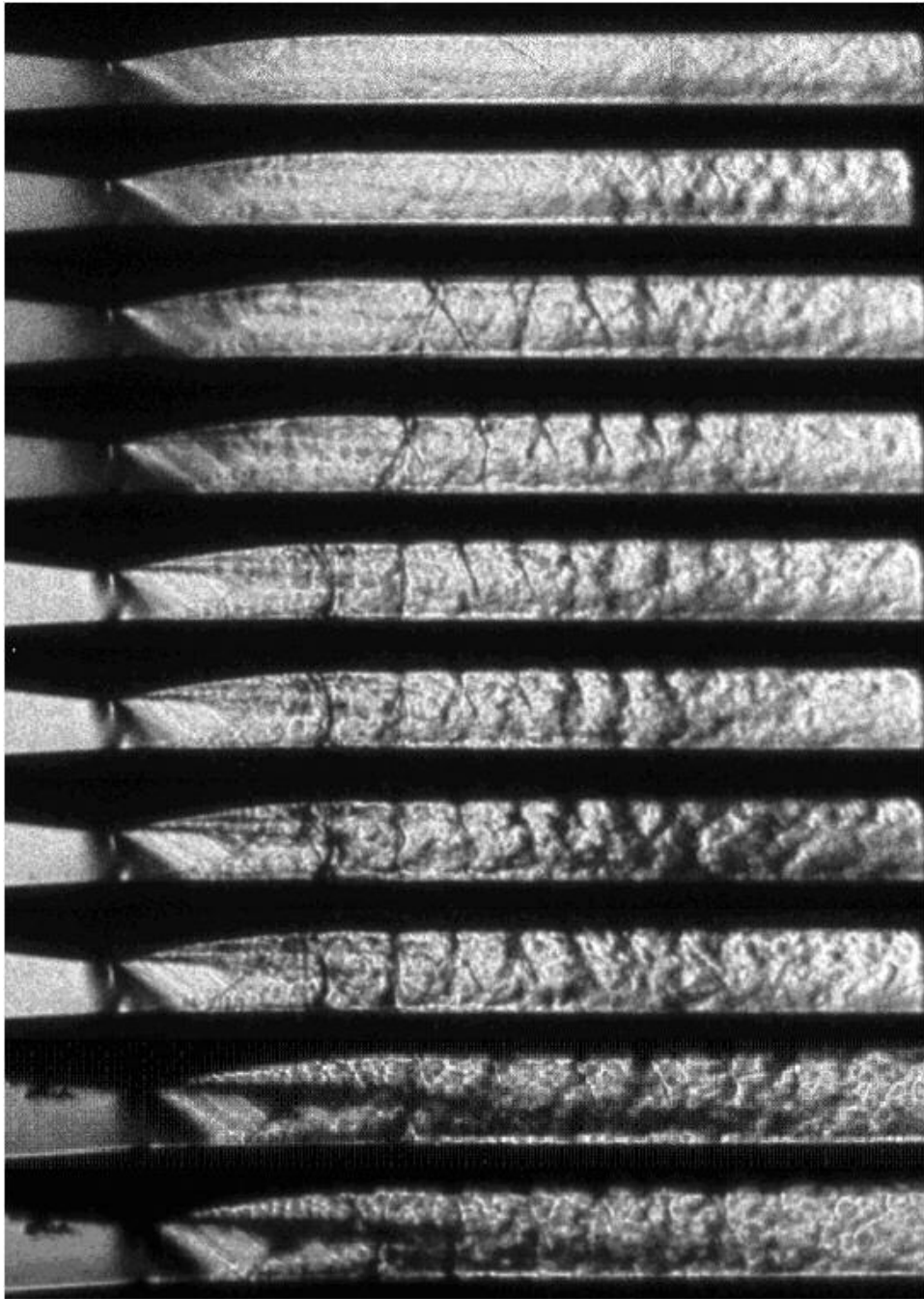
## Chapter 4: Results and Discussion

### 4.1. Internal Flowfield Characterization

#### 4.1.1. Schlieren Images of Prominent Turbulent Flow and Shock Structures

Typical schlieren images of the internal flowfield are shown in Figure 17 for various stagnation pressures using the DiCAM-ICCD Camera. The region of the supersonic duct depicted in Figure 17 is from the inlet to the end of the constant area section, or the isolator. It can be seen from Figure 17 that the detailed turbulent and shock structures of the flowfield become profound as the upstream stagnation pressure increases from 20psig to 110psig. This complicated flowfield can be attributed to the intricate interaction between the boundary layer and the shock wave taking place along the surface of the duct.<sup>25</sup> When the stagnation pressure was relatively low, the nozzle boundary layer thickness remained small and the oblique shock train was observed inside the isolator duct. At higher stagnation pressure conditions, the shock train appeared more normal than oblique due to thicker boundary layer associated with turbulent boundary flow at the nozzle exit. In essence, this is a result of the development of large adverse pressure gradient due to the normal shock train in the central core of the stream.<sup>25</sup> A backflow in the boundary layer created separation of the stream from the duct walls. These normal shock trains decelerate the core flow from supersonic to subsonic speeds, while increasing the static pressure across the shocks. In this experimental setup, the subsonic flow was reaccelerated at the end of the isolator as the upper wall expanded at a constant angle of  $3^\circ$ .

Normal shock trains, where there are a series of normal shocks in supersonic flow, are typical examples of the complex flowfield structures that frequently occur in devices such as supersonic wind-tunnels, inlets of scramjets, as well as supersonic ejectors.<sup>35</sup> At high speeds ( $M > 1$ ), these “multiple shock waves” interact with the turbulent boundary layer (as seen in Figure 17). This interaction, according to Carroll et al.<sup>25</sup> has the potential of producing a non-uniform and unsteady flow conditions downstream of the interaction.



**Figure 17. Schlieren images showing the flowfield inside the nozzle, isolator for stagnation pressures ranging from 35psi to 125psi.**

Flow conditions are affected or dominated by the undisturbed Mach number ( $M_u$ ), Reynolds number ( $Re$ ), and the flow confinement.<sup>35</sup> Similar findings were reported by Pei Lin et al.<sup>36</sup>, who suggested that the boundary layer momentum thickness ( $\theta$ ) is a key parameter that characterizes flow confinement.

The flowfield in the expansion duct, which simulates a supersonic combustor geometry was visualized as well as the free jet exiting from the duct. Figure 18 shows these images taken at various stagnation pressures. The free jet exited into an exhaust duct, which under certain conditions interacted with the exiting jet producing resonant oscillations. This allowed us to view the oscillating shock trains in a seemingly fixed position. The oscillations in what can be considered the isolator section of the supersonic duct have created possibly additional disturbances that were detected by the pressure transducers, in particular PT4 and PT5. When the upstream stagnation pressure was increased from 20psig to 60psig detailed diamond shock structures were observed propagating downstream of the duct exit, see Figure 18. With continued focus on the combustor and duct exit region, it was keenly observed that for upstream flow conditions with stagnation pressures between 70psig and 110psig produced dramatic changes at duct exit. Among the several possible observations the most prominent are the development of expansion waves from the corner of the duct exit as well as the shock wave and expansion fan reflections from the free jet shear layer, which forms a constant pressure boundary. For this experiment, flow conditions at 80psig, 90psig, 110psig were of interest due to the fact that the screech tones observed at these upstream conditions were distinct, detectable, and relatively consistent in the data acquisition process. Oscillations at the jet exit were observed for stagnation pressures between 80psig and 90psig.



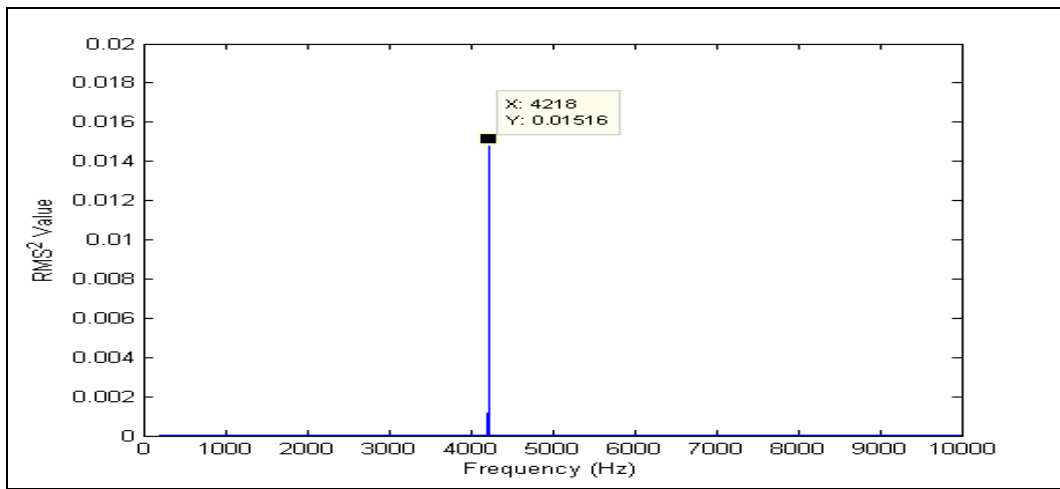
**Figure 18. Schlieren images showing the flowfield inside the expansion region of the duct for stagnation pressures ranging from 35psi to 125psi**

The prominent growth of the shear layers streaming from the duct exit, illustrated Figure 18, for stagnation pressures between 80psig and 100psig are also of interests. At this pressure range, as previously mentioned, the occurrences of screech tones were observed. This seems to support Powell's<sup>22</sup> theory of the interaction between the shear layer and shock cells to create the distinct acoustic phenomena.

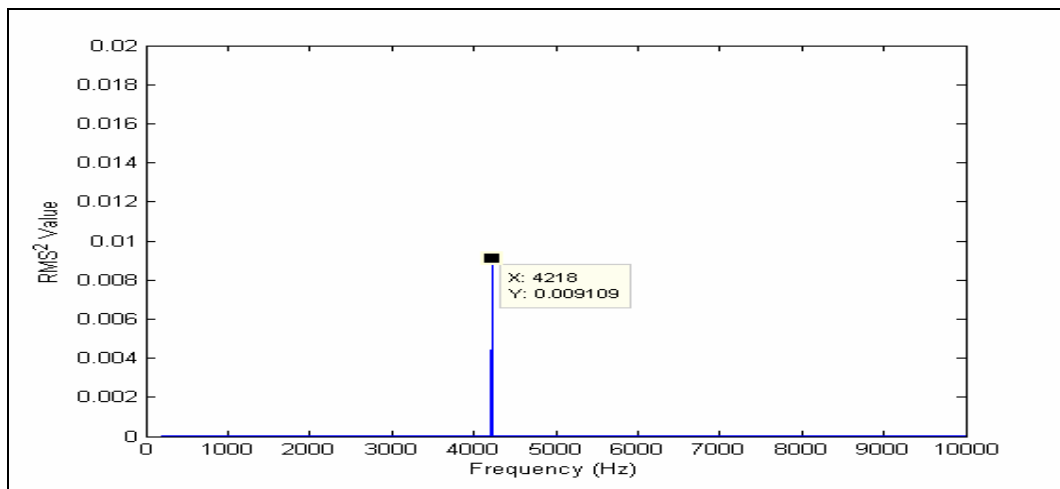
## 4.2. Acoustic Wave Amplitude Characterization

### 4.2.1 Acoustic Signature Characterization

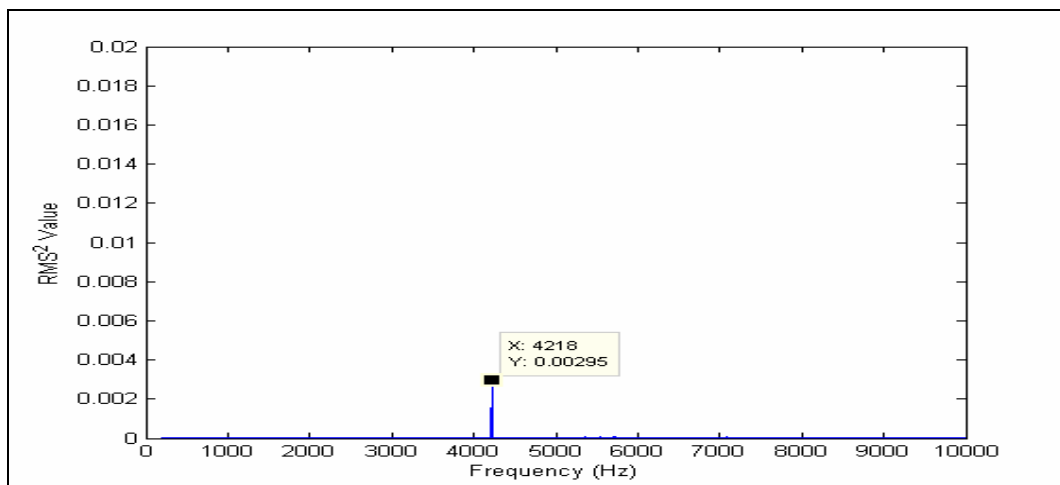
In Figures 19-22, the acoustic signatures of the supersonic jet stream for four upstream pressure conditions of interest are illustrated. The distinct acoustic signals were established by the passive acoustic excitation mechanism at the following pressures: 95psi, 105psi, 115psi, and 125psi. These absolute stagnation values were chosen because of the simple fact that at these particular stagnation pressure values (+/- 3psi) fundamental frequencies were captured repeatedly over several characterization experimental test runs. The power spectrum plots illustrated in Figures 19-22 are typical representations of the numerous spectral analysis results that were obtained for this investigation, which can be seen in Appendix section. From these figures, one is able to observe how the acoustic wave pressure amplitude, measured in RMS (root mean square) squared, diminishes as it travels through the supersonic ducted medium. With increasing upstream stagnation pressure, the frequency of the dominant acoustic signal decreased from 4.2kHz at 95psi to 2.8kHz at 125psi. It was also observed that even though there was a significant decrease in frequency going from 95psi to 125psi there was little or no change in dominant frequency values at 105psi, 115psi, and 125psi. In Figures 19-22, each spectral analysis plot starts off with power spectrum the first pressure transducer, PT2 (a) that the acoustic waves encounter in the in duct. With the subsequent plots representing PT3 (b), PT4(c), PT5 (d) as the wave travels upstream.



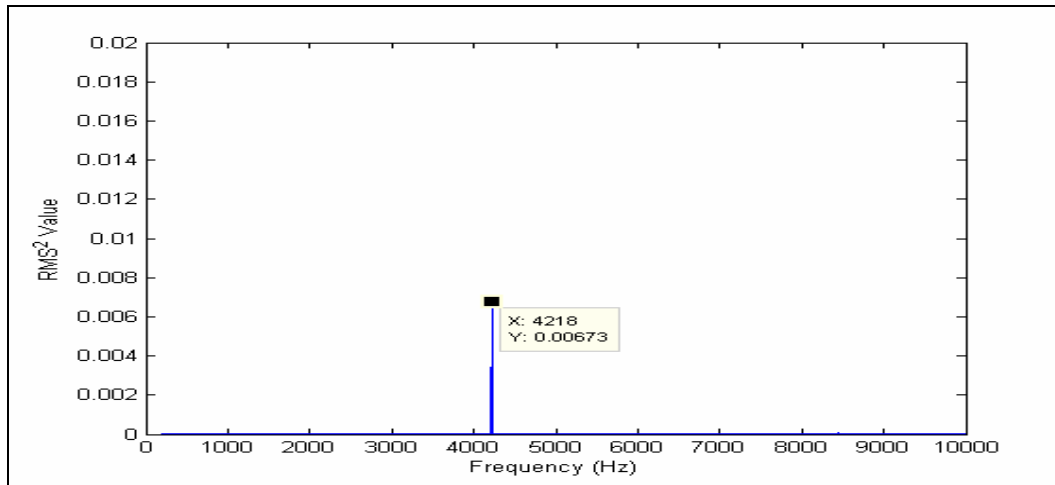
a)



b)



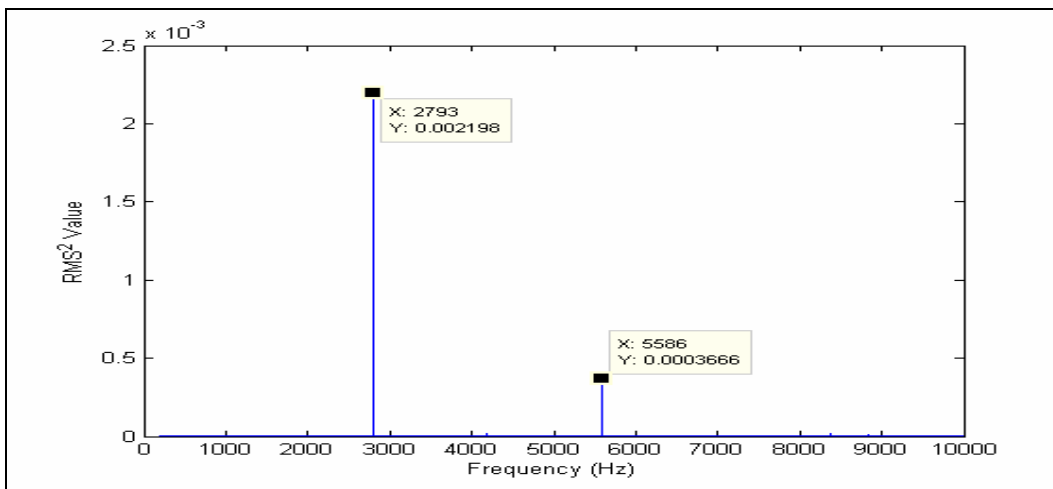
c)



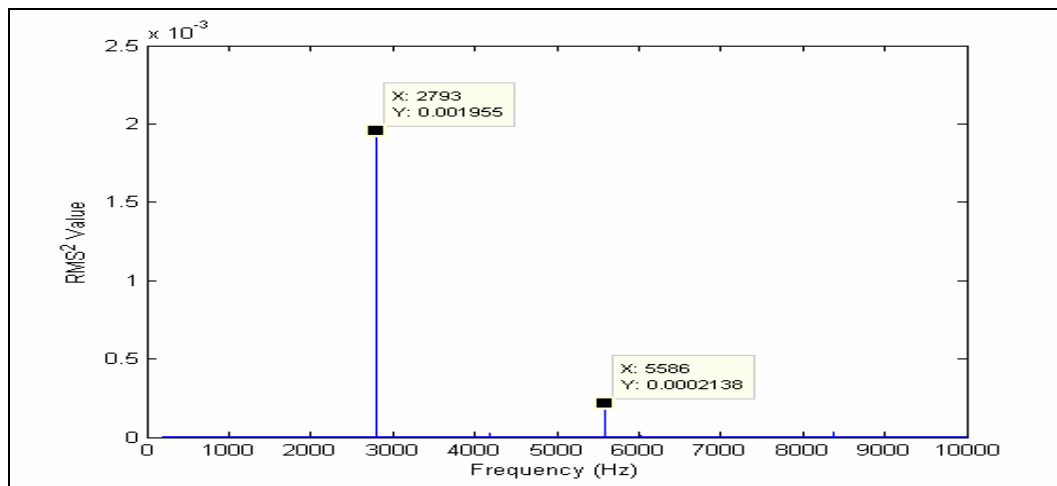
d)

**Figure 19. Power Spectrum showing the magnitude of acoustic wave at a 95psi upstream stagnation pressure with a dominant frequency of approximately 4.22kHz for a)PT2, b)PT3, c)PT4,and ,d) PT5 respectively**

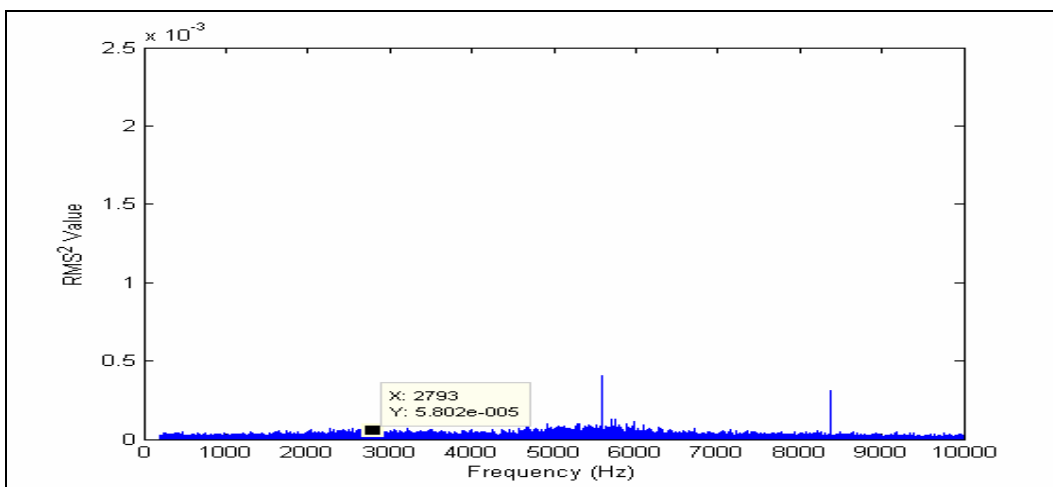
At times, it was observed that higher harmonics were also being captured by the high frequency response pressure transducers, as shown in the power spectrum plots in Figures 20-22. Frequently, the higher harmonics would be the dominant frequency for PT4 and PT5, which are located in the constant area section of the duct, which simulates the isolator section of the scramjet engine. The original amplitude of the dominant frequency from the acoustic source, which was initially tracked by PT2, would be virtually negligible in comparison to amplitude values for PT2 and PT3. This would suggest that some type of acoustic dissipative effects had occurred as the wave propagated upstream. This idea of acoustic wave dissipation or attenuation will become more evident in the subsequent sections of this chapter.



a)

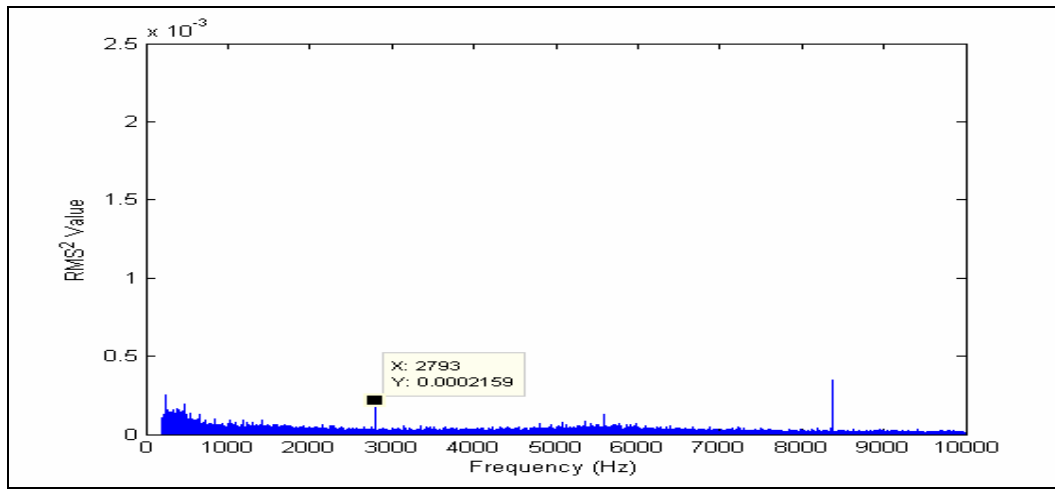


b)



c)

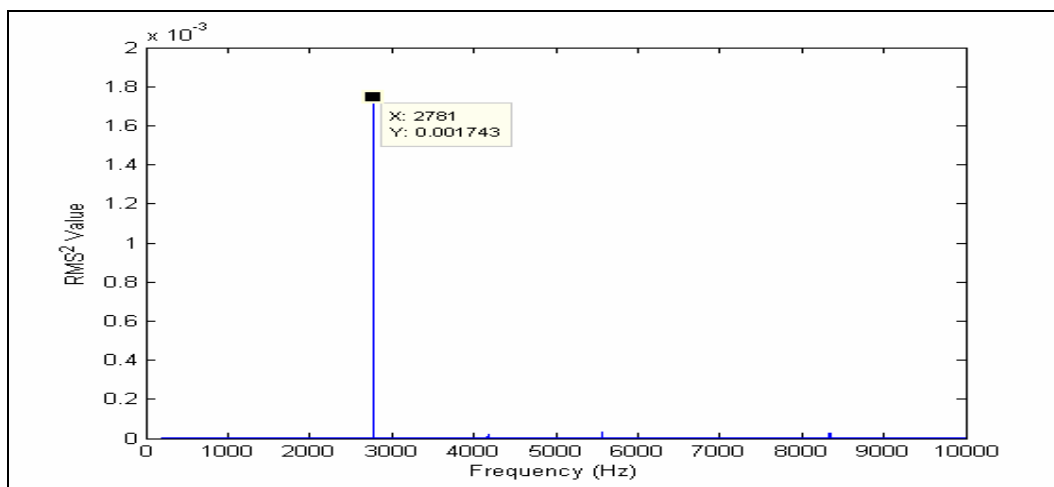




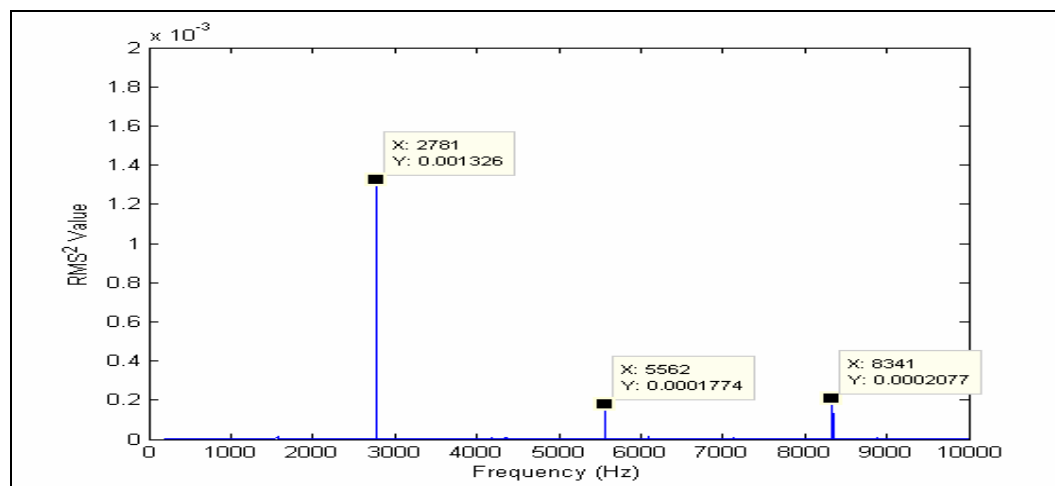
d)

**Figure 20. Power Spectrum showing the magnitude of acoustic wave at a 105psi upstream stagnation pressure with a dominant frequency of approximately 2.79kHz for a)PT2, b)PT3, c)PT4,and ,d) PT5 respectively**

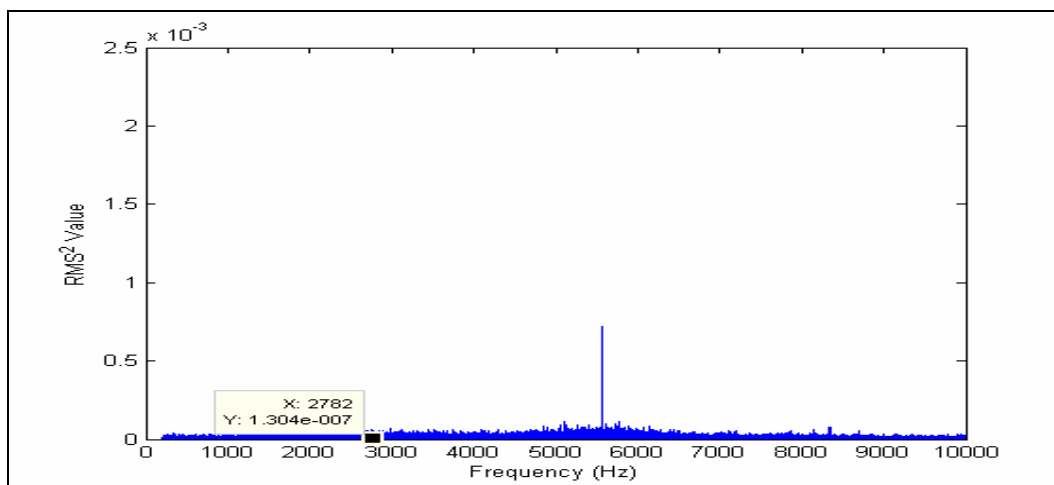
Continuing with the observations of these power spectrum plots, a closer look at the analysis for pressure transducer locations PT4(c) and PT5(d) for Figures 20-22 would suggest some interestingly acute behavior of the flowfield's affect on the acoustic wave or vice versa. One may observe the unique conditions that exist in the power spectrum plots for both PT4 and PT5 in comparison to the acoustic source, PT2, and PT3 for most of the upstream pressure conditions under investigation. At these locations, the power spectrum illustrates an increase in signal intensity, which may be attributed to the oscillations of the normal shock train in the isolator region. To clarify this point, let's take a look at Figure 21, as an example. First, one observes the general trend of decreasing amplitude of the acoustic signal from PT2 (a)-PT5(d). Even though there is an increase in amplitude for PT5, as opposed to PT4, the relative amplitude in comparison to original encounter at PT2 ( $0.0017 \text{ RMS}^2$ ) and PT5 ( $7.487 \times 10^{-5} \text{ RMS}^2$ ) has decreased significantly.



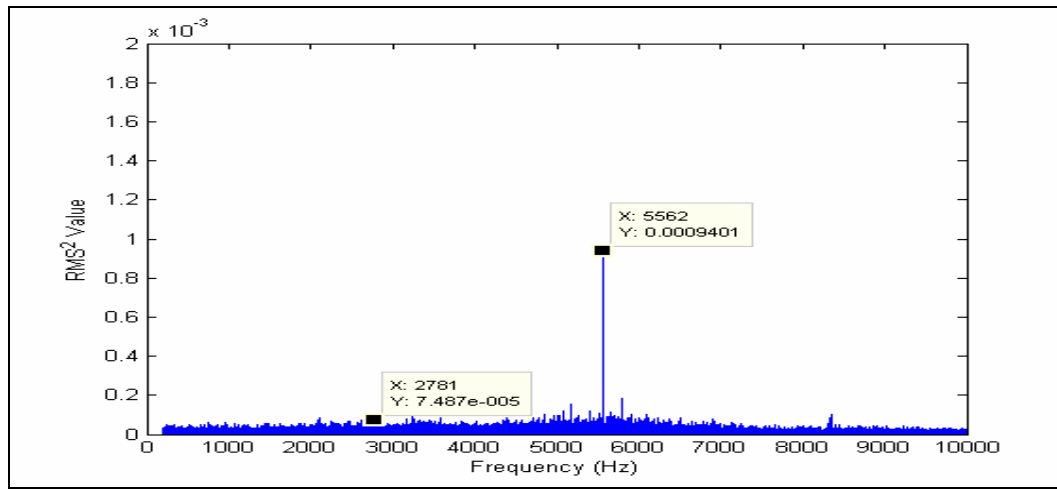
a)



b)



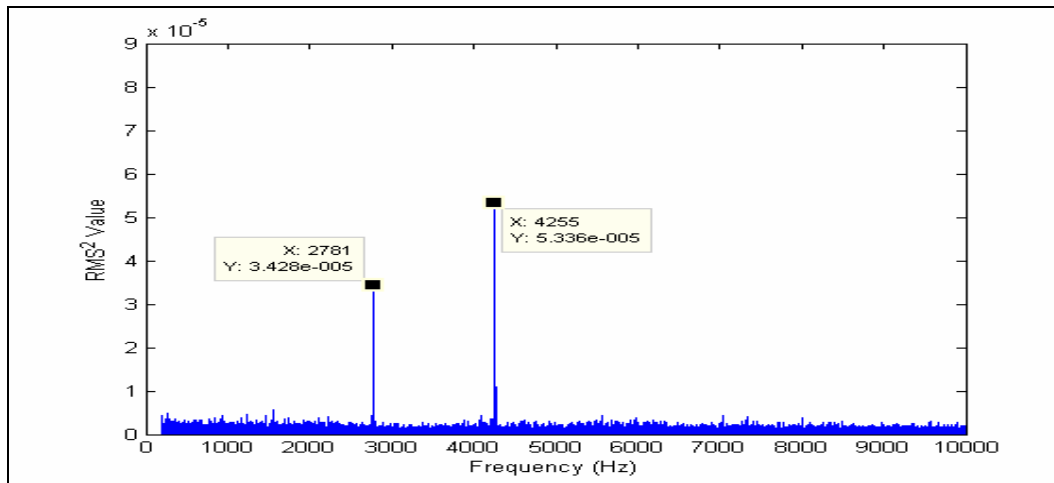
c)



d)

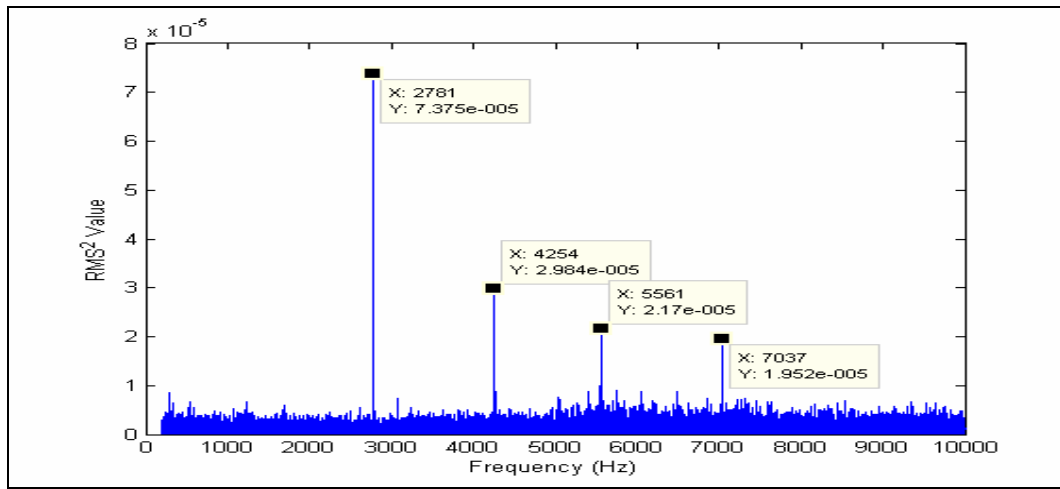
**Figure 21. Power Spectrum showing the magnitude of acoustic wave at a 115psi upstream stagnation pressure with a dominant frequency of approximately 2.78kHz for a)PT2, b)PT3, c)PT4,and ,d) PT5 respectively.**

This amount of relative dissipation will be addressed in the next section. Second, both PT4 and PT5 have a dominant frequency at higher harmonics, approximately 5.6kHz, and what may be described as amplified erratic signals. With this in mind, one can look back at the corresponding internal flowfield schlieren images in Figure 17 and see the complex dynamics within this region of inspection. The flowfield becomes extremely turbulent as the upstream stagnation pressure increases from 35psi to 125psi. For the example above, the schlieren images at 115psi hints at the complexity of the flowfield, which become numerically evident in the spectral analysis. Comparing this to a flowfield at 95psi, even though it is relatively turbulent, it is arguably more turbulent at 115psi..

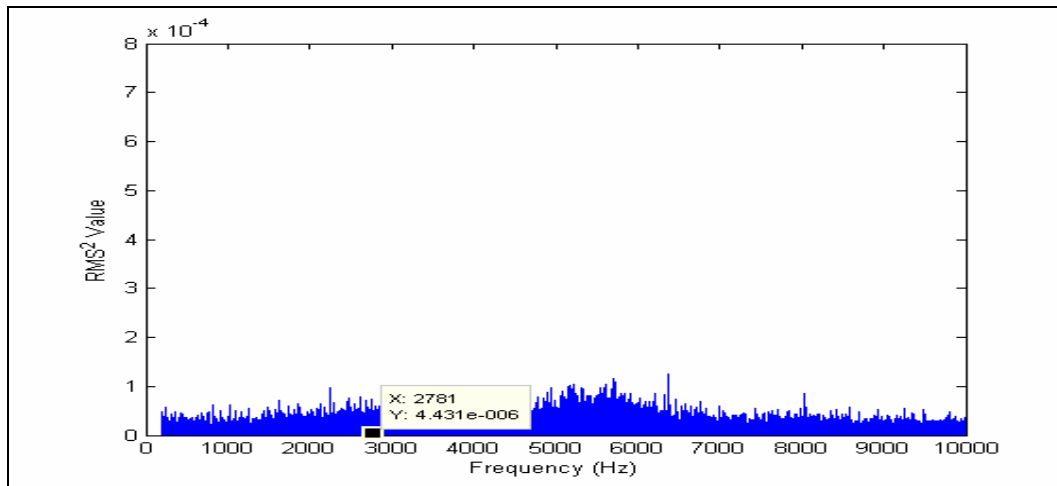


a)

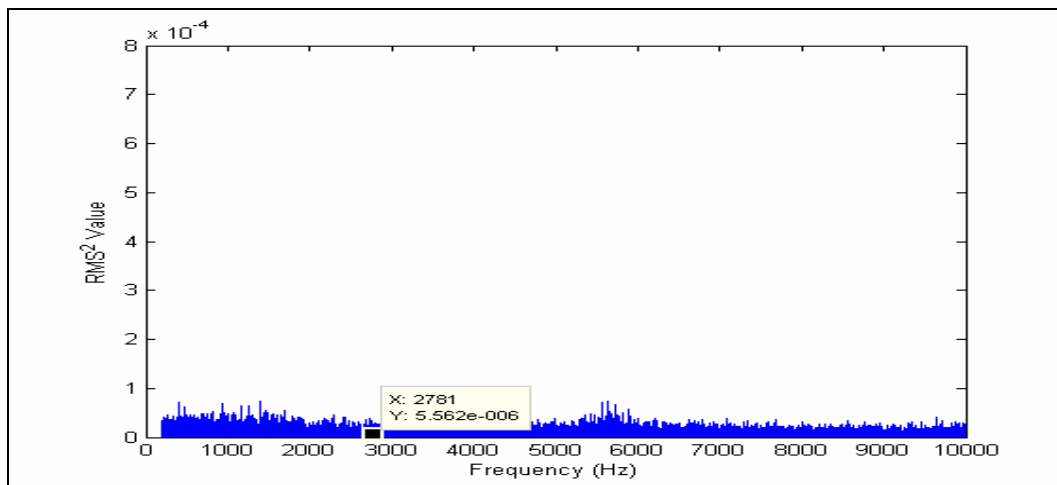
Looking at Figure 19, it can be observed that at PT4 and PT5, the signal intensity is more or less negligible in comparison to the signals at 115 as well as 125psi in Figures 21 and 22, respectively.



b)



c)



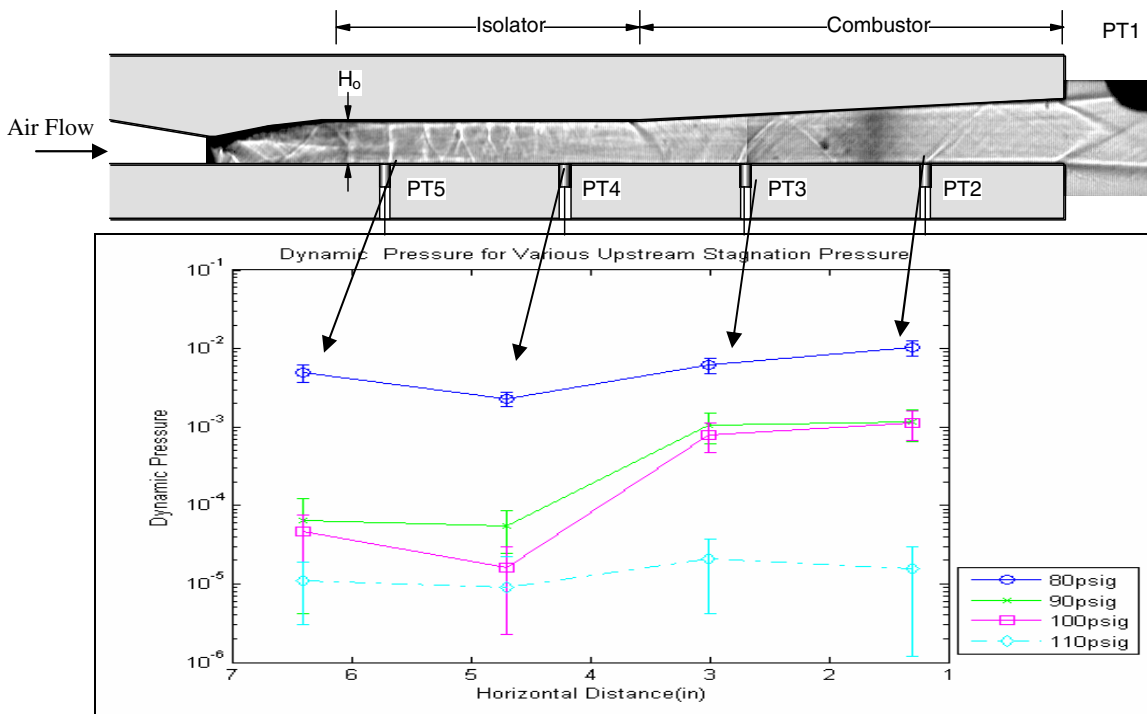
d)

**Figure 22. Power Spectrum showing the magnitude of acoustic wave at a 125psi upstream stagnation pressure with a dominant frequency of approximately 2.78kHz for a)PT2, b)PT3, c)PT4,and ,d) PT5**

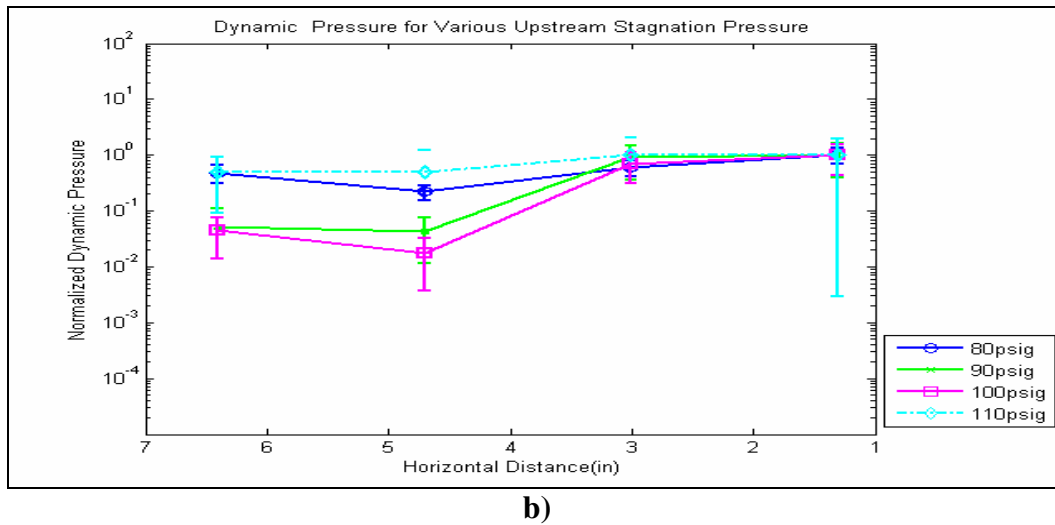
The distinct shock structures coupled with a dominant turbulent flowfield with a mixture of shock/ boundary layer interactions would be strong evidence for this amplified erratic dynamic pressure signals occurring at PT4 and PT5. What seems to be encouraging is that the acoustic intensity will decrease but over what distance of which further investigation and analysis will be needed. These power spectrums are clear indicators that the large amplitude pressure oscillations in combustors of scramjet engines will more than likely propagate through the subsonic region of the supersonic core flow, namely the boundary layer.

#### 4.2.2. Amplitude Attenuation

The general trend of the dissipation of acoustic intensity as the upstream stagnation pressure increases is illustrated in Figure 23. In this figure, the analysis of the dynamic pressure signal was carried out for absolute stagnation pressure values of 95psi, 105psi, 115psi, and 125psi. The data points are based on ten sequential experimental runs for each upstream stagnation pressure conditions stated above.



a)



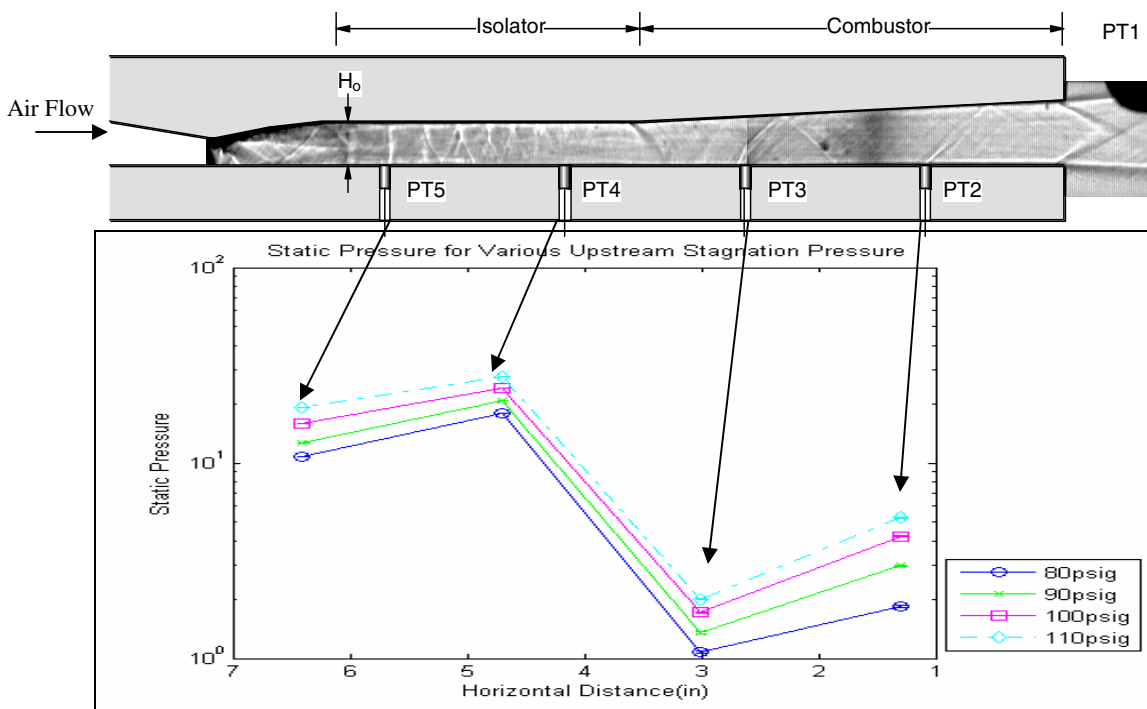
**Figure 23. Illustration of a) averaged and b) normalized dynamic pressure at each pressure transducer location for increasing upstream stagnation pressure conditions**

A schematic diagram illustrating the location of pressure transducer inside the duct relative to the acoustic source at the duct exit and simulated isolator and combustor sections is shown in Figure 23. From the series of runs, the best estimate or the arithmetic mean was calculated at each pressure transducer location. Statistical analysis was applied to the data to get an idea of the standard deviation or the error that was associated to each with each dynamic pressure transducer value, which is displayed with the vertical error bar in Figure 23(a).

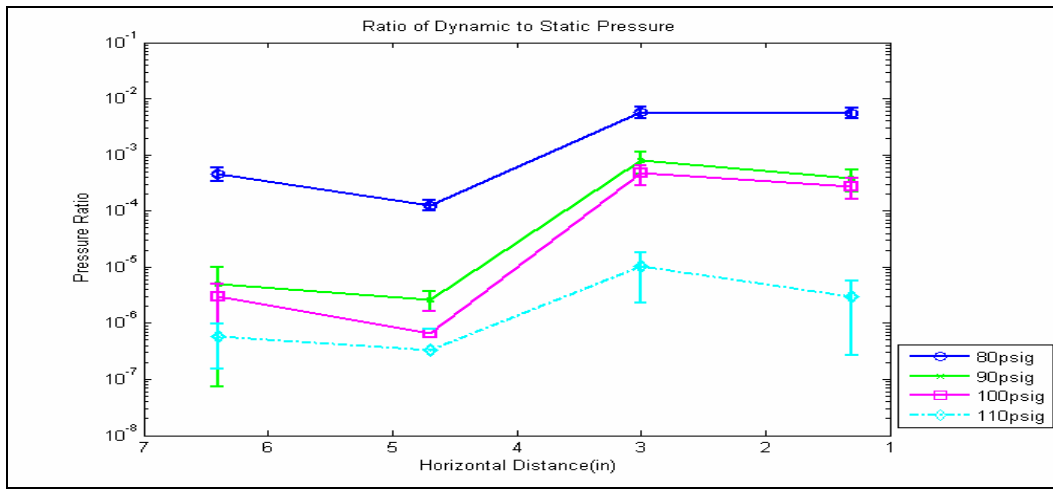
A logarithmic scale on the y-axis was applied in order to illustrate more clearly how the acoustic signal was attenuating with increasing pressure. From this figure, one can observe the decreasing of the amplitude by a couple of orders of magnitude as pressure increases from 95psi (80psig) to 125psi (110psig). In essence, one can look at this plot in terms of microcosmic and macrocosmic viewpoints. For the microcosmic perspective, one is able to observe the decrease in acoustic intensity at each pressure transducer as the signal propagates upstream. This trend seems to hold true even though there maybe slight aberrations found for 105psi (90psig), 115psi (100psig), and 125psi (110psig). In terms of the macrocosmic outlook, the trend is consistent with the fact that as the upstream stagnation pressure increases less acoustic signal intensity had propagated through the boundary layer. This revelation seems to falling directly in line with the intuitive thought process in terms of the decreasing boundary layer thickness and hence smaller area to allow for transmission of the acoustic signal, this concept will be explored in more detail in subsequent sections. In Figure 23(b), the plot illustrates how the signal intensity decreases relative to PT2 location inside the duct.

This location was the chosen point of reference, therefore the all values were normalized with respect to PT2's measured acoustic pressure. If one follows the lines on the graph going from right to left starting at PT2, it can be observed that at each subsequent location after PT2 there is an order of magnitude decrease in original intensity at PT2. The relatively small error gaps at all locations for the respective pressure conditions give a certain level of confidence in the overall trend of the attenuation effects. In essence, the tracking was done such that the fundamental frequency of the acoustic signal was detected at each pressure transducer location and the amplitude of that frequency was observed to be in general smaller than the acoustic signal at PT2. As aforementioned, the general trend is that the signal does dissipate from PT2 to PT5, if we look at the trend from PT4 to PT5 the amplitude generally increases.

A similar graphical layout to Figure 23 is shown in Figure 24 for the static pressure measurements and the ratio of the dynamic pressure to the static pressure measurements. The arithmetic mean of the static pressure was analyzed based on approximately 90 static pressure measurements taken simultaneously at each location. The standard deviations of these values were calculated and plotted with the best estimate for upstream pressure conditions under consideration, see Figure 24.



a)



b)

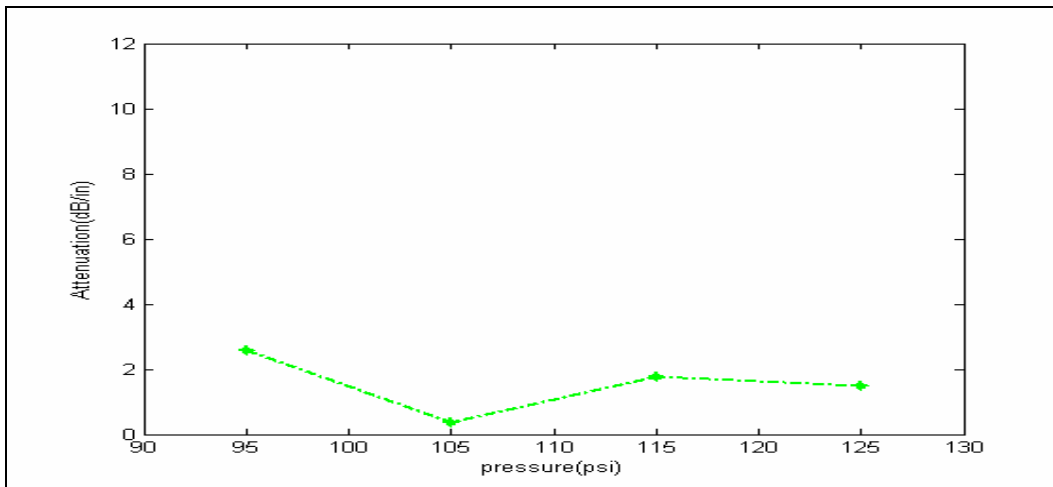
**Figure 24. Illustration of a) static pressure and b) ratio of dynamic pressure to static pressure at each PT location for increasing upstream stagnation pressure conditions**

The error was shown to be negligible, which basically illustrates the consistency of the pressure values taken at those points for a given upstream stagnation pressure. Figure 24(a) was also plotted on a logarithmic scale on the y-axis, while the x-axis was reversed in order illustrate how the static pressure changes at each PT location as the flow goes from left to right. The general trend shows that as upstream stagnation pressure increases the overall static pressure increases. For example, the static pressure value at 95psi (80psig) are approximately two orders of magnitude lower than for the static pressure at each location for upstream total pressure of 125psi(110psig) stagnation pressure, which is expected. What is also reassuring is that as the static pressure increase at a given location for a given location the dynamic pressure decreases. This relationship between the static pressure, dynamic pressure, as well as the total pressure is expected due to the fact that the stagnation upstream pressure is the sum of the dynamic pressure and static pressure. If one travels from left to right starting at PT5 and then to PT4, the pressure increases by means of shocking down the flow to subsonic with the development of normal shocks trains inside the isolator. From PT4 to PT3, the flow accelerates as it approaches the expansion region where pressure transducer three is located. This is captured in Figure 24(a) where a steep drop in static pressure is demonstrated for each upstream pressure conditions. Then finally moving from PT3 to PT2, the flow increases the pressure gradually in for the exit pressure to adjust to the ambient or back pressure.

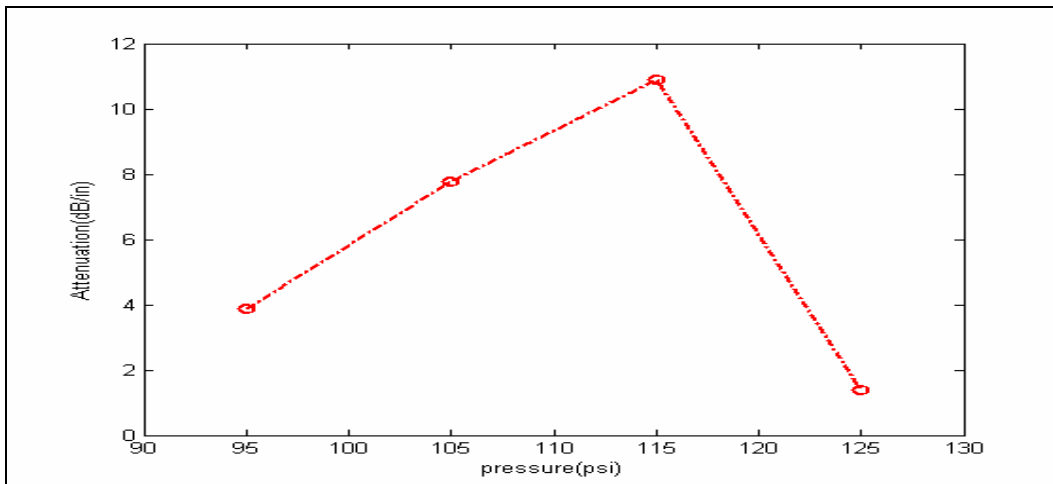


Figure 24(b) illustrates the general trend of changes for the ratio of dynamic pressure to static pressure at each PT as the upstream stagnation pressure increases. To better demonstrate how the ratio is affected along the duct, the y-axis was set to a logarithmic scale and the x-axis values were reversed. This particular plotting arrangement basically shows how the pressure ratio changes with increasing distance away from the acoustic source going from right to left. A two dimensional schematic diagram of the supersonic duct with typical internal flow schlieren image and PT location is provided to further assist in visualizing the process. The results show very similar trends as the dynamic pressure plot, seen in Figure 23(a). Where, the acoustic pressure wave transmission RMS (root mean square) squared amplitude decreasing with increasing upstream pressure. Again, one may conclude that this maybe due to the acoustic dissipation effects, which may be caused by and combination of things including the extremely turbulent flowfield.

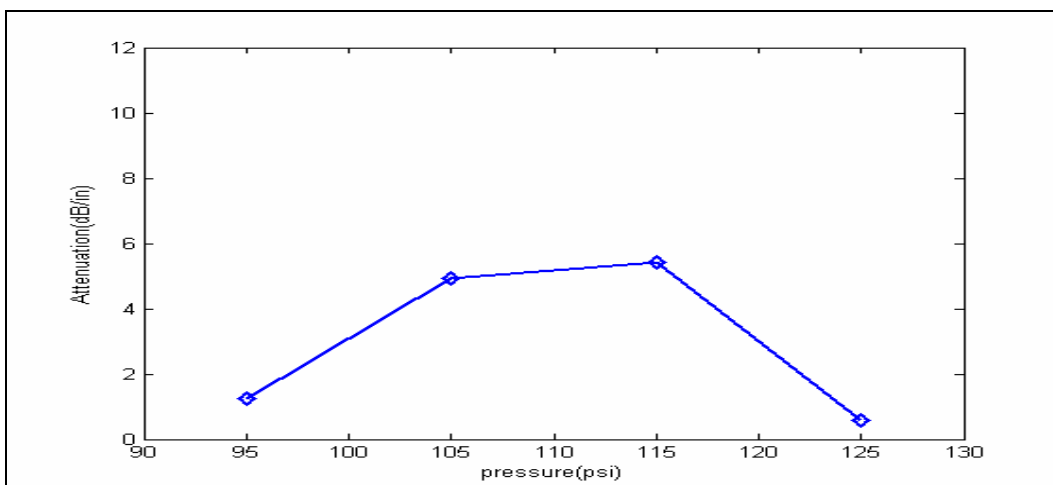
One of the objectives of this research was to characterize how far upstream the signal would propagate. In order to accomplish this goal, an effort was made to characterize the attenuation or the reduction in pressure levels between the reference pressure at PT2 and the other pressure transducers, PT3, PT4, and PT5, inside the duct. The distance between each transducer was set at an equidistance of 1.7 inches(43.18mm) such that PT3 had a factor one times this distance away from PT2, while the furthest at PT5 was set at factor of 3 times as far as PT3. The decibel levels were calculated using the formula,  $dB = 20 \cdot \log_{10}(P_x / P_{ref})$ . The individual values at PT3, PT4, PT5 were set equal to  $P_x$ , while the reference pressure ( $P_r$ ) was set a PT2, see Table 2 & 3. As seen in Figures 25 & 26, the highest amount of attenuation occurred when the signal propagated from PT2 to PT4 with the highest value occurring at 115psi and dissipation rate of approximately 11dB/in. The least amount of attenuation per inch occurred between PT2 and PT3. The complex flow field in the constant area section (or isolator) may be the key attributing factor to this substantial amount of dissipation. The schlieren images as well the power spectrum plots in Figure 19-22, with particular attention at Figure 21, illustrates how the acoustic pressure at PT4 is virtually negligible in comparison to the RMS value at PT2.



a)



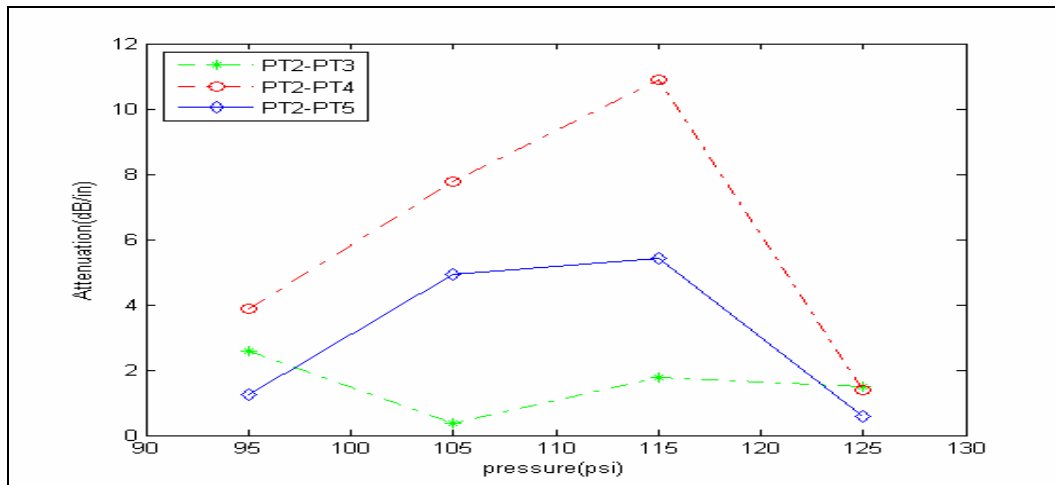
b)



c)

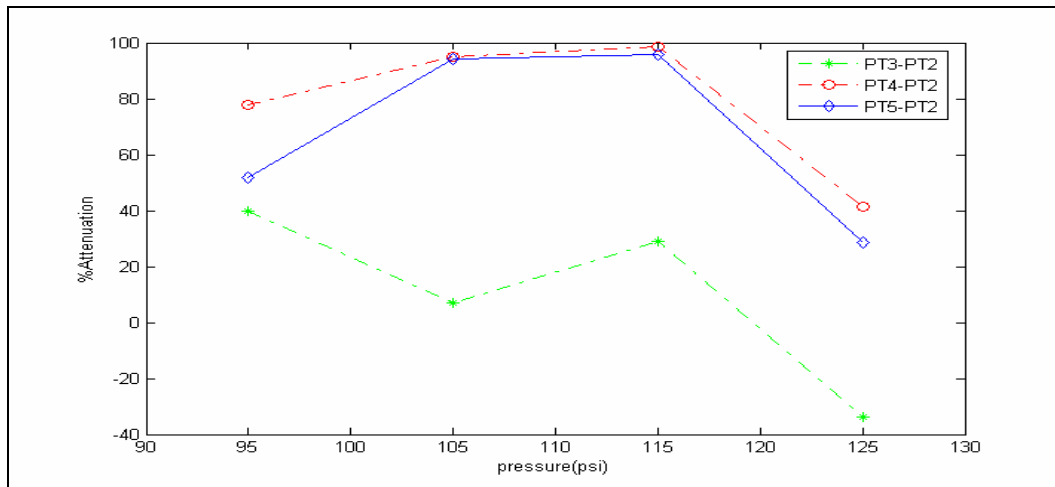
**Figure 25. Illustration of rate of the amplitude dissipation (decibels per inch) for specific upstream stagnation pressures between a)PT2-PT3, b)PT2-PT4, and c) PT2-PT5**

Figure 25 illustrates the individual plots of the attenuation rates with respect to upstream stagnation pressure. No apparent trend is immediately discerned from this perspective. Unlike Figure 26, which showcases all the cases on the same plot where the possibility of a trend becomes recognizable. In comparing the plot for attenuation rate, in Figure 26, moving from PT2-PT4 and from PT2-PT5, one can observe that the attenuation was almost doubled for PT2-PT4. Pattern is almost similar except the slighter higher dissipation rate at 115psi stagnation pressure. Again, the attenuation rate is less going from PT2 to PT3 in comparison to the other two cases.



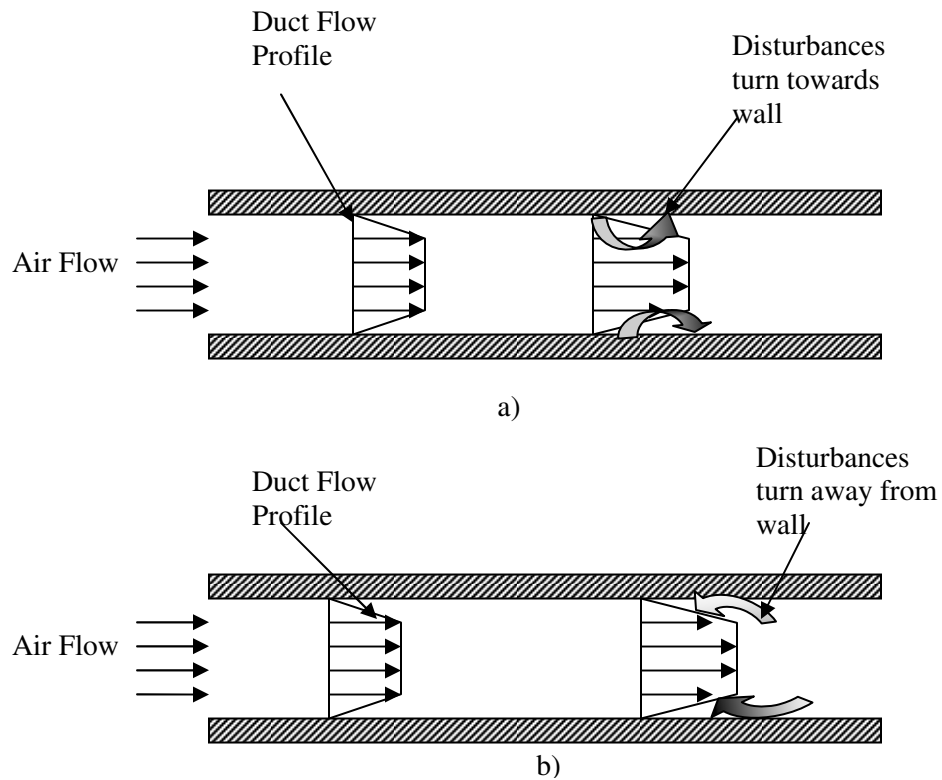
**Figure 26. Illustration of rate of the amplitude dissipation (decibels per inch) for specific upstream stagnation pressures.**

This analysis was extended by looking the percentage of attenuation with respect to the raw RMS squared values with PT2 as a reference point. As seen in Figure 27, the relative percentage attenuation plots for all three cases follow similar trend as the attenuation rates in Figure 25 and 26. With an attenuation of approximately 98% from its originally detected value at PT2 at 115psi, this coincides with the highest attenuation rate observed in Figure 26 for PT2-PT4. A similar pattern is seen between cases PT2-PT4 and from PT2-PT5. The negative value for PT2-PT3 can be interpreted as a slight amplification for that specific upstream pressure condition.



**Figure 27. Illustration of relative percentage of the dissipation of acoustic wave amplitude for various stagnation pressures.**

To shed some light on some of the possible reasons for this attenuation effect, the schematic diagram shown in Figure 28 will be used as an aid.<sup>37</sup> According to Hersh et al., Pridmore and Brown [3]<sup>37</sup> were pioneers in the late 1950's for the investigation on how acoustic waves are influenced by shear flow inside a rectangular duct.

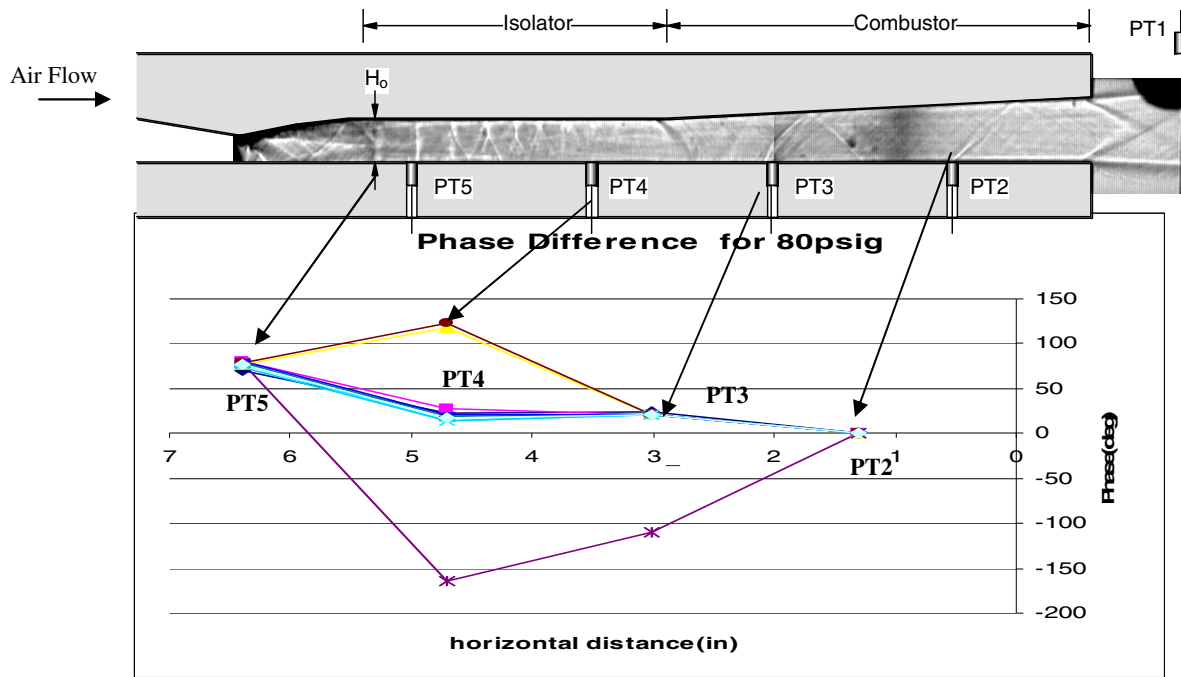


**Figure 28. Schematic diagram illustrating disturbance propagation in the a) downstream direction and b) upstream direction.**

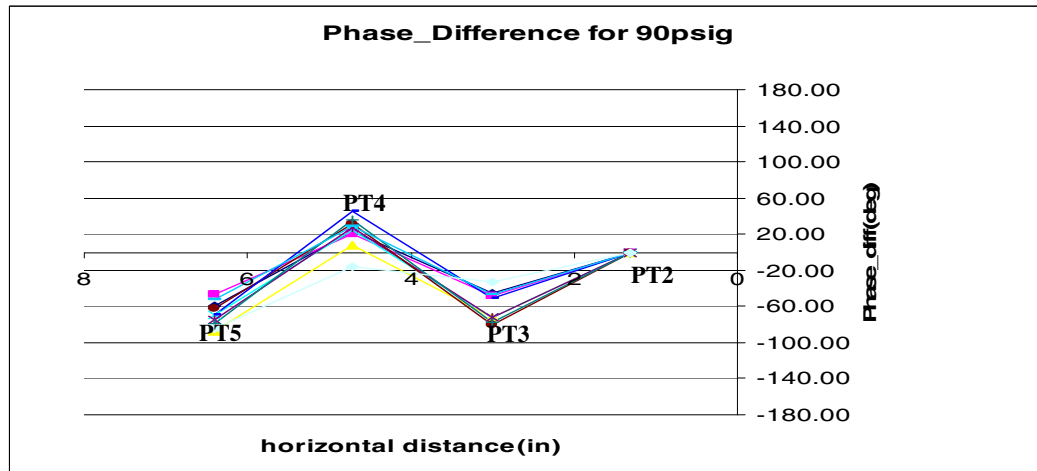
Their focus was mainly on how acoustic attenuation occurs in the downstream direction rather than upstream propagation. The pivotal findings of their research was that the velocity gradient played a crucial role in forcing the acoustic waves towards the walls of the duct as seen in Figure 28(a).<sup>37</sup> It seems that Hersh and Catton were the first to seriously tackle the idea of upstream propagation in a rectangular duct with shear flow. Even though this was not explicitly stated in their writings, it was only implied on the basis of their stated literature review and motivation for their paper.<sup>37</sup> Hersh and Catton numerically confirmed Pridmore and Brown's idea of downstream propagation and found that for upstream propagation the acoustic waves were being directed towards the core flow by the velocity changes were inside the flowfield.<sup>37</sup> Whether or not attenuation was occurring wasn't explicitly state by the authors,<sup>37</sup> which look only at the case for max Mach number of 0.5. Figure 28 may be just a display of the mechanism of what's going on inside the duct. According to Schneider<sup>38</sup> who look at the supersonic flow case numerically, the acoustic waves will dampen if viscosity effects are taken into account. In the case of this experimental research effort where the real flow conditions do imply that viscosity would play a role, one can see that attenuation effects are occurring based on the above analysis of the dynamic pressure measurements. These measurements are manifested as power spectrum and acoustic attenuation plots, which are shown above.

#### 4.3. Effects on the Phase of Acoustic Wave

In Figures 29-31, the variation of the phase differences along the duct with the stagnation pressures 95psi, 105psi, 115psi, and 125psi. The phase calculation was conducted using the FFT analysis of the dynamic pressure for all 10 sequential runs at each pressure transducer.



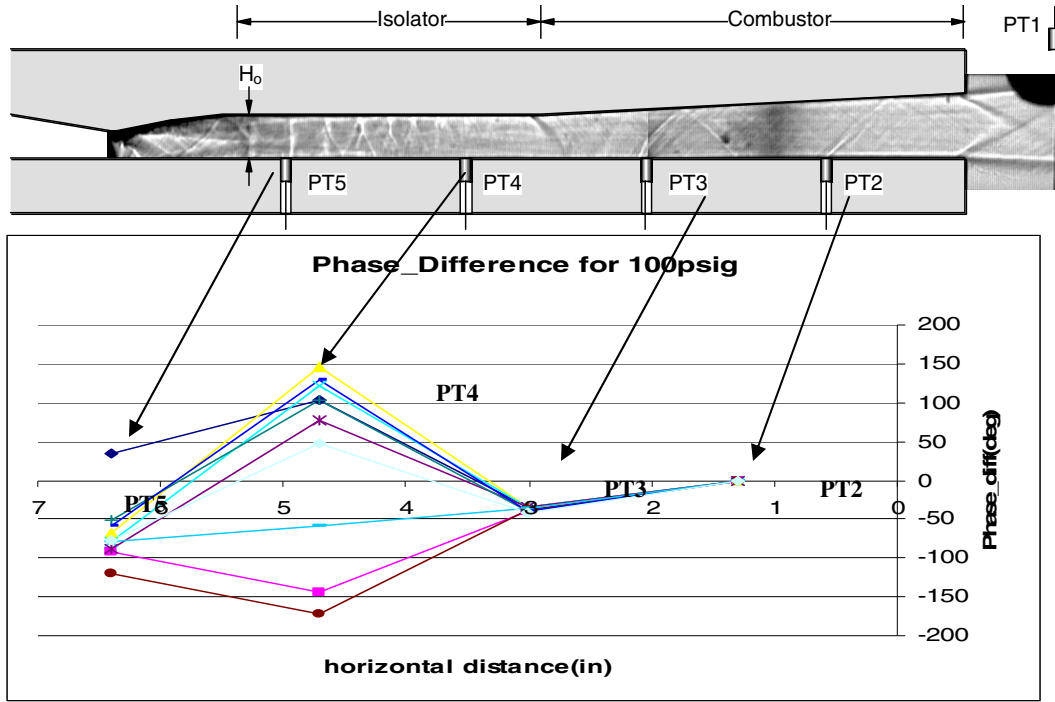
a)



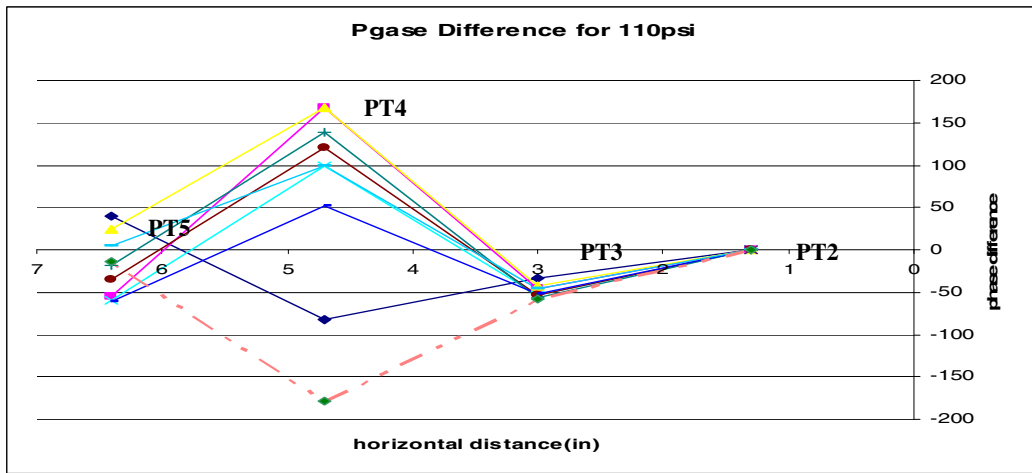
b)

**Figure 29. Illustration of phase change along the duct for upstream stagnation pressure of a) 95psi b) 105psi**

The phase of the acoustic wave at each PT with respect to the dominant frequency was extracted from the data. Of keen interest in this particular analysis was the change of phase differences along the supersonic duct. Specifically, the phase difference with respect to a reference point.



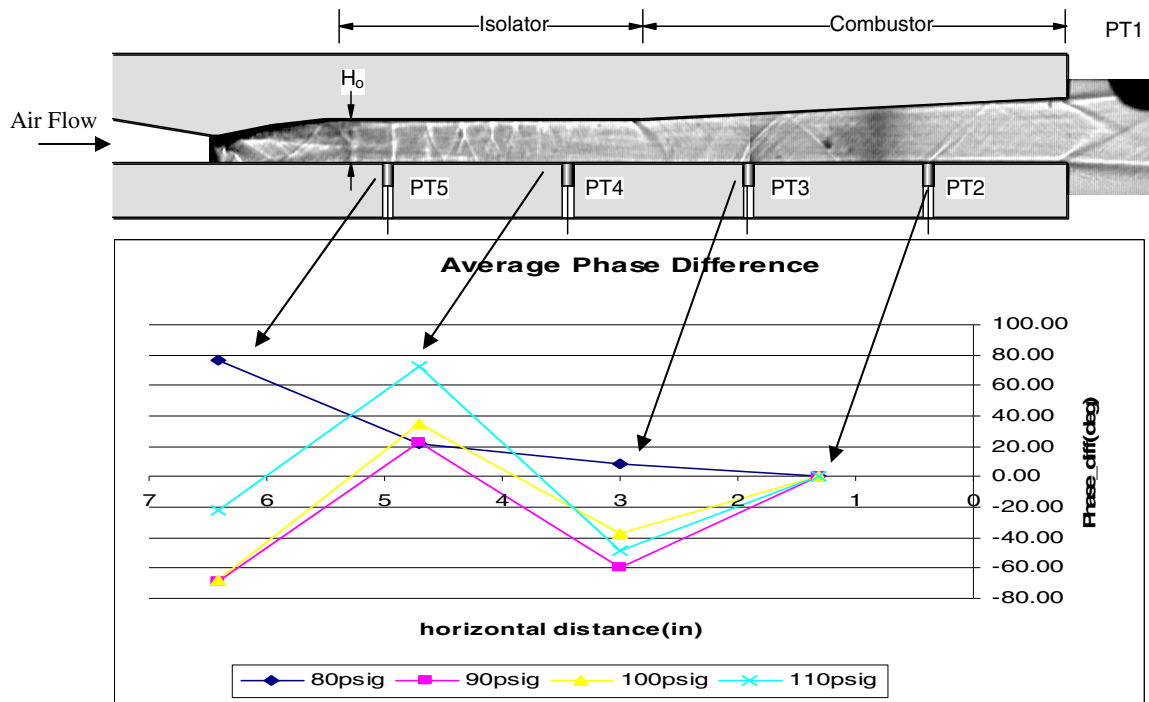
a)



b)

**Figure 30. Illustration of phase change along the duct for upstream stagnation pressure of a)115 psi b) 125psi**

The reference point chosen for this analysis was PT2. In order to explore this phase within one cycle, values were kept within (+/- )180 degree phase. In Figure 29(a), the phase difference at 95psi (80psig) is shown with a few aberrations from the basic pattern. The relative phase difference going from PT2 to PT3, for all 10 experimental case runs, was on the average 20 degrees with a dominant frequency of approximately 4.8kHz.



**Figure 31. Illustration of average phase change along the duct at each pressure transducer location for increasing upstream stagnation pressure conditions.**

Similarly, the phase differences going from PT2 to PT4 were within the range of 14-20 degrees, see Figure 29(a). The phase gap widens by approximately 50 degrees travel from PT2 to PT5. It can be observed that the trend for almost all the cases were consistent except for a few data points. Now, for the upstream conditions at 105psi (90psig) see Figure 29(b), the negative and positive phase alternation becomes prominent. Also, the average phase gap between the pressure transducers increased slightly in comparison to those found in Figure 29(a). Even though that may be the case, interestingly enough, it can be observed that all ten experimental case runs at this pressure condition more or less followed the same phase difference pattern. This could suggest some type of phase correlation between the points of interest.

In Figure 30, both phase difference trends display what may be construed as some type of oscillation change around PT4 at 115psi and 125psi. With a dominant frequency of 2.8kHz, the phase difference at 115psi between PT2 and PT3 is fairly consistent for all case runs with an average phase difference of approximately 37 degrees. In Figure 30(b) the variation gap widens with an average difference of approximately minus 48 degrees. The seemingly sporadic behavior is the same for both pressure conditions going from PT2 to PT4 and PT2 to PT5, with the largest phase difference (73 degrees) occurring at PT2-PT4. To further extending this analysis, the

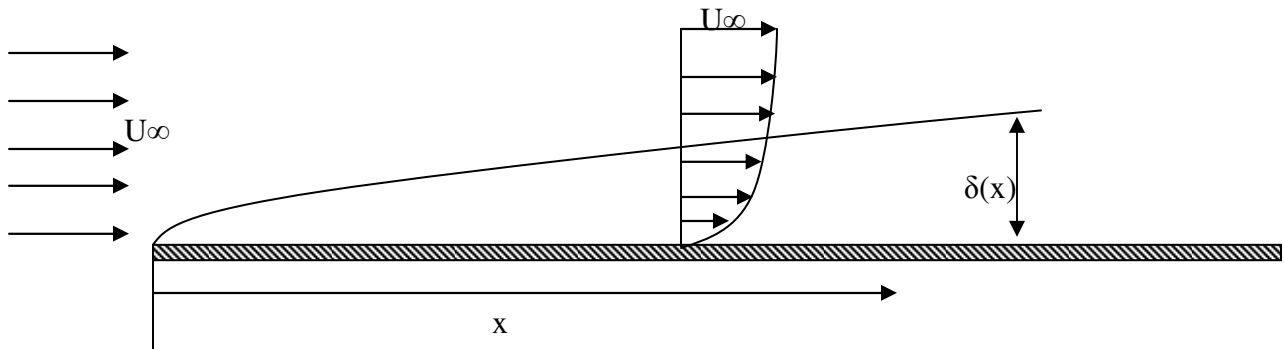


average of all experimental runs were calculated and plotted, see Figure 31. The general pattern alternated between approximately 80 degrees occurring at PT4 and minus 70 degrees, which occurred at PT5.

#### 4.4. Boundary Layer Thickness Characterization

##### 4.4.1. Upstream Flow conditions

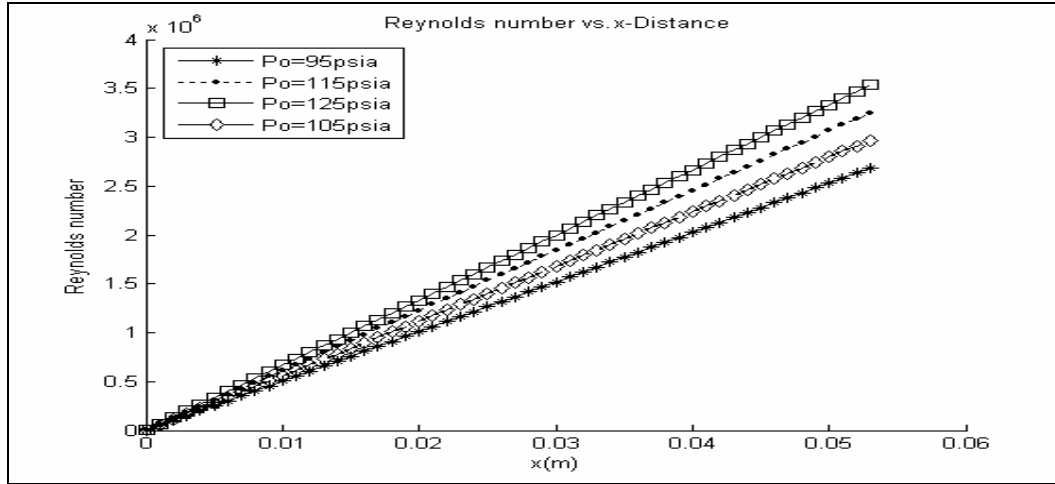
In theory, it is possible to have acoustic waves propagate upstream via boundary layer because a part of the boundary layer will always remain subsonic no matter how high the value of the core Mach number. However, it is also expected that, even if the acoustic waves can propagate upstream, there will be substantial attenuation in the amplitude because the subsonic region is relatively thin and the nature of turbulent boundary layer flow is very unsteady.



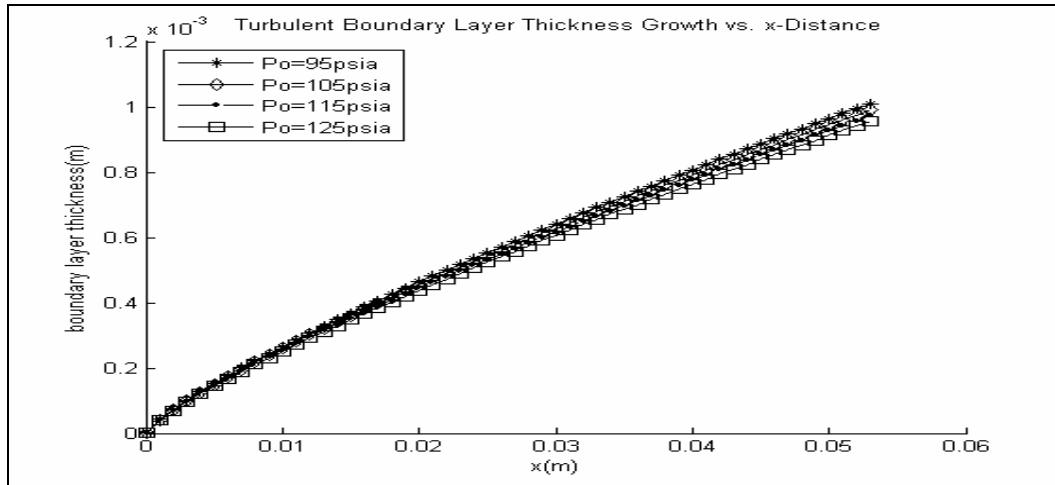
**Figure 32. Illustration of boundary layer thickness growth along a flat plate**

The turbulent boundary analysis was done on the basis of a first cut approximation modeled with flow over a flat plate. In Figure 32, a schematic drawing shows the development of the boundary layer as a function of the horizontal distance  $x$ . Given some freestream velocity, a velocity profile is created where the velocity gradient is prominent inside the boundary layer.

The velocity is assumed to observe the “non-slip condition” where the velocity component normal to the surface is equal to zero. This velocity profile also changes significantly from position to position inside the boundary layer.



a)

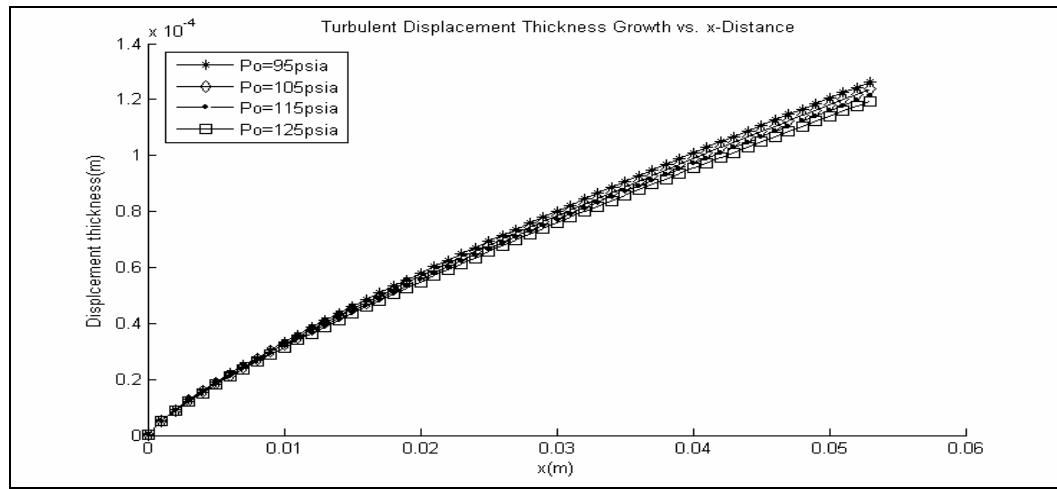


b)

**Figure 33. Illustration of a) Reynolds number and b) boundary layer thickness inside constant area section**

The boundary layer thickness ( $\delta$ ) computation was done for turbulent boundary layers only, using the one-seventh power rule<sup>34</sup>. At very low Reynolds number, this result would not be applicable as the boundary layer at the isolator entrance was expected to be laminar. Also, in the expansion region, these values are expected to vary slightly. Figure 33 shows how Reynolds number ( $Re$ ) and the boundary layer thickness develops along the

horizontal axis in the constant area section of the duct where pressure transducers 4 and 5 are located. Figure 33(a) shows how Reynolds number increases with the increase in upstream stagnation pressure with values as high as approximately  $3.6 \times 10^6$  for 125psi, while the expected turbulent boundary layer thickness decreases in the constant area section as pressure increases. Also the displacement thickness also shows the similar trend as the boundary layer thickness, where there is generally a decrease in the displacement thickness ( $\delta_{disp}^*$ ) as pressure increase, see Figure 34.

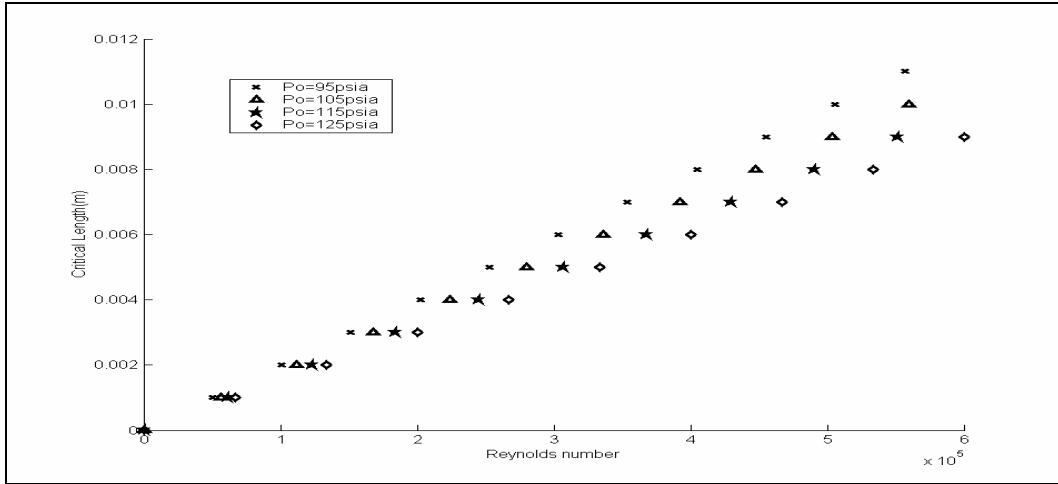


**Figure 34. Illustration of turbulent displacement thickness growth inside constant area section (isolator region)**

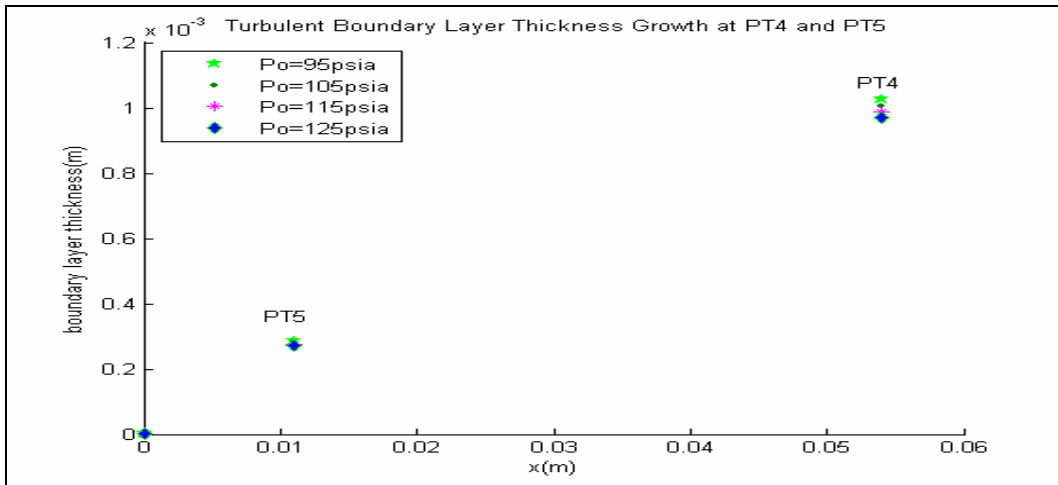
#### 4.4.2. Boundary Layer and Displacement Growth at Pressure Transducer Location

From the investigation on how the boundary layer thickness and displacement thickness changes continuously along the horizontal axis inside the constant area, the analysis was then focused more at the what were changes in boundary layer and displacement thicknesses at the individual pressure transducer locations, in particular at the specific transducer location. In order to get an idea were the flow transition from laminar became turbulent as the upstream pressure increased, the critical length was determined, see Figure 35. Typically, the transition to turbulent can be considered to have occurred when the Reynolds number<sup>34</sup> is greater than 500, 000. It was observed that the transition point from laminar to turbulent occurred less than 0.4 inches(8mm) from the c-d nozzle exit at max pressure(125psi). As shown in Figure 36, the thickness growth at PT5 with respect to all pressure conditions showed minute changes. The boundary layer thickness ( $\delta$ ) as well as the displacement

thickness ( $\delta_{disp}^*$ ) is relatively significant at PT4. Similar observations were made for the displacement thickness shown in Figure 37.

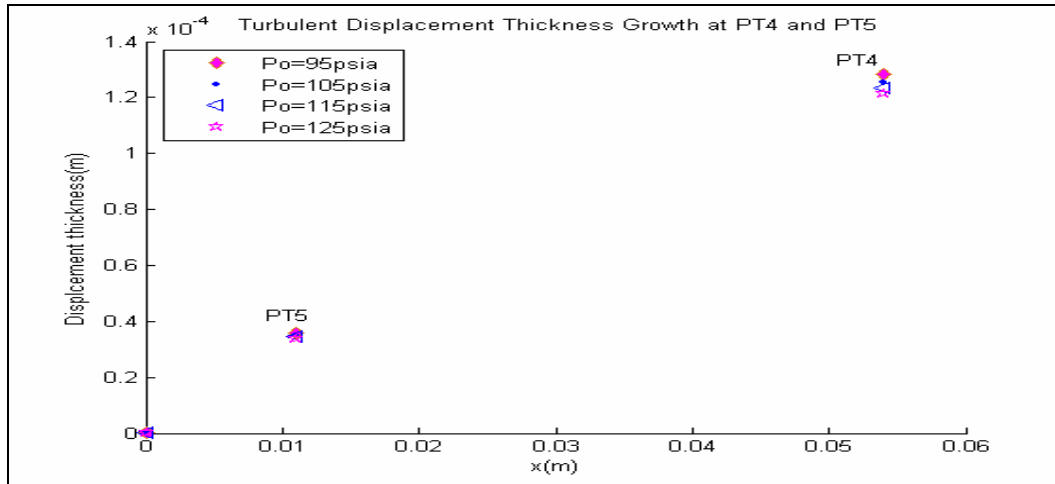


**Figure 35. Illustration of critical length as function of Reynolds number inside constant area section (isolator region)**



**Figure 36. Calculated boundary layer thickness growth at pressure transducer locations PT4 and PT5**

Where the  $\delta$  ranges from approximately ( $1.0 \times 10^{-3}$  to  $0.971 \times 10^{-3}$ ) meters, see Fig. 36, for upstream pressure 95 to 25psi, respectively. Similarly in Fig. 37, the ( $\delta_{disp}^*$ ) ranges from approximately ( $1.28 \times 10^{-4}$  to  $1.21 \times 10^{-4}$ ) meters with 95 to 125psi, respectively.



**Figure 37. Calculated displacement thickness growth at pressure transducer locations PT4 and PT5**

## Chapter 5: Summary and Conclusion

This research effort was carried out to fully examine whether or not acoustic waves could propagate upstream through a scramjet internal flowpath. Several non-reacting and passively-controlled experiments were conducted in a combined supersonic duct-resonant supersonic jet configuration, which simulated the instabilities in scramjet combustor. Coherent pressure oscillations at several distinct frequencies, which were generated from supersonic jet screech phenomena at the duct exit, were systematically observed propagating upstream using wall-mounted pressure transducers. Boundary layer analysis was based on the model of flow over a flat plate. The natural frequency of the supersonic duct was determined not to be equivalent to the resonance frequency of the passive acoustic mechanism. Hence, reducing the possible influence on the upstream propagating wave. To summarize the results of this experiment, the focus was on the following topics:

- **Internal Flowfield Characterization**

Characterization of the flowfield inside the supersonic duct was carried out using the schlieren method

It can be seen from Figure 17 that the turbulent flow and shock structures of the flowfield inside the constant area section became more profound as the upstream stagnation pressure increases from 20psig to 110psig. This complicated flowfield can be attributed to the intricate interaction between the boundary layer and the shock wave taking place along the surface of the duct.<sup>25</sup> When the stagnation pressure was relatively low, the nozzle boundary layer thickness remained small and the oblique shock train was observed inside the isolator duct. At higher stagnation pressure conditions, the shock train transitioned to normal shocks due to thicker boundary layer associated with turbulent boundary flow at the nozzle exit

The flowfield in the expansion duct and the free jet at the duct exit was visualized. Figure 18 shows these images taken at various stagnation pressures. When the upstream stagnation pressure was increased from 20psig to 60psig detailed diamond shock structures were observed propagating downstream of the duct exit, see Figure 18. Among the several possible observations the most prominent are the development of expansion waves from the corner of the duct exit as well as the shock wave and expansion fan reflections from the free jet shear layer, which forms a constant pressure boundary.

- **Acoustic Wave Amplitude Characterization**

An acoustic signature characterization was carried out by performing a spectral analysis of the acoustic signal. Results from the spectral analysis show the amount of amplitude dissipation as the acoustic wave traveled upstream. In addition, higher harmonics at time became the dominant frequency various stagnation pressures. Acute behavior in the flowfield was detected in the constant area section which could be attributable to the significant reduction in  $RMS^2$  value, which was observed at the two extreme upstream pressure transducer locations. Dynamic pressure measurements show that there is a definite general trend of acoustic attenuation that is occurring as the signal propagates upstream through the subsonic boundary layer. Normalized dynamic pressure calculations showcase this attenuation more clearly with reference point taken at PT2. High attenuation rate up to 11dB/in were observed between the extreme ends of the interrogation region. Again, the flowfield conditions inside isolator could have a significant role on these attenuation effects. On the basis of percent attenuation, high percentage (up to 98%) of signal reduction was obtained for pressures such as 105psi and 115psi.

- **Effects of Acoustic Wave Amplitude and Phase**

Phase analysis showed that there is a strong correlation between the pressure transducer locations as the signal propagates upstream. The phase calculations at stagnation pressures 95psi, 105psi, 115psi, and 125psi. were conducted using the FFT analysis of the dynamic pressure . The analysis focused on the phase difference with respect to a reference point. The reference point chosen for this analysis was at pressure transducer two (PT2). The relative phase difference going from PT2 to PT3, for all 10 experimental case runs, was on the average 20 degrees with a dominant frequency of approximately 4.8kHz. The phase gap widens by approximately 50 degrees travel from PT2 to PT5. It can be observed that the trend for almost all the cases were consistent except for a few data points. The upstream conditions at 105psi (90psig) had an increase in the average phase gap between the pressure transducers. All sequential runs followed the same general phase difference pattern. This could suggest some type of phase correlation between the points of interest. With a dominant frequency of 2.8kHz, the phase difference at 115psi between PT2 and PT3 is fairly consistent for all case runs with an average phase difference of approximately 37 degrees. In Figure 30(b) the variation gap widens with an average difference of approximately minus 48 degrees.

- **Boundary Layer Thickness Characterization**

A first cut turbulent boundary layer analysis of the constant area section shows that Reynolds number increasing with increase upstream stagnation pressure. Simultaneously, it was observed that the boundary layer thickness ( $\delta$ ) and displacement thickness ( $\delta_{\text{disp}}^*$ ) were decreasing. This could shed some light on why the dynamic pressure values were decreased overall as the pressure increased due to the reduction of the subsonic boundary layer channel area.

In theory, it is possible to have acoustic waves propagate upstream via boundary layer because a part of the boundary layer will always remain subsonic no matter how high the value of the core Mach number. However, it is also expected that, even if the acoustic waves can propagate upstream, there will be substantial attenuation in the amplitude because the subsonic region is relatively thin and the nature of turbulent boundary layer flow is very unsteady. The boundary layer thickness ( $\delta$ ) computation was done for turbulent boundary layers only, using the one-seventh power rule<sup>34</sup>. The Reynolds number increases with the increase in upstream stagnation pressure with values as high as approximately  $3.6 \times 10^6$  for 125psi, while the expected turbulent boundary layer thickness decreases in the constant area section as pressure increases. Also the displacement thickness also shows the similar trend as the boundary layer thickness, where there is generally a decrease in the displacement thickness ( $\delta_{\text{disp}}^*$ ) as pressure increase. It was observed that the transition point from laminar to turbulent occurred less than 0.4 inches(8mm) from the c-d nozzle exit at max pressure (125psi). The boundary layer thickness ( $\delta$ ) as well as the displacement thickness ( $\delta_{\text{disp}}^*$ ) is relatively significant at PT4.

- **Key Findings:**

It was found that even though the core flow of the simulated scramjet remained supersonic, pressure oscillations could still propagate upstream through the boundary layer, part of which would remain subsonic. This result suggests that a scramjet combustor may be susceptible to acoustic-feedback type self-sustained combustion instabilities.



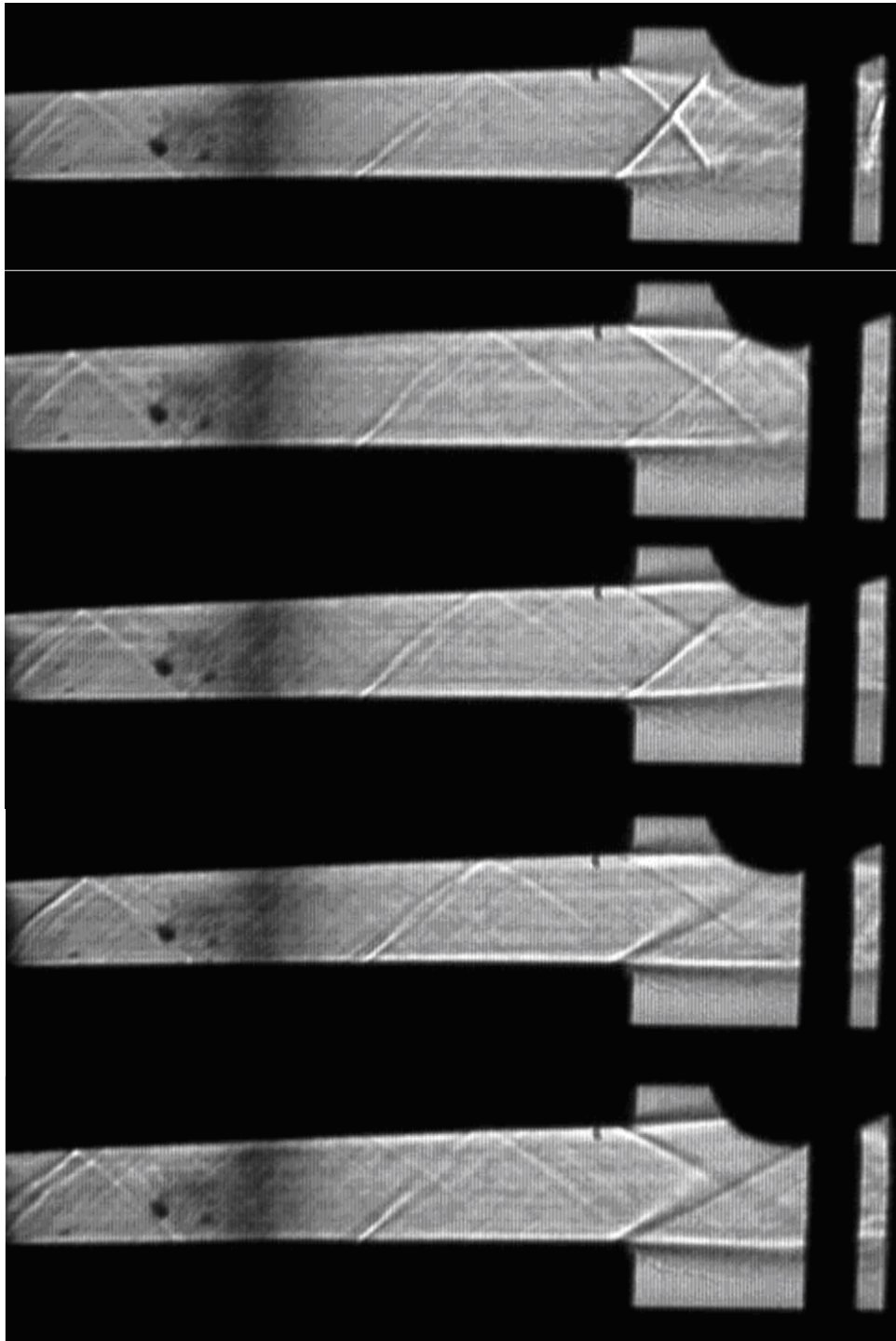
## Chapter 6: Future Work

This experimental work, as far as I know, has never been explored before experimentally. A few drawbacks existed in this preliminary effort to extract the proper physical understand of how the acoustic signal propagates through the boundary layer in a supersonic flowfield. Therefore continuous effort is needed to expand this research for more accurate description as well as prediction of propagation mechanism will be desperately needed. For example, the current experimental setup and limited scope of the experimental equipment to quantify the boundary layer thickness growth with respect to the acoustic attenuation such that an applicable relationship can be obtained. In addition, due to the dimensions of the duct, it was more or less plagued with 3-D effects, which were not accounted for in this experiment. Therefore, it was difficult to obtain a proper theoretical prediction of the neither the boundary layer growth nor displacement thickness. The ability to accurately obtain the amplitude of the acoustic source can further improve this experiment. A collaborative effort with computational analysis could produce a prediction model. In essence, this research effort has provided more of a general trend of what one could expect from similar situations.

## Appendices



**Figure 38.** A Photron Fastcam depicts schlieren images of the flowfield inside the nozzle, isolator, and expansion zone at stagnation pressure of (a) 35psi, (b) 45psi, (c) 55psi, (d) 75psi, (e) 95psi, (f) 105psi, and (g) 115psi

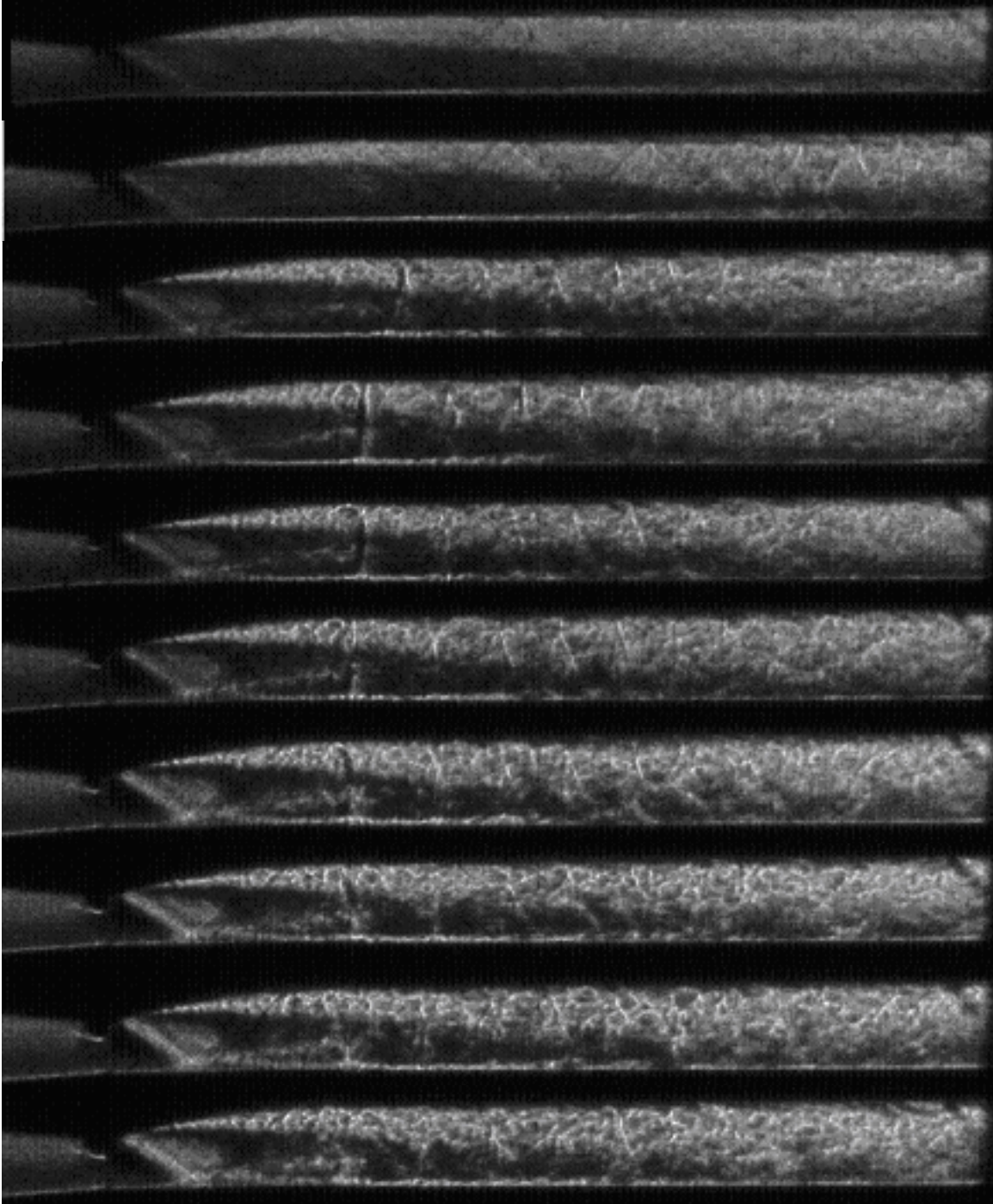


**Figure 39.** A Photron Fastcam depicts schlieren images of the flowfield inside expansion zone and duct exit for stagnation pressure of (a) 75psi, (b) 85psi, (c) 95psi, (d) 105psi, (e) 115psi

## Qualitative view of BL growth

### Throat to Isolator Region:

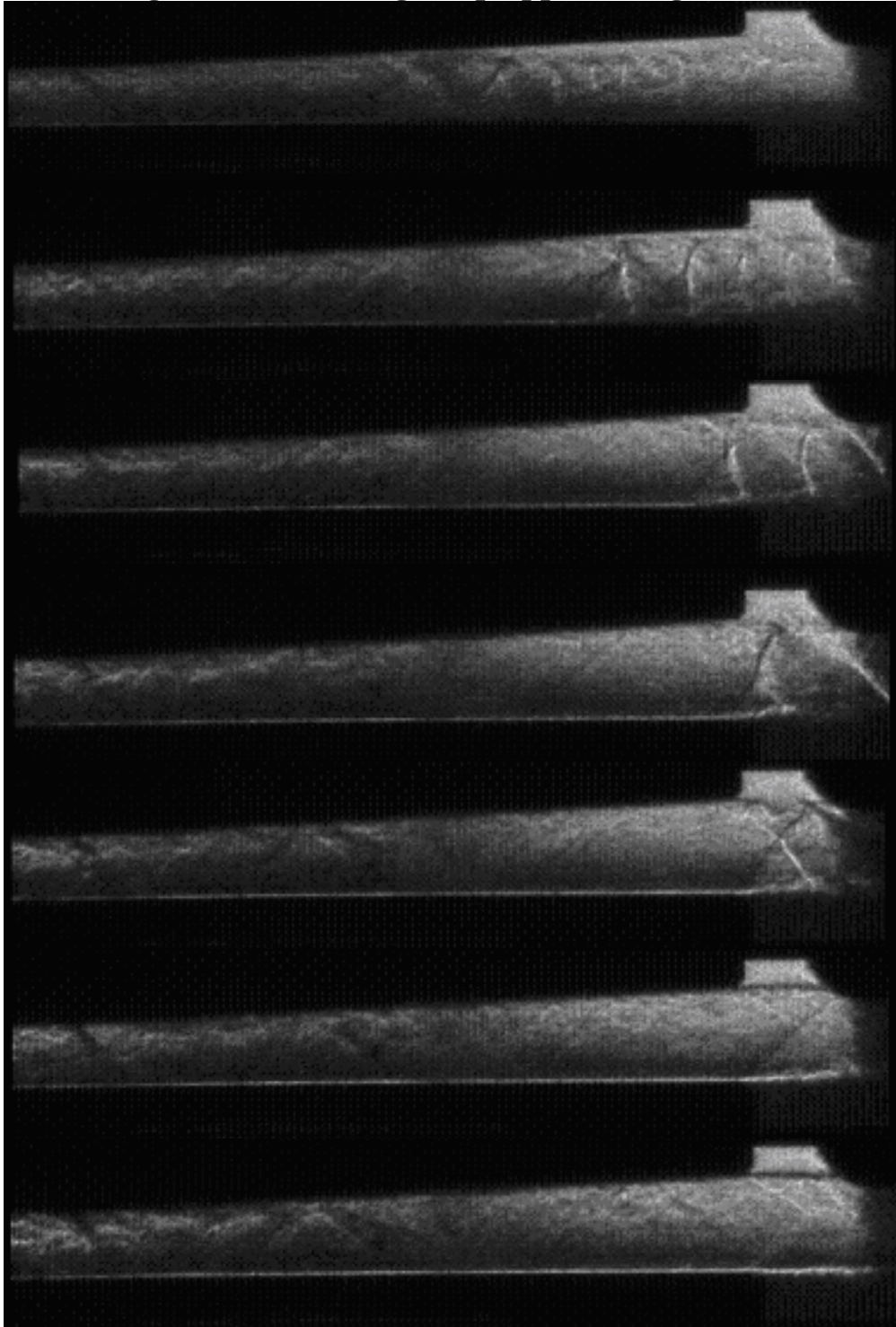
- Schlieren Images with knife edge stop(approaching from the bottom)

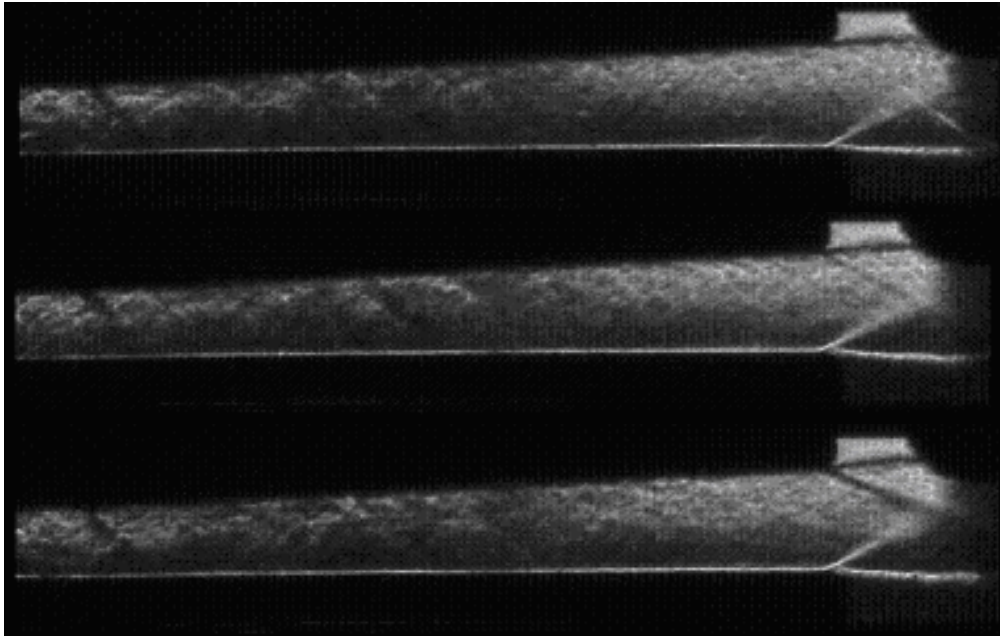


**Figure 40. Qualitative illustration of boundary layer growth inside the constant area section with upstream pressures from top to bottom: a) 35, b) 45, c)55, d) 65, e)75, f)85, g)95, h)105, i)115, j)125**

**From Isolator Exit to Combustor and Nozzle Region:**

- Schlieren Images with knife edge stop(approaching from the bottom)



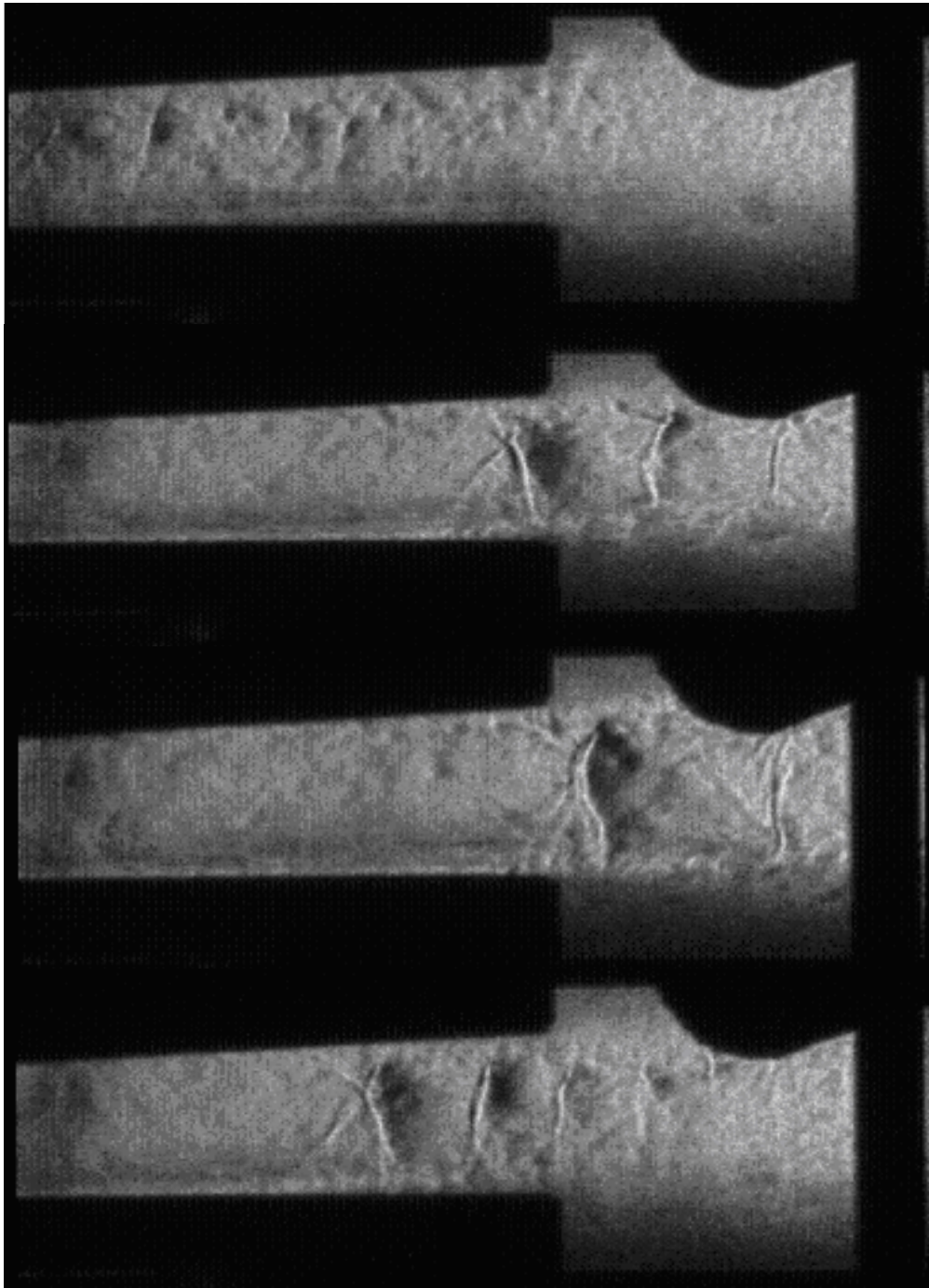


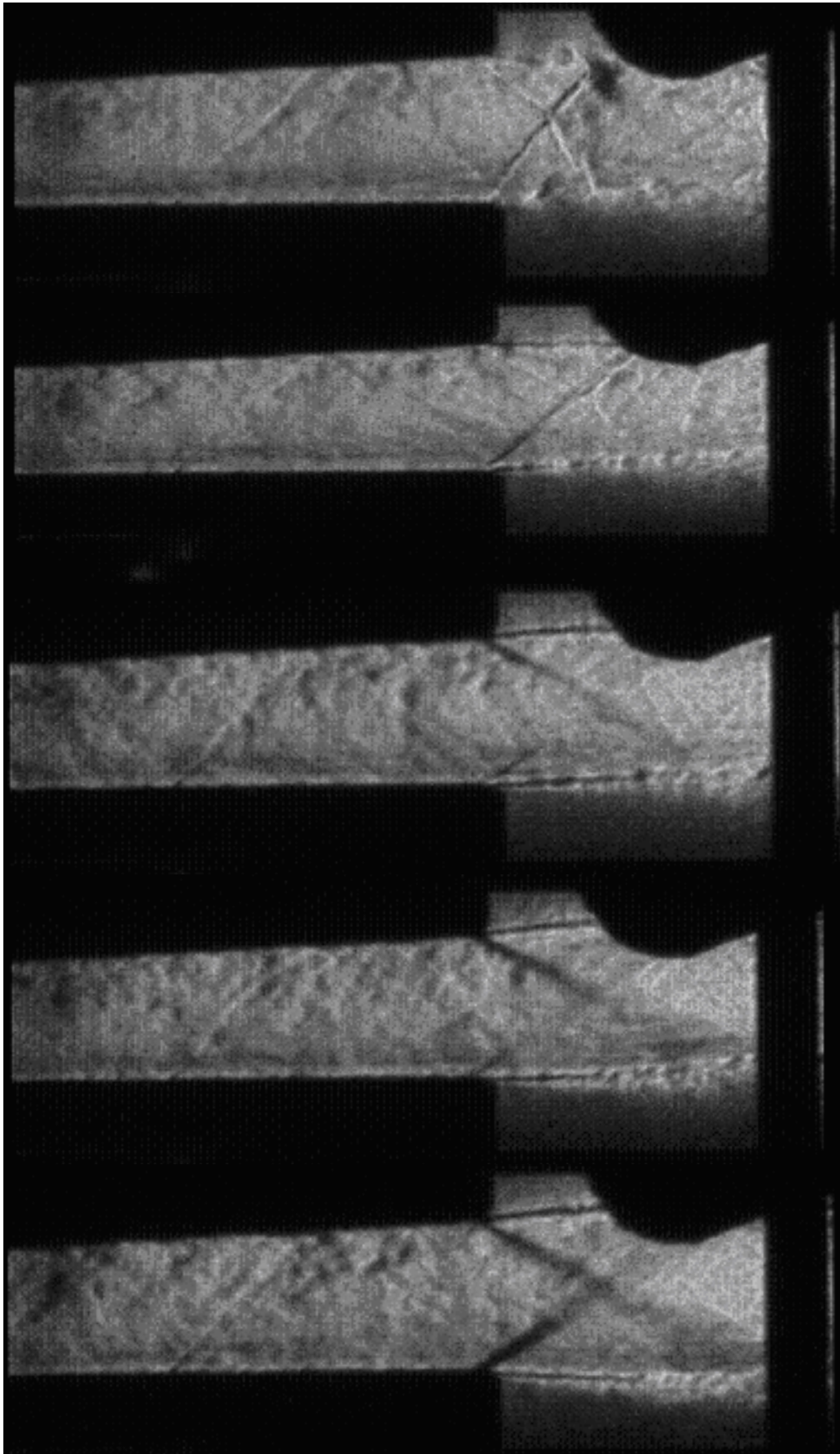
**Figure 41. Qualitative illustration of boundary layer growth inside the expansion section with upstream pressures from top to bottom: a) 35, b) 45, c)55, d) 65, e)75, f)85, g)95, h)105, i)115, j)125**



**Nozzle Exit Region:**

- Schlieren Images with apperature

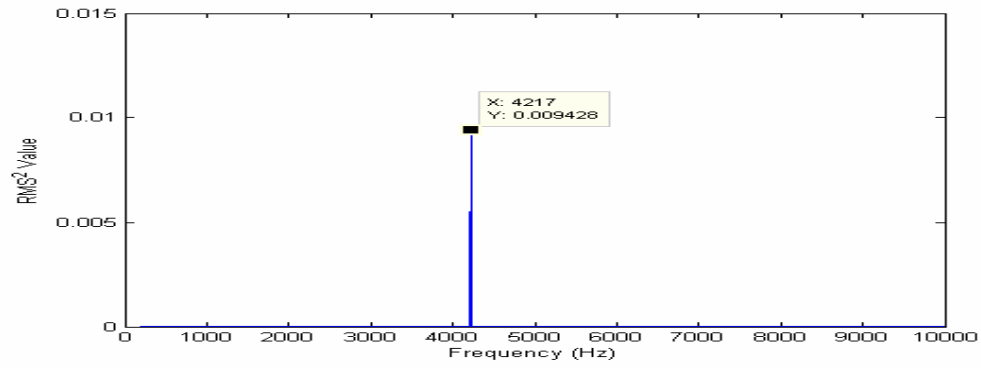




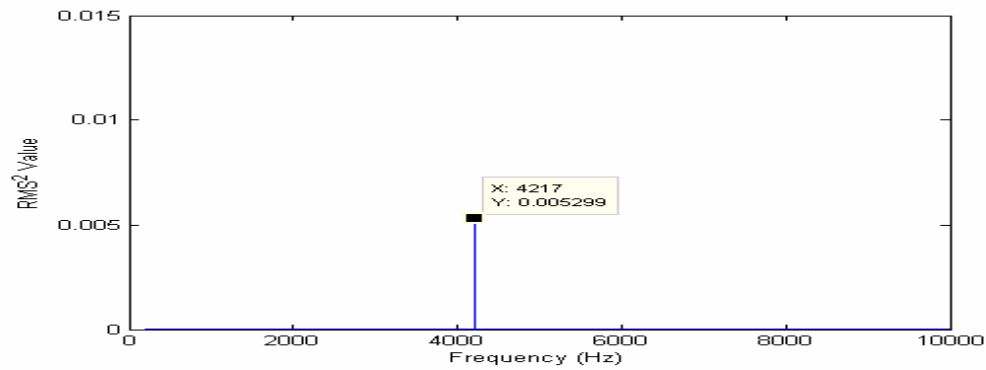
**Figure 42. Qualitative illustration of duct exit conditions with upstream pressures from top to bottom: a) 35, b) 45, c) 55, d) 65, e) 75, f) 85, g) 95, h) 105, i) 115, j) 125**



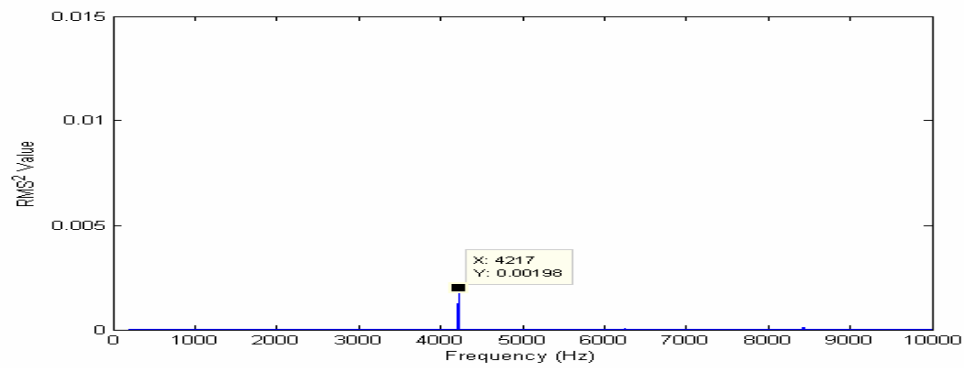
## Case Run1



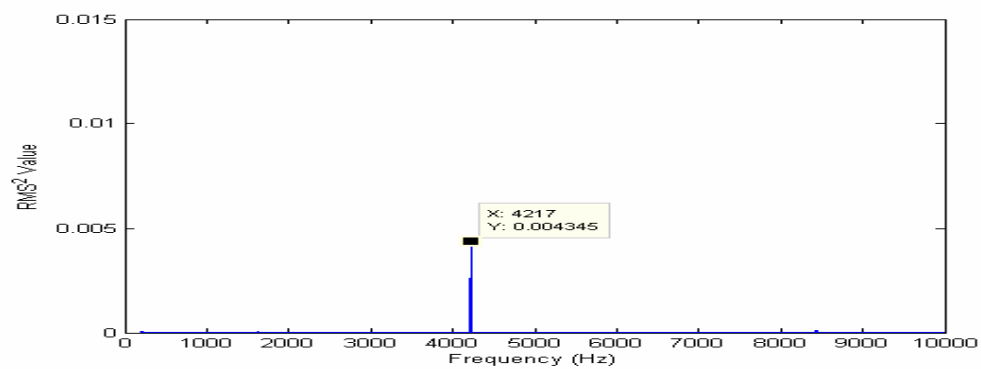
a)



b)

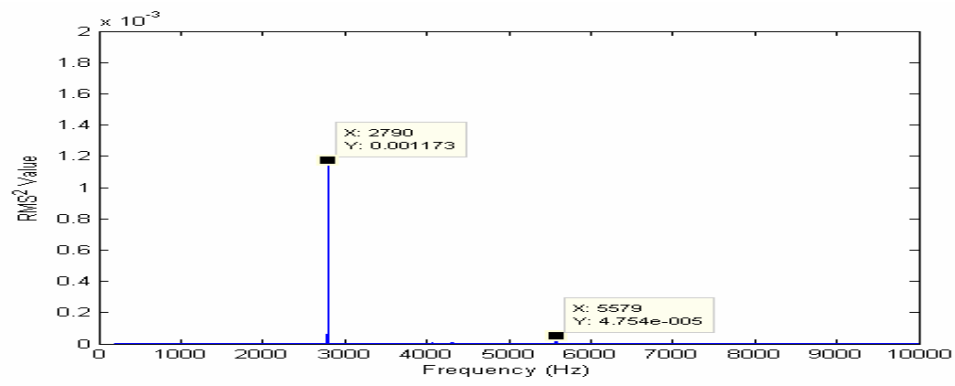


c)

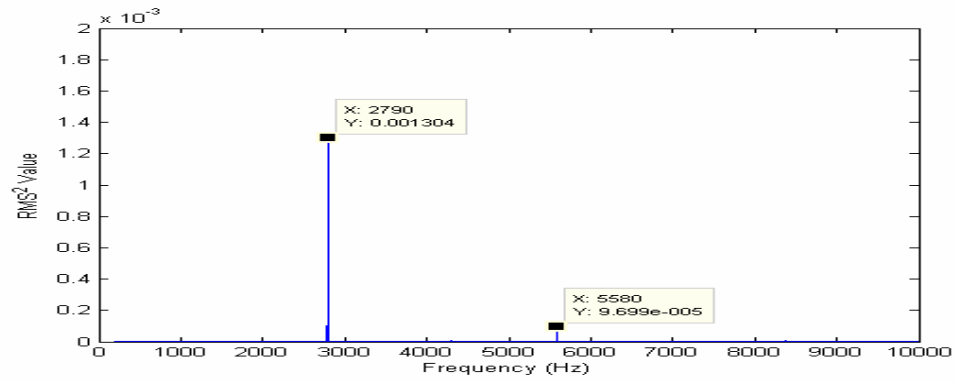


d)

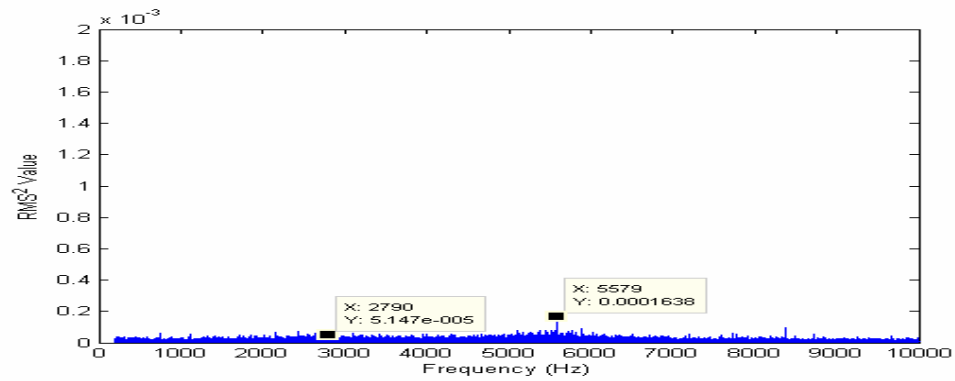
Figure 43. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 1



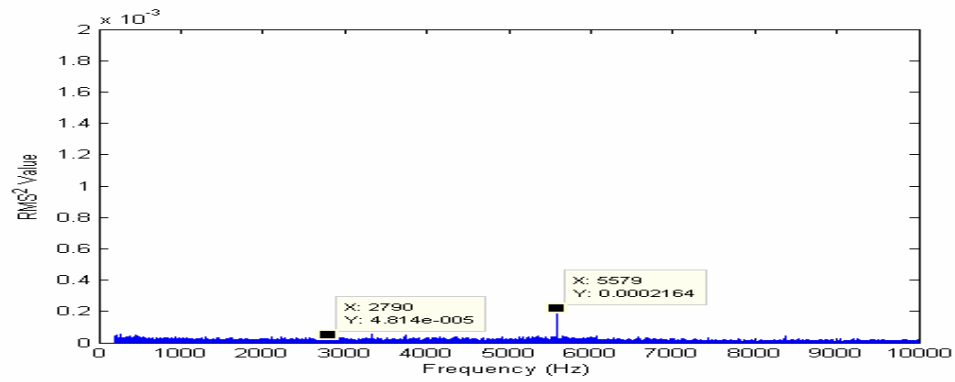
a)



b)

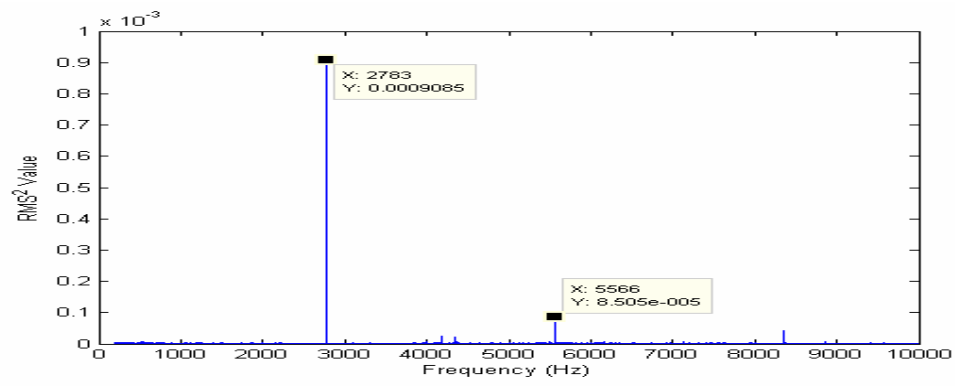


c)

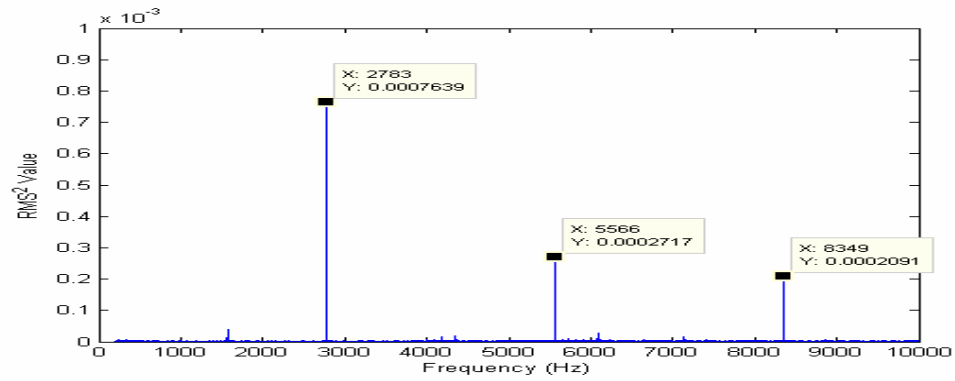


d)

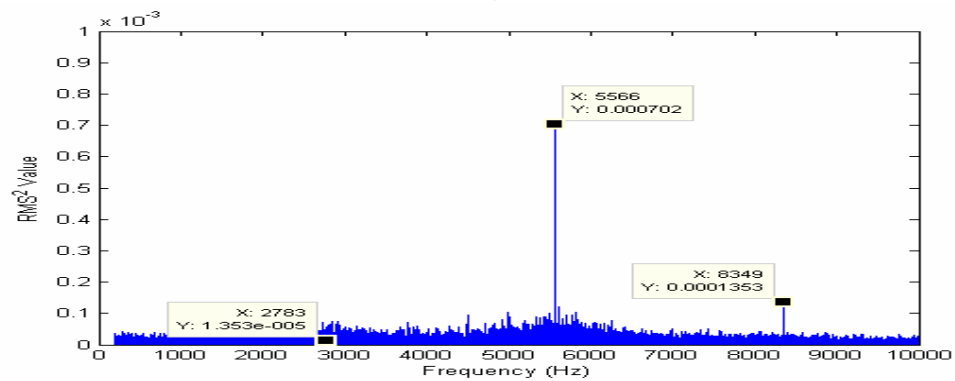
**Figure 44. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 1**



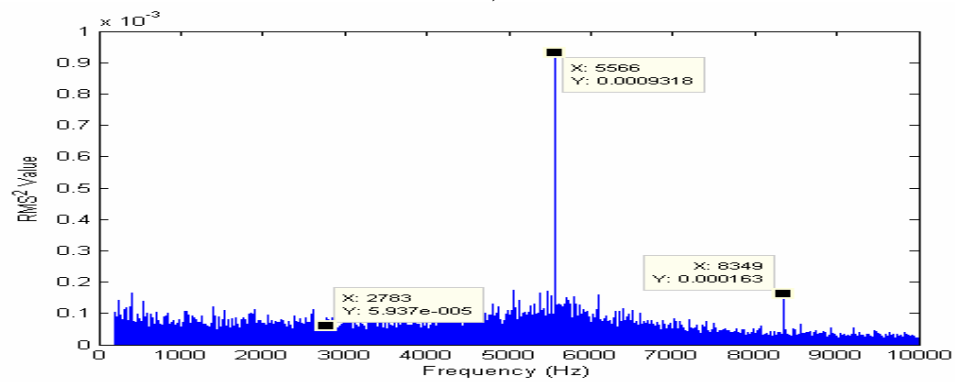
a)



b)

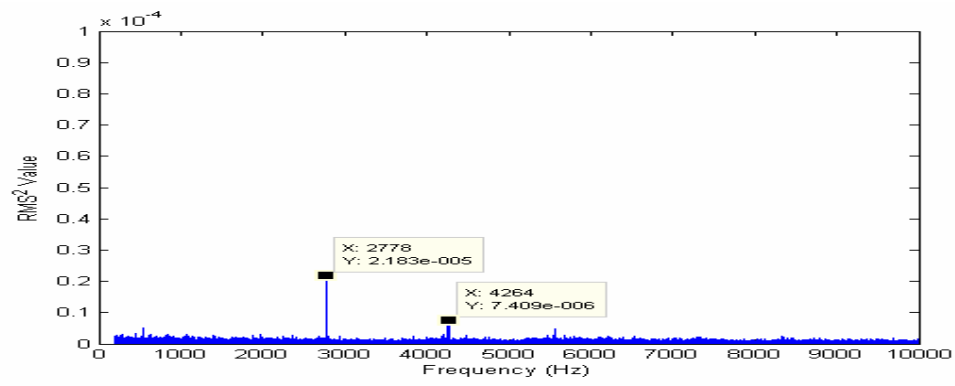


c)

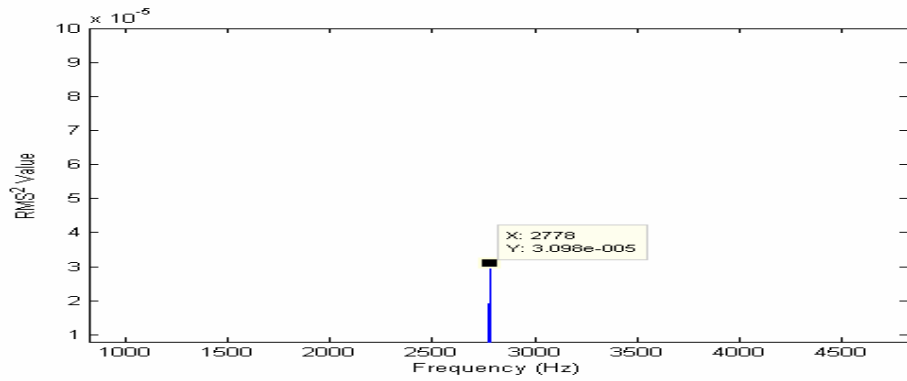


d)

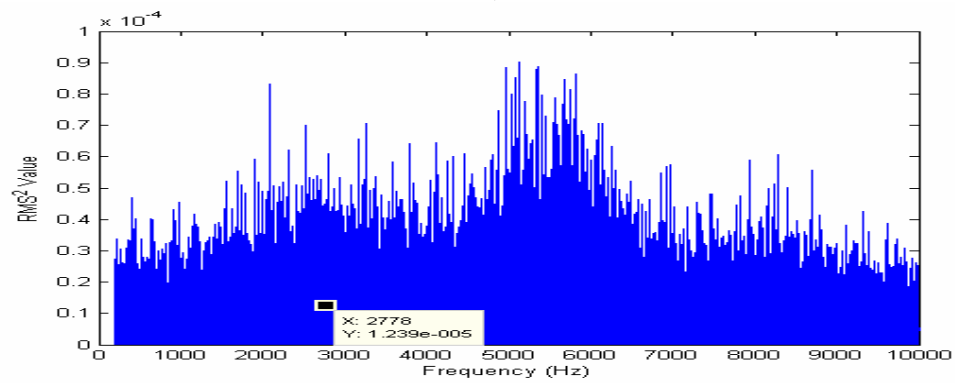
Figure 45. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 1



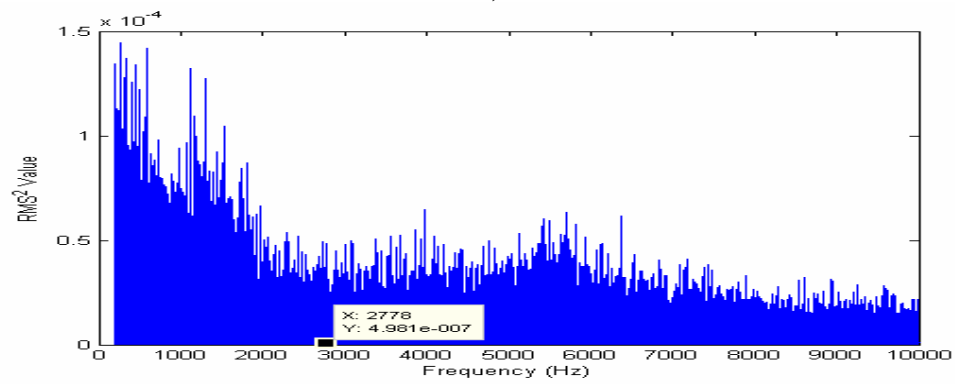
a)



b)



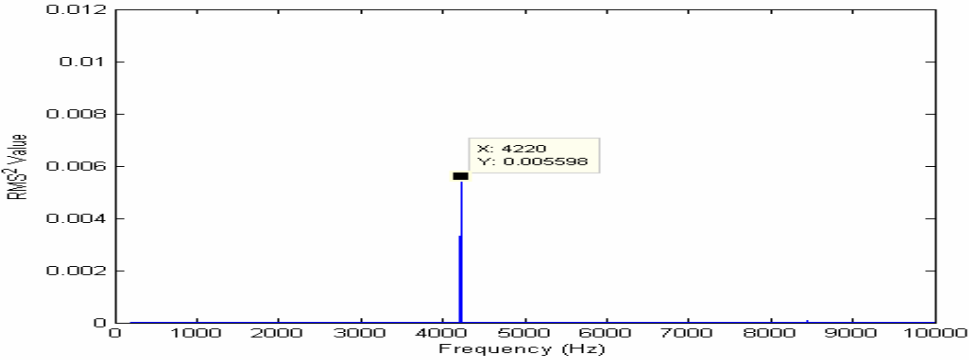
c)



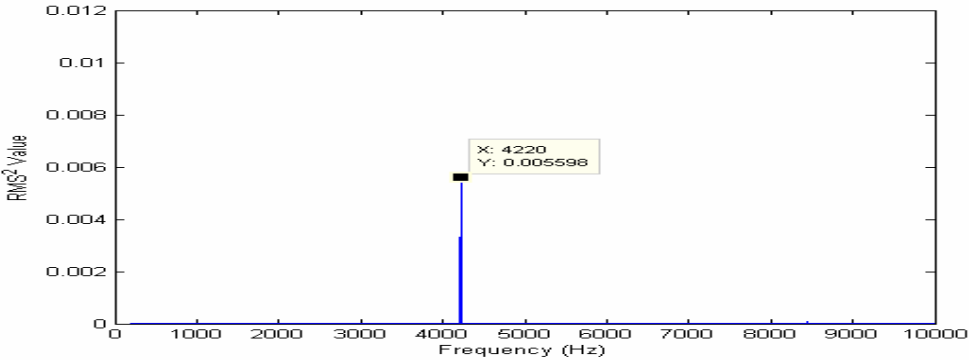
d)

**Figure 46. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d)PT5 for case run No. 1**

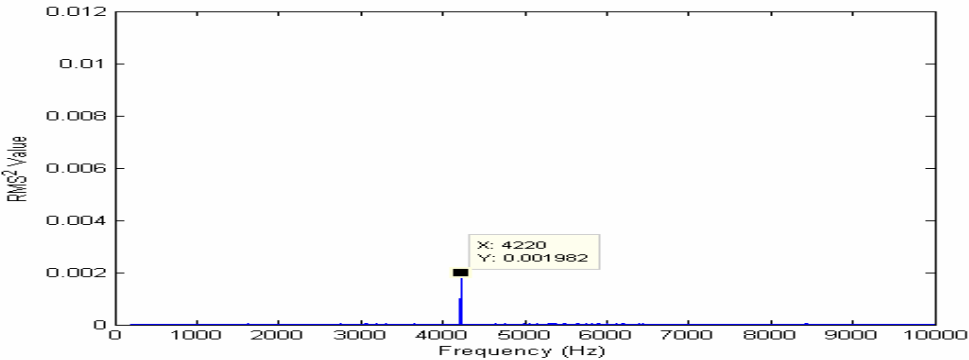
**Case Run2**



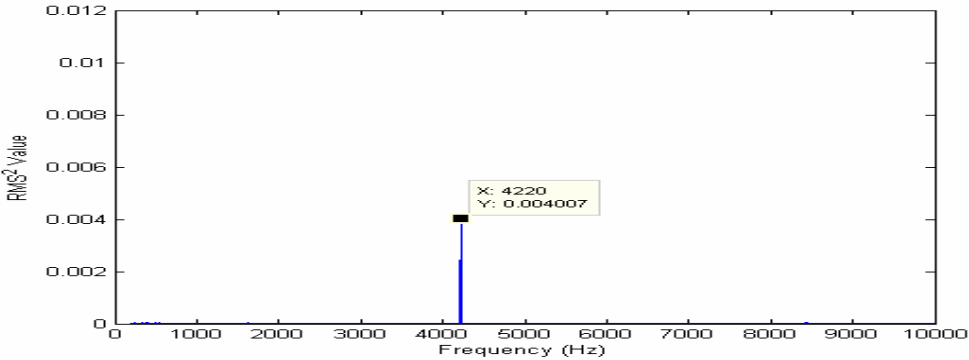
**a)**



**b)**

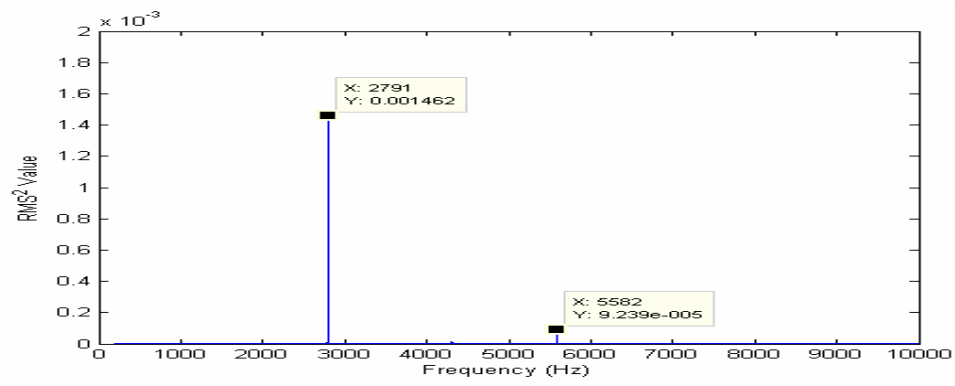


**c)**

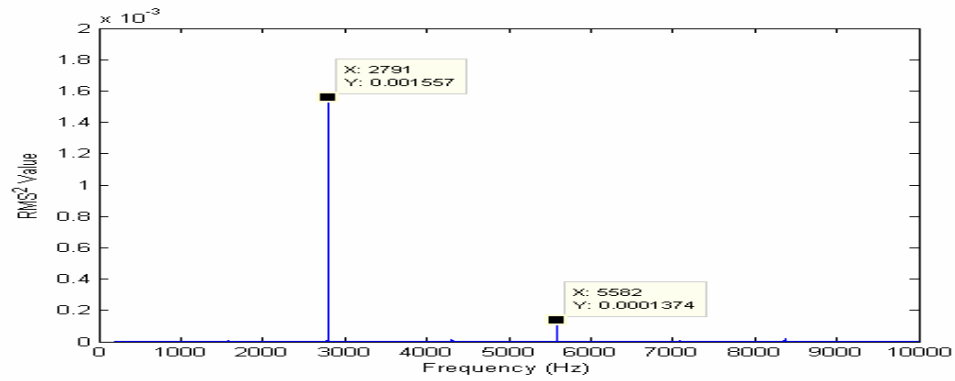


**d)**

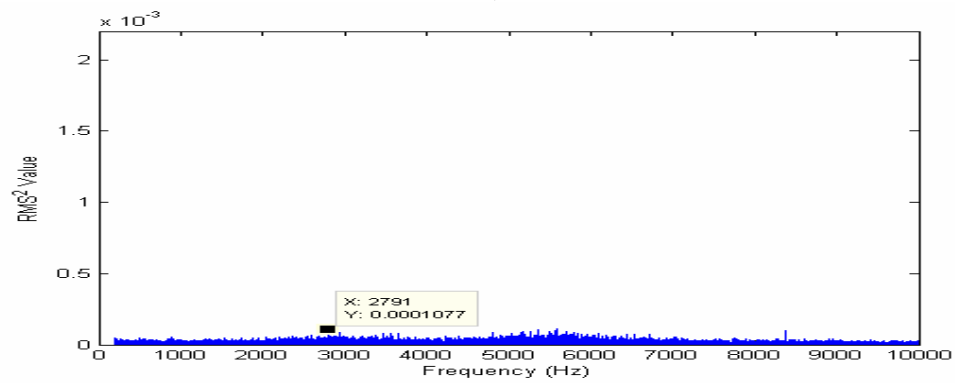
**Figure 47. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 2**



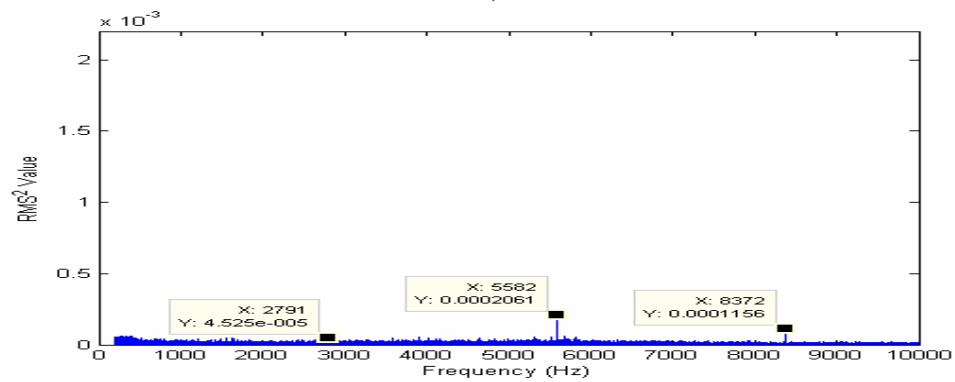
a)



b)

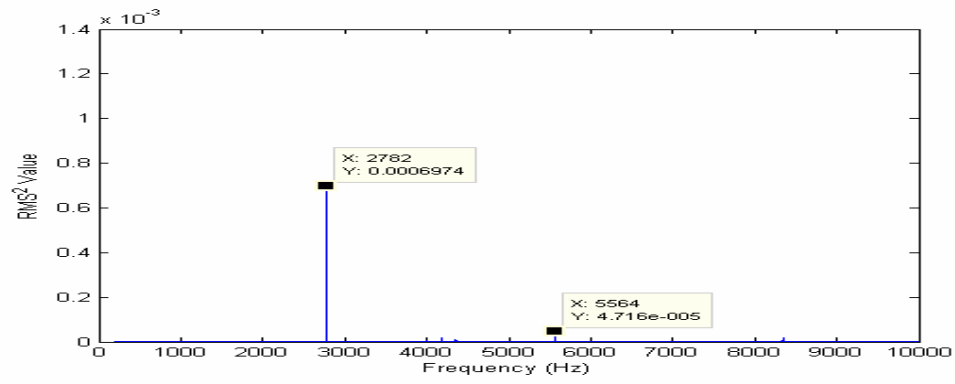


c)

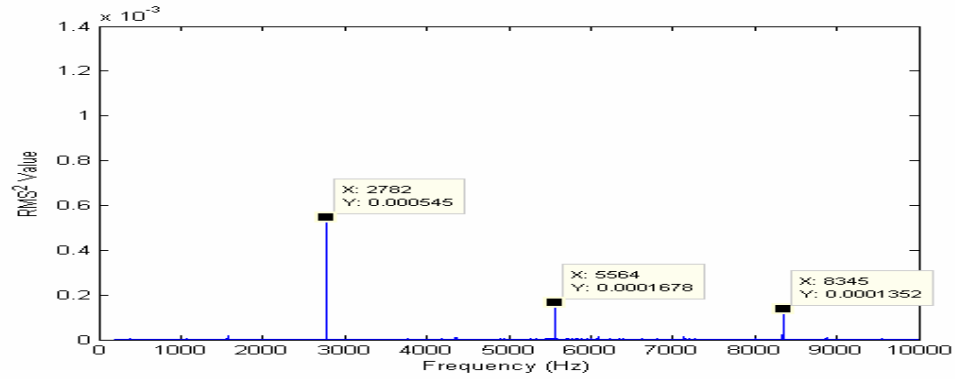


d)

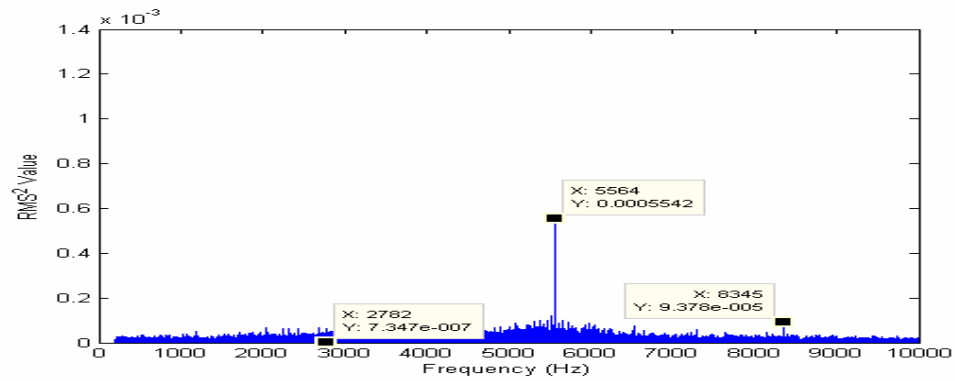
**Figure 48. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 2**



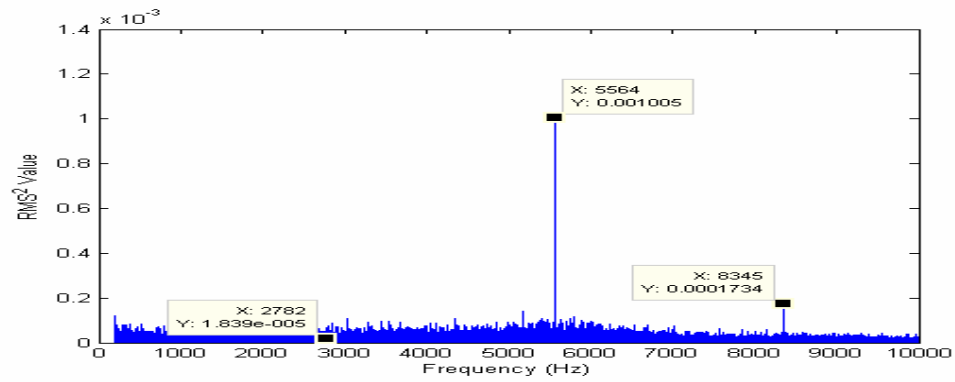
a)



b)

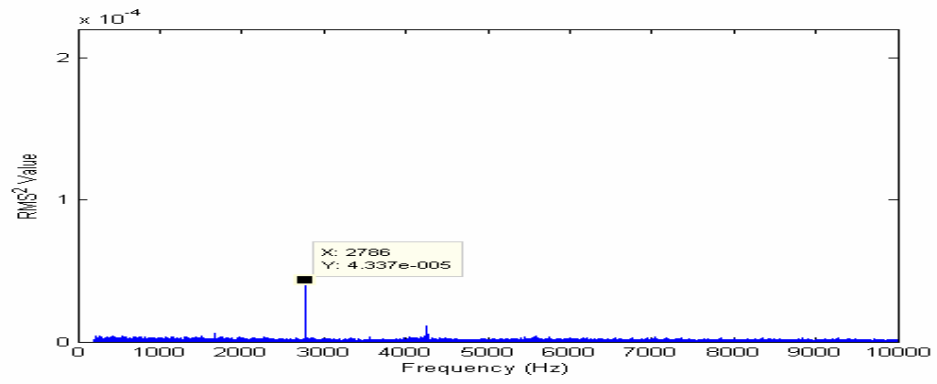


c)

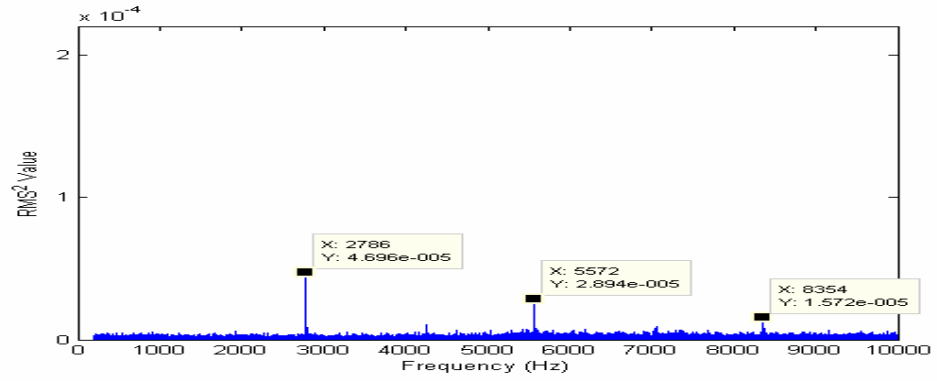


d)

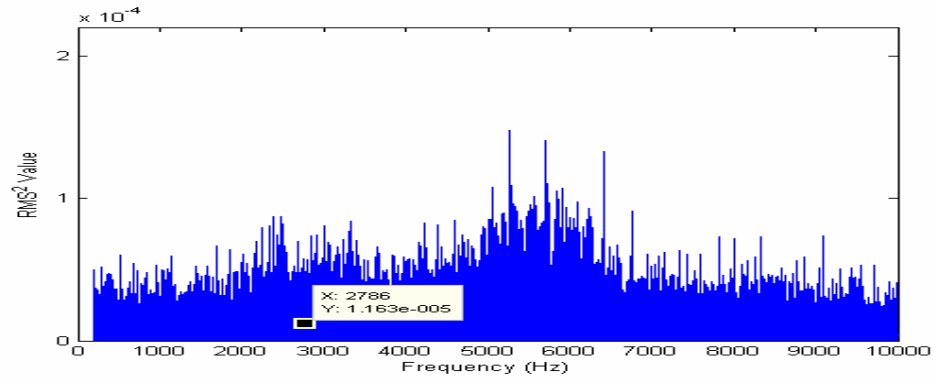
Figure 49. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d)PT5 for case run No. 2



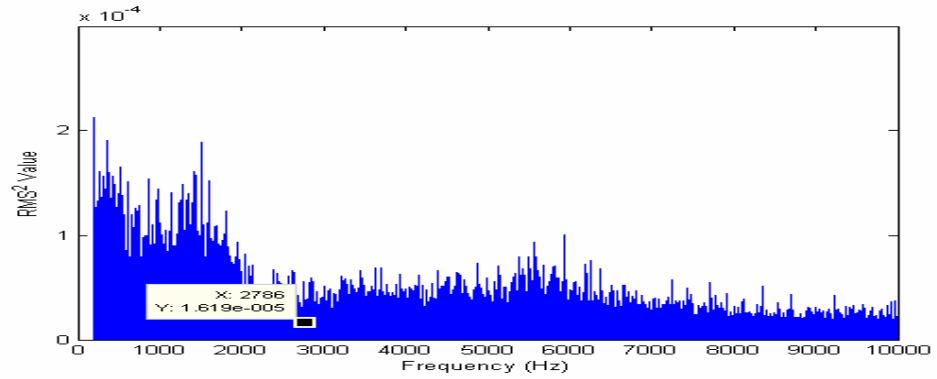
a)



b)



c)

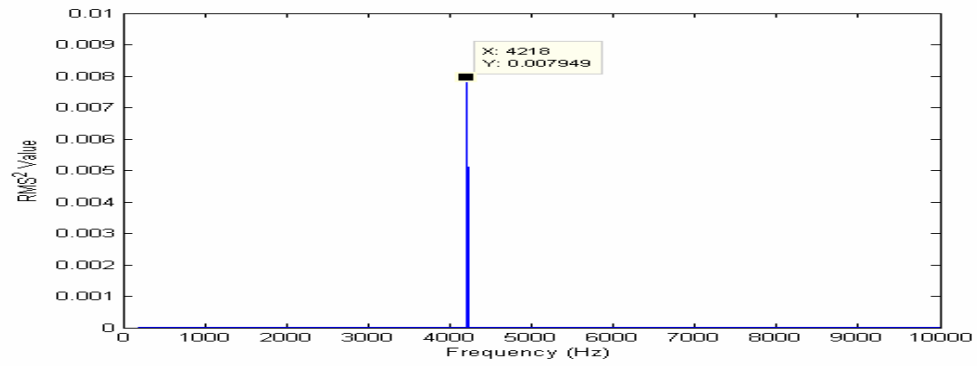


d)

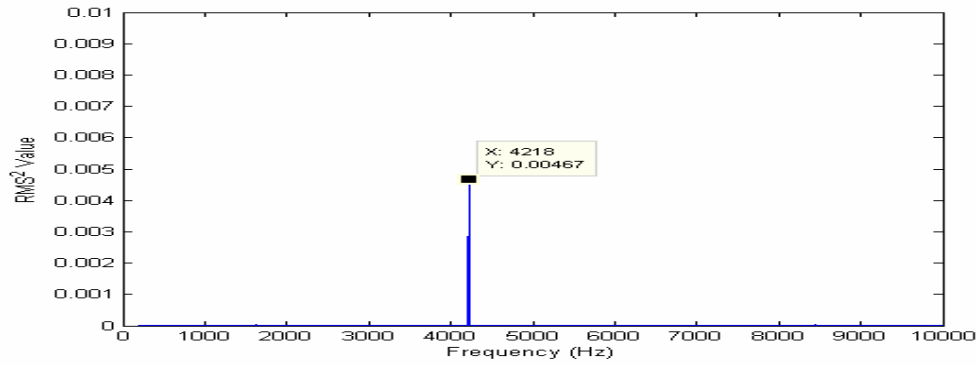
Figure 50. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 2



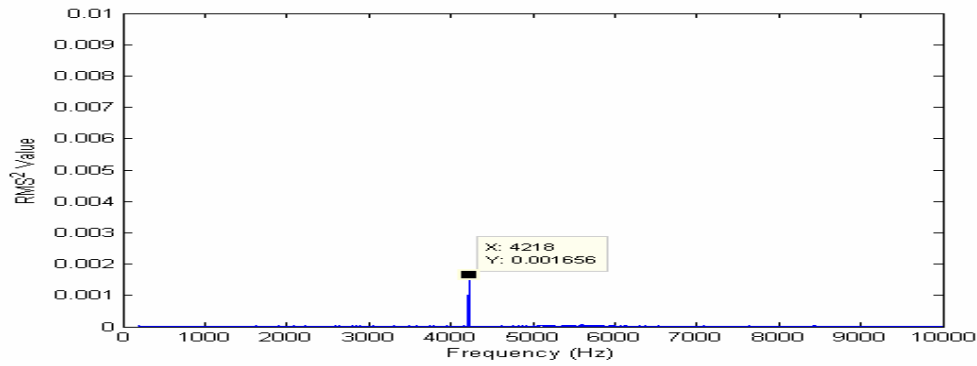
**Case Run3**



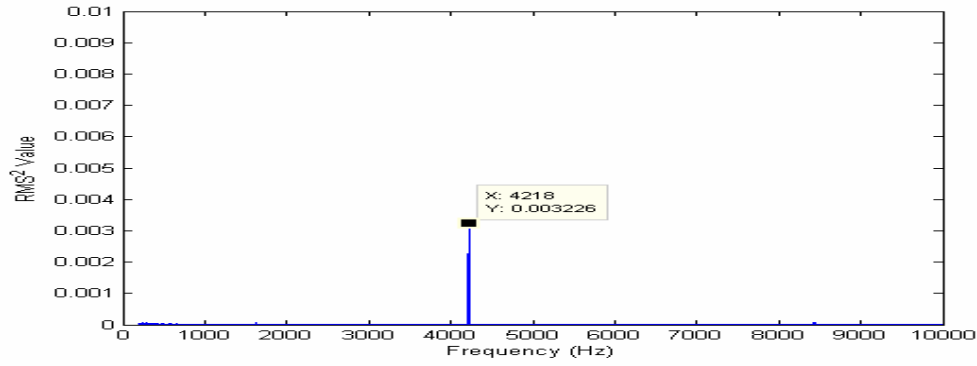
**a)**



**b)**

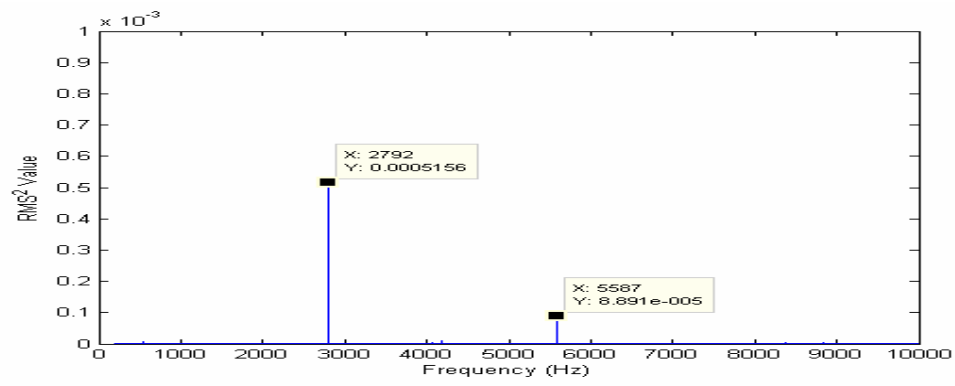


**c)**

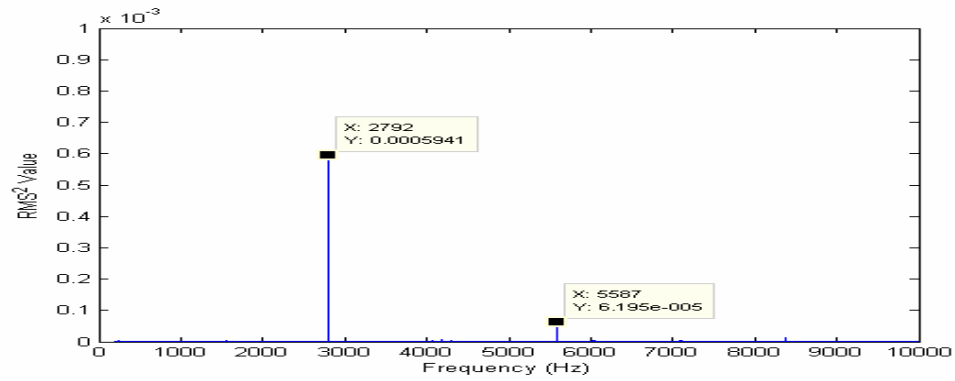


**d)**

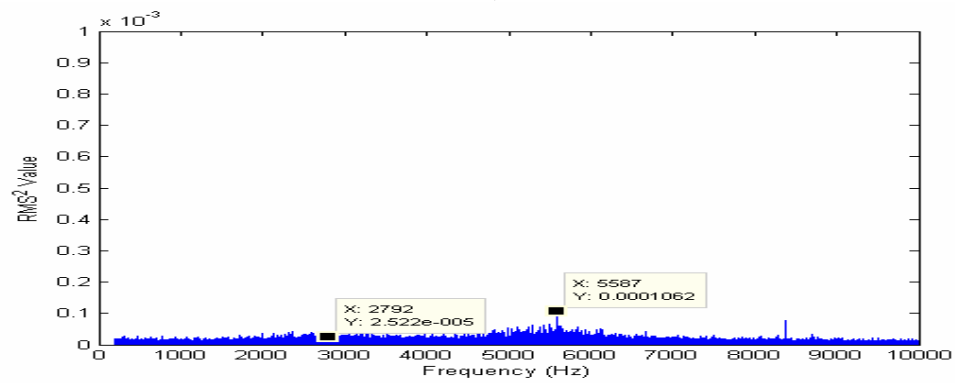
**Figure 51. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 3**



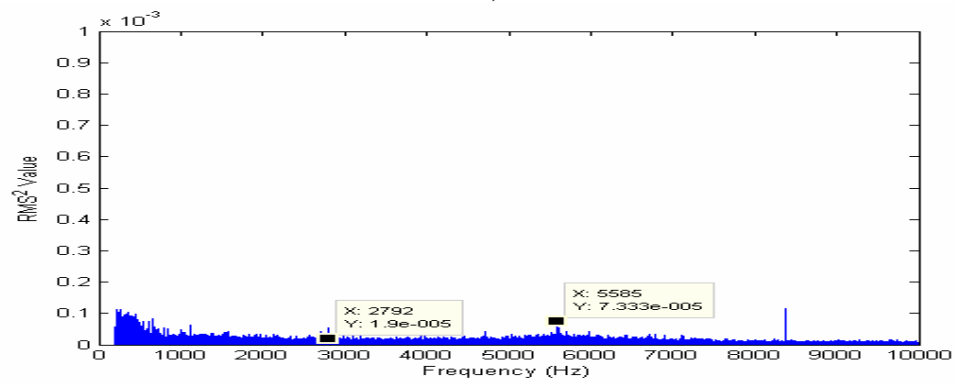
a)



b)

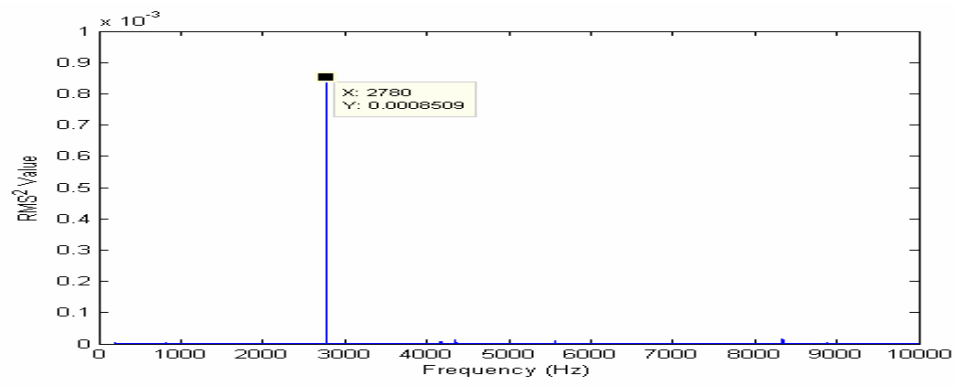


c)

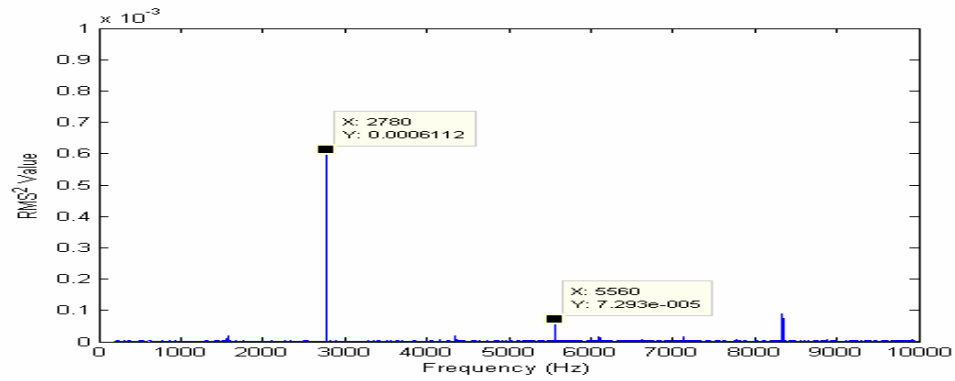


d)

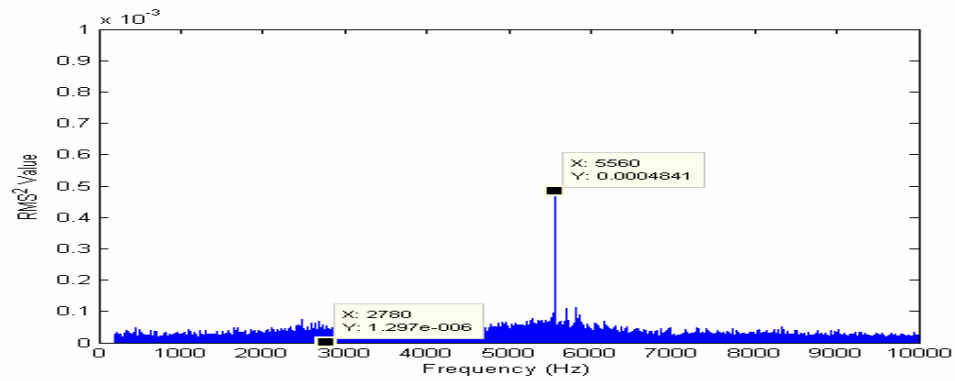
**Figure 52. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 3**



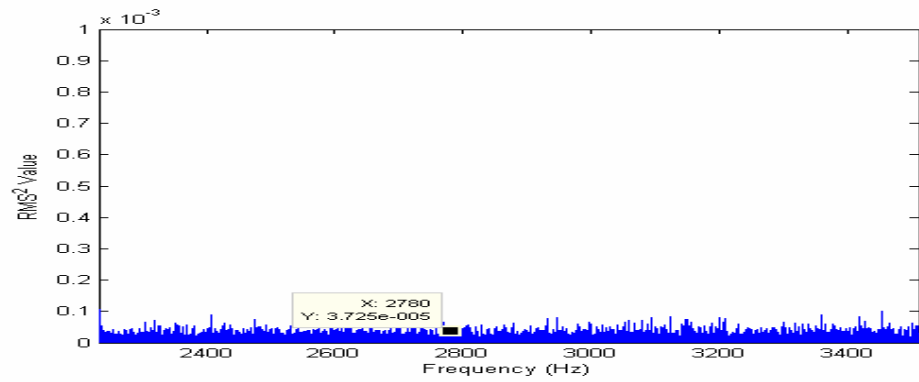
a)



b)

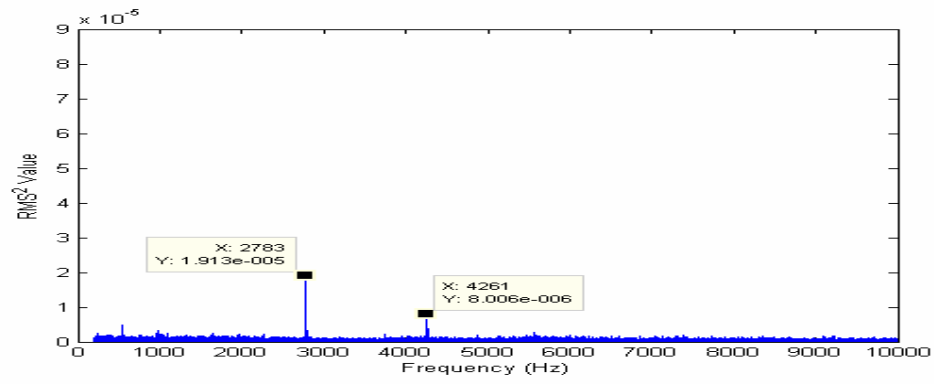


c)

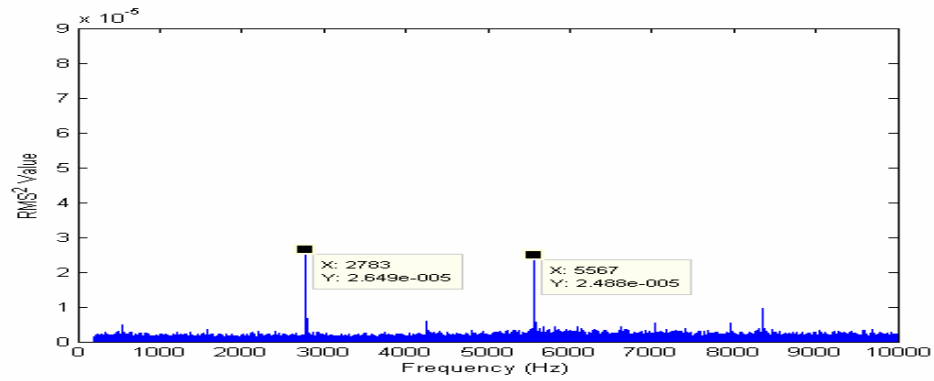


d)

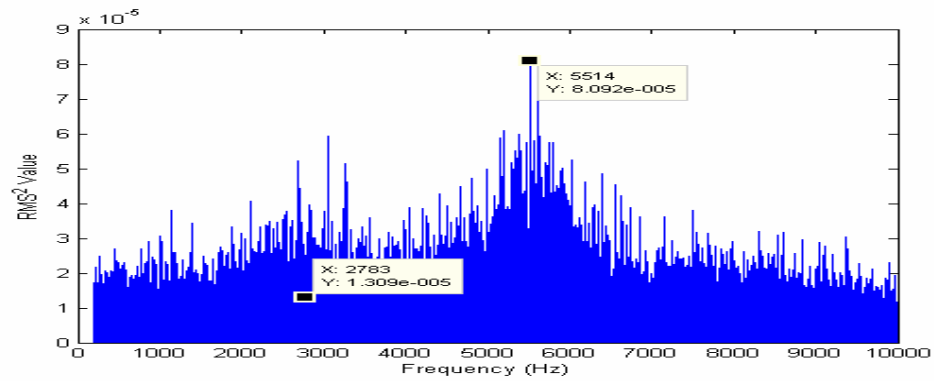
Figure 53. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 3



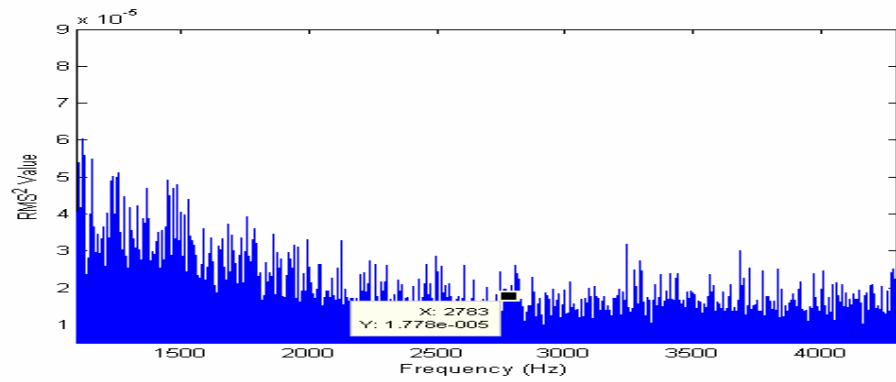
a)



b)



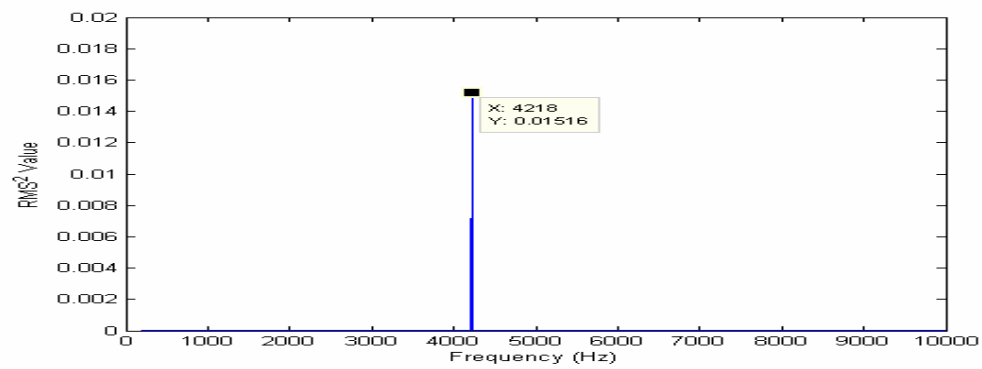
c)



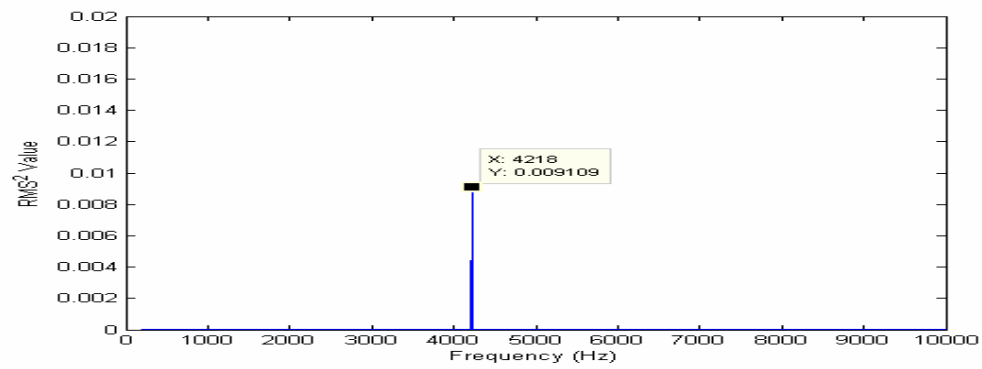
d)

Figure 54. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 3

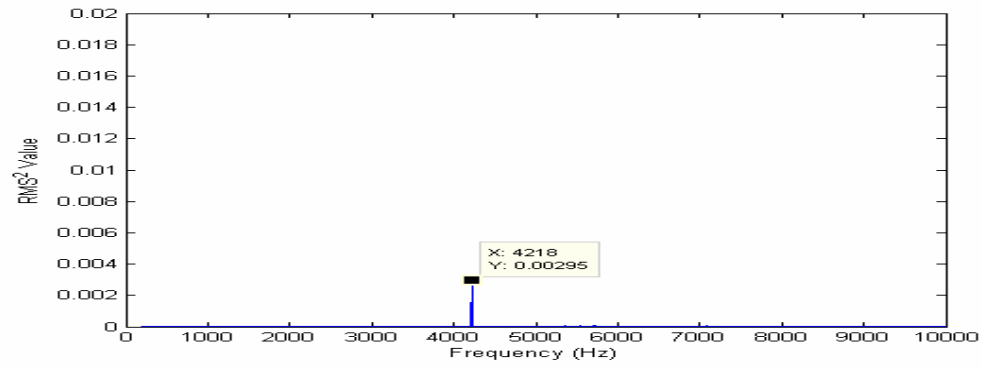
**Case Run4**



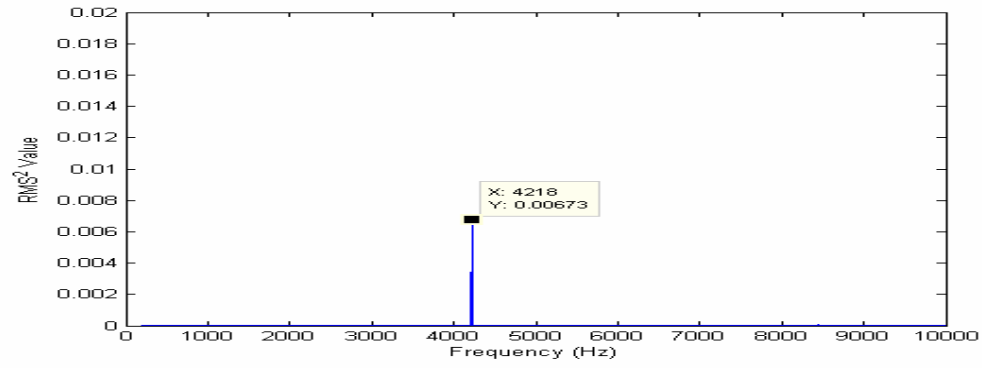
**a)**



**b)**

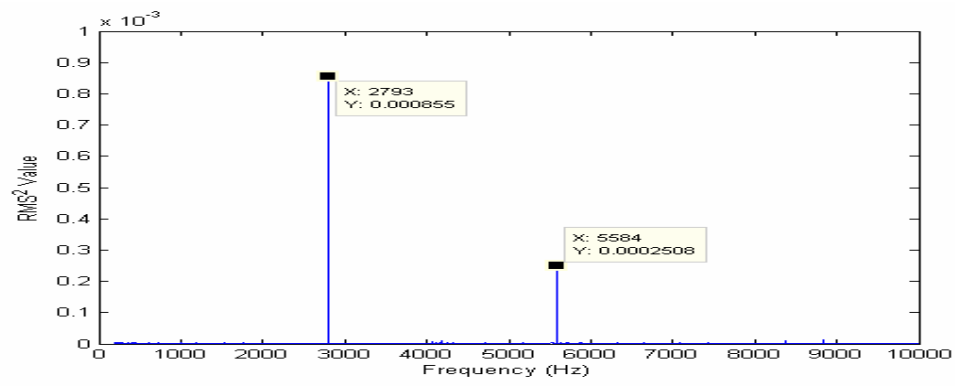


**c)**

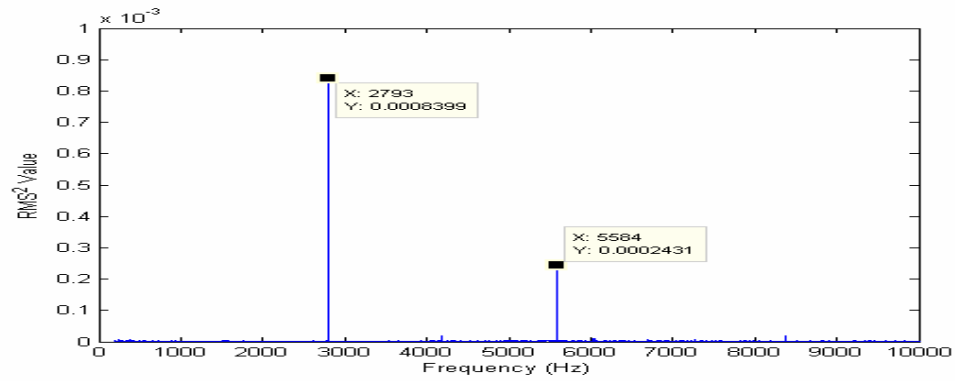


**d)**

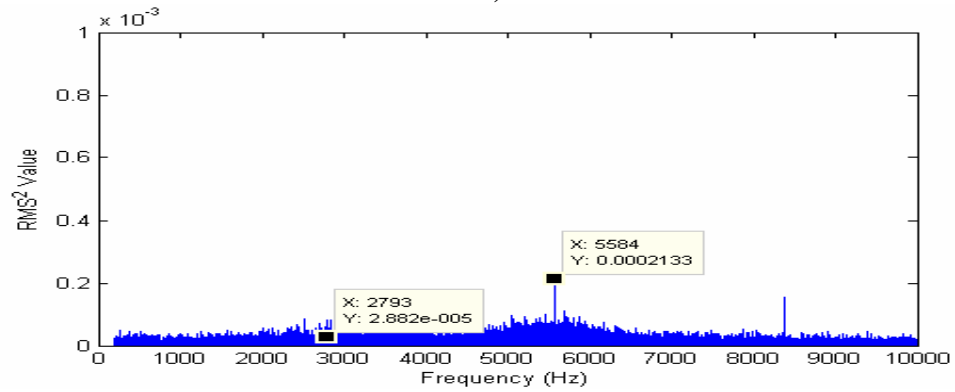
**Figure 55. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 4**



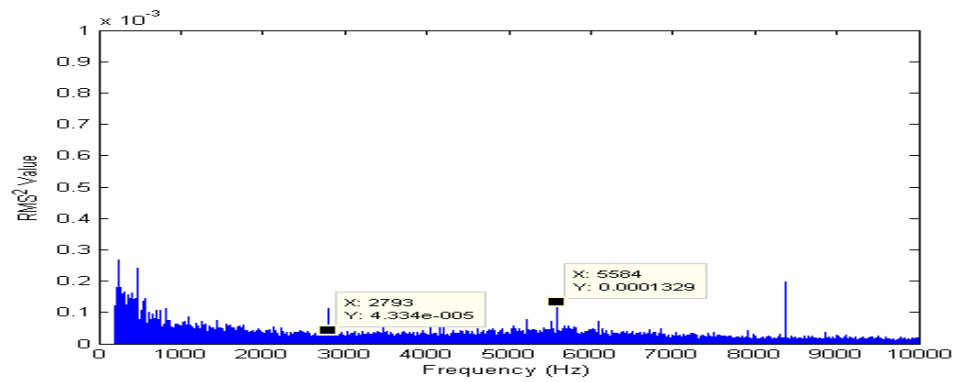
a)



b)

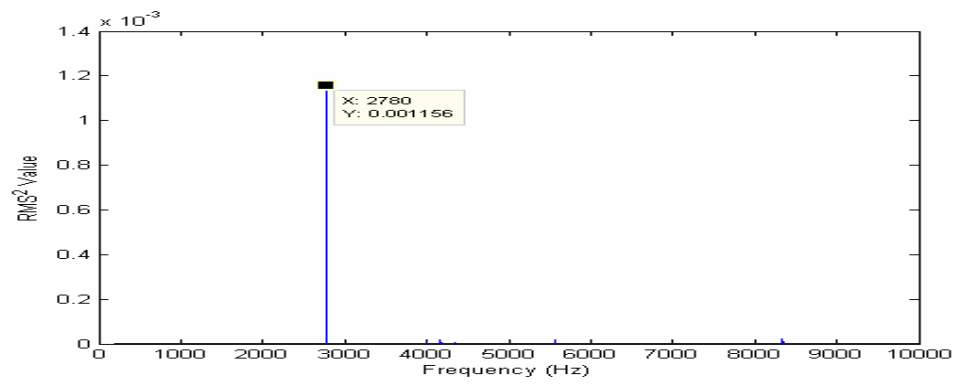


c)

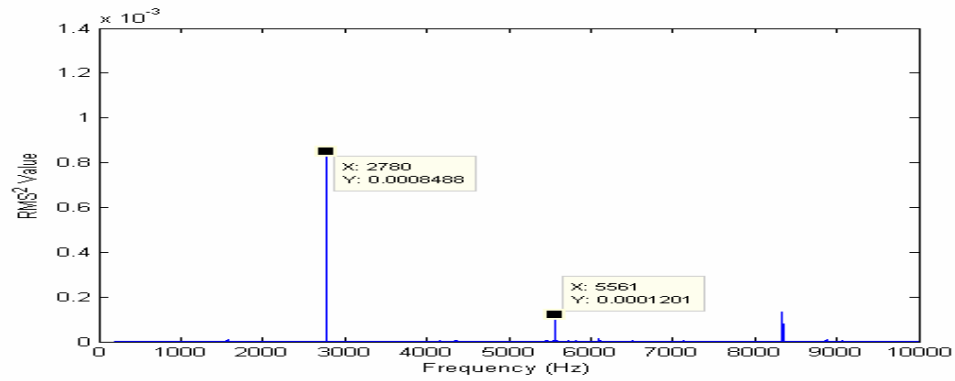


d)

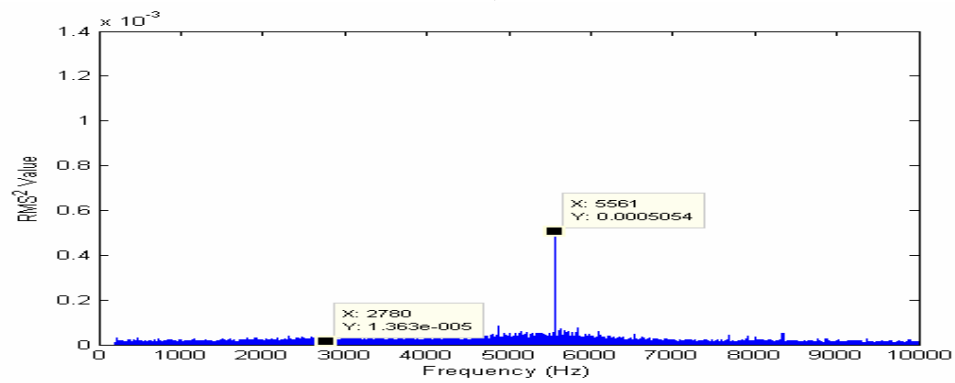
**Figure 56. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and PT5 for case run No. 4**



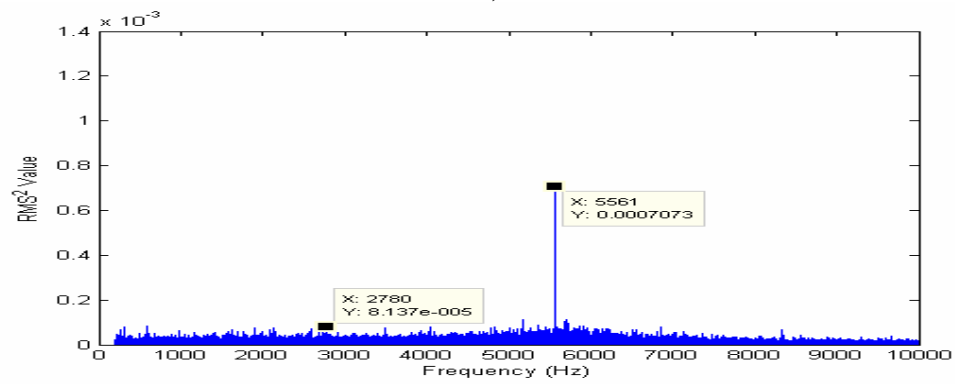
a)



b)

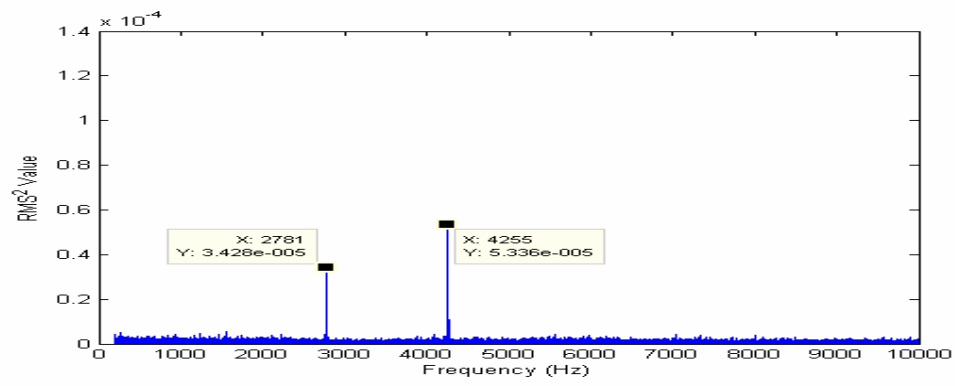


c)

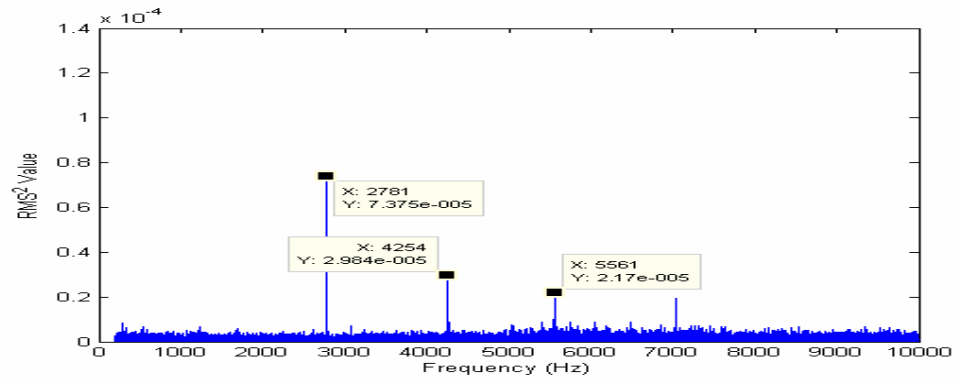


d)

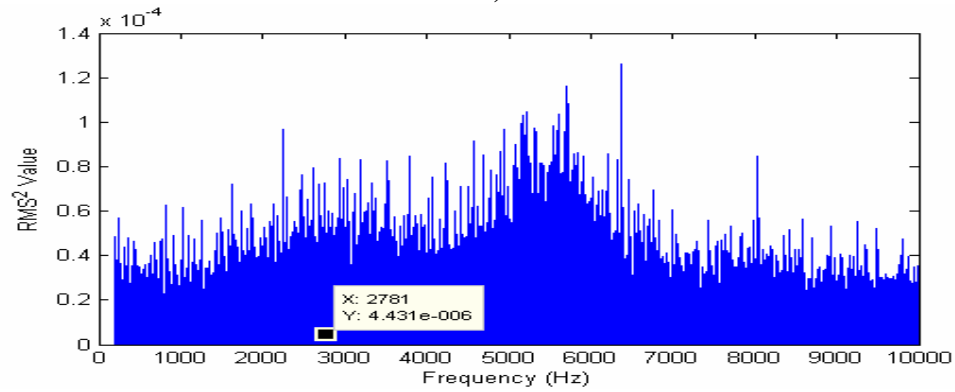
Figure 57. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 4



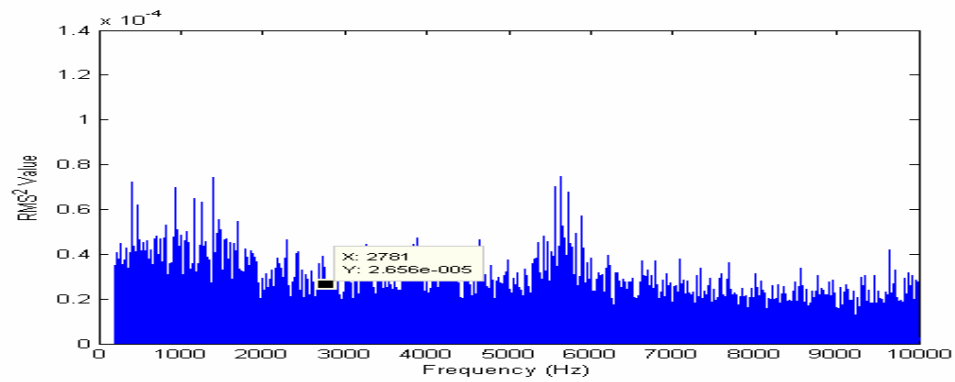
a)



b)



c)

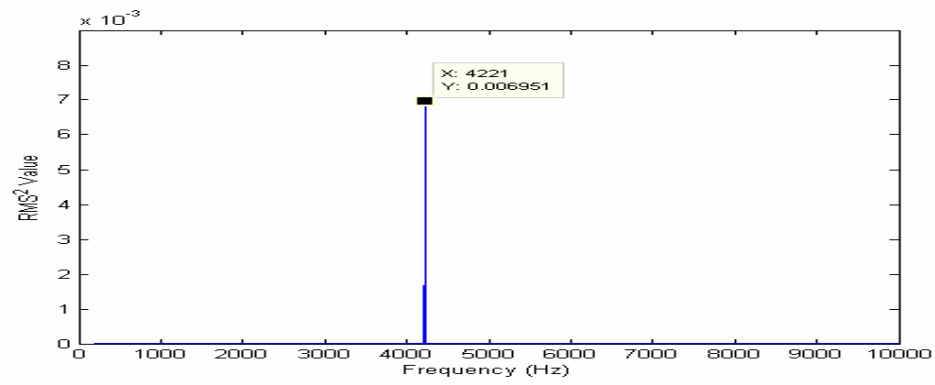


d)

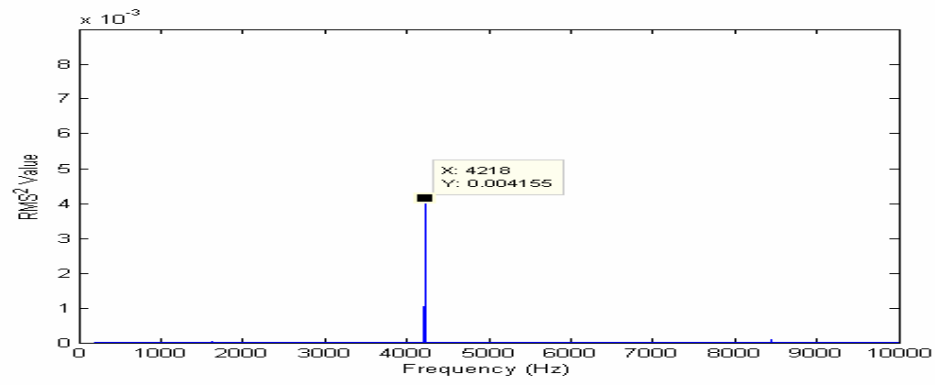
Figure 58. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 4



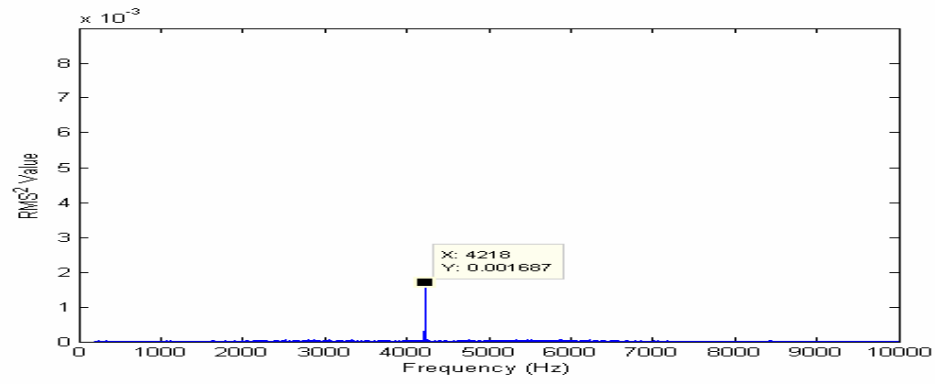
**Case Run5**



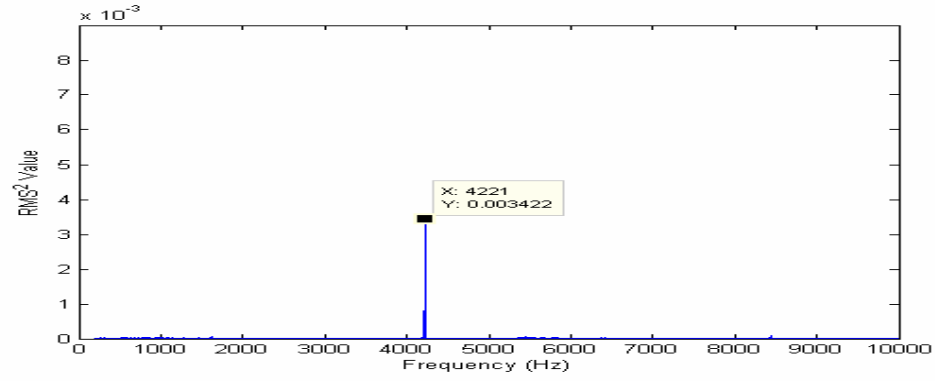
**a)**



**b)**

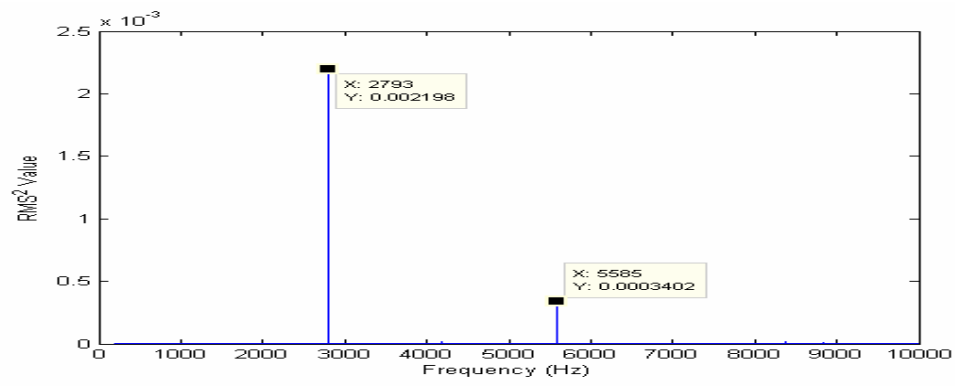


**c)**

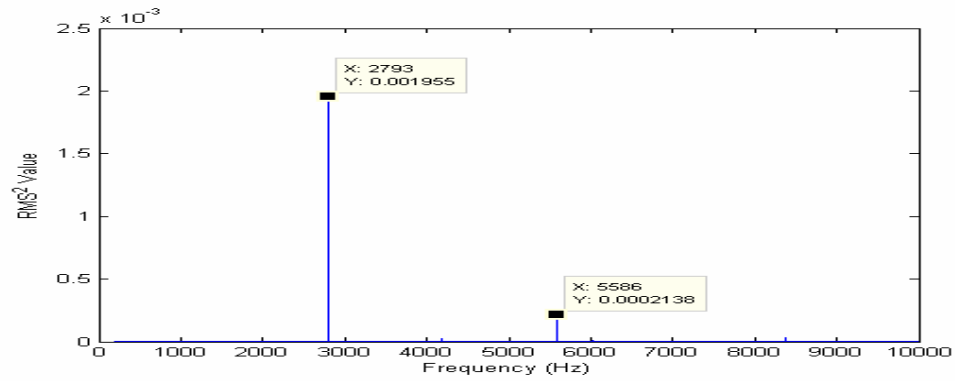


**d)**

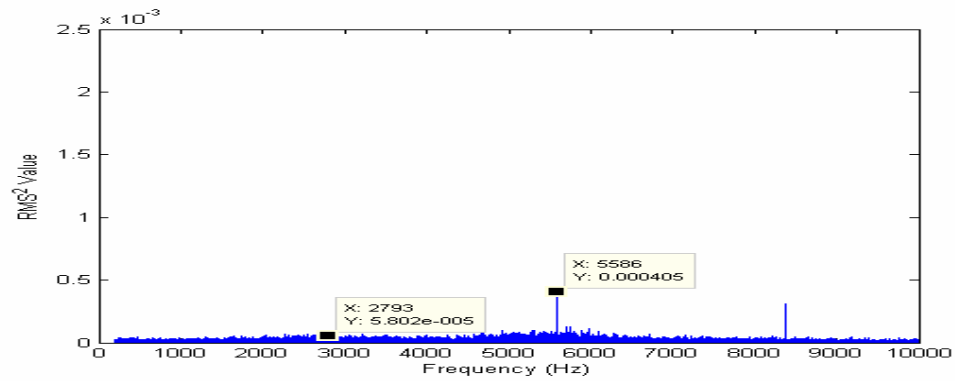
**Figure 59. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 5**



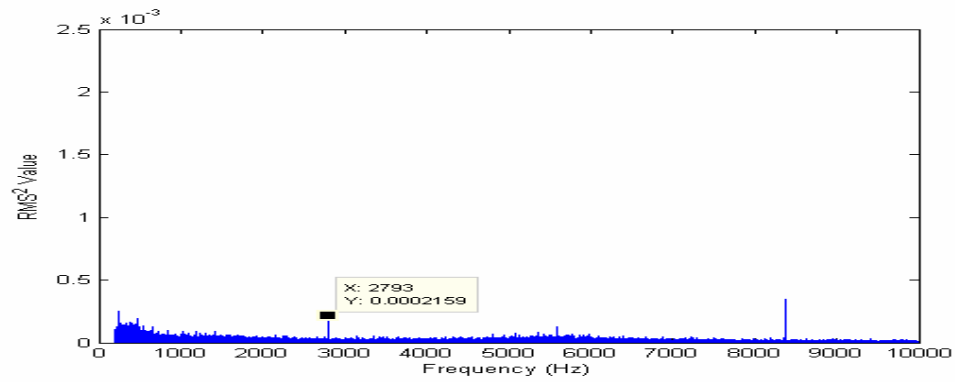
a)



b)

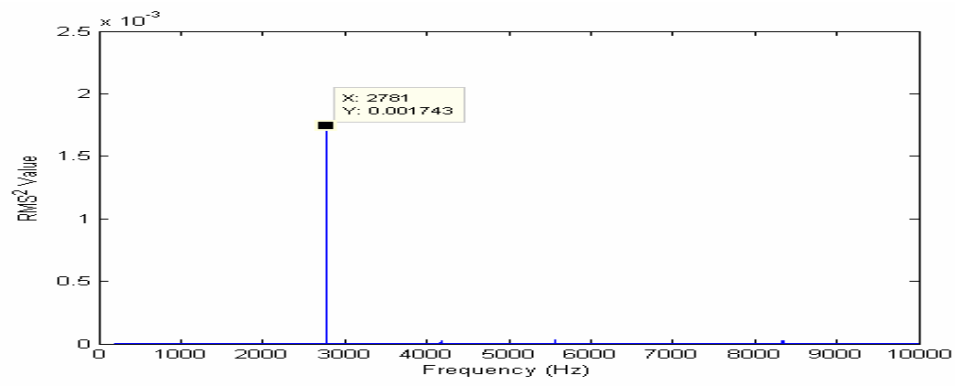


c)

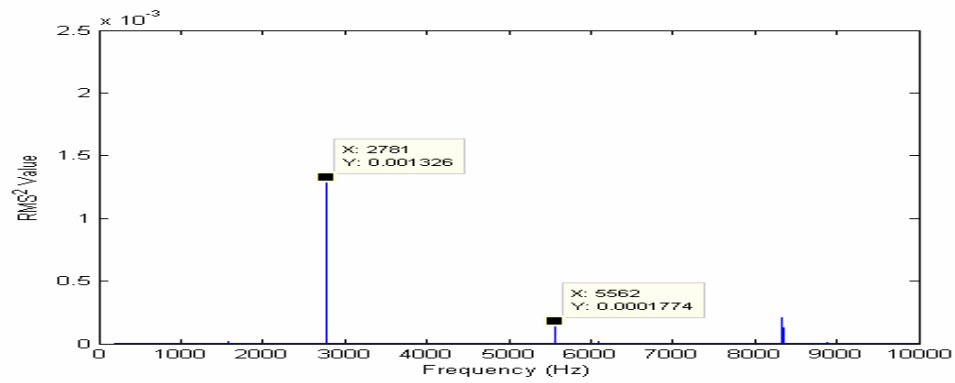


d)

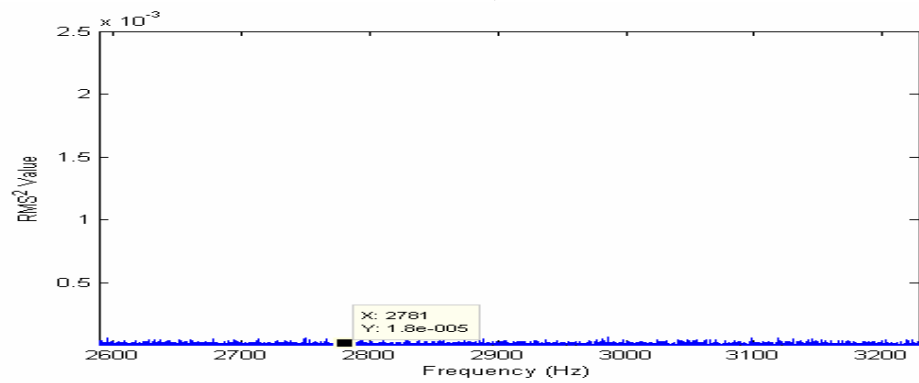
Figure 60. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 5



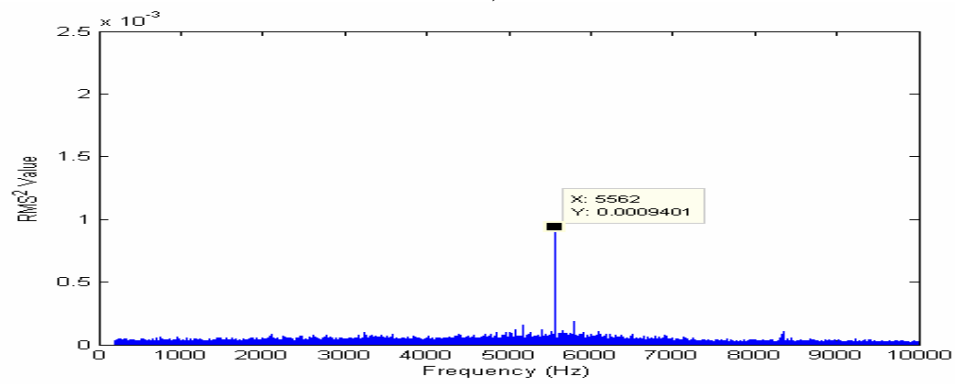
a)



b)

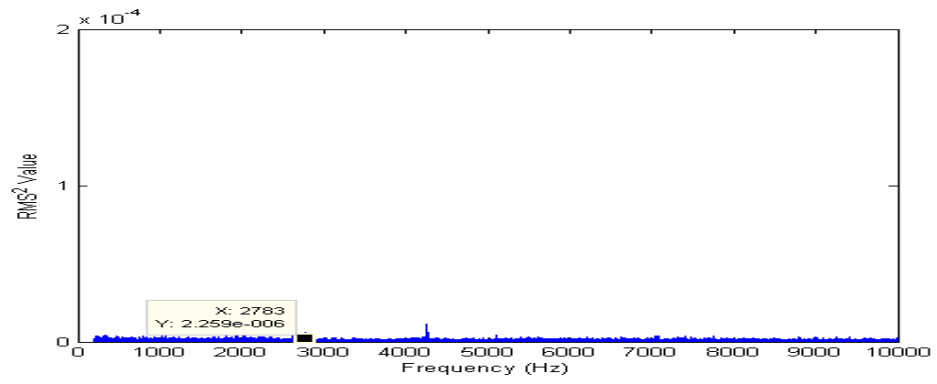


c)

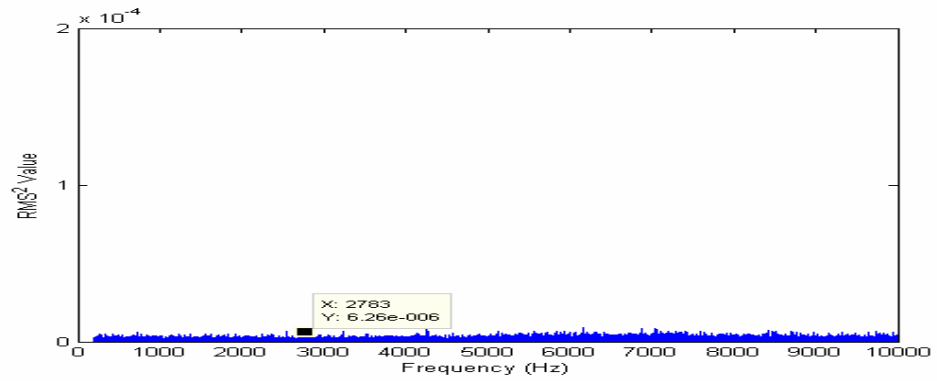


d)

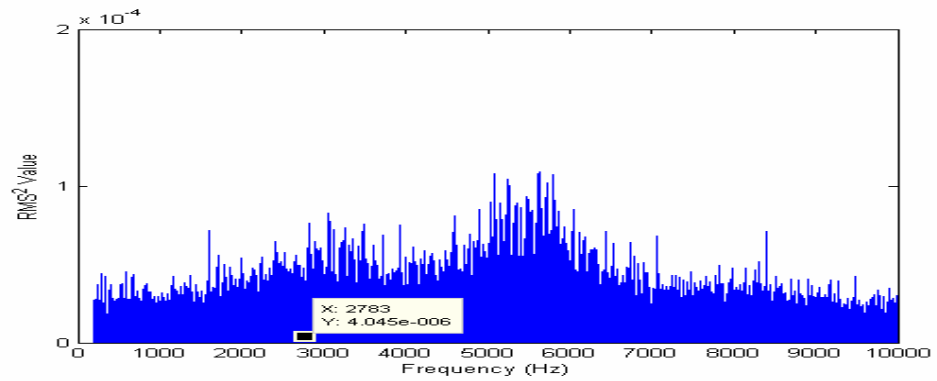
**Figure 61. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 5**



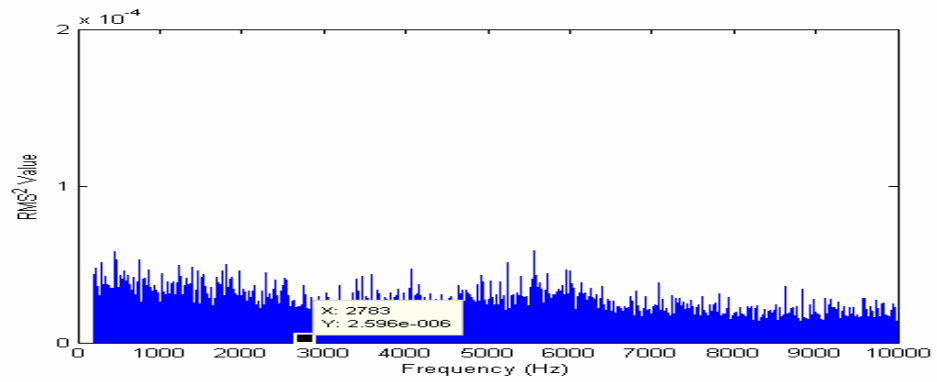
a)



b)



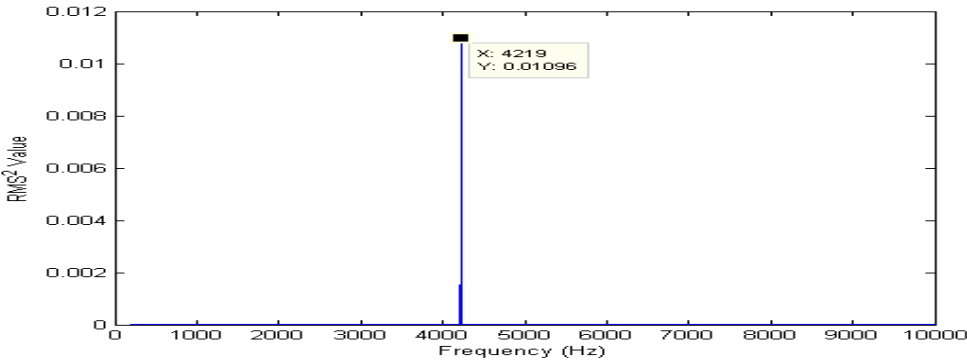
c)



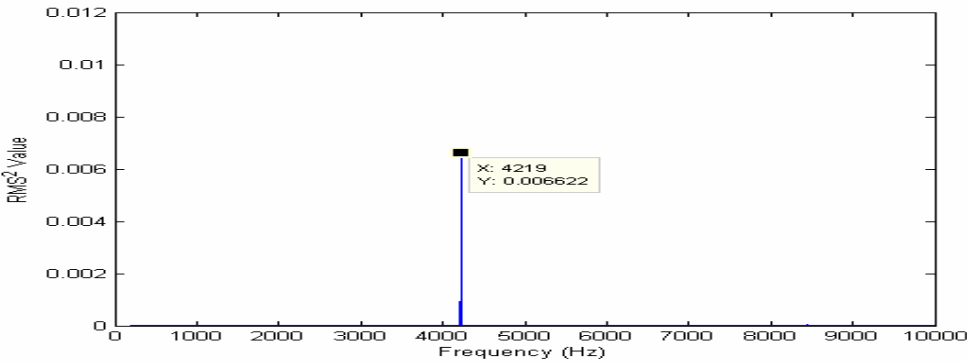
d)

**Figure 62. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 5**

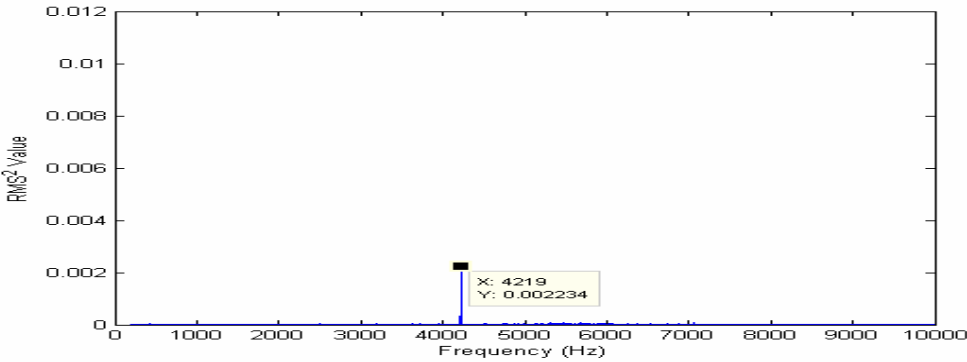
**Case Run6**



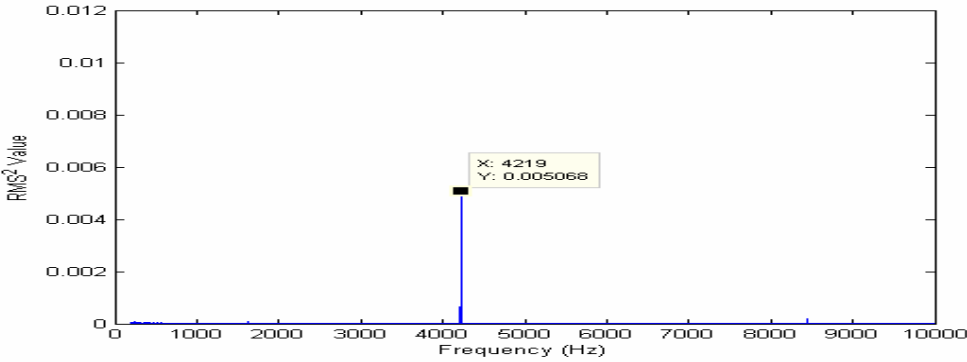
**a)**



**b)**

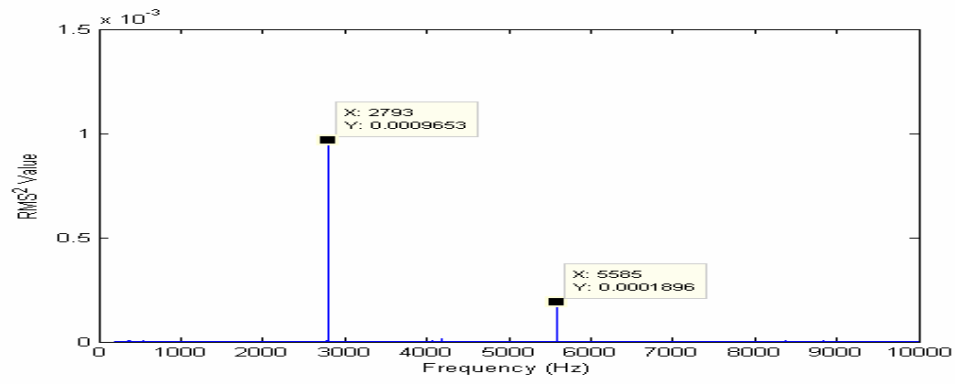


**c)**

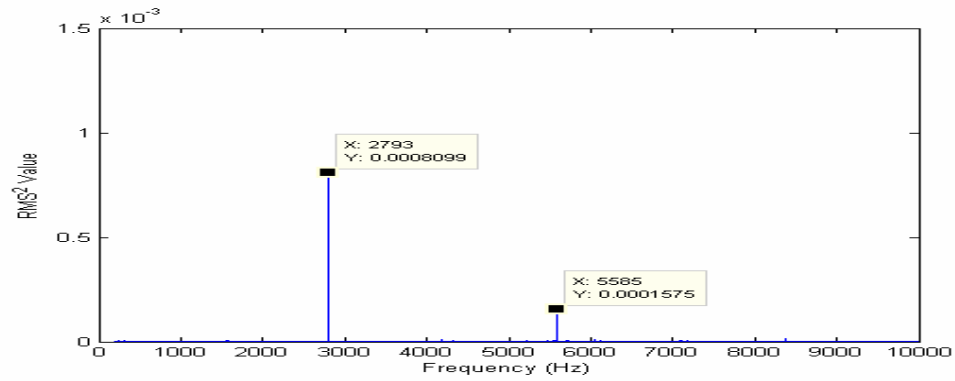


**d)**

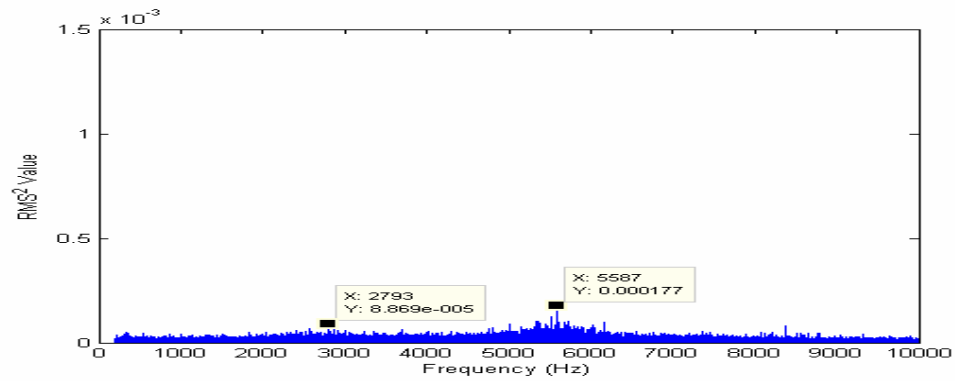
**Figure 63. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 6**



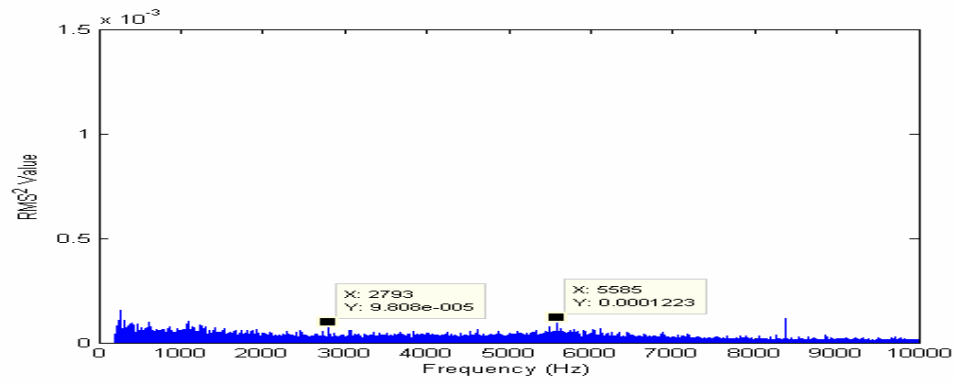
a)



b)

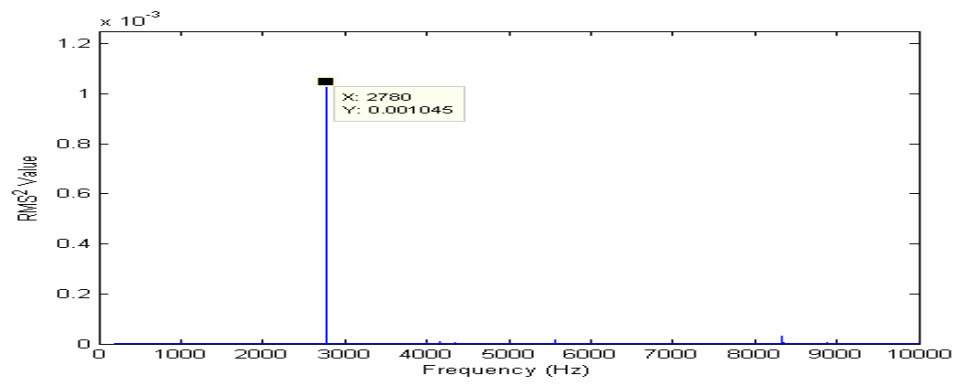


c)

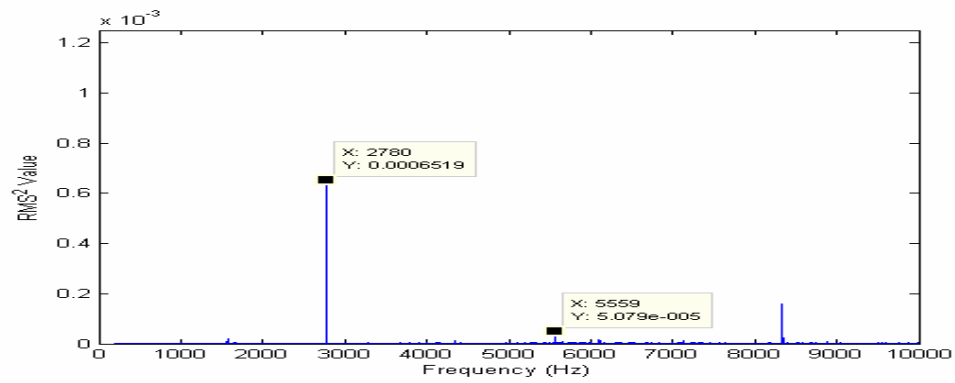


d)

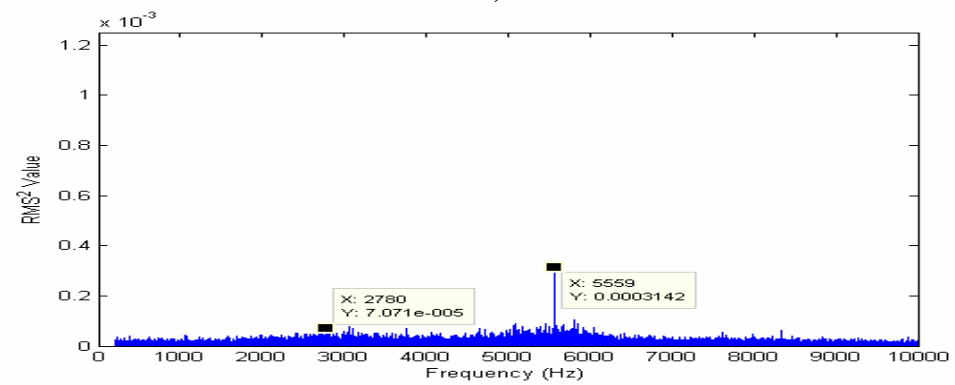
**Figure 64. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 6**



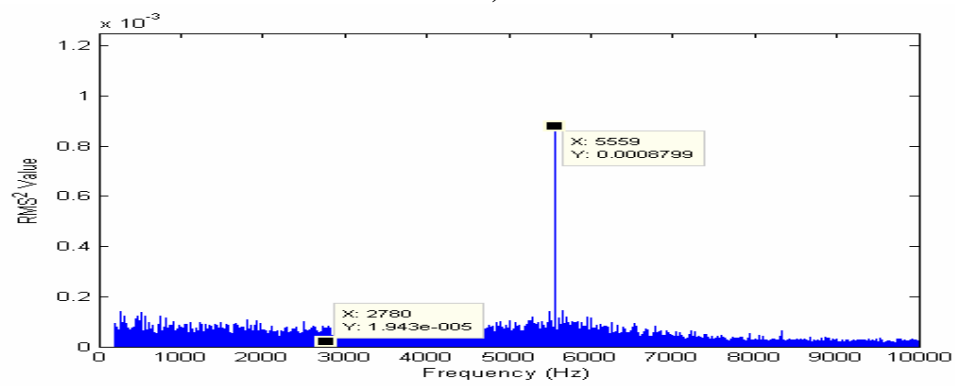
a)



b)

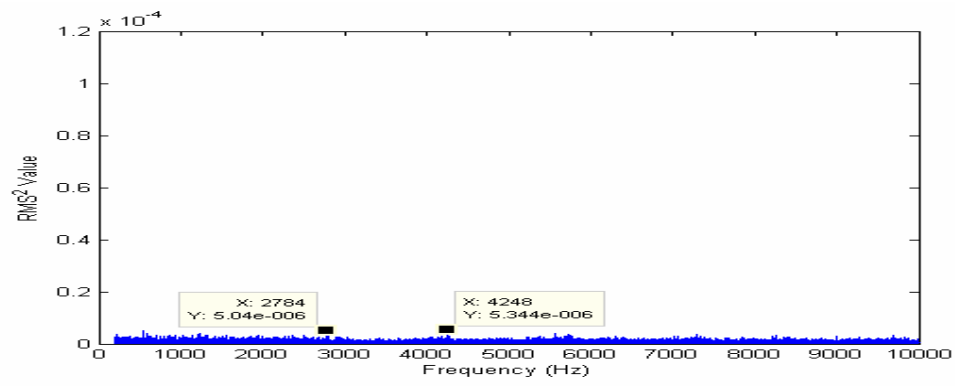


c)

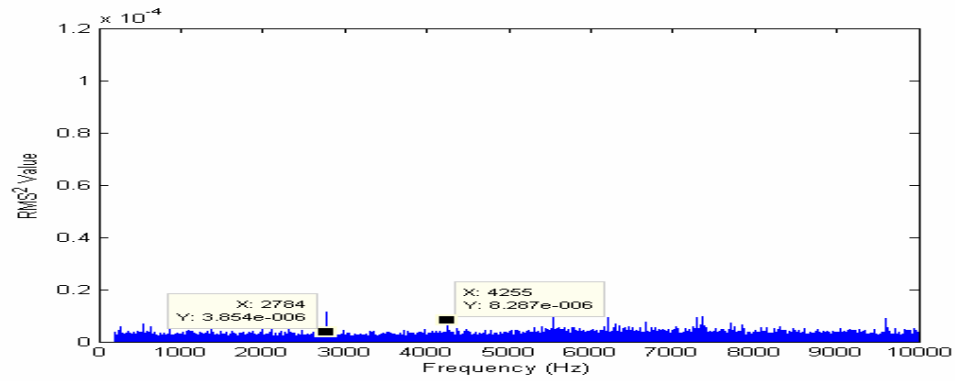


d)

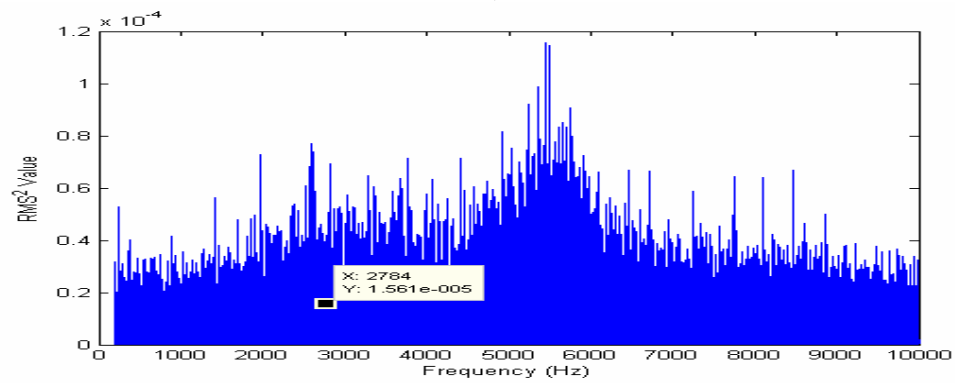
Figure 65. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 6



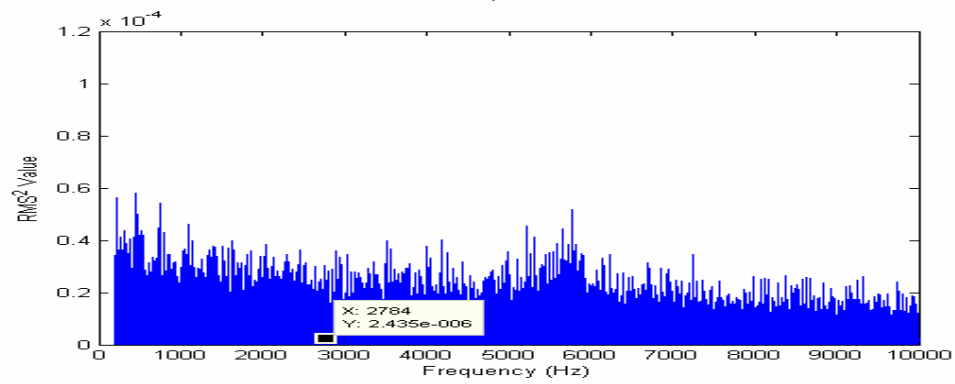
a)



b)



c)

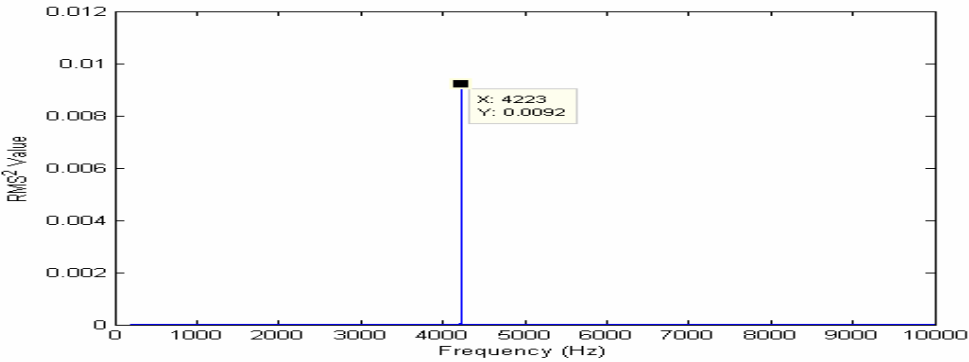


d)

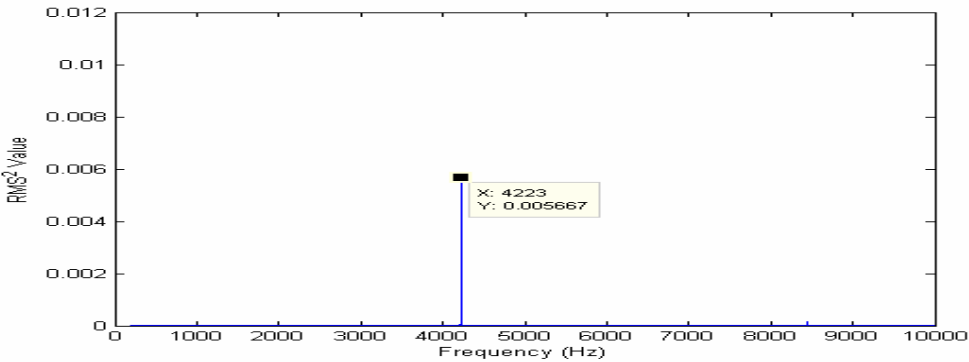
Figure 66. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 6



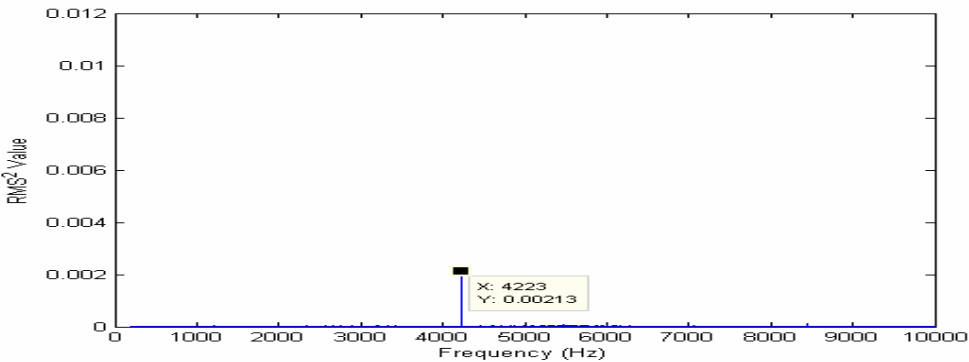
**Case Run7**



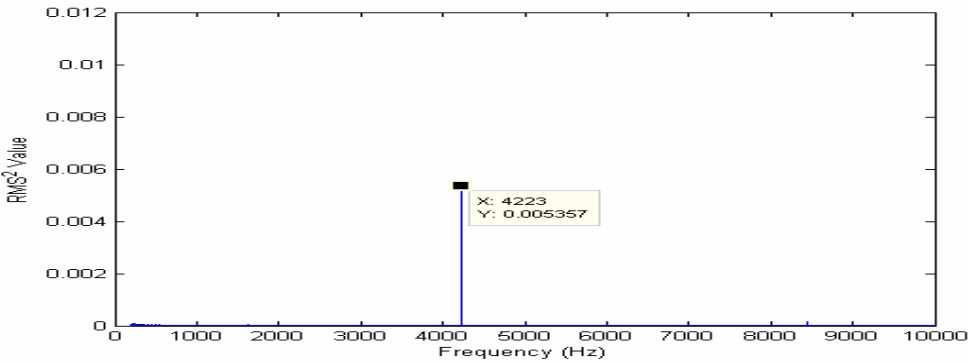
**a)**



**b)**

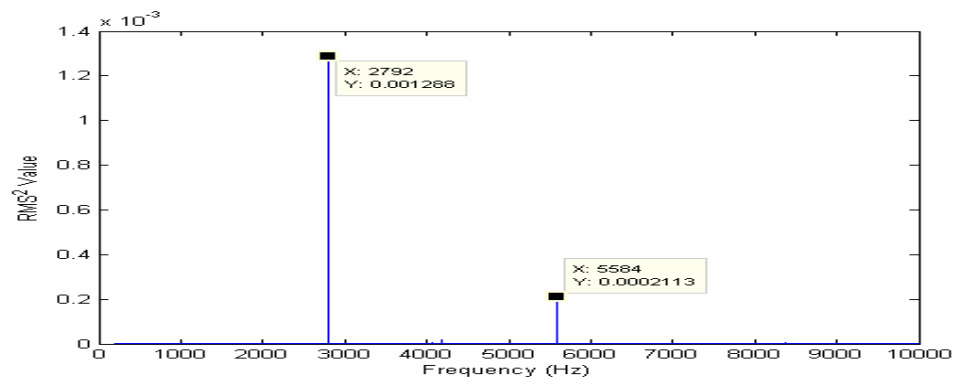


**c)**

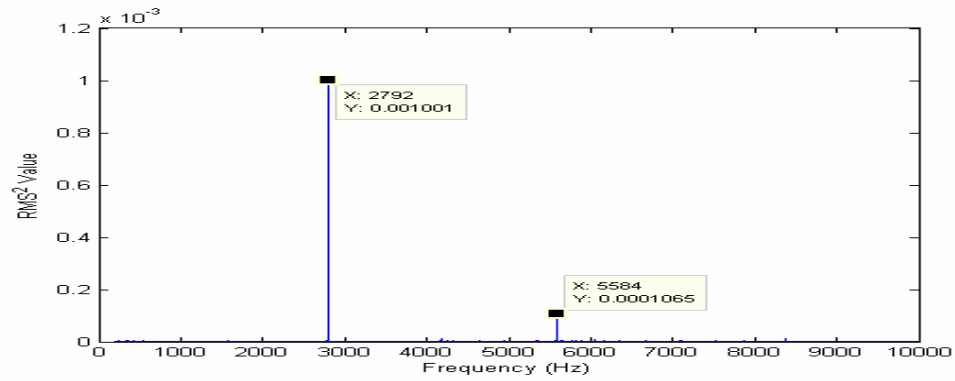


**d)**

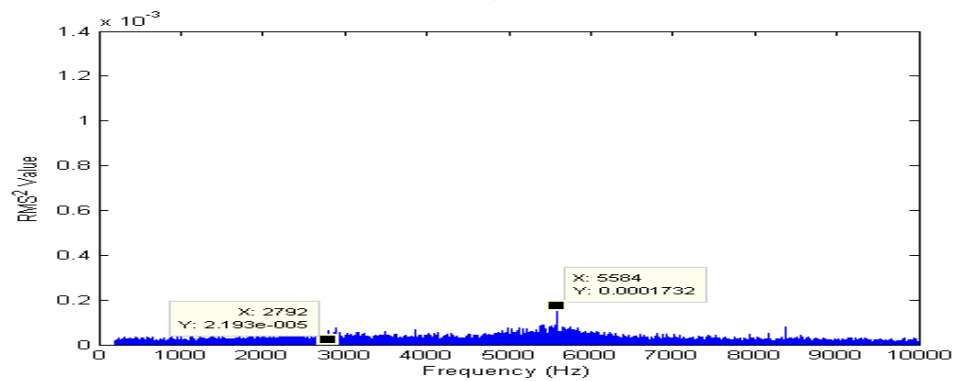
**Figure 67. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 7**



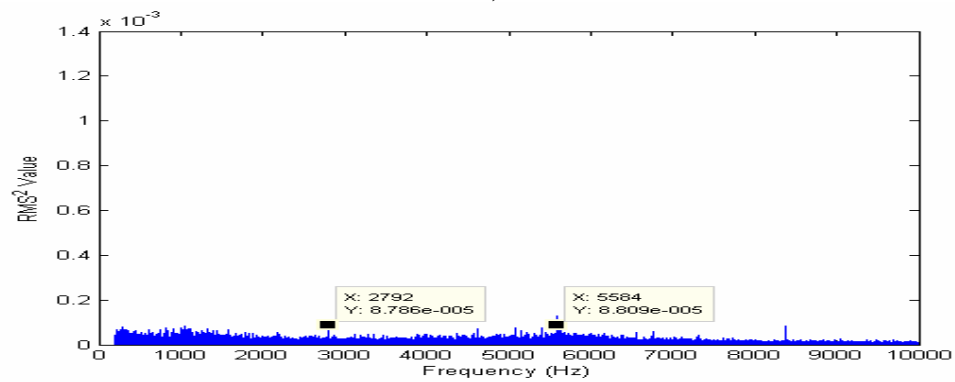
a)



b)

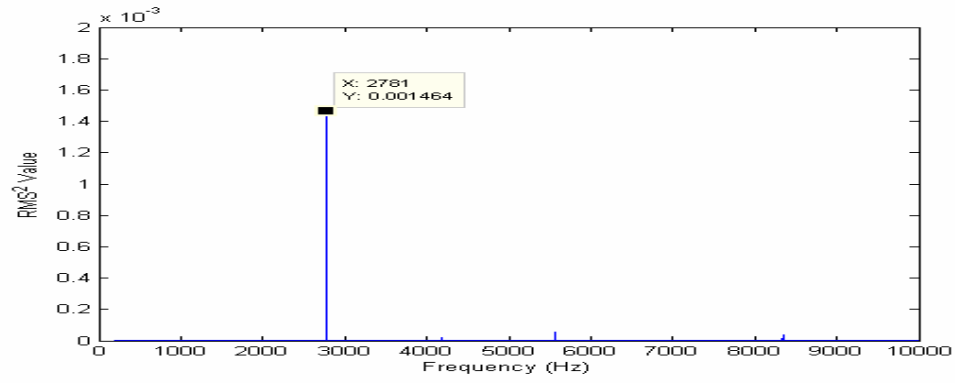


c)

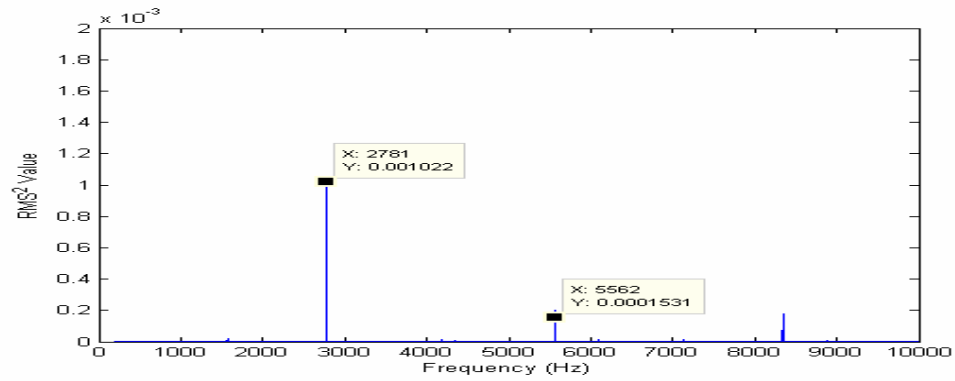


d)

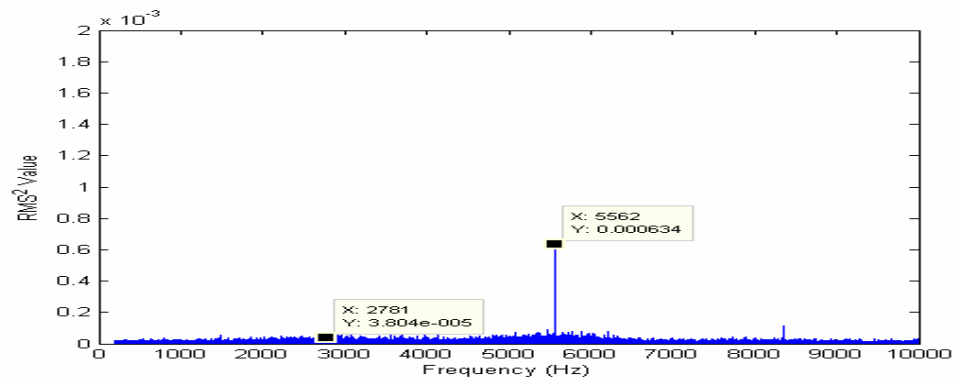
**Figure 68. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 7**



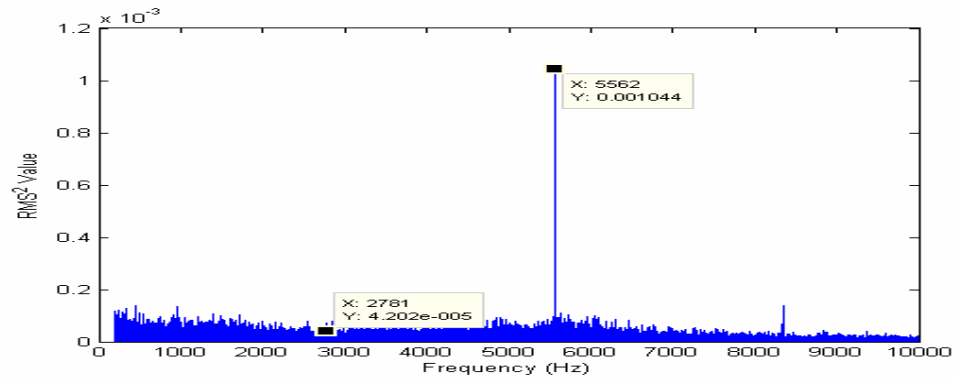
a)



b)

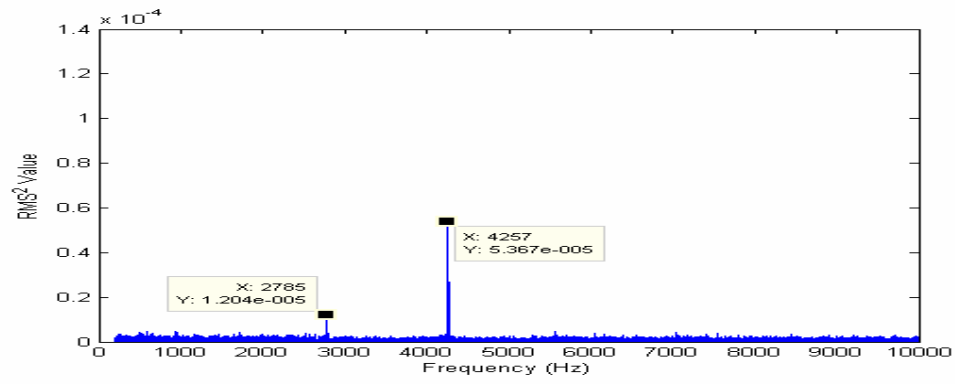


c)

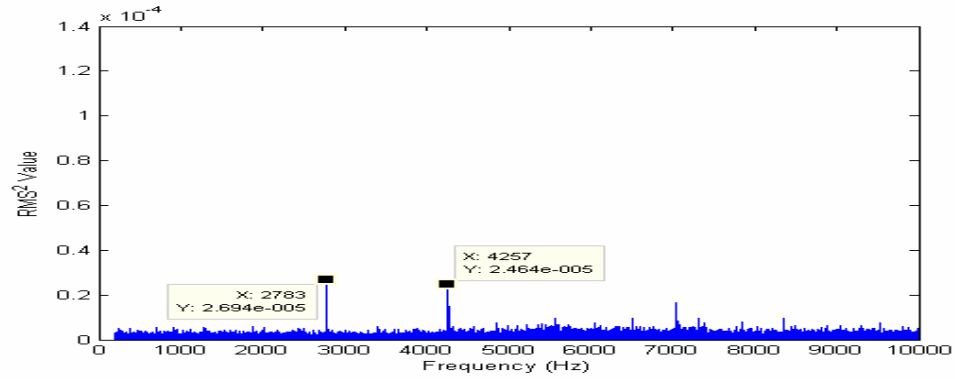


d)

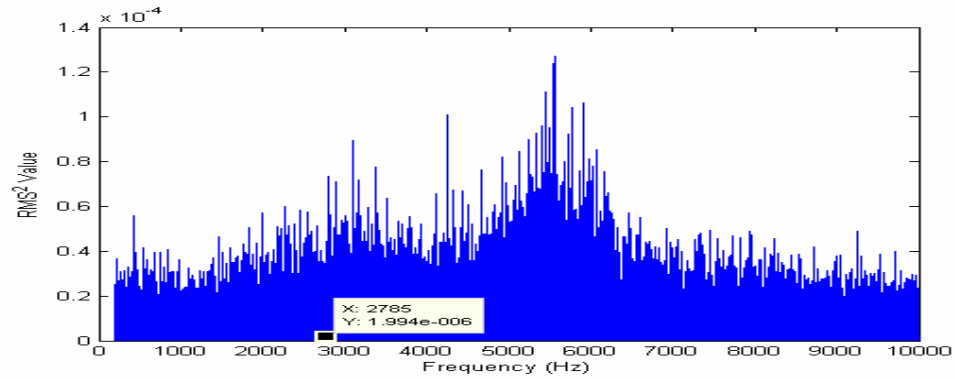
**Figure 69. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 7**



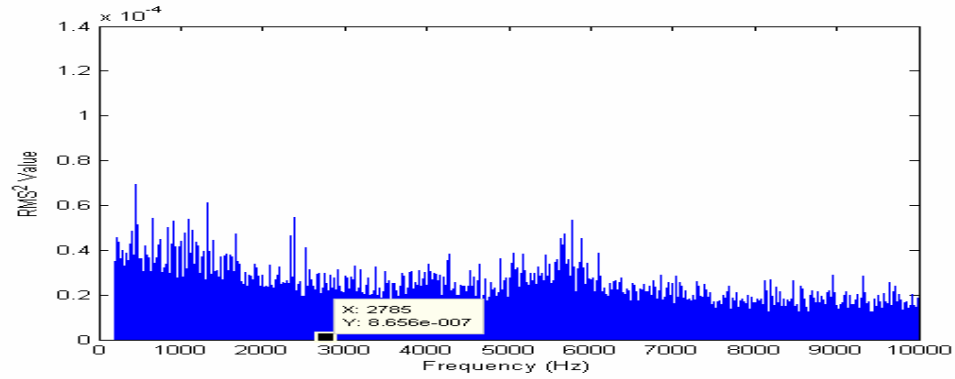
a)



b)



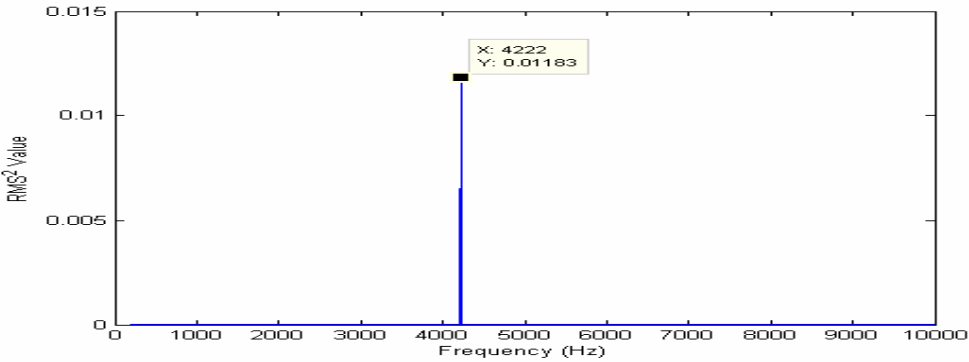
c)



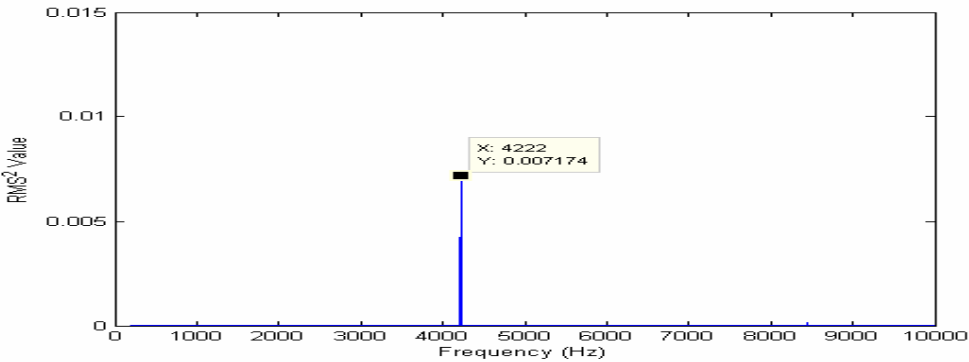
d)

**Figure 70. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 7**

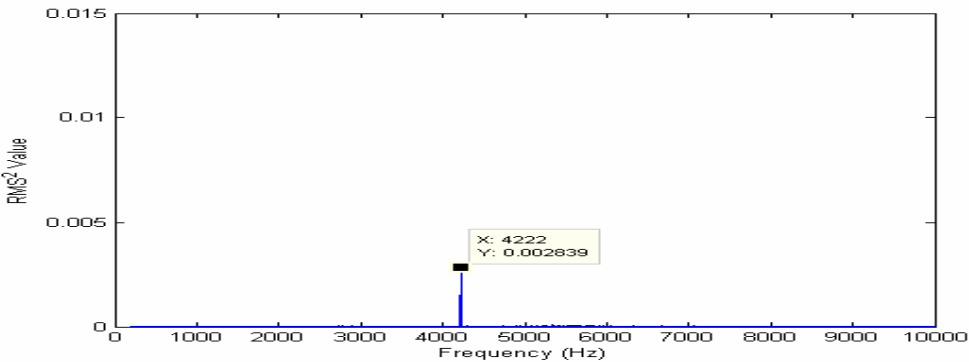
**Case Run8**



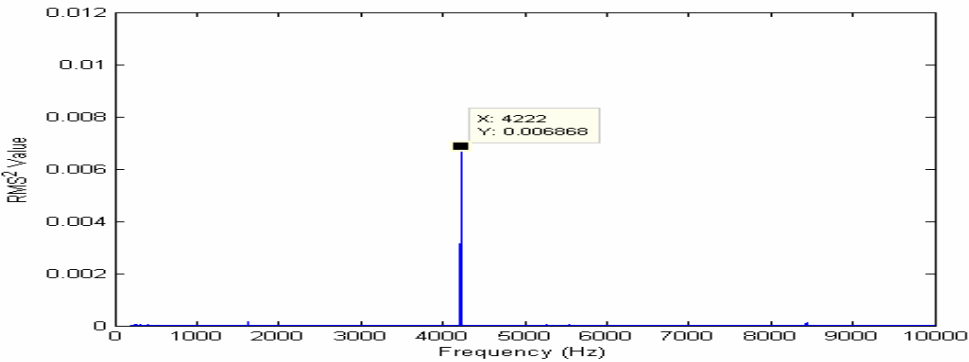
**a)**



**b)**

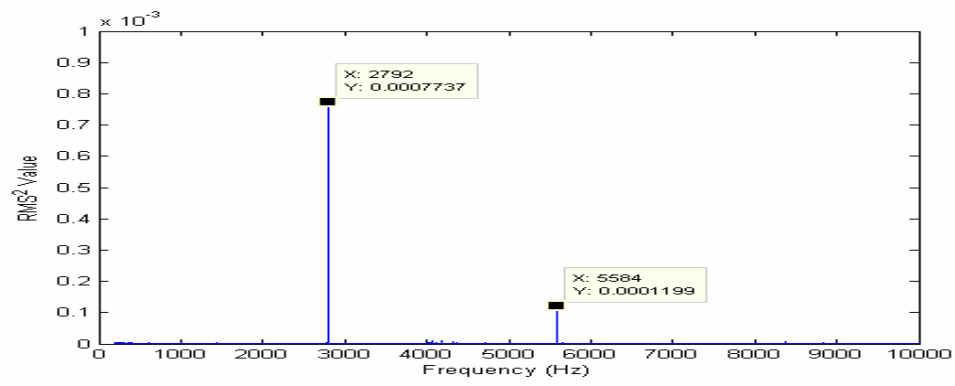


**c)**

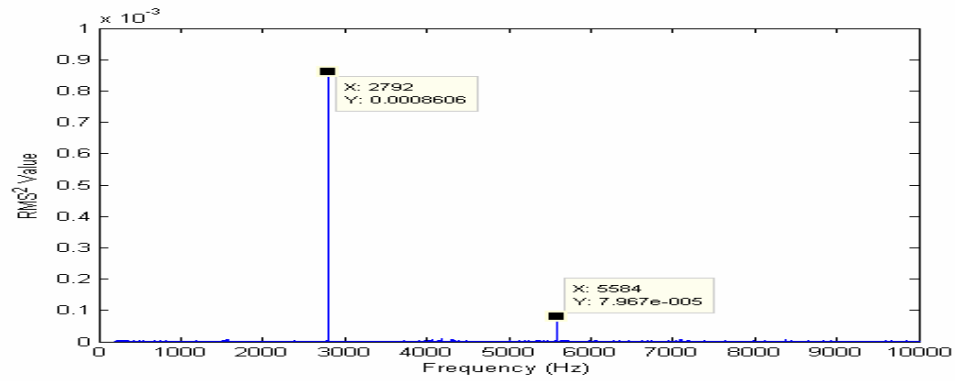


**d)**

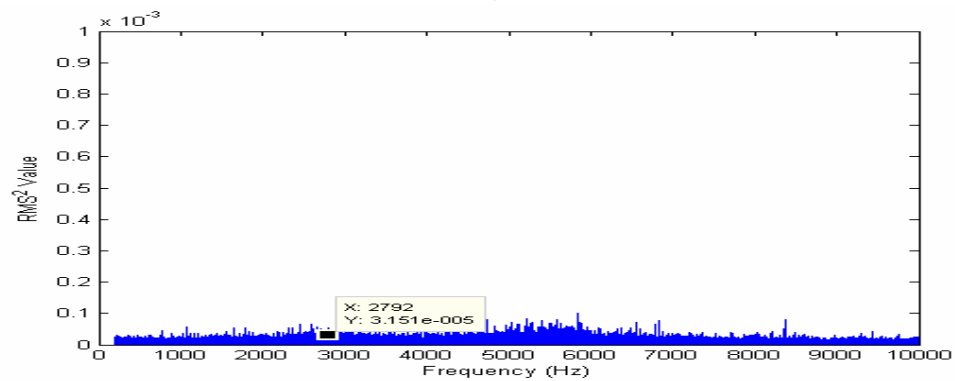
**Figure 71. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 8**



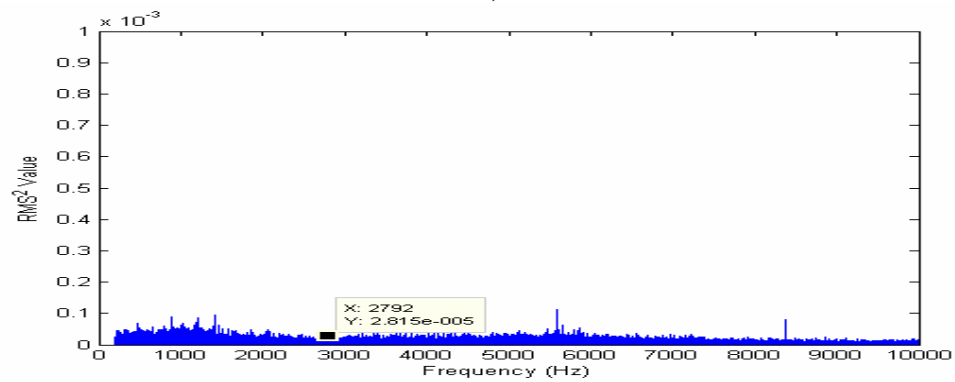
a)



b)

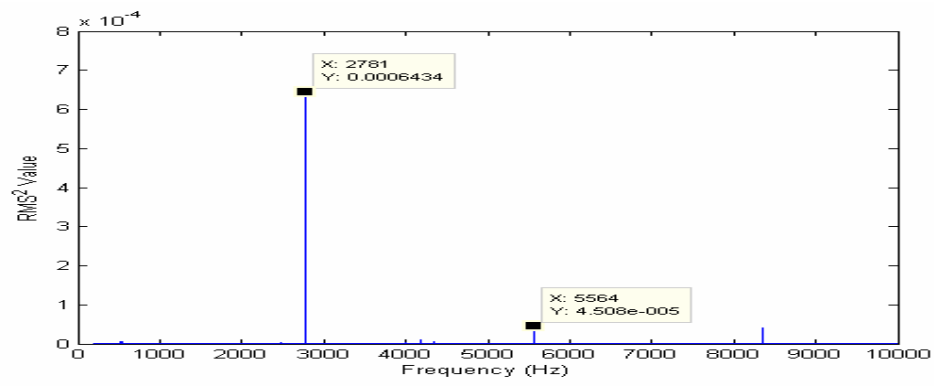


c)

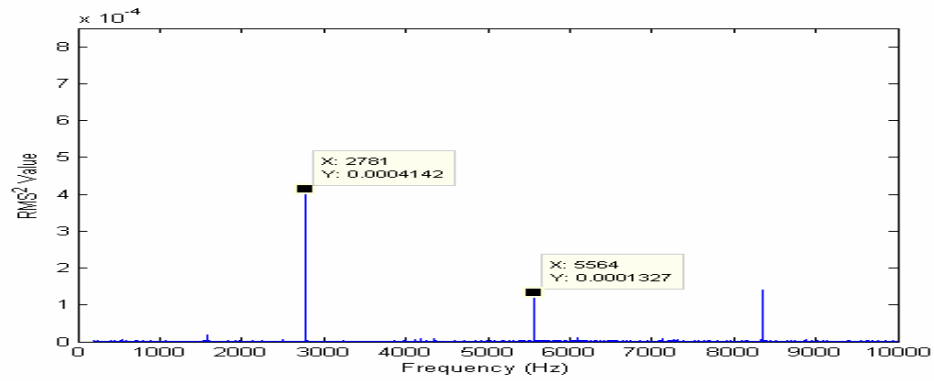


d)

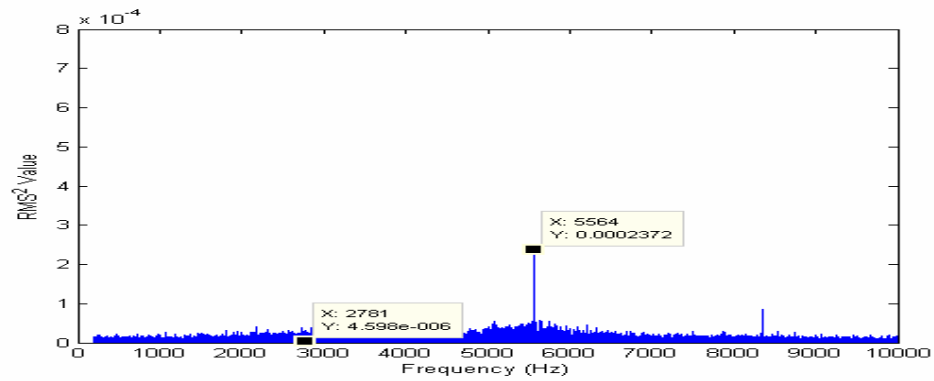
**Figure 72. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 8**



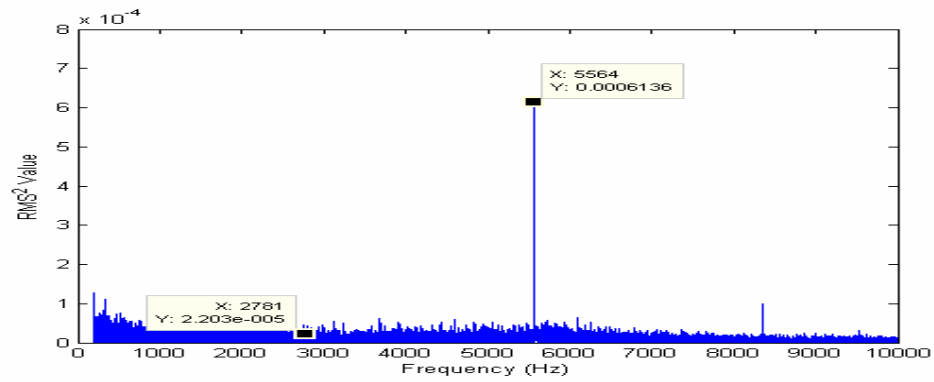
a)



b)

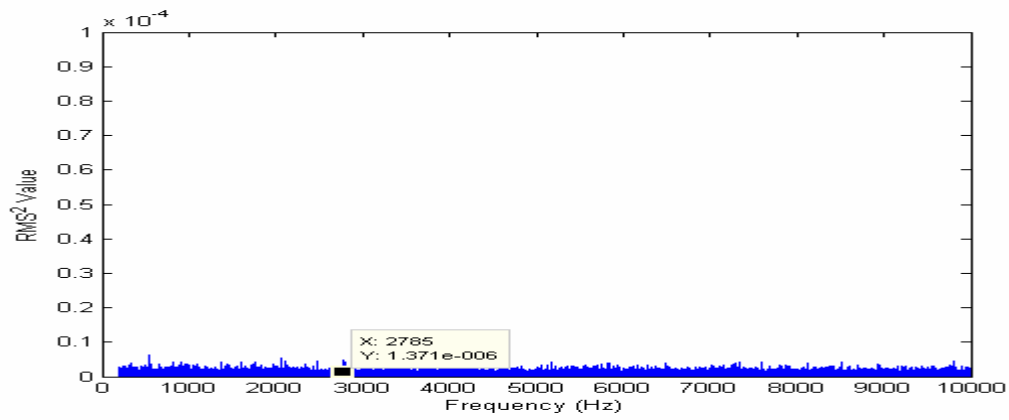


c)

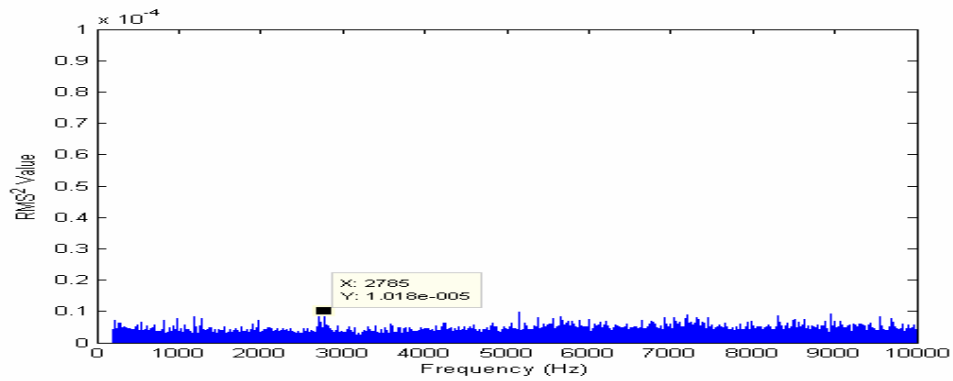


d)

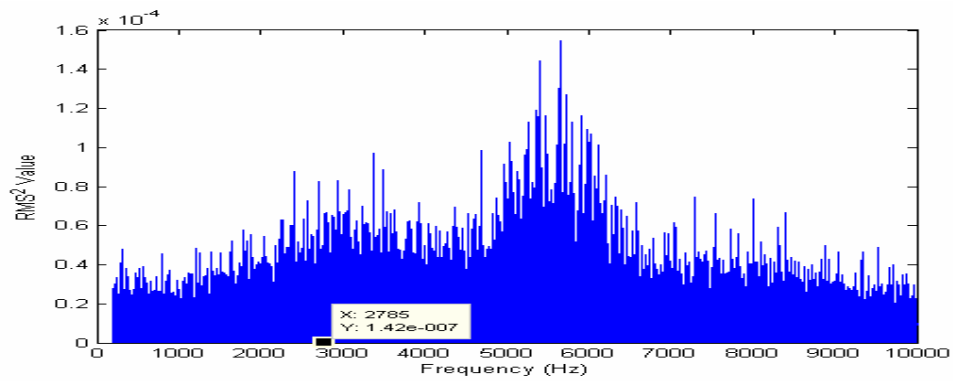
**Figure 73. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 8**



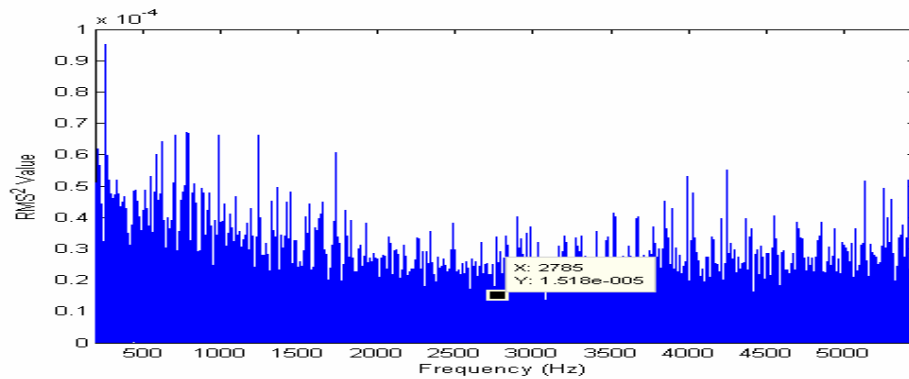
a)



b)



c)

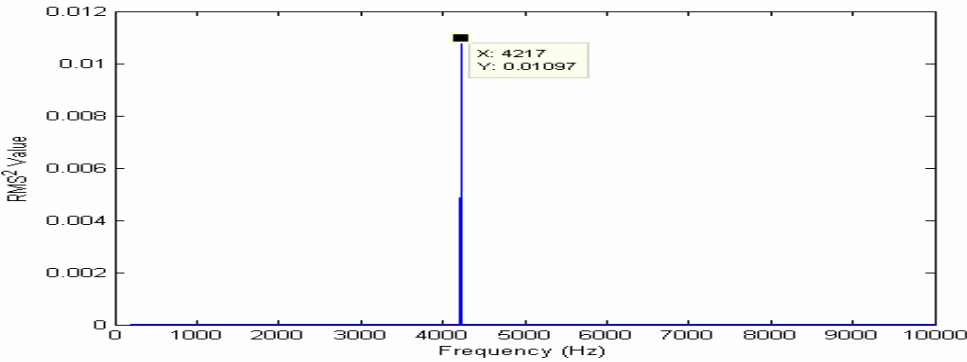


d)

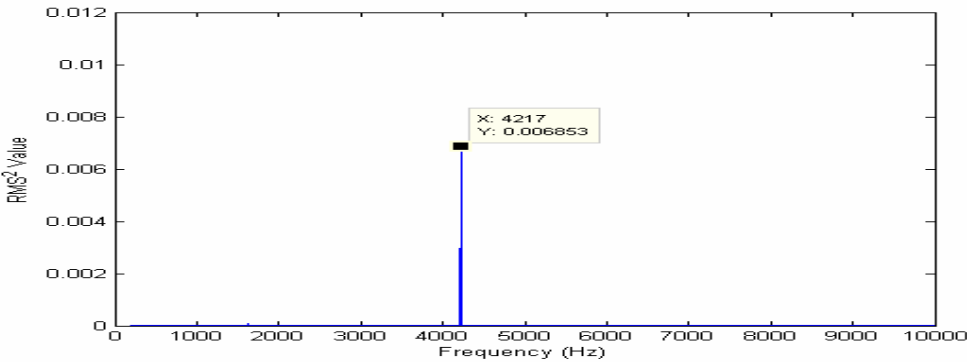
**Figure 74. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 8**



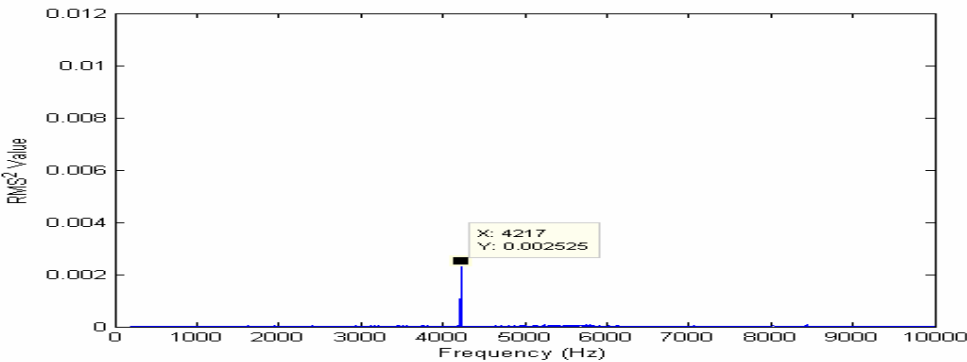
**Case Run9**



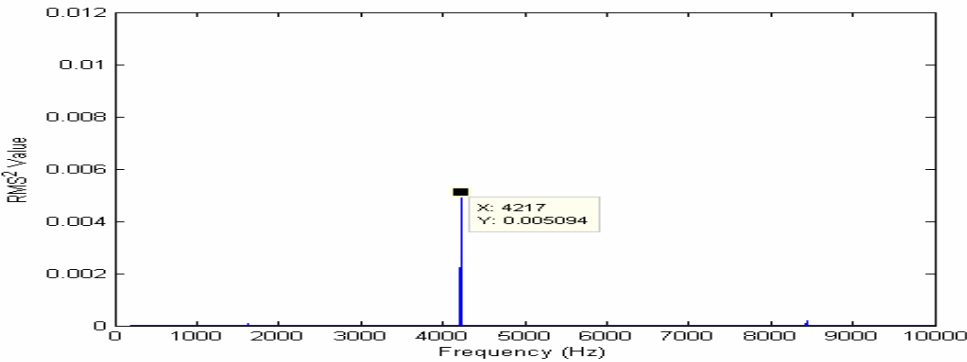
**a)**



**b)**

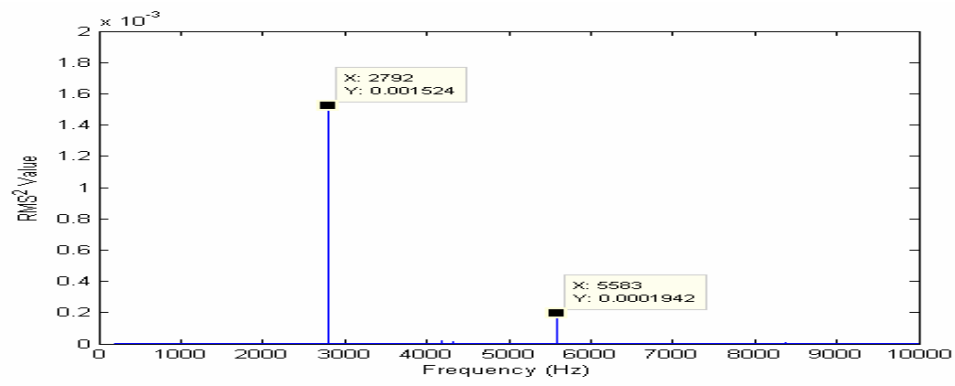


**c)**

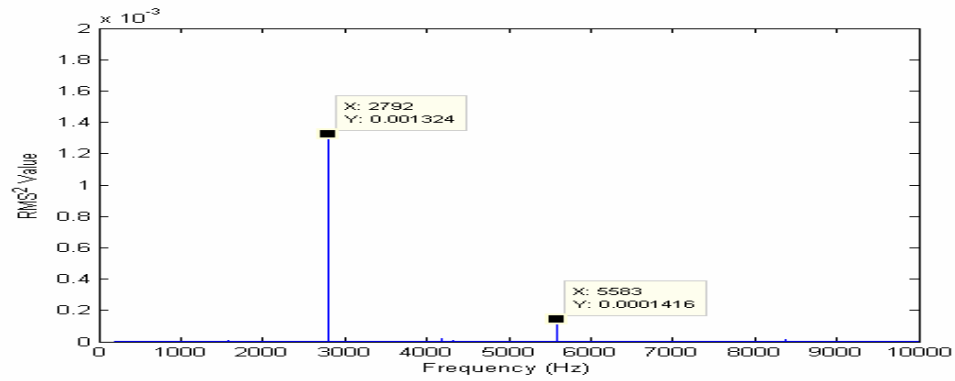


**d)**

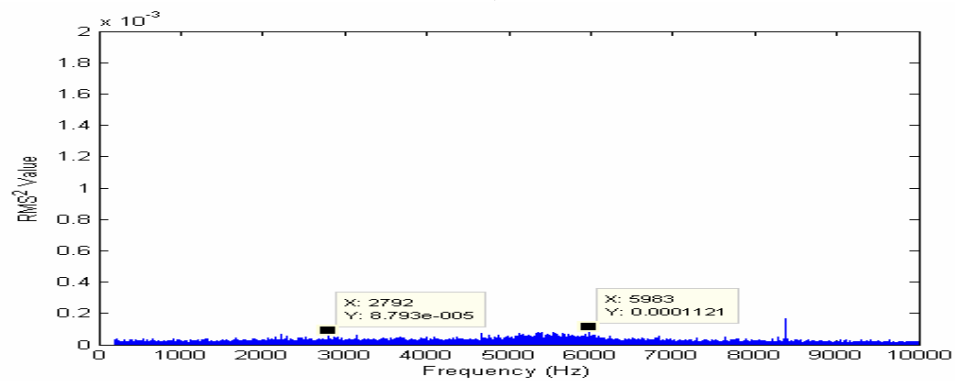
**Figure 75. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 9**



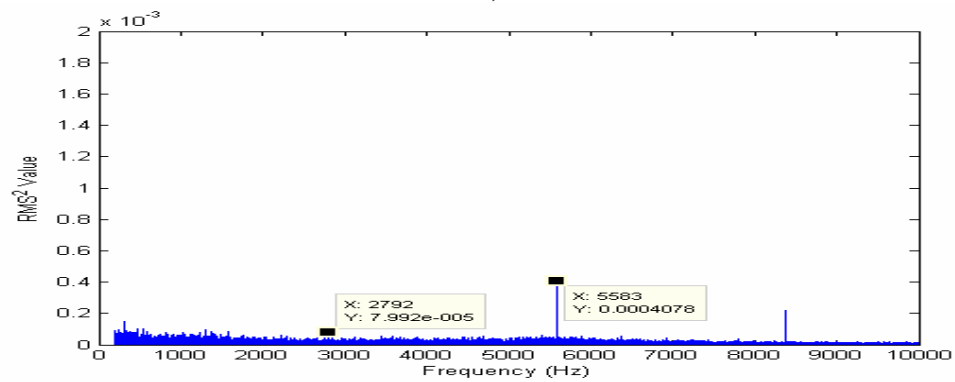
a)



b)

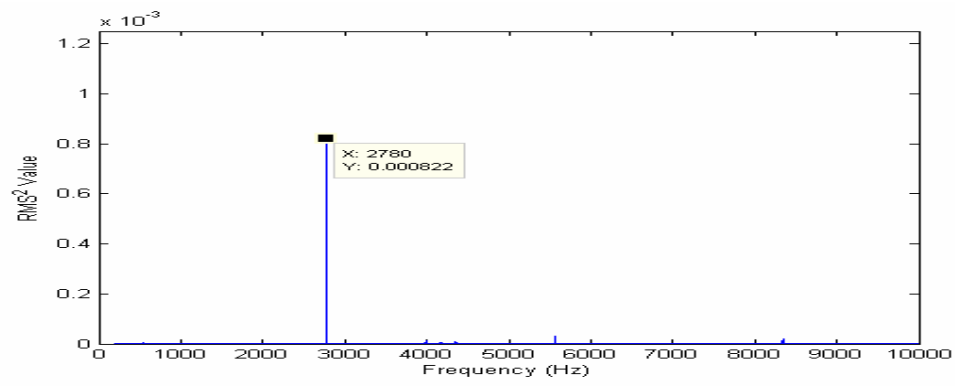


c)

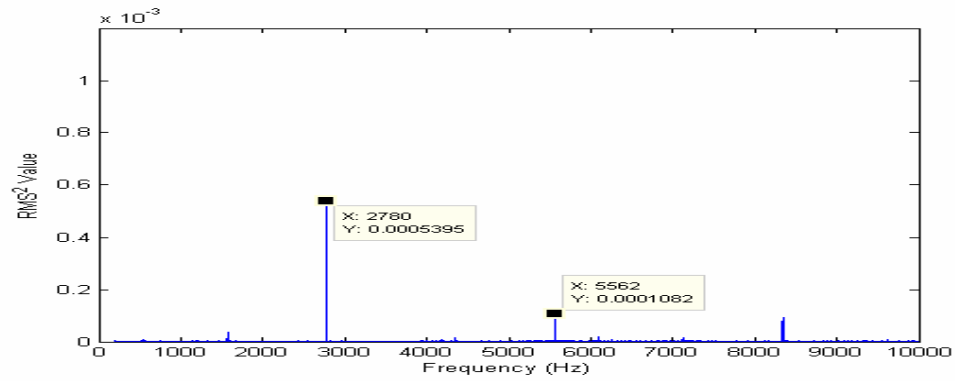


d)

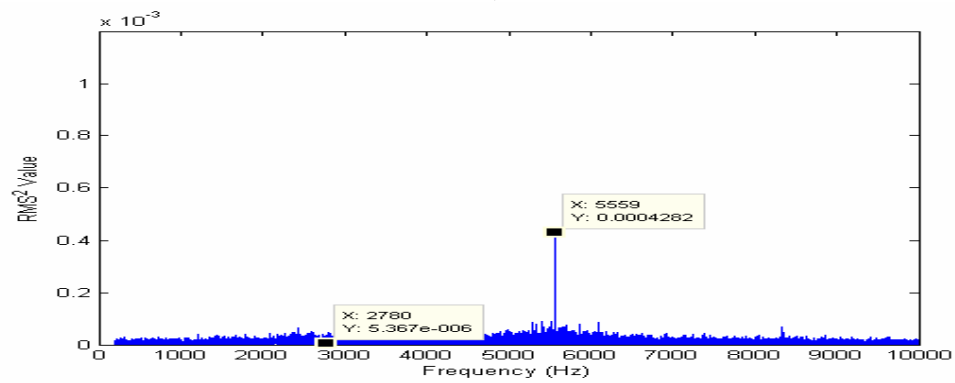
**Figure 76. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 9**



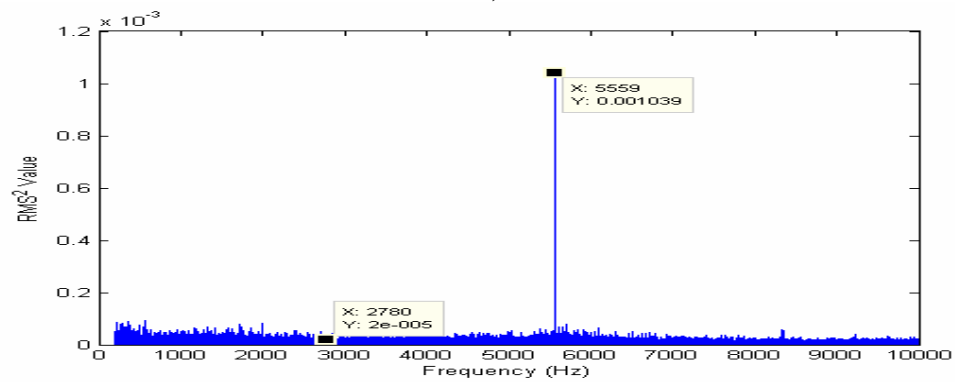
a)



b)

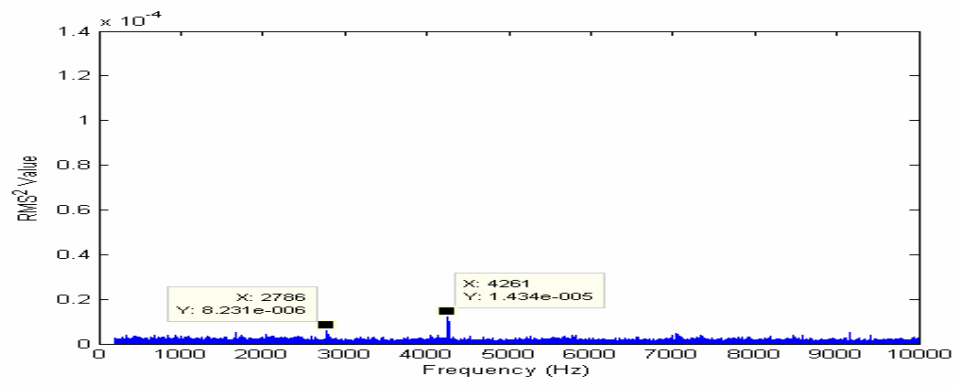


c)

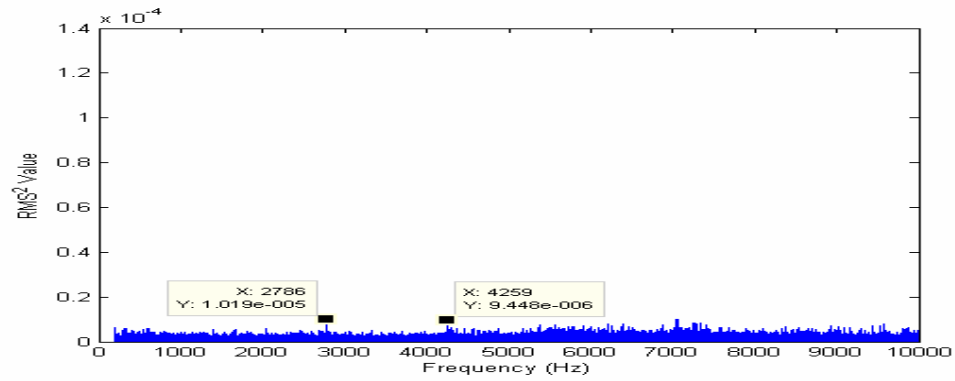


d)

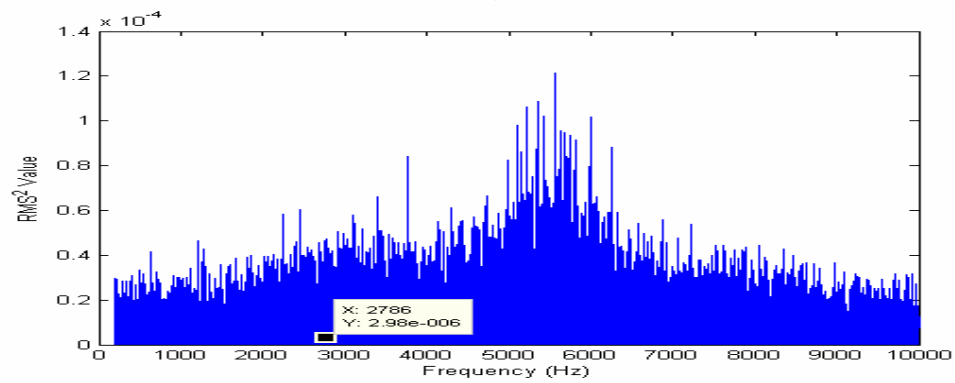
**Figure 77. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 9**



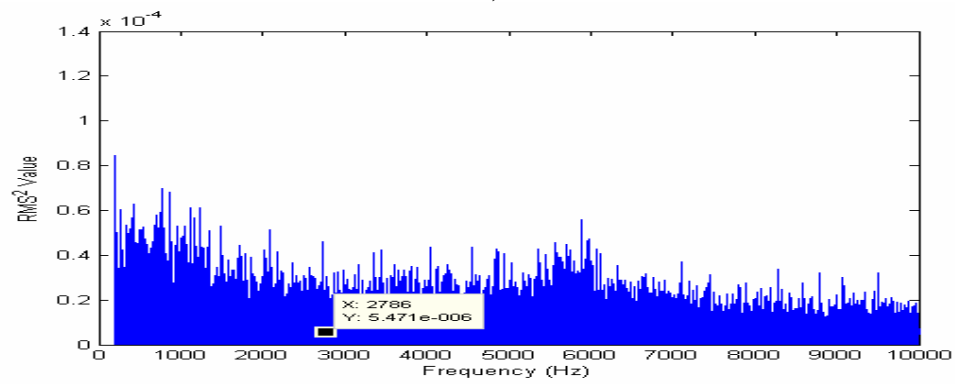
a)



b)



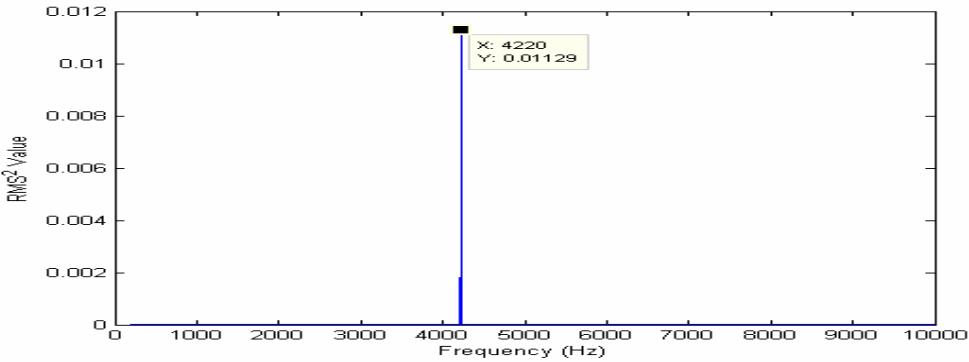
c)



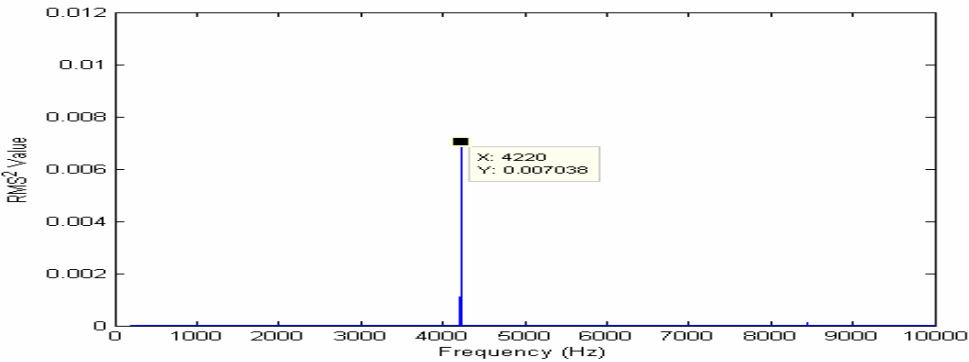
d)

**Figure 78. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 9**

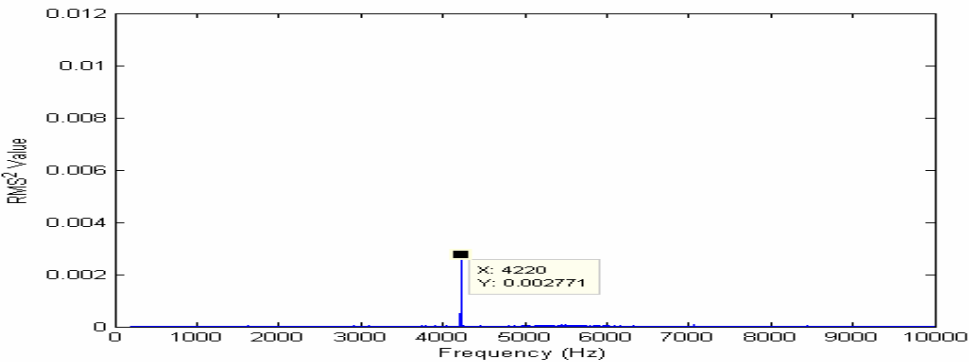
**Case Run10**



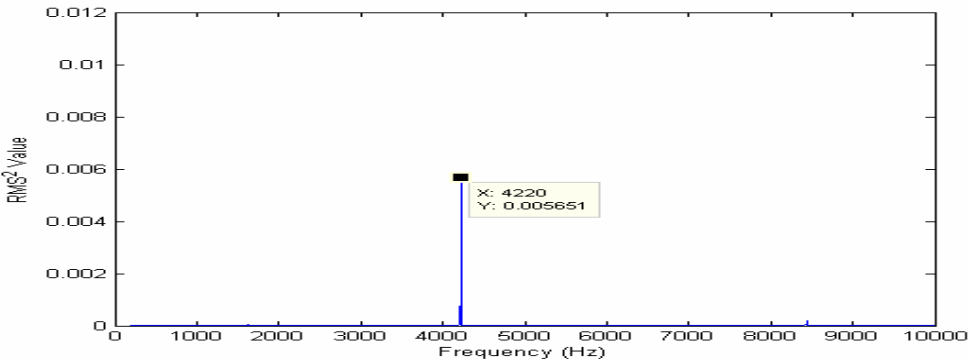
**a)**



**b)**

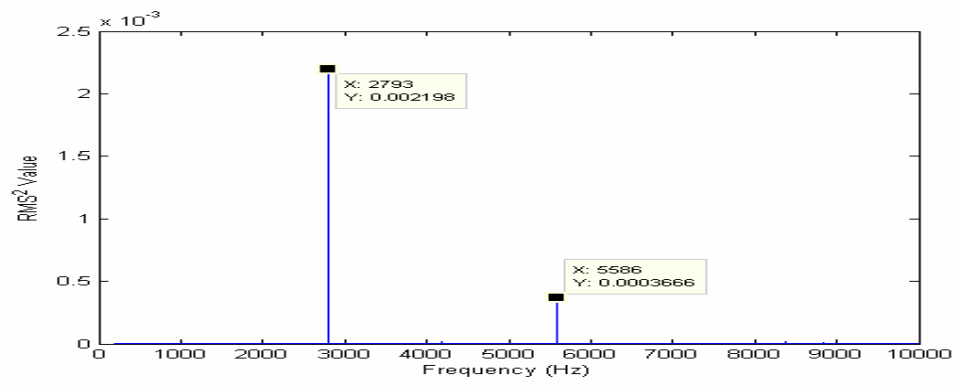


**c)**

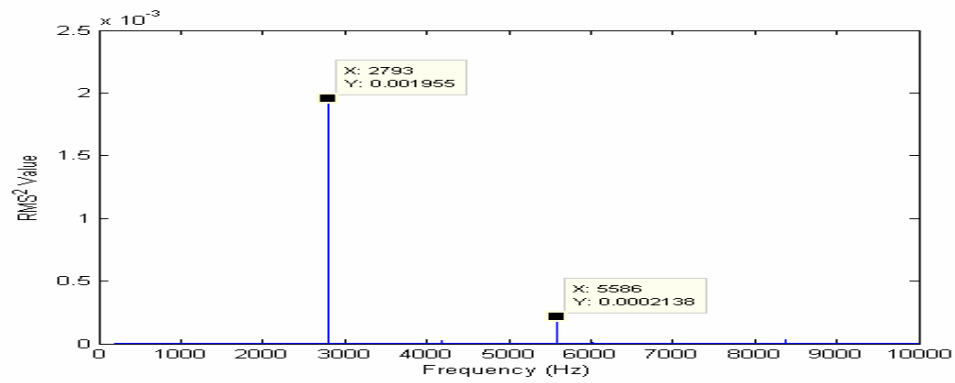


**d)**

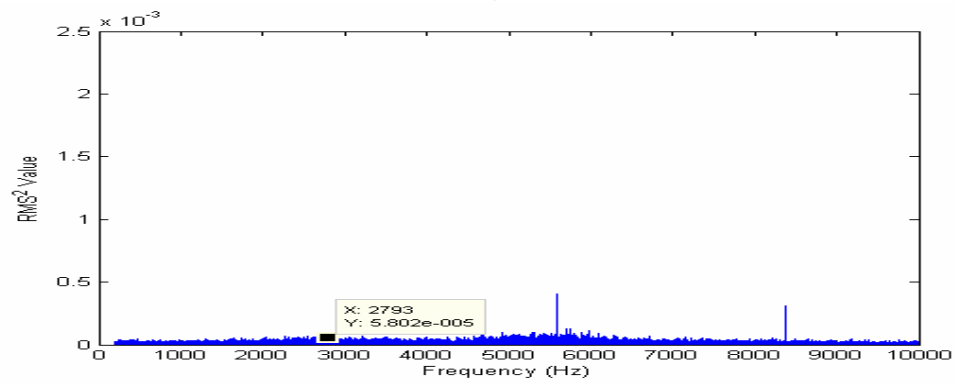
**Figure 79. Power spectrum of dynamic acoustic signal at 95psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 10**



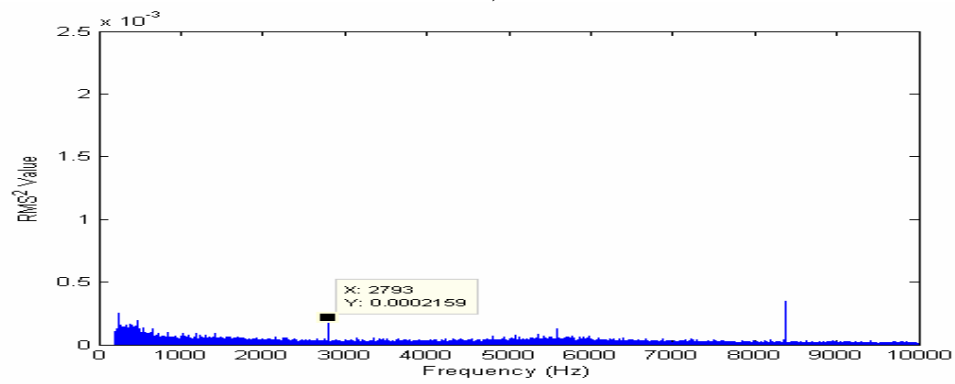
a)



b)

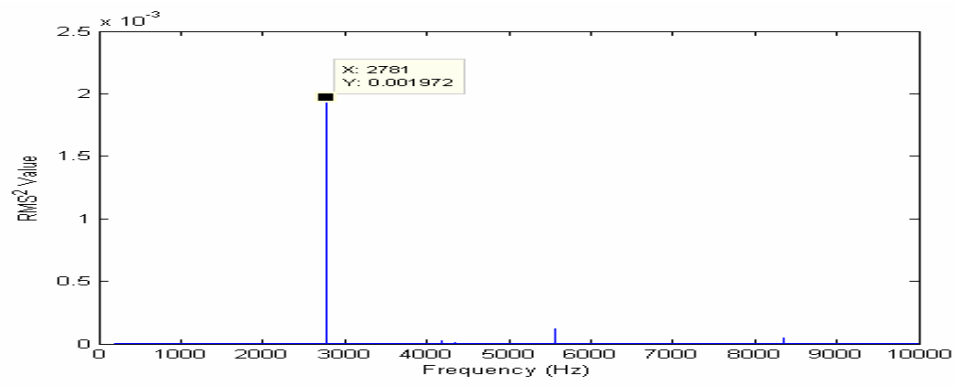


c)

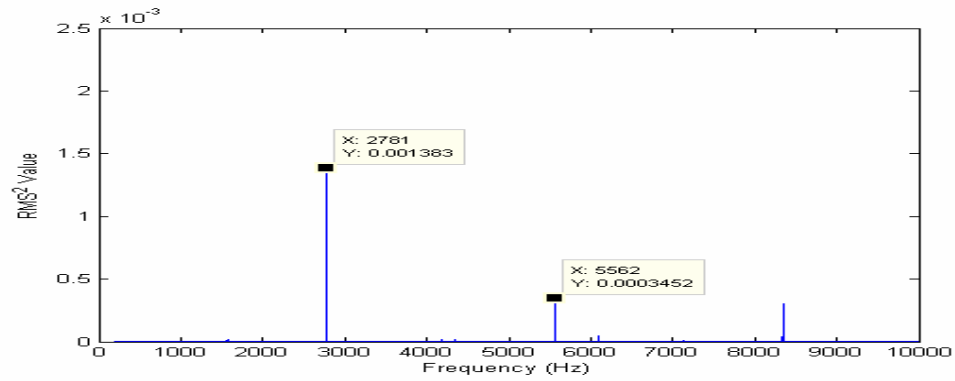


d)

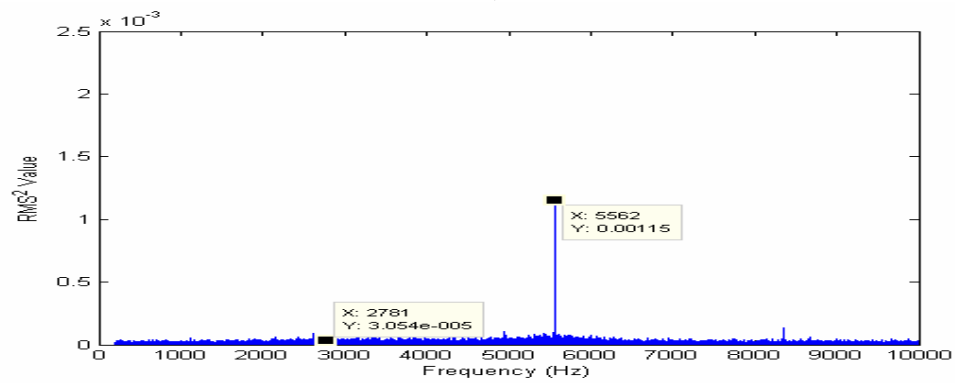
**Figure 80. Power spectrum of dynamic acoustic signal at 105psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 10**



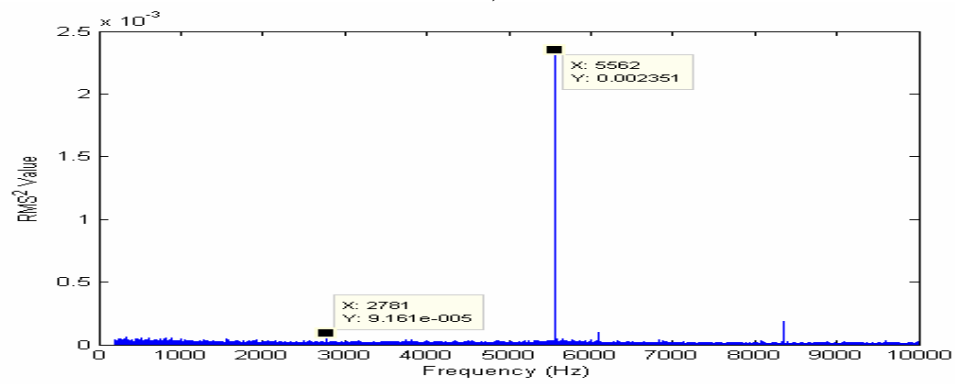
a)



b)

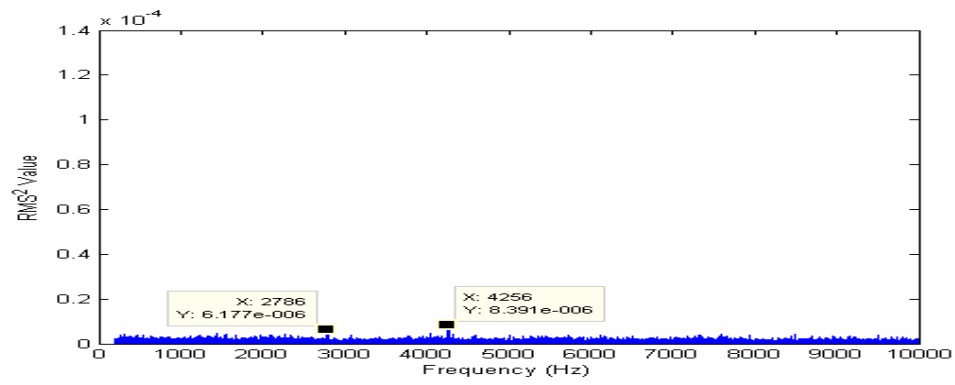


c)

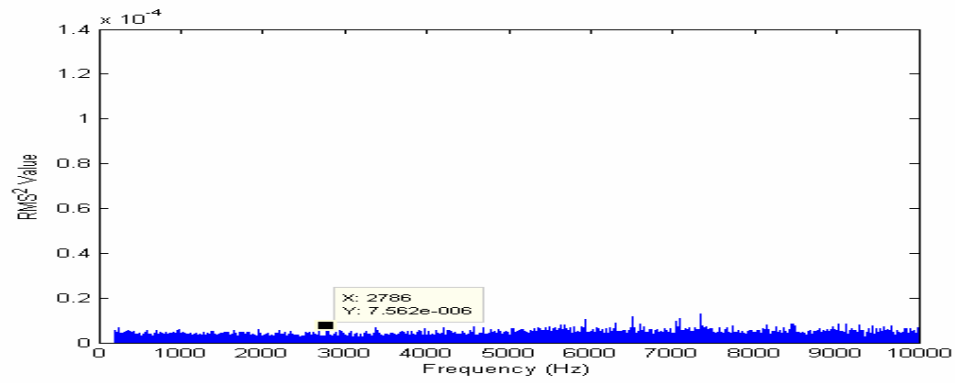


d)

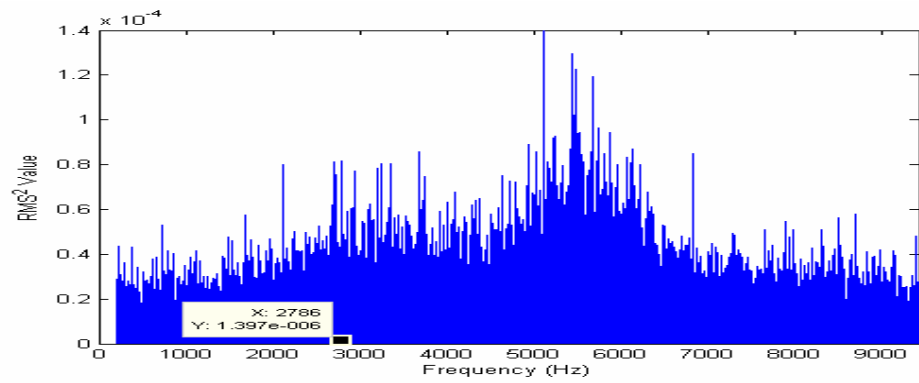
Figure 81. Power spectrum of dynamic acoustic signal at 115psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 10



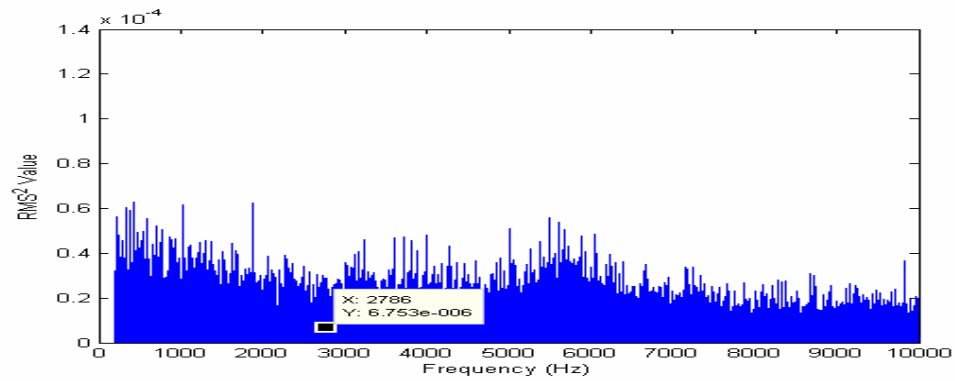
a)



b)



c)



d)

**Figure 82. Power spectrum of dynamic acoustic signal at 125psi for a)PT2, b)PT3, c)PT4, and d) PT5 for case run No. 10**



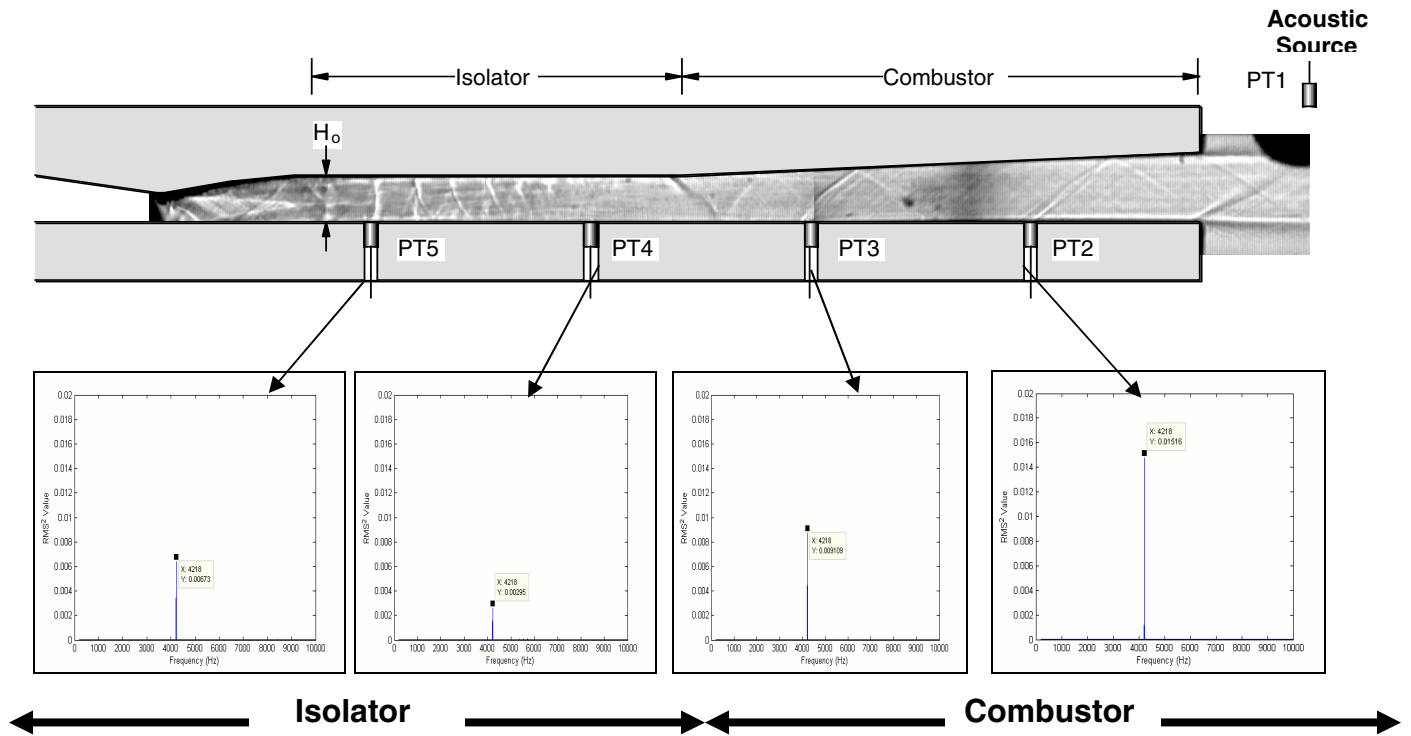


Figure 83. Power spectrum analysis at individual pressure transducers for stagnation pressure 95psi

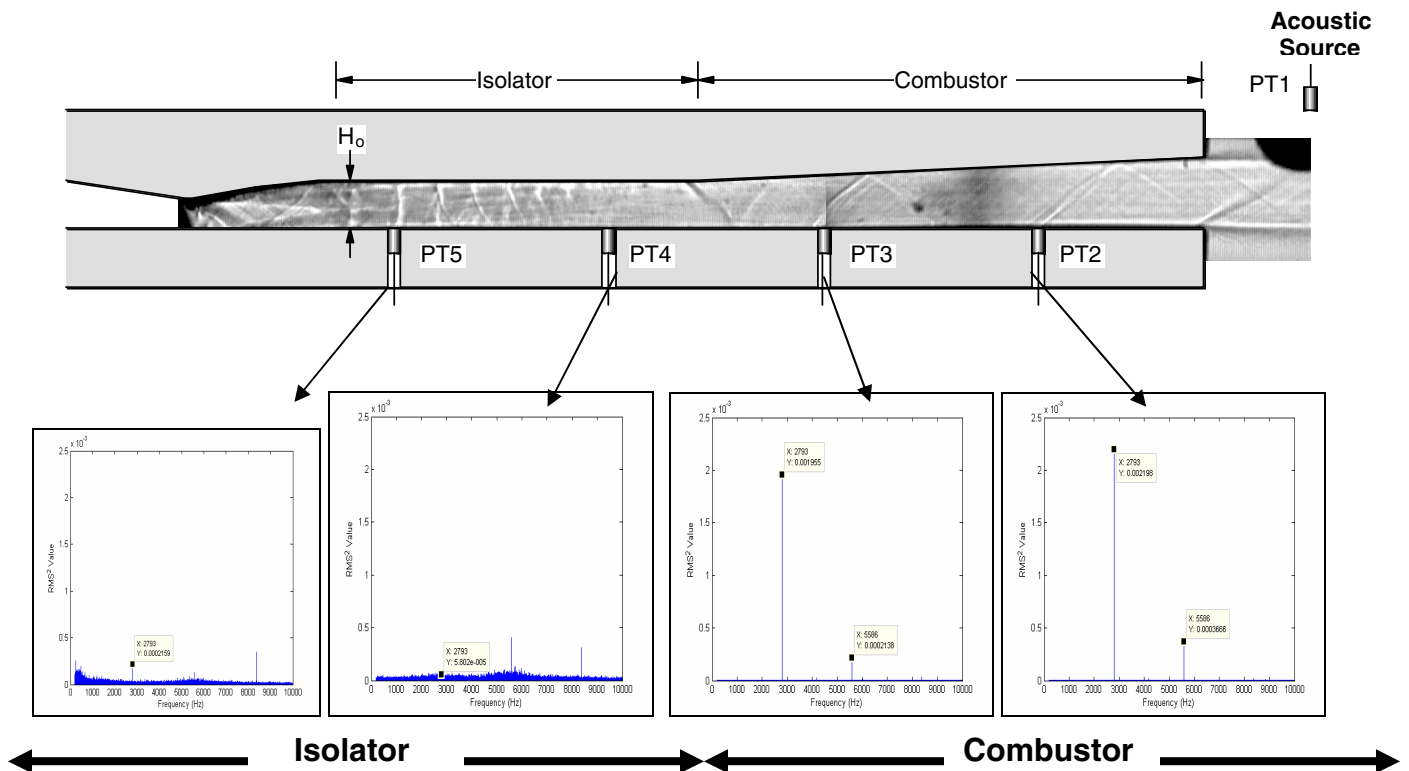
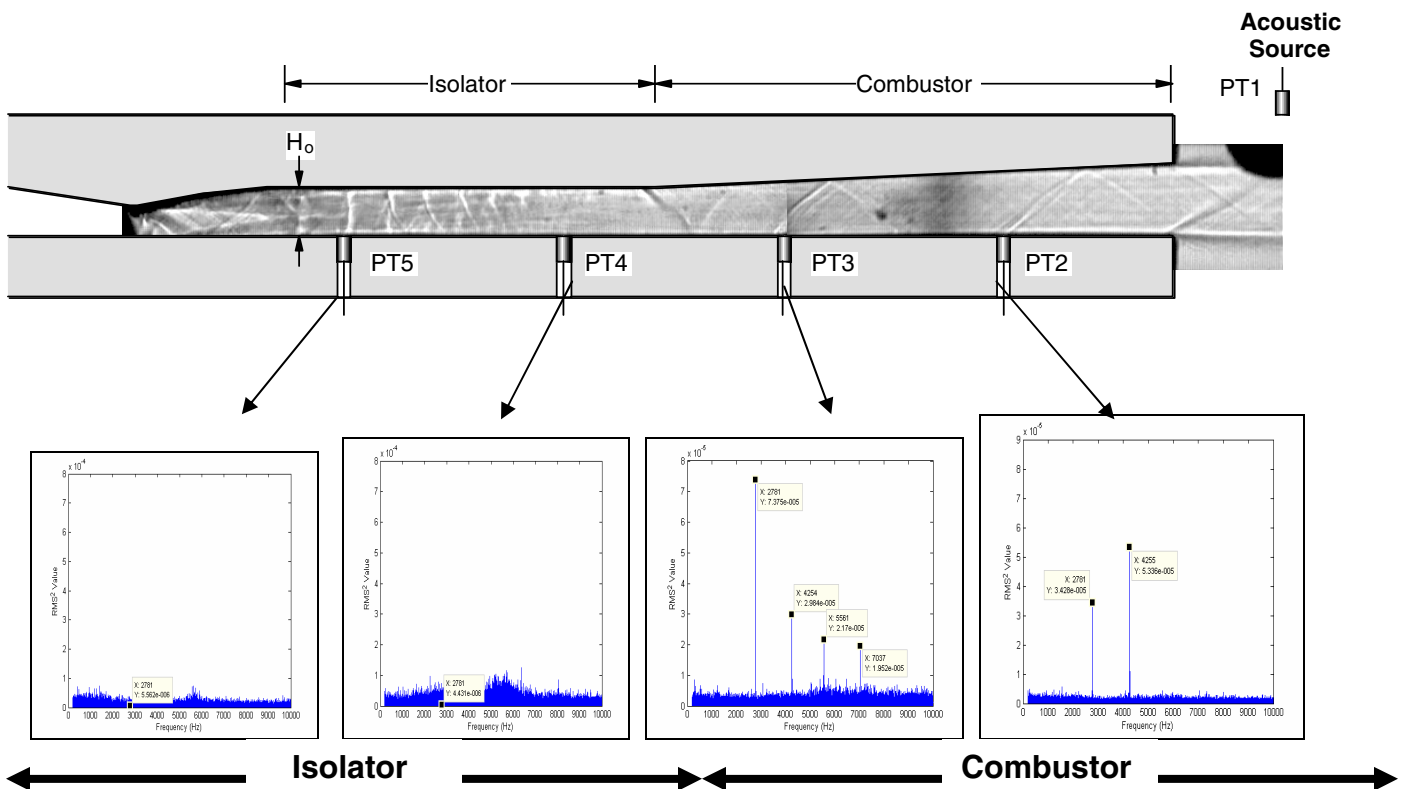
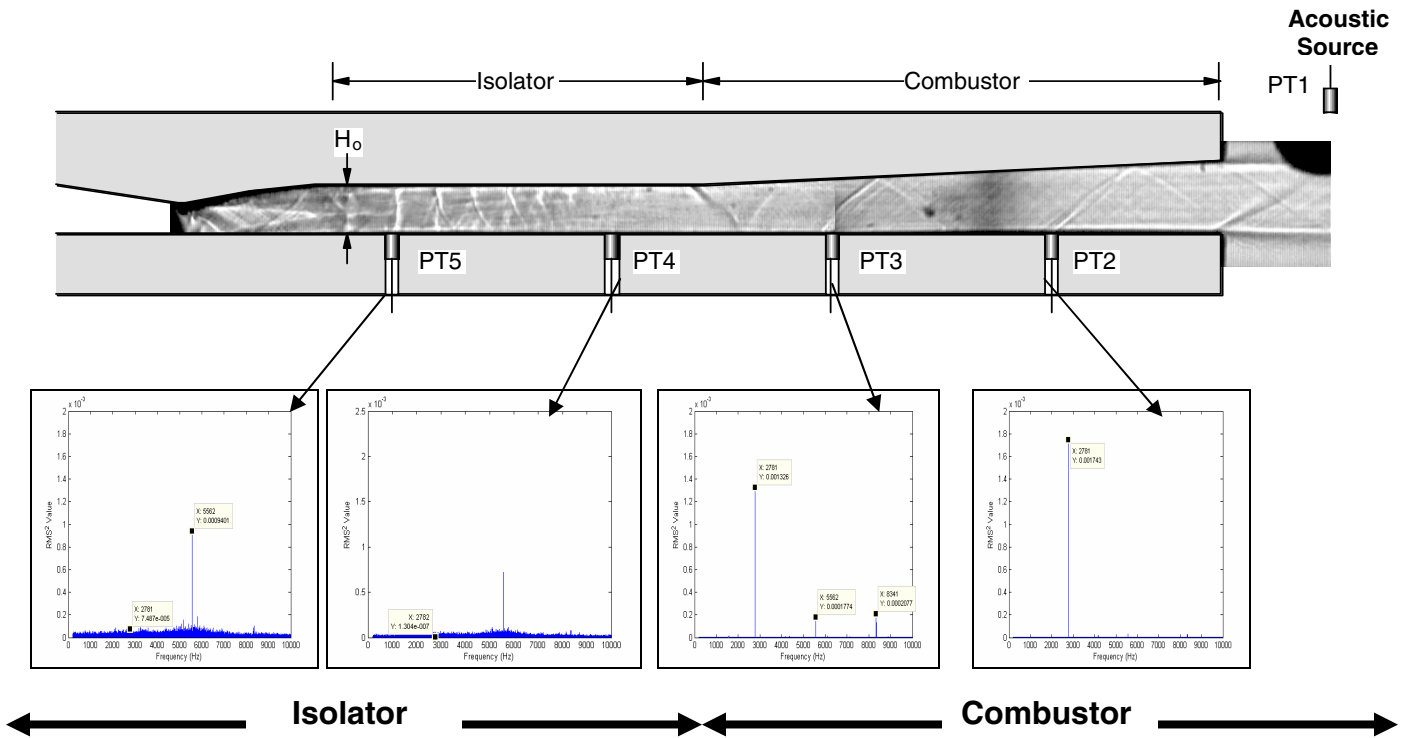


Figure 84. Power spectrum analysis at individual pressure transducers for stagnation pressure 105psi



**Table 1. Error analysis Values for ratio of dynamic pressure to static pressure**

<b>Error Analysis</b>		<b>80psig</b>						
<b>Press_Trnsd</b>	<b>x(in)</b>	<b>Static</b>	<b>Dynamic</b>	<b>%error static</b>	<b>%error dyn</b>	<b>ratio dyn stc</b>	<b>%err Quot</b>	<b>Error Quot</b>
PT2	1.305	1.845	0.01034	0.009230352	0.22244	0.005604336	0.2226286	0.0012477
PT3	3.005	1.0811	0.00623	0.015752474	0.36918	0.005762649	0.3695173	0.0021294
PT4	4.705	17.938	0.002275	0.000949381	1.01099	0.000126826	1.0109895	0.0001282
PT5	6.405	10.784	0.00499	0.001579191	0.46092	0.000462723	0.4609245	0.0002133
		<b>90psig</b>						
<b>Press_Trnsd</b>	<b>x(in)</b>	<b>Static</b>	<b>Dynamic</b>	<b>%error static</b>	<b>%error dyn</b>	<b>ratio dyn stc</b>	<b>%err Quot</b>	<b>Error Quot</b>
PT2	1.305	2.982	0.00115	0.005667337	0.429565217	0.000385647	0.4296026	0.0001657
PT3	3.005	1.357	0.00107	0.012225497	0.461682243	0.000788504	0.4618441	0.0003642
PT4	4.705	20.844	0.000055	0.000887066	8.981818182	2.63865E-06	8.9818182	2.37E-05
PT5	6.405	12.611	0.000064	0.001035287	7.71875	5.07493E-06	7.7187501	3.917E-05
		<b>100psig</b>						
<b>Press_Trnsd</b>	<b>x(in)</b>	<b>Static</b>	<b>Dynamic</b>	<b>%error static</b>	<b>%error dyn</b>	<b>ratio dyn stc</b>	<b>%err Quot</b>	<b>Error Quot</b>
PT2	1.305	4.224	0.00113	0.004034091	0.401238938	0.000267519	0.4012592	0.0001073
PT3	3.005	1.716	0.0008	0.009545455	0.400125	0.0004662	0.4002388	0.0001866
PT4	4.705	24.277	0.000016	0.00074931	0.85625	6.5906E-07	0.8562503	5.643E-07
PT5	6.405	15.827	0.000047	0.000796108	0.59787234	2.96961E-06	0.5978729	1.775E-06
		<b>110psig</b>						
<b>Press_Trnsd</b>	<b>x(in)</b>	<b>Static</b>	<b>DYN</b>	<b>%error static</b>	<b>%error dyn</b>	<b>rat_dyn_st</b>	<b>%err Quot</b>	<b>Error Quot</b>
PT2	1.305	5.247	0.0000154	0.003329522	0.922077922	2.93501E-06	0.9220839	2.706E-06
PT3	3.005	2.013	0.0000206	0.007908594	0.799514563	1.02335E-05	0.7995537	8.182E-06
PT4	4.705	27.607	0.000009	0.000532474	1.488888889	3.26004E-07	1.488889	4.854E-07
PT5	6.405	19.184	0.000011	0.000802752	0.727272727	5.73394E-07	0.7272732	4.17E-07

**Table 2. Phase analysis w.r.t upstream stagnation pressure of 95psi**

Po=80psig												
Dom_freq(Hz)	4.2kHz	run1	run2	run3	run4	run5	run6	run7	run8	run9	run10	AVG
	xpositon	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	
PT2_phase(deg)	1.305	-170.00	55.83	110.80	-103.30	165.40	-69.89	25.98	153.70	-114.30	-20.77	3.35
PT3_phase(deg)	3.005	-145.00	76.75	131.90	-82.57	56.28	-49.54	47.59	175.40	-92.70	0.95	11.91
PT4_phase(deg)	4.705	-147.00	84.10	-131.70	-88.71	1.29	52.15	45.98	175.10	-100.10	-3.72	11.26
PT5_phase(deg)	6.405	-100.00	136.90	-172.10	-26.92	-116.10	8.49	105.10	-126.00	-41.19	55.61	27.62
		phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	AVG
PT2_PT2	1.305	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
PT3_PT2	3.005	25.00	20.92	21.10	20.73	-109.12	20.35	21.61	21.70	21.60	21.72	8.56
PT4_PT2	4.705	23.00	28.27	117.50	14.59	-164.11	122.04	20.00	21.40	14.20	17.05	21.39
PT5_PT2	6.405	70.00	81.07	77.10	76.38	78.50	78.38	79.12	80.30	73.11	76.38	77.03

**Table 3. Phase analysis w.r.t upstream stagnation pressure of 105psi**

Po=90psig												
Dom_freq(Hz)	2.8kHz	run1	run2	run3	run4	run5	run6	run7	run8	run9	run10	AVG
	x_positon	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	
PT2_phase(deg)	1.305	36.75	-7.83	151.17	118.21	-111.92	76.82	54.53	136.78	18.64	115.00	58.81
PT3_phase(deg)	3.005	-8.16	-56.13	79.07	45.78	176.43	-4.22	-22.73	86.29	-28.75	81.40	34.90
PT4_phase(deg)	4.705	59.76	12.60	159.41	142.75	-84.54	108.81	90.84	-177.10	47.61	98.80	45.89
PT5_phase(deg)	6.405	-22.38	-54.66	63.08	48.71	173.81	14.61	-24.74	67.88	-34.45	29.60	26.15
		phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	AVG
PT2_PT2	1.305	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
PT3_PT2	3.005	-44.91	-48.30	-72.10	-72.43	-71.65	-81.04	-77.26	-50.49	-47.39	-33.60	59.92
PT4_PT2	4.705	23.01	20.43	8.24	24.54	27.38	31.99	36.31	46.12	28.97	-16.20	23.08
PT5_PT2	6.405	-59.13	-46.83	-88.09	-69.50	-74.27	-62.21	-79.27	-68.90	-53.09	-85.40	68.67

**Table 4. Phase analysis w.r.t upstream stagnation pressure of 115psi**

Po=100psig												
Dom_freq(Hz)	4.2kHz	run1	run2	run3	run4	run5	run6	run7	run8	run9	run10	AVG
	x_positon	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	
PT2_phase(deg)	1.305	2.07	72.48	-177.30	86.58	38.70	-24.54	57.41	139.04	-16.74	26.51	20.42
PT3_phase(deg)	3.005	-33.02	33.39	148.53	52.79	5.08	-63.28	18.13	99.85	-52.23	-14.73	19.45
PT4_phase(deg)	4.705	105.76	-71.00	-31.98	-151.48	116.09	162.81	161.06	-90.84	-76.17	73.74	19.80
PT5_phase(deg)	6.405	37.03	-19.86	115.19	7.78	-49.43	-144.16	6.58	80.89	-95.74	-50.96	11.27
		phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	AVG
PT2_PT2	1.305	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
PT3_PT2	3.005	-35.09	-39.09	-34.18	-33.79	-33.62	-38.74	-39.28	-39.19	-35.50	-41.24	36.97
PT4_PT2	4.705	103.69	-143.47	145.31	121.94	77.40	-172.65	103.65	130.12	-59.43	47.23	35.38
PT5_PT2	6.405	34.96	-92.34	-67.52	-78.80	-88.12	-119.62	-50.83	-58.15	-79.00	-77.47	67.69

**Table 5. Phase analysis w.r.t upstream stagnation pressure of 125psi**

Po=110psig												
Dom_freq(Hz)	4.2kHz	run1	run2	run3	run4	run5	run6	run7	run8	run9	run10	AVG
	x_positon	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	phase_deg	
PT2_phase(deg)	1.305	-67.35	-151.64	113.81	95.64	73.11	-56.81	72.43	179.52	143.66	-155.24	24.71
PT3_phase(deg)	3.005	-100.53	163.07	70.98	53.29	20.20	-110.54	15.94	127.79	98.07	147.27	48.55
PT4_phase(deg)	4.705	-149.54	16.57	-78.40	-120.31	172.13	63.50	-149.25	-127.88	-117.06	26.52	46.37
PT5_phase(deg)	6.405	-27.55	153.65	137.96	51.54	13.94	-91.66	54.20	117.72	147.57	-168.49	38.89
		phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	phase_diff	AVG
PT2_PT2	1.305	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00	0.00
PT3_PT2	3.005	-33.18	-45.29	-42.84	-42.35	-52.92	-53.73	-56.49	-51.73	-45.59	-57.50	48.16
PT4_PT2	4.705	-82.19	168.20	167.78	144.05	99.01	120.31	138.32	52.61	99.28	-178.24	72.91
PT5_PT2	6.405	39.80	-54.71	24.15	-44.10	-59.17	-34.85	-18.23	-61.80	3.91	-13.26	21.83

**Table 6. Static pressure at each pressure transducer location w.r.t upstream stagnation pressure of 80psig**

80psig	Upstream(psig)	pt2(psig)	pt3(psig)	pt4(psig)	pt5(psig)
	80.23	1.83	1.08	17.93	10.77
	80.25	1.86	1.10	17.94	10.77
	80.25	1.83	1.09	17.93	10.77
	80.18	1.88	1.10	17.91	10.77
	80.23	1.87	1.09	17.92	10.76
	80.23	1.85	1.09	17.93	10.77
	80.22	1.82	1.10	17.93	10.77
	80.23	1.84	1.09	17.93	10.77
	80.22	1.86	1.05	17.93	10.80
	80.21	1.84	1.07	17.94	10.79
	80.16	1.83	1.10	17.94	10.76
	80.17	1.82	1.07	17.96	10.76
	80.19	1.85	1.08	17.93	10.76
	80.18	1.85	1.10	17.90	10.78
	80.15	1.86	1.12	17.93	10.79
	80.15	1.85	1.08	17.93	10.78
	80.16	1.88	1.08	17.92	10.76
	80.11	1.88	1.08	17.90	10.79
	80.15	1.85	1.07	17.93	10.79
	80.18	1.85	1.05	17.94	10.77
	80.16	1.86	1.07	17.93	10.80
	80.12	1.85	1.10	17.93	10.79
	80.10	1.85	1.09	17.97	10.77
	80.12	1.85	1.07	17.95	10.80
	80.12	1.85	1.08	17.94	10.79
	80.16	1.85	1.12	17.93	10.79
	80.09	1.88	1.11	17.95	10.79
	80.09	1.84	1.09	17.93	10.78
	80.08	1.85	1.09	17.93	10.76
	80.15	1.84	1.07	17.95	10.77
	80.12	1.84	1.05	17.93	10.79
	80.11	1.82	1.08	17.92	10.78
	80.15	1.85	1.09	17.95	10.82
	80.16	1.84	1.08	17.94	10.81
	80.12	1.83	1.06	17.96	10.76
	80.12	1.82	1.07	17.97	10.80
	80.13	1.83	1.07	17.96	10.81
	80.09	1.85	1.10	17.93	10.77
	80.13	1.86	1.09	17.93	10.79
	80.14	1.87	1.09	17.92	10.80
	80.16	1.86	1.08	17.93	10.78
	80.18	1.85	1.04	17.93	10.80
	80.14	1.83	1.07	17.93	10.78
	80.18	1.85	1.07	17.94	10.80
	80.19	1.85	1.07	17.94	10.81
	80.15	1.84	1.07	17.91	10.82

	80.15	1.85	1.06	17.96	10.79
	80.21	1.84	1.08	17.95	10.79
	80.13	1.85	1.08	17.93	10.79
	80.19	1.85	1.11	17.93	10.79
	80.20	1.84	1.08	17.93	10.79
	80.19	1.85	1.09	17.93	10.79
	80.23	1.87	1.06	17.94	10.77
	80.22	1.88	1.08	17.93	10.79
	80.19	1.85	1.06	17.94	10.79
	80.25	1.84	1.07	17.95	10.80
	80.23	1.83	1.08	17.95	10.81
	80.22	1.86	1.09	17.93	10.79
	80.19	1.85	1.07	17.96	10.79
	80.20	1.84	1.09	17.95	10.79
	80.18	1.81	1.07	17.93	10.79
	80.25	1.83	1.11	17.93	10.79
	80.16	1.84	1.08	17.93	10.79
	80.16	1.87	1.09	17.94	10.79
	80.22	1.86	1.07	17.92	10.76
	80.20	1.83	1.09	17.93	10.78
	80.20	1.81	1.07	17.93	10.79
	80.24	1.86	1.08	17.93	10.79
	80.19	1.83	1.06	17.95	10.80
	80.22	1.83	1.07	17.93	10.79
	80.20	1.85	1.08	17.96	10.77
	80.19	1.82	1.06	17.95	10.79
	80.17	1.86	1.10	17.94	10.77
	80.22	1.86	1.08	17.93	10.78
	80.22	1.82	1.10	17.93	10.79
	80.19	1.87	1.07	17.93	10.79
	80.17	1.84	1.07	17.92	10.78
	80.24	1.86	1.06	17.96	10.79
	80.19	1.83	1.08	17.94	10.78
	80.15	1.82	1.06	17.94	10.79
	80.17	1.84	1.07	17.95	10.79
	80.16	1.86	1.08	17.93	10.78
	80.11	1.88	1.07	17.96	10.78
	80.19	1.85	1.10	17.96	10.79
	80.17	1.85	1.09	17.94	10.78
	80.16	1.85	1.09	17.93	10.78
Avg	80.17	1.85	1.08	17.94	10.78
STDV	0.04	0.02	0.01	0.01	0.01

**Table 7. Static pressure at each pressure transducer location w.r.t upstream stagnation pressure of 90psig**

90psig	Upstream(psig)	pt2(psig)	pt3(psig)	pt4(psig)	pt5(psig)
	90.25	3.00	1.36	20.84	12.61
	90.26	2.98	1.33	20.85	12.64
	90.25	2.99	1.39	20.85	12.59
	90.30	3.01	1.38	20.86	12.61
	90.30	2.97	1.37	20.87	12.63
	90.24	3.02	1.36	20.84	12.61
	90.33	2.99	1.35	20.85	12.58
	90.28	2.99	1.34	20.84	12.60
	90.33	2.97	1.36	20.86	12.61
	90.31	2.96	1.39	20.82	12.63
	90.33	2.96	1.36	20.85	12.63
	90.32	2.98	1.36	20.85	12.62
	90.34	2.99	1.38	20.85	12.61
	90.24	2.99	1.34	20.86	12.60
	90.33	2.98	1.37	20.86	12.61
	90.32	2.98	1.36	20.85	12.61
	90.34	2.99	1.39	20.81	12.64
	90.36	3.02	1.34	20.84	12.62
	90.37	2.99	1.37	20.83	12.59
	90.39	2.99	1.34	20.86	12.61
	90.39	2.99	1.39	20.84	12.63
	90.38	2.96	1.37	20.84	12.61
	90.38	2.96	1.35	20.82	12.61
	90.37	2.97	1.37	20.80	12.61
	90.36	2.98	1.36	20.86	12.59
	90.37	2.97	1.36	20.82	12.61
	90.37	2.98	1.38	20.85	12.61
	90.40	2.99	1.37	20.82	12.60
	90.38	3.00	1.36	20.81	12.61
	90.35	2.99	1.34	20.82	12.61
	90.37	2.99	1.35	20.81	12.61
	90.36	2.99	1.34	20.84	12.64
	90.35	2.97	1.33	20.85	12.63
	90.33	2.96	1.32	20.85	12.61
	90.33	2.97	1.37	20.83	12.61
	90.32	2.95	1.36	20.84	12.58
	90.30	2.98	1.33	20.84	12.61
	90.32	2.96	1.36	20.81	12.61
	90.32	2.98	1.35	20.85	12.60
	90.33	2.97	1.35	20.82	12.61
	90.31	2.98	1.36	20.84	12.64
	90.30	2.96	1.35	20.84	12.62
	90.28	2.98	1.35	20.85	12.61
	90.27	2.96	1.35	20.85	12.61
	90.27	2.96	1.34	20.81	12.61
	90.30	2.99	1.36	20.85	12.59



	90.28	2.96	1.39	20.85	12.61
	90.27	2.96	1.39	20.85	12.60
	90.26	2.99	1.36	20.85	12.61
	90.25	3.00	1.34	20.85	12.59
	90.22	2.96	1.36	20.86	12.61
	90.28	3.00	1.37	20.82	12.60
	90.24	2.99	1.35	20.84	12.61
	90.22	2.99	1.33	20.84	12.62
	90.21	2.97	1.32	20.85	12.61
	90.19	3.01	1.34	20.86	12.61
	90.23	2.97	1.36	20.85	12.60
	90.17	2.97	1.36	20.85	12.60
	90.20	2.97	1.36	20.81	12.61
	90.24	2.97	1.36	20.83	12.63
	90.20	2.99	1.35	20.85	12.59
	90.15	2.99	1.36	20.88	12.61
	90.22	2.95	1.37	20.86	12.61
	90.24	2.96	1.36	20.83	12.63
	90.15	2.96	1.36	20.83	12.61
	90.18	2.99	1.33	20.81	12.63
	90.20	2.99	1.37	20.89	12.61
	90.24	2.97	1.36	20.85	12.62
	90.20	2.97	1.35	20.86	12.61
	90.21	2.99	1.36	20.87	12.62
	90.24	3.00	1.38	20.86	12.61
	90.21	2.99	1.36	20.85	12.62
	90.19	2.99	1.34	20.83	12.61
	90.19	2.99	1.35	20.85	12.61
	90.23	2.96	1.36	20.85	12.61
	90.24	3.00	1.35	20.85	12.61
	90.25	2.98	1.33	20.85	12.63
	90.27	2.99	1.36	20.85	12.63
	90.25	2.97	1.36	20.85	12.59
	90.26	3.01	1.35	20.84	12.62
	90.24	3.01	1.36	20.88	12.61
	90.25	2.96	1.35	20.85	12.61
	90.27	2.98	1.38	20.87	12.63
	90.26	3.00	1.39	20.86	12.64
	90.24	3.02	1.37	20.86	12.60
	90.21	3.00	1.37	20.86	12.61
Avg	90.28	2.98	1.36	20.84	12.61
STDV	0.06	0.02	0.02	0.02	0.01

**Table 8. Static pressure at each pressure transducer location w.r.t upstream stagnation pressure of 100psig**

100psig	upstream	pt2	pt3	pt4	pt5
	100.31	4.20	1.72	24.29	15.85
	100.29	4.22	1.74	24.28	15.84
	100.32	4.22	1.72	24.28	15.85
	100.29	4.23	1.71	24.30	15.82
	100.33	4.22	1.71	24.30	15.82
	100.36	4.20	1.74	24.29	15.83
	100.32	4.23	1.74	24.30	15.83
	100.38	4.23	1.72	24.28	15.83
	100.35	4.26	1.72	24.27	15.83
	100.32	4.25	1.74	24.29	15.83
	100.38	4.22	1.70	24.28	15.83
	100.36	4.26	1.72	24.29	15.83
	100.39	4.22	1.71	24.27	15.83
	100.38	4.20	1.72	24.29	15.84
	100.42	4.22	1.71	24.29	15.84
	100.38	4.20	1.73	24.30	15.81
	100.37	4.22	1.71	24.30	15.83
	100.40	4.22	1.74	24.30	15.83
	100.38	4.18	1.74	24.31	15.83
	100.42	4.23	1.72	24.30	15.84
	100.42	4.23	1.72	24.27	15.83
	100.42	4.21	1.73	24.27	15.84
	100.40	4.23	1.73	24.29	15.81
	100.38	4.21	1.70	24.28	15.82
	100.38	4.20	1.71	24.29	15.84
	100.42	4.21	1.75	24.28	15.83
	100.38	4.22	1.71	24.30	15.84
	100.37	4.23	1.71	24.29	15.83
	100.37	4.22	1.74	24.30	15.82
	100.36	4.22	1.73	24.28	15.81
	100.34	4.20	1.74	24.28	15.81
	100.36	4.21	1.71	24.27	15.82
	100.30	4.23	1.72	24.27	15.84
	100.35	4.25	1.71	24.28	15.83
	100.32	4.23	1.70	24.27	15.82
	100.32	4.22	1.68	24.29	15.83
	100.32	4.19	1.71	24.24	15.83
	100.32	4.22	1.72	24.28	15.83
	100.32	4.21	1.73	24.26	15.83
	100.34	4.23	1.73	24.29	15.82
	100.29	4.23	1.71	24.29	15.81
	100.28	4.21	1.74	24.27	15.81
	100.25	4.24	1.73	24.27	15.81
	100.23	4.24	1.72	24.21	15.84

	100.22	4.23	1.73	24.28	15.82
	100.21	4.23	1.71	24.24	15.83
	100.19	4.23	1.67	24.29	15.84
	100.22	4.23	1.69	24.28	15.83
	100.20	4.23	1.71	24.27	15.82
	100.16	4.18	1.72	24.27	15.85
	100.16	4.26	1.72	24.24	15.80
	100.12	4.24	1.71	24.28	15.82
	100.12	4.21	1.74	24.27	15.80
	100.13	4.24	1.72	24.27	15.83
	100.15	4.26	1.71	24.29	15.81
	100.13	4.23	1.73	24.27	15.81
	100.13	4.23	1.73	24.26	15.82
	100.11	4.22	1.71	24.23	15.83
	100.12	4.23	1.72	24.28	15.83
	100.13	4.20	1.69	24.27	15.83
	100.14	4.23	1.69	24.28	15.82
	100.11	4.22	1.74	24.29	15.80
	100.13	4.23	1.69	24.28	15.80
	100.10	4.23	1.71	24.27	15.82
	100.15	4.23	1.73	24.24	15.81
	100.15	4.23	1.71	24.27	15.80
	100.13	4.26	1.71	24.30	15.81
	100.18	4.25	1.69	24.27	15.83
	100.20	4.23	1.71	24.27	15.84
	100.16	4.22	1.70	24.29	15.83
	100.14	4.23	1.71	24.30	15.83
	100.19	4.23	1.70	24.25	15.83
	100.19	4.21	1.74	24.26	15.83
	100.20	4.23	1.73	24.28	15.83
	100.17	4.23	1.72	24.26	15.82
	100.21	4.26	1.74	24.28	15.83
	100.23	4.24	1.73	24.28	15.81
	100.26	4.21	1.70	24.27	15.83
	100.31	4.23	1.69	24.26	15.84
	100.30	4.24	1.73	24.26	15.84
	100.28	4.26	1.71	24.27	15.84
	100.32	4.22	1.71	24.28	15.86
	100.33	4.20	1.68	24.26	15.85
	100.32	4.23	1.70	24.29	15.84
	100.31	4.21	1.73	24.28	15.83
	100.35	4.22	1.74	24.25	15.82
	100.27	4.22	1.72	24.28	15.83
	0.10	0.02	0.02	0.02	0.01

**Table 9. Static pressure at each pressure transducer location w.r.t upstream stagnation pressure of 110psig**

110psig	upstream	pt2	pt3	pt4	pt5
	110.20	5.24	2.01	27.63	19.20
	110.18	5.25	2.02	27.60	19.21
	110.21	5.26	2.02	27.62	19.19
	110.22	5.26	2.03	27.59	19.20
	110.24	5.26	2.07	27.63	19.17
	110.23	5.25	2.01	27.63	19.19
	110.21	5.27	2.01	27.62	19.21
	110.24	5.24	2.02	27.60	19.18
	110.24	5.26	2.02	27.60	19.21
	110.20	5.27	1.98	27.61	19.19
	110.22	5.27	2.02	27.60	19.20
	110.28	5.27	1.97	27.63	19.18
	110.24	5.25	2.02	27.62	19.22
	110.20	5.25	1.99	27.60	19.18
	110.25	5.27	2.01	27.62	19.19
	110.21	5.26	2.02	27.60	19.18
	110.21	5.28	2.01	27.62	19.17
	110.21	5.26	2.02	27.60	19.18
	110.22	5.25	2.00	27.62	19.18
	110.24	5.28	2.01	27.59	19.16
	110.24	5.26	2.00	27.59	19.20
	110.21	5.26	2.00	27.60	19.19
	110.23	5.27	2.02	27.60	19.18
	110.23	5.25	2.02	27.60	19.18
	110.24	5.25	2.00	27.63	19.18
	110.21	5.22	2.01	27.60	19.18
	110.17	5.24	2.02	27.60	19.19
	110.21	5.25	2.02	27.59	19.18
	110.26	5.26	2.02	27.61	19.20
	110.21	5.24	2.02	27.60	19.18
	110.24	5.23	2.02	27.60	19.16
	110.24	5.24	1.99	27.59	19.18
	110.27	5.22	2.04	27.60	19.21
	110.19	5.26	1.99	27.59	19.20
	110.23	5.23	2.03	27.57	19.19
	110.21	5.25	1.99	27.63	19.19
	110.22	5.24	2.01	27.61	19.18
	110.19	5.21	2.01	27.61	19.20
	110.19	5.23	2.01	27.62	19.18
	110.14	5.22	2.01	27.61	19.16
	110.18	5.24	2.00	27.62	19.18
	110.18	5.24	2.02	27.58	19.16
	110.19	5.24	2.01	27.60	19.17
	110.13	5.21	2.00	27.60	19.16
	110.18	5.25	2.01	27.58	19.18
	110.16	5.22	2.01	27.58	19.18

	110.18	5.24	2.02	27.62	19.17
	110.15	5.26	1.99	27.63	19.18
	110.15	5.19	2.00	27.61	19.19
	110.15	5.25	2.03	27.60	19.17
	110.13	5.24	2.03	27.61	19.17
	110.15	5.25	2.02	27.59	19.18
	110.12	5.25	2.02	27.60	19.18
	110.11	5.26	1.99	27.60	19.15
	110.16	5.27	2.01	27.60	19.15
	110.11	5.27	2.01	27.60	19.18
	110.15	5.27	2.00	27.60	19.19
	110.14	5.27	2.00	27.60	19.18
	110.11	5.26	2.00	27.60	19.16
	110.11	5.26	2.01	27.62	19.17
	110.08	5.27	2.02	27.62	19.19
	110.10	5.24	2.02	27.62	19.18
	110.10	5.25	2.02	27.62	19.16
	110.10	5.25	2.02	27.60	19.19
	110.07	5.27	1.99	27.62	19.17
	110.09	5.26	2.00	27.60	19.15
	110.12	5.24	2.03	27.60	19.18
	110.08	5.24	2.00	27.60	19.18
	110.14	5.24	2.01	27.61	19.19
	110.15	5.27	2.00	27.59	19.20
	110.15	5.25	1.99	27.63	19.19
	110.15	5.24	2.01	27.64	19.18
	110.12	5.24	2.00	27.61	19.19
	110.14	5.21	2.02	27.62	19.21
	110.18	5.25	2.05	27.62	19.19
	110.18	5.24	2.00	27.61	19.18
	110.20	5.25	2.04	27.61	19.18
	110.16	5.25	2.03	27.60	19.15
	110.20	5.22	2.01	27.62	19.20
	110.21	5.24	2.01	27.57	19.22
	110.20	5.24	2.01	27.61	19.20
	110.18	5.24	2.00	27.60	19.19
	110.21	5.25	1.99	27.62	19.19
	110.16	5.23	2.02	27.63	19.18
	110.20	5.22	2.03	27.63	19.20
	110.23	5.23	2.05	27.62	19.19
	110.18	5.25	2.01	27.61	19.18
	0.05	0.02	0.02	0.01	0.02

**Table 10. Calculated values fro amplitude attenuation in decibels**

press(psia)	x(in)	PT2_ampd	PT3_ampd	Px_P3/Pr_P2	log10(P3/P2)	dB	dB/in	abs(dB/in)
95	1.305	0.01034	0.00623	0.6025	-0.2200	-4.40	-2.59	2.59
105	3.005	0.00115	0.00107	0.9304	-0.0313	-0.63	-0.37	0.37
115	4.705	0.00113	0.0008	0.7080	-0.1500	-3.00	-1.76	1.76
125	6.405	0.0000154	0.0000206	1.3377	0.1263	2.53	1.49	1.49

press(psia)	x(in)	PT2_ampd	PT4_ampd	Px_P4/Pr_P2	log10(P4/P2)	dB	dB/in	abs(dB/in)
95	1.305	0.01034	0.002275	0.2200	-0.6575	-13.15	-3.87	3.87
105	3.005	0.00115	0.000055	0.0478	-1.3203	-26.41	-7.77	7.77
115	4.705	0.00113	0.000016	0.0142	-1.8490	-36.98	-10.88	10.88
125	6.405	0.0000154	0.000009	0.5844	-0.2333	-4.67	-1.37	1.37

press(psia)	x(in)	PT2_ampd	PT5_ampd	Px_P5/Pr_P2	log10(P5/P2)	dB	dB/in	abs(dB/in)
95	1.305	0.01034	0.00499	0.4826	-0.3164	-6.33	-1.24	1.24
105	3.005	0.00115	0.000064	0.0557	-1.2545	-25.09	-4.92	4.92
115	4.705	0.00113	0.000047	0.0416	-1.3810	-27.62	-5.42	5.42
125	6.405	0.0000154	0.000011	0.7143	-0.1461	-2.92	-0.57	0.57

**Table 11. Calculated values for percent dissipation**

Press(psia)	x(in)	PT2_ampd	PT3_ampd	diff_Pt2-pt3	%attenuation	Atten(%/inch)
95	1.305	0.01034	0.00623	0.00411	39.7	23
105	3.005	0.00115	0.00107	0.00008	7.0	4
115	4.705	0.00113	0.0008	0.00033	29.2	17
125	6.405	0.0000154	0.0000206	-0.00001	-33.8	-20

Press(psia)	x(in)	PT2_ampd	PT4_ampd	diff_Pt3-pt4	%attenuation	Atten(%/inch)
95	1.305	0.01034	0.002275	0.00807	78.0	23
105	3.005	0.00115	0.000055	0.00110	95.2	28
115	4.705	0.00113	0.000016	0.00111	98.6	29
125	6.405	0.0000154	0.000009	0.00001	41.6	12

Press(psia)	x(in)	PT2_ampd	PT5_ampd	diff_Pt4-pt5	%attenuation	Atten(%/inch)
95	1.305	0.01034	0.00499	0.00535	51.7	10
105	3.005	0.00115	0.000064	0.00109	94.4	19
115	4.705	0.00113	0.000047	0.00108	95.8	19
125	6.405	0.0000154	0.000011	0.00000	28.6	6

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